

**NISTIR 7336**

# **Influence of Concentration and Additives on R123/Paraffinic Material Oil Boiling Heat Transfer Performance**

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# Influence of Additives and Concentration on R123/Paraffinic Mineral Oil Pool Boiling Heat Transfer

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## **ABSTRACT**

This report investigates the effect that oil concentration had on the boiling performance of an R123/paraffinic mineral oil mixture on a roughened, horizontal flat surface. For all compositions (0.5 %, 1 %, and 2 %), the lubricant caused a heat transfer degradation relative to the heat transfer of pure R123 of between 2 % and 70 % for the range of measured heat fluxes. The heat transfer degradation was shown to increase with lubricant mass fraction. The minimum heat transfer degradation for each mixture ranged between 2 % and 12 % and occurred at approximately  $20 \text{ kW/m}^2$ . For a given composition, the heat transfer degradation increased as the heat flux increased from roughly  $20 \text{ kW/m}^2$  to  $90 \text{ kW/m}^2$ . In addition, the effect of two trial additives on the pool boiling heat transfer of an R123/paraffinic mineral oil mixture was examined in order to test the validity of a theory for choosing oil additives to enhance boiling performance. The verification tests were inconclusive. More research with lubricants and additives with greater differences in surface tensions is required to develop a more rigorous and quantifiable theory for designing additives that improve boiling heat transfer.

Keywords: refrigerants, additives, boiling, enhanced heat transfer, refrigerant/lubricant mixtures, paraffinic mineral oil

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## INTRODUCTION

The quest for understanding how lubricants and lubricant additives influence pool boiling has always been important in terms of how lubricants affect the design of efficient, affordable and reliable equipment. Improving the efficiency of existing air-conditioning equipment has been significantly emphasized with the recent blackout of the U.S. Northeast and escalating energy and copper costs. Increased accuracy in the sizing of evaporators affords the opportunity for using less copper in evaporators. Consequently, every trustworthy refrigerant/lubricant pool boiling heat transfer measurement can contribute to the pursuit of a fundamental boiling model. In addition, as illustrated in Appendix A, a refrigerant oil additive that improves water chiller efficiency could be a cost-effective and immediate means of reducing operating costs and improving the reliability of (reduced demand on) our nation's electricity grid. Because of this, agencies in the US federal government have been interested in the possibility of using additives to improve chiller performance. Toward this end, NIST<sup>1</sup> (Kedzierski, 2004 & 2006) has proposed a theory with a purpose of selecting lubricants that when added to the base lubricant of an existing chiller will improve the pool boiling performance.

As shown by Kedzierski (2001c), the viscosity, miscibility and concentration of the lubricant strongly influence refrigerant/lubricant pool boiling. The properties of the lubricant closest to the wall essentially control the boiling. Consequently, if an additive (or added lubricant) is to have an impact on a given refrigerant/lubricant system, there would be a greater likelihood for influence if the additive can exist as a monolayer on the surface. If the additive is well-mixed in the lubricant excess layer (which is likely for a chemically similar additive and lubricant), it will have minimal influence on the heat transfer if it is only 2 % by mass of the lubricant (typical additive charge). For this reason, two of the three basic rules for an additive to potentially enhance boiling heat transfer, according to the NIST hypothesis, are concerned with determining if a monolayer forms or not. According to the theory, the two requirements for a monolayer to form are: (1) the liquid-vapor surface tension of the additive should be larger than that of the base lubricant, and (2) the base lubricant and the additive must be "chemically dissimilar." At this point in time, these rules are not well defined. For example, it is not known how large of a difference between surface-tensions is required. In addition, synthetic lubricant and mineral oil are sufficiently different to satisfy the "chemically dissimilar" definition, but this may not be true for all cases. The third and final rule for enhancement is that the viscosity of the additive must be larger than that of the base lubricant. However, an additive with larger viscosity than the base lubricant will potentially induce an enhancement in boiling heat transfer only if it forms a monolayer. In other words, all three rules or conditions must be satisfied before a lubricant can have the potential of being an effective additive.

The primary goals of this study were to (1) attempt to verify or disprove the NIST rules for selecting lubricants to improve pool boiling heat transfer, and (2) to contribute R123/lubricant pool boiling data to the literature in support of improved boiling modeling. The base lubricant chosen for use with R123 was a paraffinic mineral oil with a nominal kinematic viscosity of  $85 \mu\text{m}^2/\text{s}$  at 297.8 K. Paraffinic oils are typically used for ammonia applications and less typically with other low pressure refrigerants like R123. The primary

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<sup>1</sup> NIST (National Institute of Standards and Technology) is part of the U.S. Department of Commerce.



reason for the selection of this particular paraffinic mineral was because it had the smallest liquid-vapor surface tension with a room temperature viscosity larger than  $60 \mu\text{m}^2/\text{s}$  of all lubricants that were found in an extensive search of various mineral oils. By choosing a base lubricant with a small surface tension and a large viscosity, the opportunity for finding additives to enhance the boiling performance was improved. Following the NIST additive rules for enhancing pool boiling heat transfer, two synthetic oils were chosen (PG and LP1) as additives that had both larger viscosity and larger surface tension than the paraffinic base lubricant. Table 1 shows that the viscosity of the two oils that were used as additives were approximately  $150 \mu\text{m}^2/\text{s}$  and  $2000 \mu\text{m}^2/\text{s}$  at 297.8 K for PG and LP1, respectively. Also shown in Table 1 is that the surface tensions are only nominally 5 % larger than that of the paraffinic ( $30.6 \mu\text{m}^2/\text{s} \pm 0.7 \mu\text{m}^2/\text{s}$ ).

## APPARATUS

Figure 1 shows a schematic of the apparatus that was used to measure the pool boiling data of this study. More specifically, the apparatus was used to measure the liquid saturation temperature ( $T_s$ ), the average pool-boiling heat flux ( $q''$ ), and the wall temperature ( $T_w$ ) of the test surface. The three principal components of the apparatus were the test chamber, the condenser, and the purger. The internal dimensions of the test chamber were  $25.4 \text{ mm} \times 257 \text{ mm} \times 1.54 \text{ m}$ . The test chamber was charged with approximately 7 kg of refrigerant, giving a liquid height of approximately 80 mm above the test surface. As shown in Fig. 1, the test section was visible through two opposing, flat  $150 \text{ mm} \times 200 \text{ mm}$  quartz windows. The bottom of the test surface was heated with high velocity (2.5 m/s) water flow. The vapor produced by liquid boiling on the test surface was condensed by the brine-cooled, shell-and-tube condenser and returned as liquid to the pool by gravity. Further details of the test apparatus can be found in Kedzierski (2002a) and Kedzierski (2001a).

## TEST SURFACE

Figure 2 shows the oxygen-free high-conductivity (OFHC) copper flat test plate used in this study. The test plate was machined out of a single piece of OFHC copper by electric discharge machining (EDM). A tub grinder was used to finish the heat transfer surface of the test plate with a crosshatch pattern. Average roughness measurements were used to estimate the range of average cavity radii for the surface to be between  $12 \mu\text{m}$  and  $35 \mu\text{m}$ . The relative standard uncertainty of the cavity measurements were approximately  $\pm 12 \%$ . Further information on the surface characterization can be found in Kedzierski (2001a).

## MEASUREMENTS AND UNCERTAINTIES

The standard uncertainty ( $u_i$ ) is the positive square root of the estimated variance  $u_i^2$ . The individual standard uncertainties are combined to obtain the expanded uncertainty ( $U$ ), which is calculated from the law of propagation of uncertainty with a coverage factor. All measurement uncertainties are reported at the 95 % confidence level except where specified otherwise. For the sake of brevity, only an outline of the basic measurements and uncertainties is given below. Complete detail on the heat transfer measurement techniques and uncertainties can be found in Kedzierski (2000a) and Appendix B, respectively.

All of the copper-constantan thermocouples and the data acquisition system were calibrated against a glass-rod standard platinum resistance thermometer (SPRT) and a reference voltage to a residual standard deviation of 0.005 K. Considering the fluctuations in the saturation temperature during the test and the standard uncertainties in the calibration, the expanded uncertainty of the average saturation temperature was no greater than 0.04 K. Consequently, it is believed that the expanded uncertainty of the temperature measurements was less than 0.1 K.

Twenty 0.5 mm diameter thermocouples were force fitted into the wells of the side of the test plate shown in Fig. 2. The heat flux and the wall temperature were obtained by regressing the measured temperature distribution of the block to the governing two-dimensional conduction equation (Laplace equation). In other words, rather than using the boundary conditions to solve for the interior temperatures, the interior temperatures were used to solve for the boundary conditions following a backward stepwise procedure given in Kedzierski (1995)<sup>2</sup>. Fourier's law and the fitted constants from the Laplace equation were used to calculate the average heat flux ( $q''$ ) normal to and evaluated at the heat transfer surface based on its projected area. The average wall temperature ( $T_w$ ) was calculated by integrating the local wall temperature ( $T$ ). The wall superheat was calculated from  $T_w$  and the measured temperature of the saturated liquid ( $T_s$ ). Considering this, the relative expanded uncertainty in the heat flux ( $U_{q''}$ ) was greatest at the lowest heat fluxes, approaching 10 % of the measurement near 10 kW/m<sup>2</sup>. In general, the  $U_{q''}$  remained approximately within 3 % and 6 % for heat fluxes greater than 30 kW/m<sup>2</sup>. The average random error in the wall superheat ( $U_{T_w}$ ) was between 0.04 K and 0.1 K. Plots of  $U_{q''}$  and  $U_{T_w}$  versus heat flux can be found in Appendix B.

## EXPERIMENTAL RESULTS

The heat flux was varied roughly between 10 kW/m<sup>2</sup> and 100 kW/m<sup>2</sup> to simulate a range of possible operating conditions for R123 chillers. All pool-boiling tests were taken at 277.6 K saturated conditions. The data were recorded consecutively starting at the largest heat flux and descending in intervals of approximately 4 kW/m<sup>2</sup>. The descending heat flux procedure minimized the possibility of any hysteresis effects on the data, which would have made the data sensitive to the initial operating conditions. Table 3 presents the measured heat flux and wall superheat for all the data of this study. Table 4 gives the number of test days and data points for each fluid.

The mixtures were prepared by charging the test chamber (see Fig. 1) with pure R123 to a known mass. Next, a measured mass of lubricant was injected with a syringe through a port in the test chamber. The refrigerant/lubricant solution was mixed by flushing pure refrigerant through the same port where the lubricant was injected. After the tests with the refrigerant/lubricant mixture were completed, the additive was added to the existing test chamber charge in the same manner as for the lubricant. Additives were added to the refrigerant/lubricant (98/2) mixture as roughly 10 % of the existing mass of lubricant in the system giving an R123/PARA/ PG (97.8/2/0.2) mixture and an R123/PARA/LP1 (97.8/2/0.2) mixture. All compositions were determined from the masses of the charged components and

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<sup>2</sup> For the record, Table 2 provides functional forms of the Laplace equation that were used in this study in the same way as was done in Kedzierski (1995) and in similar studies by this author.

are given on a mass fraction percent basis. The maximum uncertainty of the composition measurement is approximately 0.02 %, e.g., the range of a 2.0 % composition is between 1.98 % and 2.02 %. Nominal or target mass compositions are used in the discussion. For example, the “actual” mass composition of the PARA in the R123/PARA (99.5/0.5) mixture was  $0.54 \% \pm 0.02 \%$ . Likewise, the PARA mass fractions for R123/PARA (99/1) and the R123/PARA (98/2) mixtures were  $0.98 \% \pm 0.02 \%$  and  $1.99 \% \pm 0.02 \%$ , respectively. Using the same uncertainties, the additive mixtures as tested were R123/PARA/PG (97.90/1.97/0.13) and R123/PARA/LP1 (97.87/1.94/0.19).

Figure 3 is a plot of the measured heat flux ( $q''$ ) versus the measured wall superheat ( $T_w - T_s$ ) for pure R123 at a saturation temperature of 277.6 K. The opened circles represent six days of boiling measurements made over a period of approximately one week. The solid lines shown in Fig. 3 are cubic best-fit regressions or estimated means of the data. Three of the 107 measurements were removed before fitting because they were identified as “outliers” based on having both high influence and high-leverage (Belsley et al., 1980). Table 5 gives the constants for the cubic regression of the superheat versus the heat flux for all of the fluids tested here. The residual standard deviation of the regressions - representing the proximity of the data to the mean - are given in Table 6. The dashed lines to either side of the mean represent the lower and upper 95 % simultaneous (multiple-use) confidence intervals for the mean. From the confidence intervals, the expanded uncertainty of the estimated mean wall superheat was 0.26 K and 0.04 K for superheats less than and greater than 14.5 K, respectively. Table 7 provides the average magnitude of 95 % multi-use confidence interval for the fitted wall superheat for all of the test data.

Figure 4 plots the measured heat flux ( $q''$ ) versus the measured wall superheat ( $T_w - T_s$ ) at a saturation temperature of 277.6 K for the three R123/PARA mixtures. Comparison of the three boiling curves shows that they are within  $2 \text{ kW/m}^2$  of each other and the pure R123 performance at a superheat of approximately 11 K. For mean superheats greater than 11 K, the boiling heat transfer degradation with respect to pure R123 boiling heat transfer increases with increasing lubricant concentration. This is an expected trend as outlined in Kedzierski (2001c) because the bubble size becomes smaller for increasing lubricant mass fraction, which leads to a reduction in heat transfer if the site density does not increase enough to compensate for the loss of vapor generation per bubble.

Figure 5 is a plot of the measured heat flux ( $q''$ ) versus the measured wall superheat ( $T_w - T_s$ ) for the R123/PARA/PG (97.8/2/0.2) mixture at a saturation temperature of 277.6 K. The open diamonds represent over 50 boiling measurements made over three consecutive days. For comparison, the mean of the R123/PARA (98/2) mixture boiling curve is provided as a coarsely dashed, gray line. The average expanded uncertainty of the estimated mean wall superheat for superheats greater and less than 15.5 K was 0.14 K and 0.46 K, respectively.

Figure 6 is a plot of the measured heat flux ( $q''$ ) versus the measured wall superheat ( $T_w - T_s$ ) for the R123/PARA/LP1 (97.8/2/0.2) mixture at a saturation temperature of 277.6 K. The open stars represent 126 boiling measurements made over four days within a week. For comparison, the mean of the R123/PARA (98/2) mixture boiling curve is provided as a coarsely dashed, gray line. The average expanded uncertainty of the estimated mean wall superheat for superheats greater and less than 15.5 K was 0.11 K and 0.31 K, respectively.

A more detailed comparison of the R123/PARA and the R123/PARA/additive heat transfer performances the relative to R123 and R123/PARA (98/2), respectively, is given in Figs. 7 and 8. Figure 7 plots the ratio of the R123/PARA mixture heat flux to the pure R123 heat flux ( $q''_m/q''_p$ ) versus the pure R123 heat flux ( $q''_p$ ) at the same wall superheat. Figure 7 illustrates the influence of lubricant mass composition on the R123/PARA boiling curve with solid lines representing the mean heat flux ratios for each mixture. Overall, lubricant for all compositions has caused a heat transfer degradation relative to the heat transfer of pure R123 for all measured heat fluxes. The degradation is shown to increase with lubricant mass fraction. For example, the average heat flux ratio for the R123/PARA (99.5/0.5), the R123/PARA (99/1), and the R123/PARA (98/2) mixture from approximately 15 kW/m<sup>2</sup> to 100 kW/m<sup>2</sup> was 0.64, 0.55, and 0.48, respectively. The minimum heat transfer degradation for each mixture (or the maximum heat transfer) is shown on Fig. 7 to be at approximately 20 kW/m<sup>2</sup>. For 20 kW/m<sup>2</sup>, the heat flux ratio for the R123/PARA (99.5/0.5), the R123/PARA (99/1), and the R123/PARA (98/2) mixture is  $0.98 \pm 0.02$ ,  $0.90 \pm 0.025$ , and  $0.88 \pm 0.02$ , respectively. The lubricant effect becomes more pronounced as the heat flux increases from roughly 20 kW/m<sup>2</sup> to 90 kW/m<sup>2</sup> producing heat flux ratios of approximately 0.52, 0.4, and 0.3 at 90 kW/m<sup>2</sup> for the R123/PARA (99.5/0.5), the R123/PARA (99/1), and the R123/PARA (98/2) mixtures, respectively.

Figure 8 details the effect that the additives had on the R123/PARA (98/2) boiling curve. The figure plots the ratio of the R123/PARA/PG (97.8/2/0.2) heat flux to the R123/PARA (98/2) heat flux ( $q''_m/q''_{2\%}$ ) versus the R123/PARA (98/2) mixture heat flux ( $q''_{2\%}$ ) at the same wall superheat. Likewise, the R123/PARA/LP1 (97.8/2/0.2) heat flux to the R123/PARA (98/2) heat flux ( $q''_m/q''_{2\%}$ ) versus the R123/PARA (98/2) mixture heat flux ( $q''_{2\%}$ ) at the same wall superheat is also plotted. A heat transfer enhancement exists where the heat flux ratio is greater than one and the 95 % simultaneous confidence intervals (depicted by the shaded regions) do not include the value one. Figure 8 shows that for heat fluxes less than 85 kW/m<sup>2</sup>, neither additive causes an enhancement of R123/PARA (98/2) pool boiling heat transfer. For the most part, either a degradation or no measurable difference is observed compared to the R123/PARA (98/2) mixture. A small 5 % enhancement of the heat flux is obtained for the R123/PARA/LP1 (97.8/2/0.2) mixture at a heat flux of approximately 90 kW/m<sup>2</sup>. Similarly, the R123/PARA/PG (97.8/2/0.2) mixture shows a maximum heat flux ratio of 1.03 at a heat flux of approximately 27 kW/m<sup>2</sup>. However, because the confidence intervals include the value of one at the maximum, no difference between the mixture/additive performance and that of the mixture can be established. Overall, the average heat flux ratio for the R123/PARA/LP1 (97.8/2/0.2) mixture from approximately 7 kW/m<sup>2</sup> to 89 kW/m<sup>2</sup> was 0.95. Likewise, the average heat flux ratio for the R123/PARA/PG (97.8/2/0.2) mixture from approximately 9 kW/m<sup>2</sup> to 81 kW/m<sup>2</sup> was 0.97.

## ADDITIVE DISCUSSION

The heat transfer results summarized in Fig. 8, for the most part, show that the lubricants chosen as additives had little or no effect on the boiling performance of the R123/mineral oil (98/2) system. However, considering that the rules provided by the NIST additive hypothesis are not precisely quantified, the preceding heat transfer results cannot be used to prove or disprove the basic premise of the hypothesis. The main reason for this is that it cannot be

known if the surface-tension difference between the additive and the base lubricant was sufficient enough to cause the formation of an additive monolayer. As Table 1 shows, the differences in the mean kinematic viscosity appears to be significant enough and with satisfactorily small uncertainties for a viscosity effect to be present. However, the surface-tension differences between the base lubricant and the additives are marginal at best. In fact, the LP1 lubricant exhibits nearly no difference in surface tension between it and the PARA ( $1.1 \text{ mN/m} \pm 1 \text{ mN/m}$ ) when the measurement uncertainties are taken into consideration. As a result, when considering LP1 as a potential additive, the evidence is not sufficient to suggest that the surface tension difference was significant enough for the LP1 to form a monolayer at the surface. Consequently, if a monolayer is not formed, then the benefits of a larger viscosity cannot be realized because LP1 is only 10 % of the mass of the base lubricant.<sup>3</sup> Also, although the positive difference in surface tension between PG and the base lubricant is statistically significant ( $1.6 \text{ mN/m} \pm 1 \text{ mN/m}$ ), the difference may not be large enough. There may be a critical difference in surface tensions that must be attained before a monolayer can be formed. In support of this, when Kedzierski (2004) observed a heat transfer enhancement with additives, there was an accompanying difference in surface tension of  $3 \text{ mN/m} \pm 1.4 \text{ mN/m}$ , which is approximately 88 % larger than the surface tension difference between the present base mineral oil and PG.

Future research is required to investigate the influence of the magnitude of the surface-tension difference on boiling heat transfer performance. In addition, there may be other primary lubricant property effects that govern an additive's influence on boiling other than surface tension, viscosity, and chemical similarity with the base lubricant. For example, additive polarity and miscibility may play more of a primary role along with heat transfer. In other words, how the thermal boundary layer interacts with the excess layer and its properties should be studied. Further investigation into the above effects may lead to a more rigorous and quantifiable theory that can be used to develop additives that improve boiling heat transfer for the benefit of the refrigeration and air-conditioning industry.

## CONCLUSIONS

The effect of paraffinic mineral oil concentration on the boiling performance of R123/paraffinic mineral oil mixtures on a roughened, horizontal flat surface was investigated. For all compositions, the lubricant caused a heat transfer degradation relative to the heat transfer of pure R123 for all measured heat fluxes. The heat transfer degradation was shown to increase with lubricant mass fraction. The minimum heat transfer degradation for each mixture occurred at approximately  $20 \text{ kW/m}^2$ . For a given composition, the heat transfer degradation increased as the heat flux increases from roughly  $20 \text{ kW/m}^2$  to  $90 \text{ kW/m}^2$ .

The effect of two trial additives on the pool boiling heat transfer of an R123/paraffinic mineral oil mixture was examined in order to test the validity of a theory for choosing oil additives to enhance boiling performance. In general, the specific lubricants chosen as additives had little or no effect on the boiling performance of the R123/mineral oil mixture.

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<sup>3</sup> In addition, the Kedzierski (2001) showed a linear heat transfer enhancement with respect to increasing viscosity only between roughly 5 cSt and 200 cSt. Consequently, extrapolation of the effect of a 2000 cSt on heat transfer is not justified. In other words, a heat transfer maximum may exist with respect to viscosity for some viscosity greater than 200 cSt.

However, because the rules provided by the NIST additive theory are not precisely quantified, the heat transfer results could not be used to prove or disprove the basic premise of the hypothesis. The main reason for this is that it is not known if the surface-tension difference between the additive and the base lubricant was sufficient enough to cause the formation of an additive monolayer. Future research with greater differences in surface tensions is required to investigate the influence of the magnitude of the surface-tension difference on boiling heat transfer performance in order to develop a more rigorous and quantifiable theory that can be used to design additives that improve boiling heat transfer.

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## NOMENCLATURE

### English Symbols

$A_n$	regression constant in Table 5 $n=0,1,2,3$
$C_o$	viscometer calibration constant, $\mu\text{m}^2 \text{s}^{-1}$
COP	Coefficient of Performance
$d$	liquid pool height, m
$E_{\text{COP}}$	percent increase in COP
$E_h$	percent increase in $h$
$g$	gravitational acceleration, $\text{m s}^{-2}$
$h$	heat transfer coefficient, $\text{W K}^{-1} \text{m}^{-2}$
$l$	capillary rise height, m
$L_y$	length of test surface (Fig. 2), m
LMTD	Log-Mean-Temperature-Difference, K
$P$	vapor pressure, kPa
$q''$	average wall heat flux, $\text{W m}^{-2}$
$r$	radius of capillary tube, m
$T$	temperature, K
$T_w$	temperature at roughened surface, K
$U$	expanded uncertainty
$u_i$	standard uncertainty
$X$	model terms given in Table 2

### Greek symbols

$\Delta T$	temperature difference, K
$\Delta T_s$	wall superheat: $T_w - T_s$ , K
$\Delta \nu$	difference between base lubricant and additive kinematic viscosity, $\text{m}^2 \text{s}^{-1}$
$\Delta \rho$	difference between liquid and vapor density, $\text{kg m}^{-3}$
$\Delta \sigma$	difference between base lubricant and additive surface tension, $\text{kg s}^{-2}$
$\nu$	kinematic viscosity, $\text{m}^2 \text{s}^{-1}$
$\sigma$	surface tension, $\text{kg s}^{-2}$

### English Subscripts

B	baseline
Bc	condenser baseline
Be	evaporator baseline
c	condenser
e	evaporator
m	mixture
p	pure R123
$q''$	heat flux
r	refrigerant-side
s	saturated state
$T_w$	wall temperature
v	vapor
w	wall or water-side
2%	R123/PARA (98/2)

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**Table 1 Base lubricant and additive properties<sup>4</sup>**

$T = 297.8 \text{ K}$					
Lubricant	$\nu$ ( $\mu\text{m}^2/\text{s}$ )	$\Delta\nu$ ( $\mu\text{m}^2/\text{s}$ )	$\sigma$ (mN/m)	$\Delta\sigma$ (mN/m)	$\rho$ ( $\text{kg}/\text{m}^3$ )
Paraffinic Mineral oil (PARA)	$85 \pm 1.5$	0	$30.6 \pm 0.7$	0	877
DGLF118 (PG)	$150.3 \pm 0.8$	$65 \pm 1.8$	$32.2 \pm 0.7$	$1.6 \pm 1$	996
LP1	$2000 \pm 500$	$1915 \pm 500$	$31.7 \pm 0.7$	$1.1 \pm 1$	920

**Table 2 Conduction model choice**

$X_0 = \text{constant (all models)}$ $X_1 = x$ $X_2 = y$ $X_3 = xy$ $X_4 = x^2 - y^2$ $X_5 = y(3x^2 - y^2)$ $X_6 = x(3y^2 - x^2)$ $X_7 = x^4 + y^4 - 6(x^2)y^2$ $X_8 = yx^3 - xy^3$	
Fluid	Most frequent models
Pure R123 (file: pR23ad.dat)	$X_1, X_5$ (40 of 100) 40 % $X_1, X_2, X_4$ (38 of 100) 38 % $X_1, X_2$ (9 of 100) 9 %
R123/PARA (99.5/0.5) (file: 5PARA.dat)	$X_1, X_2, X_4$ (42 of 88) 48 % $X_1, X_5$ (18 of 88) 20 % $X_1, X_5, X_6$ (13 of 88) 15 %
R123/PARA (99/1) (file: PARA1.dat)	$X_1, X_2, X_4$ (46 of 84) 55 % $X_1, X_5, X_6$ (18 of 84) 21 % $X_1, X_2, X_4, X_6$ (8 of 84) 9 % $X_1, X_2, X_5, X_6$ (8 of 84) 9 %
R123/PARA (98/2) (file: PARA2.dat)	$X_1, X_2, X_4$ (43 of 158) 27 % $X_1, X_2$ (30 of 158) 19 % $X_1, X_2, X_4, X_6$ (28 of 158) 18 %
R123/PARA/PG (97.8/2/0.2) (file: PARAPG.dat)	$X_1, X_2$ (13 of 52) 25 % $X_1, X_2, X_4$ (12 of 52) 23 % $X_1, X_2, X_4, X_6$ (12 of 52) 23 %
R123/PARA/LP1 (97.8/2/0.2) (file: PARALP1.dat)	$X_1, X_2$ (26 of 117) 22 % $X_1, X_2, X_6$ (26 of 117) 22 % $X_1, X_2, X_4, X_6$ (25 of 117) 21 %

<sup>4</sup> Density, surface tension, and viscosity measurements and measurement methodologies are given in Appendixes C, D, and E, respectively.

**Table 3 Pool boiling data**

**Pure R123**

**File: PR123AD.dat**

$\Delta T_s$ (K)	$q''$ (W/m <sup>2</sup> )
16.17	100368.4
15.45	78680.3
15.12	67248.9
14.77	56119.3
14.30	37624.5
13.95	31120.5
12.47	22070.1
10.76	16734.0
8.58	14901.5
16.64	112156.3
16.18	98384.2
15.92	90476.4
15.51	76815.4
15.13	64511.2
14.81	54653.3
14.55	43565.7
13.95	31386.2
12.56	22460.8
16.35	103370.6
16.23	100019.3
16.08	96101.5
15.80	88879.3
15.63	84008.8
15.48	78704.8
15.33	73348.2
15.21	69058.7
15.09	64392.0
14.99	60255.1
14.86	55006.2
14.70	48673.3
14.58	44497.1
14.38	38647.8
14.26	36376.0
14.04	32718.7
13.79	29494.6
13.23	25765.9
12.78	23379.9
12.22	21538.6
11.67	19322.6
10.98	17817.9
10.25	16245.2
9.40	15030.1
8.33	14709.5
7.68	13942.2
16.37	102380.8
15.93	91408.9
15.50	75842.0

15.21	66140.9
15.10	62536.8
14.97	57355.2
14.68	46509.4
14.53	41071.2
14.12	32630.9
13.63	27627.0
12.68	23272.9
11.20	17978.9
9.65	15246.2
8.71	14019.7
16.07	98119.3
15.81	90121.4
15.63	84306.2
15.43	76463.5
15.25	69072.8
15.09	62320.9
14.88	52351.0
14.59	42825.2
14.11	33951.4
13.51	27336.2
12.40	21992.4
11.62	19572.1
10.62	16715.3
9.45	15004.9
15.99	94152.9
15.68	84632.4
15.42	75696.8
15.17	65575.6
15.02	58770.2
14.71	45918.8
14.33	35843.3
13.87	29357.6
12.89	23521.2
12.16	20811.2
11.28	18084.1
10.50	16298.5
9.21	15759.4
7.46	12867.5
15.99	94152.9
15.68	84632.4
15.42	75696.8
15.17	65575.6
15.02	58770.2
14.71	45918.8
14.33	35843.3
13.87	29357.6
12.89	23521.2
12.16	20811.2
11.28	18084.1
10.50	16298.5

9.206	15759.4
7.460	12867.5

**R123/PARA (99.5/0.5)**

**File: 5PARA.dat**

$\Delta T_s$ (K)	$q''$ (W/m <sup>2</sup> )
17.664	86194.2
17.232	76725.6
16.911	69740.7
16.507	60527.9
16.183	52047.5
15.817	42105.5
15.278	31945.6
14.461	26278.3
13.058	21748.3
11.481	18073.6
17.110	73547.5
16.903	70136.6
16.642	64810.8
16.459	60912.1
16.332	56797.3
16.268	56529.1
16.308	58731.8
16.183	58840.4
16.055	54914.5
15.872	49986.0
15.582	43960.3
15.302	38495.0
15.015	33827.9
14.682	29441.2
14.270	26576.7
13.510	23877.1
12.782	22250.2
11.999	19818.2
11.461	17850.9
10.613	16655.9
10.021	15740.2
17.416	81537.8
16.956	72659.9
16.607	65639.2
16.240	55665.9
15.907	48123.2
15.410	38185.6
14.695	30002.1
13.719	24581.0
12.221	20327.8
10.717	17344.8
18.035	92753.1
17.750	90300.5
17.491	85067.9
17.378	82792.4

17.25	79952.9
16.96	75044.6
16.75	70968.0
16.58	67178.5
16.39	62727.5
16.26	58627.3
15.94	51559.9
15.68	46759.2
15.46	41663.5
15.25	37751.8
14.91	32855.5
14.52	29218.9
14.11	26473.1
13.76	24825.1
12.87	22337.7
12.15	20477.8
11.50	18253.8
10.72	16771.0
9.92	15538.7
17.85	90809.5
17.65	85605.7
17.44	82533.2
17.24	78095.9
17.00	73202.1
16.85	70359.4
16.63	65716.1
16.46	61887.3
16.32	56457.7
16.14	52339.8
15.89	47270.1
15.73	43672.6
15.38	35947.9
15.13	32716.9
14.57	28877.7
14.16	26596.6
13.61	24508.0
12.90	22830.8
12.12	20499.8
11.28	17806.6
10.87	17195.3
9.76	15340.7
8.91	15609.1
8.26	14183.3

**R123/PARA (99/1)**  
**File: PARA1.dat**

$\Delta T_s$ (K)	$q''$ (W/m <sup>2</sup> )
17.68	75831.4
17.63	71742.8
17.35	69074.2
17.13	62849.3
16.95	58990.7

16.78	55396.2
16.51	50041.3
16.23	43677.3
16.03	39797.6
15.74	35256.6
15.43	31264.7
15.02	28124.8
14.40	25478.0
13.87	23623.7
13.06	21604.0
12.48	18678.9
11.69	17106.4
11.04	16371.9
17.41	64517.6
17.32	61863.5
17.12	56781.2
16.95	53041.5
16.76	48928.1
16.53	44841.3
16.13	38162.6
15.74	32984.7
15.52	31178.9
15.00	27672.6
14.42	25976.2
13.68	23317.2
12.90	21472.4
12.30	19314.3
11.67	18122.7
10.72	16335.0
17.22	64996.6
17.05	60878.2
16.89	55928.6
16.78	52846.7
16.64	50047.8
16.39	45036.8
16.00	38143.9
15.69	33788.0
15.39	30959.9
14.94	27888.8
14.28	25700.9
13.84	23574.9
12.98	21268.0
12.27	18893.0
11.41	17142.5
10.71	16247.3
17.18	66620.4
17.03	63457.5
16.88	58939.9
16.70	55018.3
16.50	51612.1
16.22	44861.1
16.05	41313.2

15.71	36456.3
15.56	31728.8
15.16	28466.4
14.67	25443.1
14.08	24533.2
13.31	21221.9
12.79	19039.8
18.43	84802.5
18.11	82192.9
17.94	79721.8
17.72	75552.6
17.53	69660.9
17.37	65943.8
17.16	61173.9
16.94	56753.6
16.73	53066.8
16.43	46066.2
16.31	42924.9
16.04	38178.5
15.66	33027.6
15.36	30603.3
14.77	26860.2
14.20	24435.6
13.34	21353.5
12.89	19534.6
12.44	17915.7
11.90	17695.6

**R123/PARA (98/2)**  
**File: PARA2.dat**

$\Delta T_s$ (K)	$q''$ (W/m <sup>2</sup> )
19.03	83387.4
18.77	79517.8
18.51	78362.9
18.29	74699.3
18.10	70731.8
17.89	66273.3
17.77	63057.3
17.61	56225.6
17.33	50990.2
17.23	44741.7
16.98	40372.8
16.82	36872.2
16.43	31815.0
15.97	28419.7
15.49	25853.3
14.72	23156.1
14.13	20519.0
13.31	18755.6
12.88	18562.9
19.78	95822.0
19.52	92292.0

19.19	87838.2
18.93	84015.5
18.26	74593.4
18.02	71477.4
17.87	69277.5
17.59	62637.3
17.56	56851.1
17.48	52302.9
17.45	48898.7
17.24	41903.4
17.01	37837.1
16.60	33814.1
16.18	30798.7
15.63	26934.0
15.05	24590.6
14.42	23108.4
13.56	19595.5
12.91	18480.1
12.05	16921.1
11.25	15891.5
10.61	14821.6
9.81	13390.9
8.81	13599.6
7.90	11941.8
7.15	10606.2
6.16	9127.1
5.27	8011.9
19.42	88436.1
19.02	84910.8
18.76	81030.3
18.51	77337.9
18.26	72599.3
18.15	70010.0
17.96	65204.6
17.83	61350.6
17.60	54246.4
17.45	50539.8
17.31	46453.7
16.97	39327.8
16.81	35554.5
16.44	32084.5
15.96	28182.7
15.49	25665.9
14.76	23358.0
13.91	20772.3
13.35	19171.3
12.53	17979.8
11.80	16548.0
10.94	15306.8
10.23	14232.3
9.30	14241.0
8.28	12582.3

7.47	11296.8
6.32	9213.9
19.24	85892.6
19.10	84642.4
18.64	79518.1
18.47	77001.8
18.19	72456.9
18.02	69026.1
17.88	65529.0
17.72	59168.1
17.63	55344.3
17.40	49898.0
17.26	46552.3
16.88	39142.2
16.63	35488.3
16.38	32455.1
15.90	28873.8
15.41	25719.0
14.74	23656.2
14.13	22044.5
13.25	19225.7
12.82	18730.2
11.88	16996.5
11.14	15829.8
10.26	14265.3
9.52	14849.8
8.61	13192.6
7.52	11152.0
6.71	9958.1
5.71	8222.8
4.83	6959.6
19.81	93462.0
19.53	89920.9
19.22	85222.4
18.89	80249.8
18.52	76806.3
18.33	73813.5
18.09	68867.9
17.93	63321.6
17.67	56584.9
17.45	50035.4
17.23	46732.0
17.11	41355.3
16.91	36750.4
16.49	32462.1
16.08	29618.5
15.44	25913.0
14.73	23778.7
13.77	20145.0
13.35	19221.5
12.70	18377.0
11.96	16964.0

11.10	15608.3
10.38	14639.0
9.51	14787.0
8.77	13531.7
7.33	10996.5
6.42	9841.2
5.58	8364.7
4.98	7316.8
18.01	60573.2
17.77	55233.9
17.35	42191.6
16.84	35597.2
16.03	29410.9
14.80	23427.8
13.31	18698.2
11.67	15529.4
9.65	14070.5
19.10	88600.6
18.54	78050.7
18.12	69518.7
17.99	61993.4
17.72	54150.4
17.48	48425.1
17.17	40315.8
16.56	32595.3
15.64	26737.2
14.67	23216.7
13.04	18663.5
11.66	16062.9
10.15	13633.6
8.71	12798.0
6.39	9744.0
4.38	6160.1

**R123/PARA/PG**  
**File: PARAPG.dat**

$\Delta T_s$ (K)	$q''$ (W/m <sup>2</sup> )
18.52	75389.6
18.23	67941.5
17.99	61941.4
17.86	56776.8
17.56	48877.0
17.34	42323.0
16.82	34914.2
16.10	29003.0
14.79	23543.3
13.77	20023.4
12.76	17490.1
11.83	16423.1
10.71	14579.4
9.36	13928.2
7.80	11373.6

19.02	84018.0
18.65	75652.9
18.15	66693.6
17.92	58932.5
17.61	51510.6
17.41	46376.2
16.87	35430.5
16.03	28370.9
15.03	24682.7
14.18	20935.8
12.82	17658.9
11.74	16640.8
10.40	14333.0
9.33	14362.0
7.93	11714.6
6.77	10059.1
5.32	7513.8
19.16	87538.1
18.76	78591.6
18.40	73569.1
18.18	69209.6
17.90	60218.3
17.45	50465.8
17.11	42807.8
16.71	35346.1
16.17	30489.8
15.34	25240.7
14.43	22394.0
13.72	20650.6
12.75	17390.0
11.41	15767.1
10.20	13738.0
8.92	13443.6
7.58	11163.3
6.30	9265.5
5.14	7398.3
4.05	5600.9

**R123/PARA/LP1**  
**File: PARALP1.dat**

$\Delta T_s$ (K)	$q''$ (W/m <sup>2</sup> )
18.93	89371.8
18.59	84675.3
18.45	81828.4
18.15	75327.7
18.12	74006.9
17.98	69674.6
17.92	66965.8
17.81	60515.6
17.74	56478.3
17.66	52470.4
17.44	47686.9

17.06	40883.3
16.80	37061.0
16.35	32252.7
15.85	28673.4
15.34	26367.2
14.78	24009.5
14.25	21373.1
13.53	19633.5
12.74	17577.6
12.10	17057.8
11.23	15305.1
10.63	13994.6
9.69	14196.0
8.70	12823.5
7.85	11436.2
6.61	9505.7
5.99	8574.8
5.01	6623.5
19.23	91803.1
19.04	88783.1
18.90	86129.4
18.58	78884.7
18.52	78047.5
18.36	72119.5
18.21	68119.0
18.07	63441.8
17.95	58799.6
17.74	52206.4
17.67	49223.3
17.59	46295.3
17.35	41618.4
17.01	37346.6
16.54	32411.0
16.12	29224.1
15.48	25891.6
14.90	23755.9
13.97	20633.4
13.43	19148.5
12.64	17142.7
12.00	16381.3
11.149	14920.9
10.52	13956.0
9.63	13938.6
8.63	12325.7
7.84	11077.7
6.83	9357.8
5.73	7869.8
19.08	87687.1
18.95	85005.1
18.72	80083.1
18.54	73853.9
18.45	71163.1

18.31	67215.6
18.27	64663.6
18.06	56897.8
17.99	55930.6
17.76	49788.2
17.53	45156.3
17.30	40516.9
17.09	37004.0
16.76	34149.3
16.19	29284.3
15.51	25451.6
14.84	21753.2
14.38	20863.4
13.70	19321.7
12.88	17273.9
12.13	15616.9
11.46	15193.7
10.63	13767.7
9.84	14429.7
9.08	12974.0
8.02	11342.2
7.16	10146.2
6.18	8621.0
5.34	7167.0
19.25	91824.0
19.06	87816.6
18.96	83937.5
18.75	80362.6
18.59	76691.3
18.44	70509.9
18.33	67611.1
18.20	63075.5
18.07	58622.3
17.91	54519.7
17.69	49531.2
17.54	46166.8
17.16	39532.2
16.97	37176.3
16.523	32528.1
16.15	29667.3
15.46	25655.2
14.78	23302.4
14.11	20965.2
13.56	19763.9
12.78	17707.7
12.10	16904.5
11.35	15062.3
10.51	13720.9
9.74	14207.6
8.91	12817.9
7.64	11120.5
7.02	9972.6

6.02	8178.3
4.85	6413.4

**Table 4 Number of test days and data points**

Fluid (% mass fraction)	Number of days	Number of data points
Pure R123 $7.5 \text{ K} \leq \Delta T_s \leq 16.5 \text{ K}$	6	100
R123/PARA (99.5/0.5) $9 \text{ K} \leq \Delta T_s \leq 18 \text{ K}$	5	88
R123/PARA (99/1) $10.5 \text{ K} \leq \Delta T_s \leq 18 \text{ K}$	5	84
R123/PARA (98/2) $5 \text{ K} \leq \Delta T_s \leq 20 \text{ K}$	7	158
R123/PARA/PG (97.8/2/0.2) $5 \text{ K} \leq \Delta T_s \leq 19 \text{ K}$	3	52
R123/PARA/LP1 (97.8/2/0.2) $5 \text{ K} \leq \Delta T_s \leq 19.5 \text{ K}$	4	117

**Table 5 Estimated parameters for cubic boiling curve fits for plain copper surface**

$$\Delta T_s = A_0 + A_1 q'' + A_2 q''^2 + A_3 q''^3$$

$\Delta T_s$  in Kelvin and  $q''$  in  $\text{W/m}^2$

Fluid	$A_0$	$A_1$	$A_2$	$A_3$
Pure R123 $6 \text{ K} \leq \Delta T_s \leq 14.5 \text{ K}$ $14.5 \text{ K} \leq \Delta T_s \leq 16.5 \text{ K}$	-13.5834 11.9188	$2.48902 \times 10^{-3}$ $9.93197 \times 10^{-5}$	$-7.70987 \times 10^{-8}$ $-1.13967 \times 10^{-9}$	$8.18448 \times 10^{-13}$ $5.77704 \times 10^{-15}$
R123/PARA (99.5/0.5) $9 \text{ K} \leq \Delta T_s \leq 15.5 \text{ K}$ $14.5 \text{ K} \leq \Delta T_s \leq 18 \text{ K}$	-6.90657 11.7559	$1.60725 \times 10^{-3}$ $1.44116 \times 10^{-4}$	$-3.93167 \times 10^{-8}$ $-1.63992 \times 10^{-9}$	$3.27067 \times 10^{-13}$ $8.78834 \times 10^{-15}$
R123/PARA (99/1) $10.5 \text{ K} \leq \Delta T_s \leq 16 \text{ K}$ $15.5 \text{ K} \leq \Delta T_s \leq 18 \text{ K}$	-1.62317 11.9169	$1.11913 \times 10^{-3}$ $1.68801 \times 10^{-4}$	$-2.31487 \times 10^{-8}$ $-2.05499 \times 10^{-9}$	$1.54542 \times 10^{-13}$ $1.11276 \times 10^{-14}$
R123/PARA (98/2) $5 \text{ K} \leq \Delta T_s \leq 16.5 \text{ K}$ $15 \text{ K} \leq \Delta T_s \leq 20 \text{ K}$	-2.68251 8.98216	$1.04636 \times 10^{-3}$ $3.90307 \times 10^{-4}$	$-8.12573 \times 10^{-9}$ $-6.10689 \times 10^{-9}$	$-1.92917 \times 10^{-13}$ $3.41756 \times 10^{-14}$
R123/PARA/PG (97.8/2/0.2) $5 \text{ K} \leq \Delta T_s \leq 16 \text{ K}$ $15 \text{ K} \leq \Delta T_s \leq 19 \text{ K}$	3.79045 8.90035	$-3.55922 \times 10^{-4}$ $3.95287 \times 10^{-4}$	$8.78467 \times 10^{-8}$ $-6.14104 \times 10^{-9}$	$-2.24414 \times 10^{-12}$ $3.43387 \times 10^{-14}$
R123/PARA/LP1 (97.8/2/0.2) $5 \text{ K} \leq \Delta T_s \leq 17 \text{ K}$ $15 \text{ K} \leq \Delta T_s \leq 19.5 \text{ K}$	-3.67750 8.87760	$1.34398 \times 10^{-3}$ $3.85489 \times 10^{-4}$	$-2.67061 \times 10^{-8}$ $-5.62173 \times 10^{-9}$	$1.38544 \times 10^{-13}$ $2.89907 \times 10^{-14}$

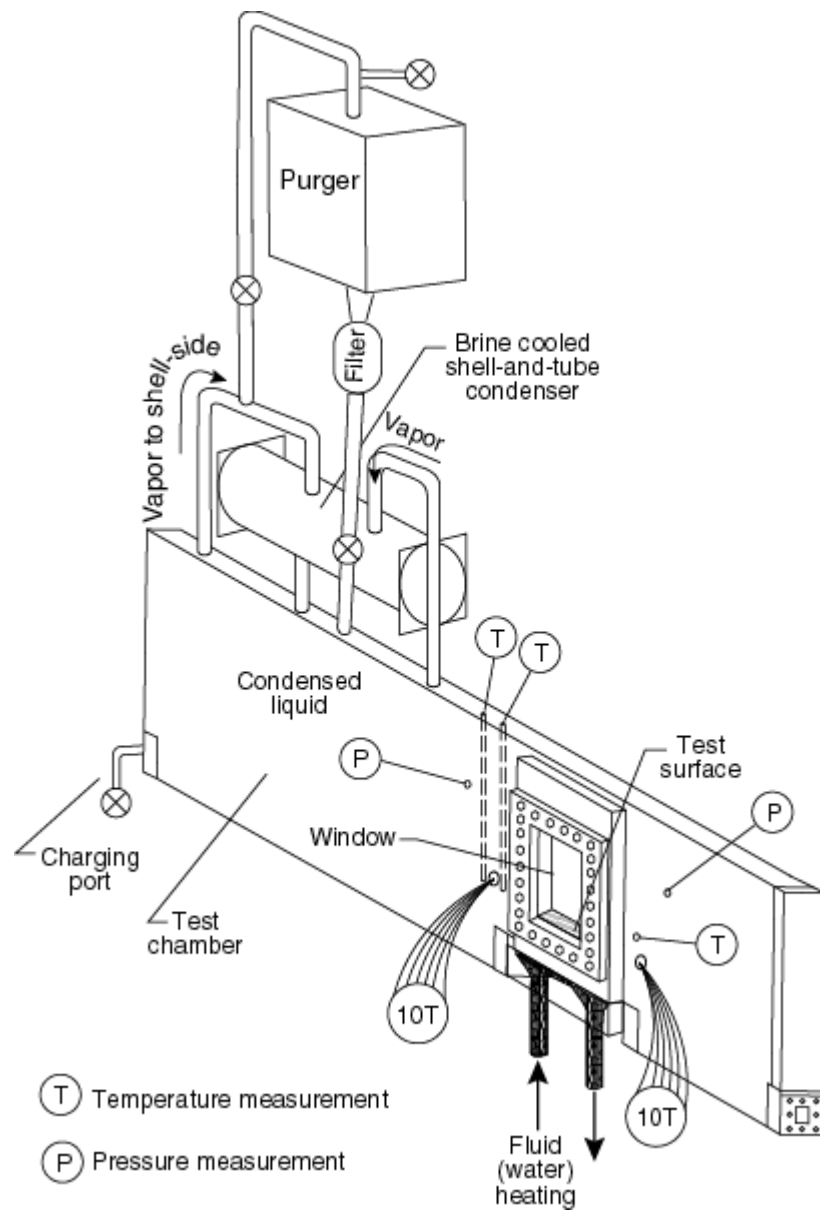
**Table 6 Residual standard deviation of  $\Delta T_s$** 

Fluid	$u$ (K)
PureR123	
6 K $\leq \Delta T_s \leq$ 14.5 K	0.27
14.5 K $\leq \Delta T_s \leq$ 16.5 K	0.05
R123/PARA (99.5/0.5)	
9 K $\leq \Delta T_s \leq$ 15.5 K	0.23
14.5 K $\leq \Delta T_s \leq$ 18 K	0.10
R123/PARA (99/1)	
10.5 K $\leq \Delta T_s \leq$ 16 K	0.21
15.5 K $\leq \Delta T_s \leq$ 18 K	0.11
R123/PARA (98/2)	
5 K $\leq \Delta T_s \leq$ 16.5 K	0.35
15 K $\leq \Delta T_s \leq$ 20 K	0.11
R123/PARA/PG (97.8/2/0.2)	
5 K $\leq \Delta T_s \leq$ 16 K	0.33
15 K $\leq \Delta T_s \leq$ 19 K	0.10
R123/PARA/LP1 (97.8/2/0.2)	
5 K $\leq \Delta T_s \leq$ 17 K	0.38
15 K $\leq \Delta T_s \leq$ 19.5 K	0.13

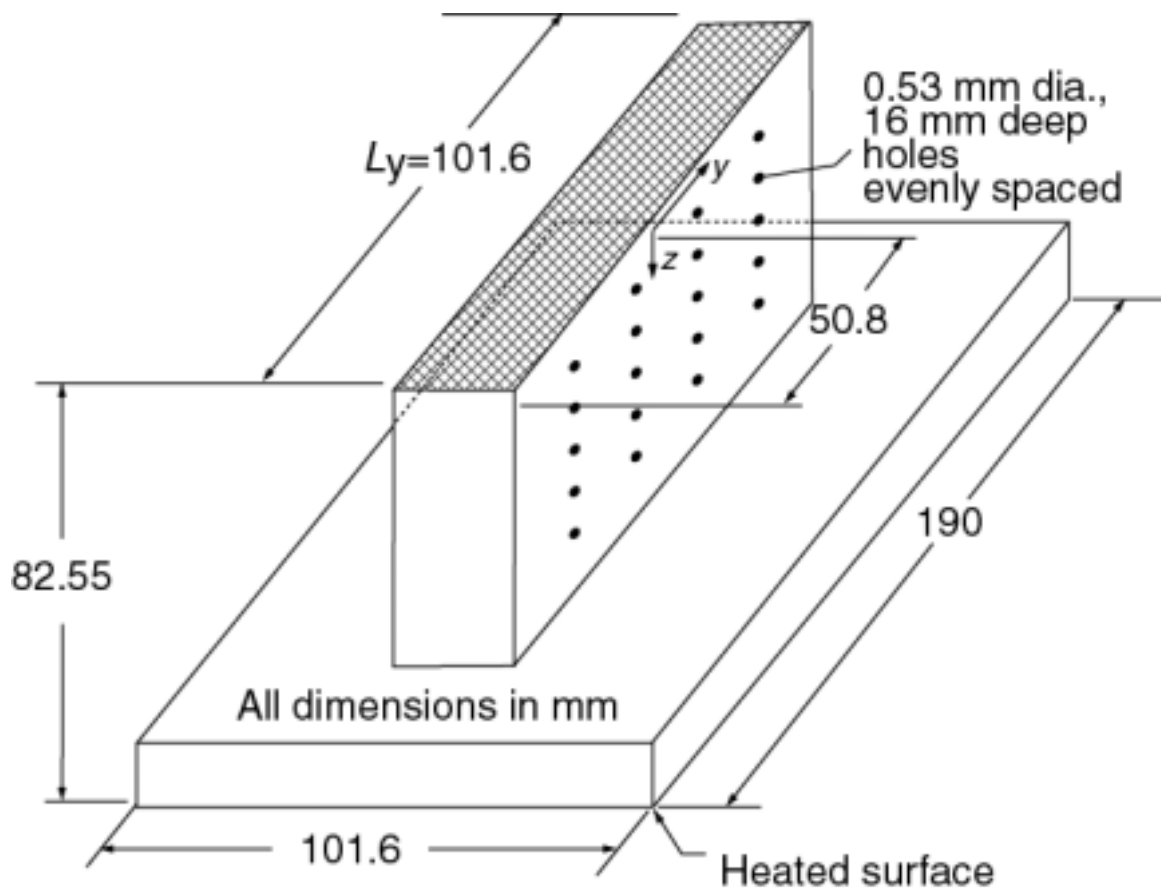


**Table 7** Average magnitude of 95 % multi-use confidence interval for mean  $T_w - T_s$ (K)

Fluid	$u$ (K)
PureR123	
6 K $\leq \Delta T_s \leq$ 14.5 K	0.26
14.5 K $\leq \Delta T_s \leq$ 16.5 K	0.04
R123/PARA (99.5/0.5)	
9 K $\leq \Delta T_s \leq$ 15.5 K	0.24
14.5 K $\leq \Delta T_s \leq$ 18 K	0.09
R123/PARA (99/1)	
10.5 K $\leq \Delta T_s \leq$ 16 K	0.22
15.5 K $\leq \Delta T_s \leq$ 18 K	0.10
R123/PARA (98/2)	
5 K $\leq \Delta T_s \leq$ 16.5 K	0.26
15 K $\leq \Delta T_s \leq$ 20 K	0.08
R123/PARA/PG (97.8/2/0.2)	
5 K $\leq \Delta T_s \leq$ 16 K	0.46
15 K $\leq \Delta T_s \leq$ 19 K	0.14
R123/PARA/LP1 (97.8/2/0.2)	
5 K $\leq \Delta T_s \leq$ 17 K	0.31
15 K $\leq \Delta T_s \leq$ 19.5 K	0.11



**Fig. 1 Schematic of test apparatus**



**Fig. 2 OFHC copper flat test plate with cross-hatched surface and thermocouple coordinate system**

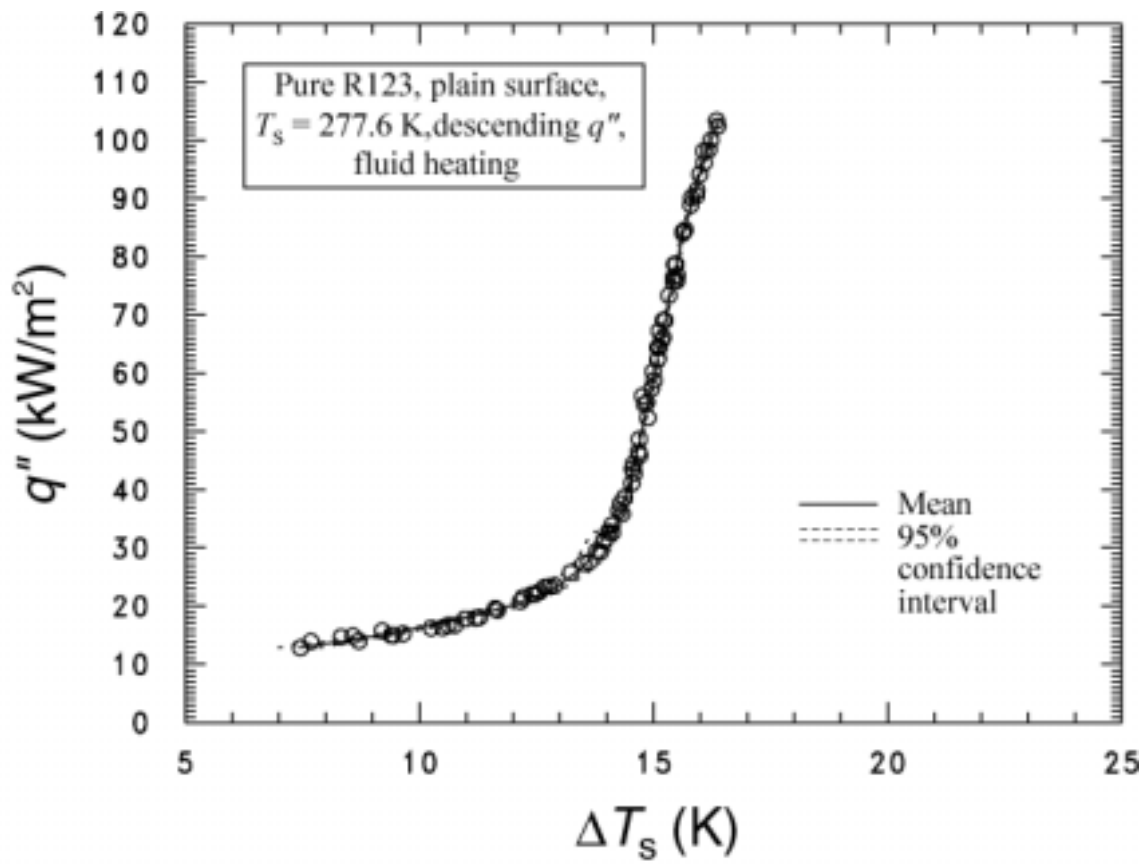


Fig. 3 Pure R123 boiling curve for plain surface

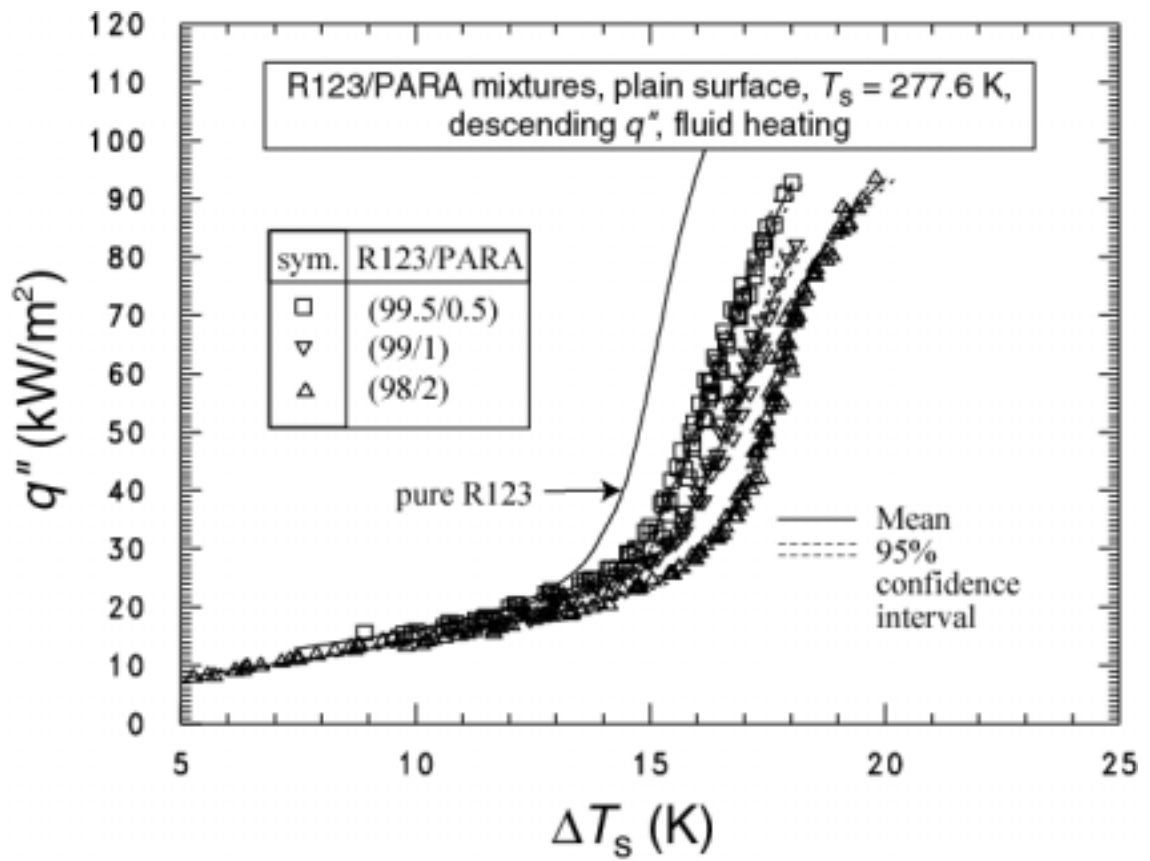


Fig. 4 R134a /PARA mixtures boiling curves for plain surface

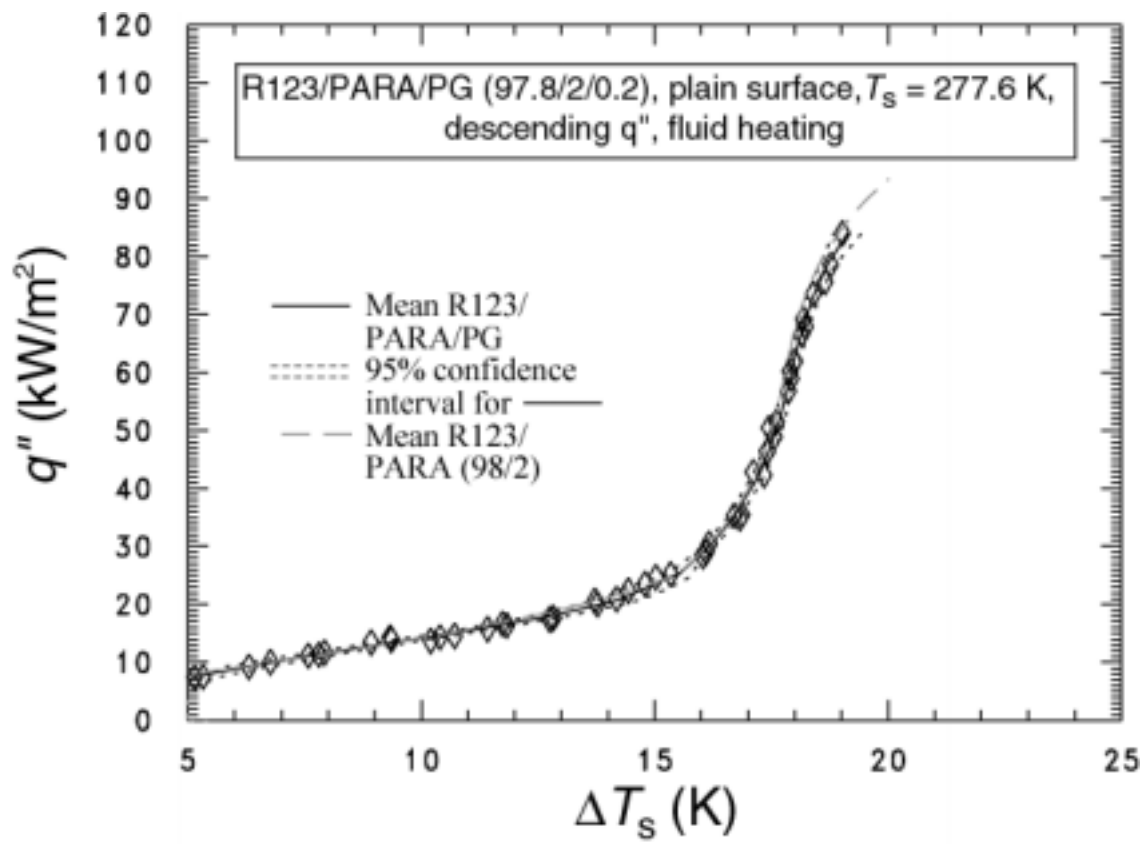


Fig. 5 R123/PARA/PG (97.8/2/0.2) boiling curve for plain surface

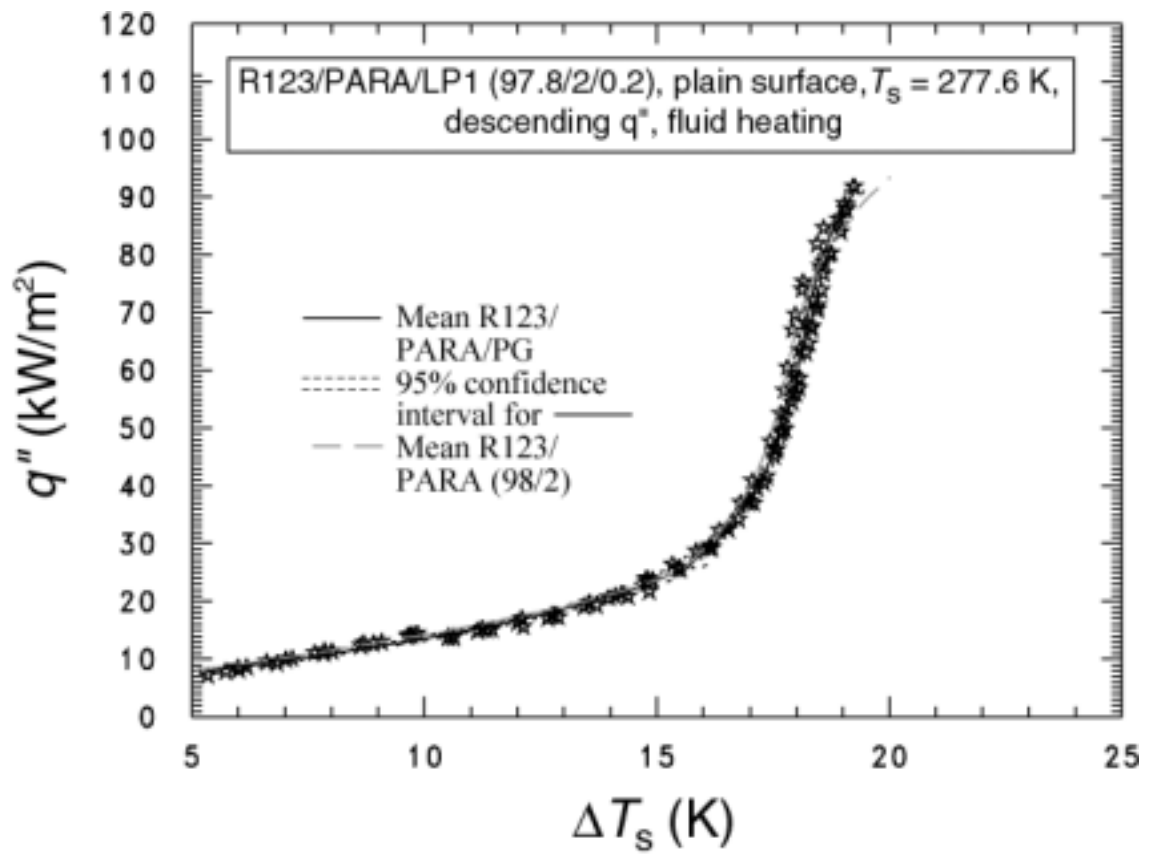
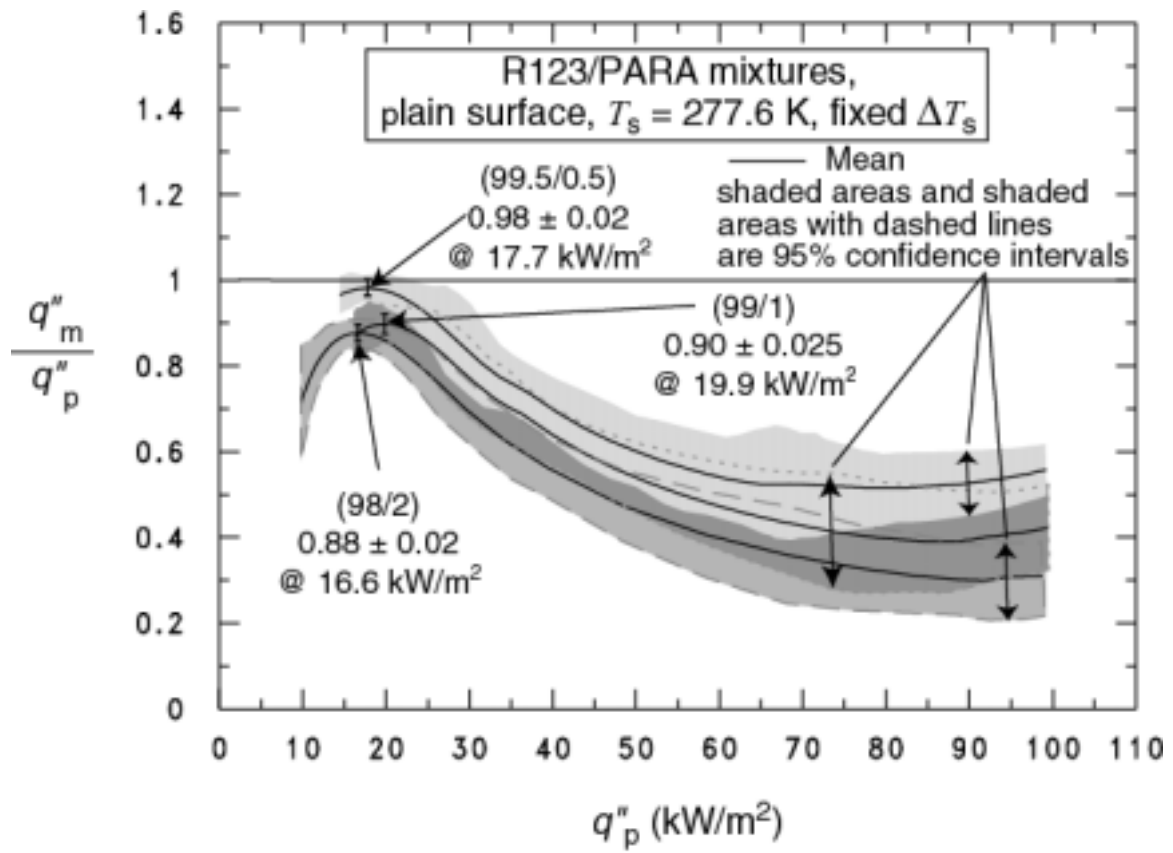


Fig. 6 R123/PARA/LP1 (97.8/2/0.2) boiling curve for plain surface



**Fig. 7 R123/PARA mixture heat fluxes relative to that of pure R123 for a plain surface**



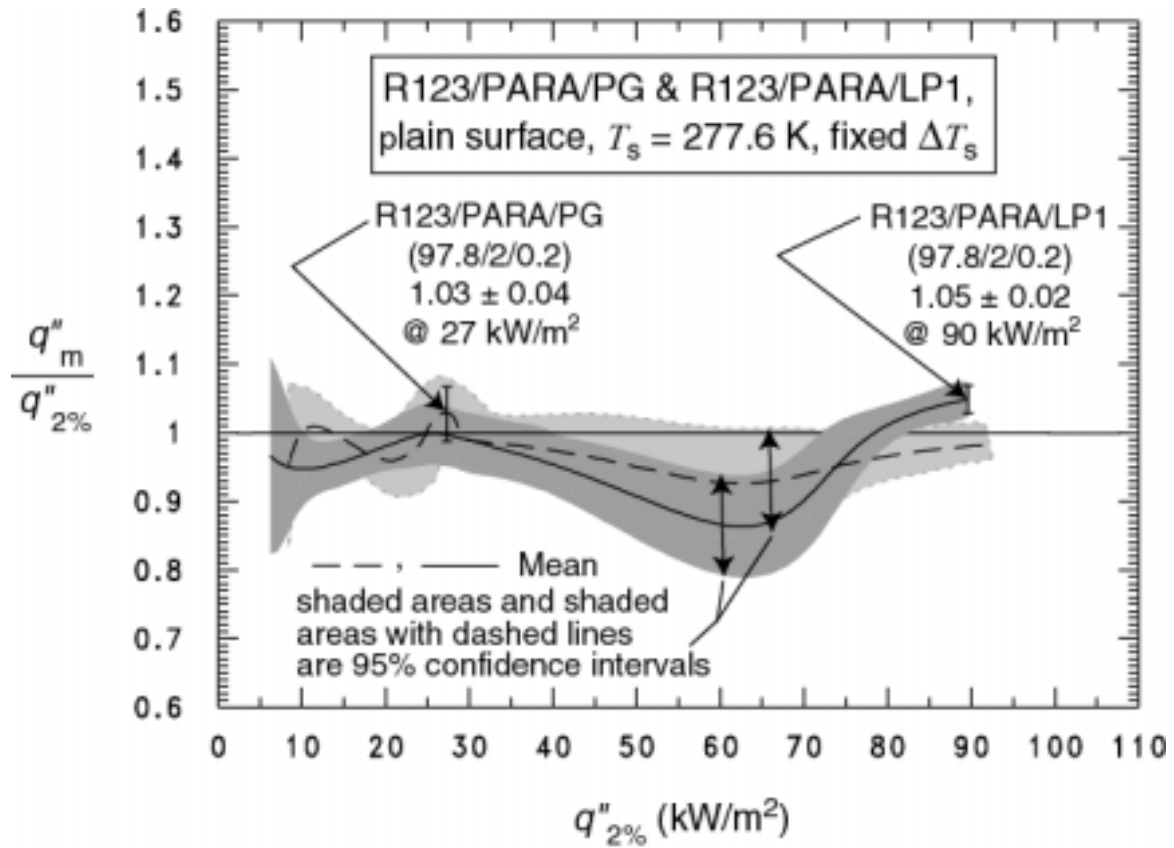


Fig. 8 R123/PARA/PG (97.8/2/0.2) and R123/PARA/LP1 (97.8/2/0.2) mixture heat fluxes relative to that of the R123/PARA (98/2) mixture for a plain surface

## APPENDIX A: ADDITIVE POTENTIAL

This appendix presents the results of a thermodynamic cycle and heat transfer analysis to determine the potential improvement in chiller coefficient of performance (COP) as caused by a given improvement in refrigerant side boiling heat transfer. The design conditions and operating specifications of a representative 1110 kW (315 ton), R11 chiller manufactured in the year 1985 was chosen as the baseline and taken from the manufacturer's catalog data. The entering and exiting sink and source temperatures were taken from design conditions for the chiller and held constant throughout the analysis. The entering and leaving condenser water temperature was 302.6 K and 307.9 K, respectively. The entering and leaving evaporator water temperature was 285.4 K and 279.8 K, respectively. The baseline Log-Mean-Temperature-Difference of the condenser ( $\text{LMTD}_{\text{Bc}}$ ) and that of the evaporator ( $\text{LMTD}_{\text{Be}}$ ) before additive treatment was 5.6 K and 3.4 K, respectively. Figure A.1 taken from Fischer et al. (1994), roughly confirms the representative value of the reported COP for the baseline chiller at approximately 5.65 for the year 1985. The analysis assumed isenthalpic expansion and adiabatic compression at an isentropic efficiency of 0.721. The compressor efficiency was chosen so that the cycle would match the quoted COP of the baseline chiller and was held constant throughout the analysis.

The split of the water-side/refrigerant-side heat transfer resistance for the evaporator of the baseline chiller was found to be 68 %/32 %. In other words, the refrigerant-side temperature difference ( $\Delta T_r$ ) and the water-side temperature difference ( $\Delta T_w$ ) were 68 % and 32 % of the LMTD, respectively:

$$\text{LMTD} = \Delta T_r + \Delta T_w = 0.32\text{LMTD}_B + 0.68\text{LMTD}_B \quad (\text{A.1})$$

The percent increase in the refrigerant-side heat transfer coefficient ( $E_h$ ) was used to reduce the magnitude of the refrigerant-side heat transfer resistance while maintaining the magnitude of the original baseline water-side resistance. Hence, the improved LMTD for the evaporator was calculated as:

$$\text{LMTD}_e = \left( \frac{0.32}{1 + \frac{E_h}{100}} + 0.68 \right) \text{LMTD}_B \quad (\text{A.2})$$

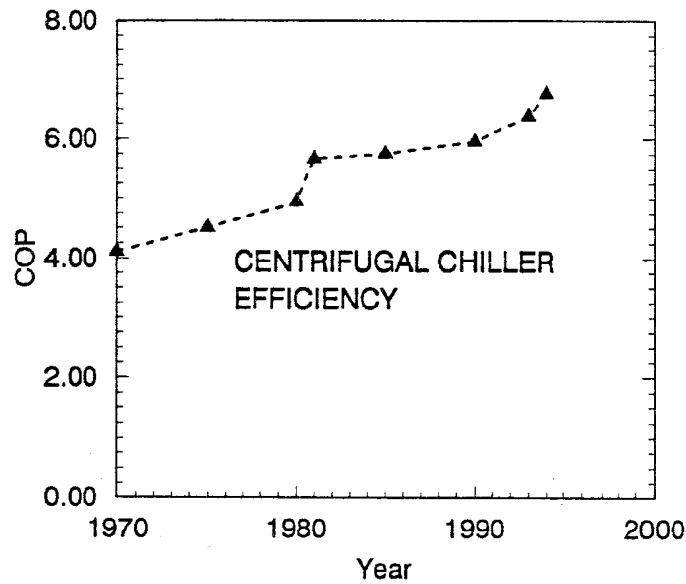
Kedzierski (2000b) has shown that a boiling additive can degrade the condensation heat transfer by as much as 10 %. Based on this, it was assumed that the additive caused a 10 % reduction in the refrigerant-side heat transfer coefficient in the condenser. The LMTD for the condenser was calculated as:

$$\text{LMTD}_c = 0.68\text{LMTD}_{\text{Bc}} + \frac{0.32}{0.9}\text{LMTD}_{\text{Bc}} \approx 1.04\text{LMTD}_{\text{Bc}} \quad (\text{A.3})$$

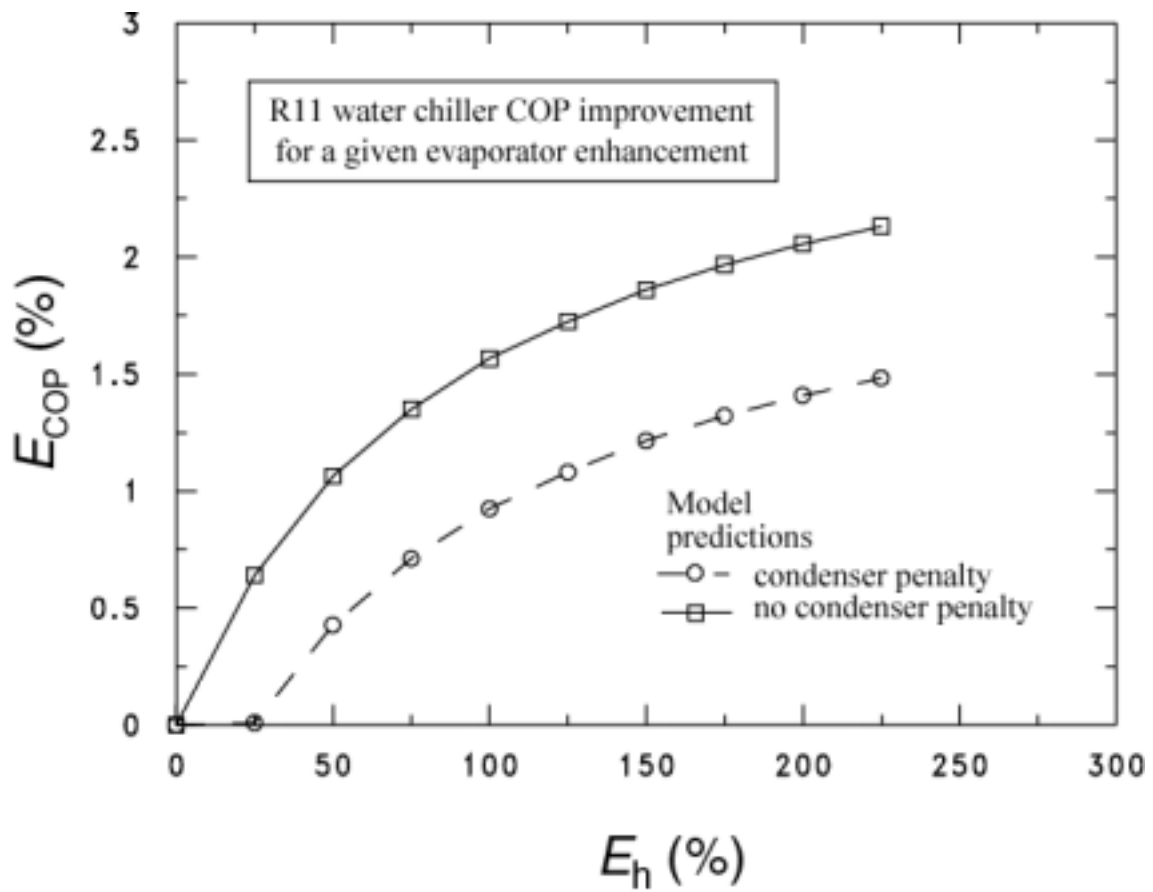
Figure A.2 shows the percent increase in the COP ( $E_{\text{COP}}$ ) for a given  $E_h$  and the assumptions of the analysis. Because of the heat transfer degradation in the condenser,

performance improvements are not realized until the evaporator heat transfer enhanced by more than 25 %. The figure also shows that if the boiling performance of a representative 1985 chiller could be improved by 100 %, the chiller would realize an improvement of approximately 1 % in the COP. The chiller COP would be improved by approximately 1.5 % for an  $E_h$  of 225 %.

Figure A.2 also shows the  $E_h$  for the case where there is no condenser penalty for heat transfer degradation as caused by the additive. A 10 % reduction in the refrigerant-side heat transfer coefficient in the condenser has caused approximately a 0.7 % loss in COP for  $E_h$  between 25 % and 275 %.



**Fig. A.1 Efficiency of centrifugal chillers at industry rating conditions (Fischer et al., 1994)**



**Fig. A.2 R11 water chiller COP improvement for a given evaporator enhancement**

## APPENDIX B: UNCERTAINTIES

Figure B.1 shows the relative (percent) uncertainty of the heat flux ( $U_{q''}$ ) as a function of the heat flux. Figure B.2 shows the uncertainty of the wall temperature as a function of heat flux. The uncertainties shown in Figs. B.1 and B.2 are "within-run uncertainties." These do not include the uncertainties due to "between-run effects" or differences observed between tests taken on different days. The "within-run uncertainties" include only the random effects and uncertainties associated with one particular test. All other uncertainties reported in this study are "between-run uncertainties" which include all random effects such as surface past history or seeding.

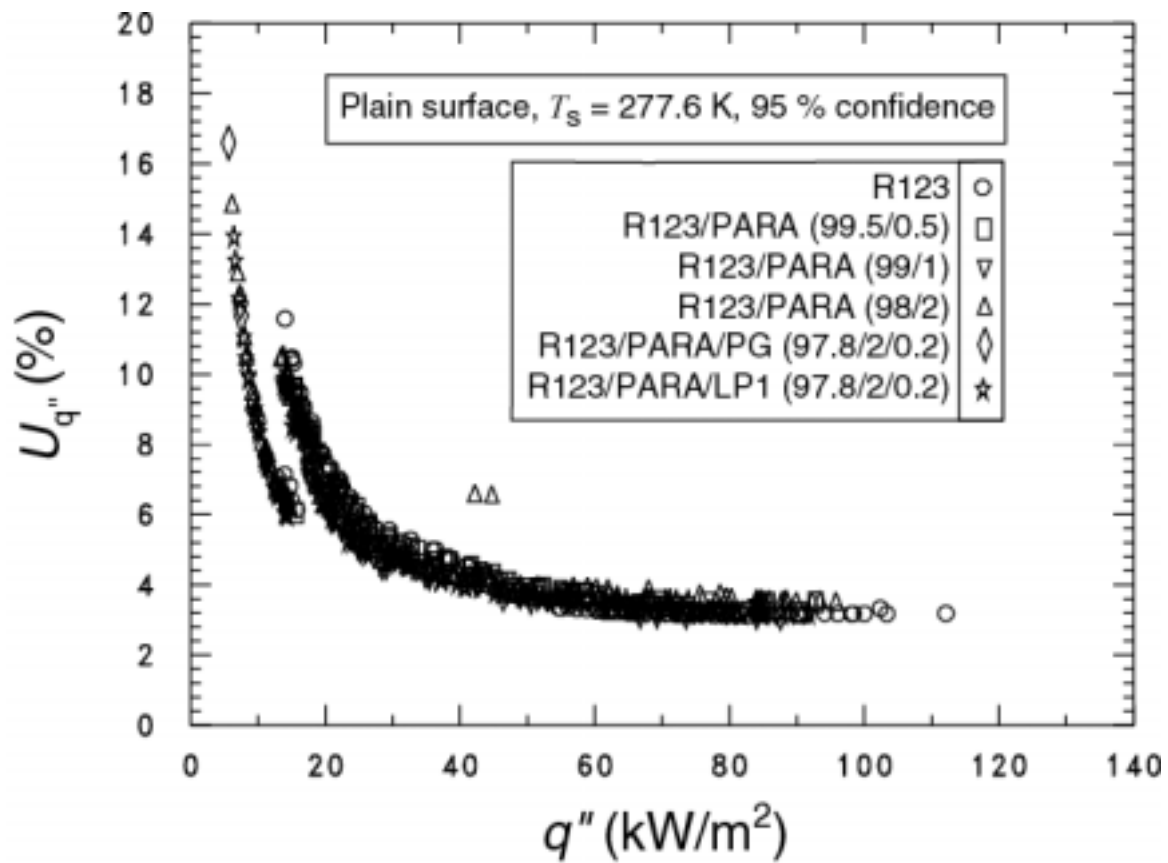
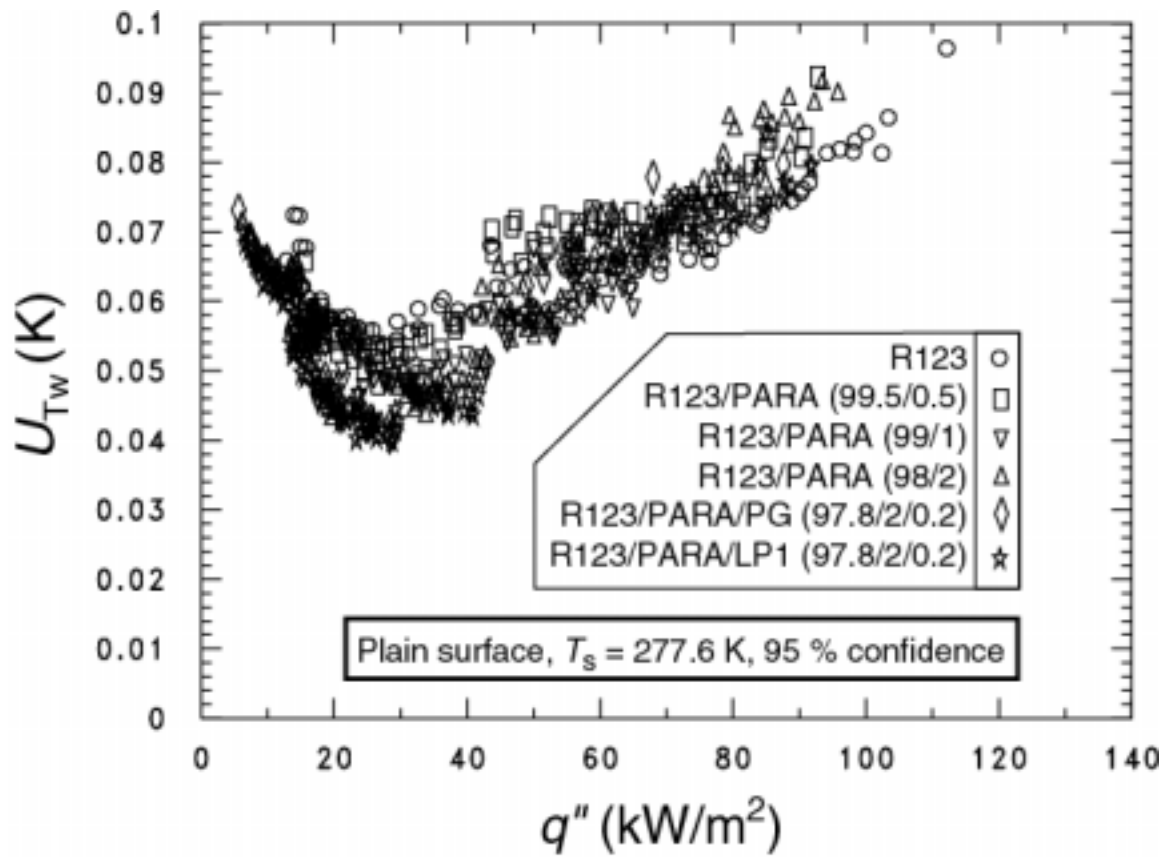


Fig. B.1 Expanded relative uncertainty in the heat flux of the surface at the 95 % confidence level



**Fig. B.2 Expanded uncertainty in the temperature of the surface at the 95 % confidence level**

## APPENDIX C: LUBRICANT LIQUID DENSITY MEASUREMENTS

This appendix presents the measurements and the correlation of the PARA, PG, and LP1 lubricant liquid densities ( $\rho_l$ ). The density of the liquid lubricant was measured as a function of temperature with a glass pycnometer. The pycnometer was factory instrumented with a glass mercury thermometer with a range of 14 °C to 38 °C in 0.2 ° graduations, accurate to within  $\pm 0.2$  K. The pycnometer was filled with distilled water and its volume was calculated from the known density of water. The volume was found with 29 trials to be 9.588 ml with a standard uncertainty of 0.002 ml. The average mass of the pycnometer after nine trials was  $28.794 \text{ g} \pm 0.001 \text{ g}$ .

The pycnometer containing the test lubricant was cooled in an ice bath and then removed from the bath and allowed to warm on the balance to room temperature over approximately one hour. The standard uncertainty of the balance was approximately 1 mg. The outside of the pycnometer was wiped clean before each measurement to remove the lubricant that was expelled through the pipette due to volume expansion with temperature increase.

The Biot number for the warming pycnometer was estimated to be approximately 0.5, which is greater than the recommended limit of 0.1 (Incropera and Dewitt, 1985) for a uniform temperature in fluid. It is difficult to estimate the error introduced in the measurements due to temperature gradients that existed in the lubricant. However, the data regression shows that the residuals are independent of temperature, which suggests that the error due to temperature gradients in the liquid had a negligible effect on the density measurements.

Tables C.1 through C.3 shows the recorded measurements for the three test lubricants. Equation C.1 gives the fit of the liquid lubricant density ( $\rho_l$ ) in  $\text{kg/m}^3$  versus temperature ( $T$ ) in Kelvin:

$$\rho_l = a_0 - a_1 T \quad \text{C.1}$$

Where the coefficients for each lubricant are given in Table C.4. The expanded uncertainty of the fits were approximately  $\pm 1 \text{ kg/m}^3$  for 95 % confidence.

**Table C.1 PARA liquid density measurements**

$T$ (°C)	Lubricant mass (g)	$\rho_l$ (kg/m <sup>3</sup> )
14.6	8.466	882.96
17.2	8.449	881.19
18	8.446	880.88
19	8.441	880.36
20.2	8.436	879.83
21.4	8.428	879.00
22	8.424	878.58
22.8	8.418	877.96
23.6	8.414	877.54
24.2	8.411	877.23
15.2	8.452	881.50
18.2	8.442	880.46
19.2	8.439	880.15
20.4	8.433	879.52
21	8.429	879.10
21.8	8.426	878.79
22.6	8.419	878.06
23.2	8.417	877.85
24	8.411	877.23
14.8	8.453	881.61
17.6	8.447	880.98
18.6	8.441	880.36
19.8	8.435	879.73
20.8	8.429	879.10
21.8	8.422	878.37

**Table C.2 PG liquid density measurements**

$T$ (°C)	Lubricant mass (g)	$\rho_l$ (kg/m <sup>3</sup> )
14.2	9.609	997.38
15.8	9.604	996.44
16.8	9.595	995.71
17.8	9.589	995.29
18.8	9.585	994.77
19.8	9.573	994.36
20.8	9.567	993.73
21.8	9.56	993.21
22.8	9.553	997.48
23.6	9.549	996.96
24.2	9.543	996.34
14.6	9.599	995.82
16.8	9.592	995.40
18	9.59	994.98
18.8	9.584	994.25
19.6	9.579	993.73
20.4	9.575	993.10



21.2	9.571	997.38
21.8	9.567	996.96
22.4	9.56	996.44
23.2	9.556	995.82
24.2	9.548	995.19
15	9.6	994.77
16	9.595	994.36
18.2	9.588	993.63
19.2	9.581	993.10
20	9.578	997.38
21	9.572	996.44
21.8	9.566	995.71
22.8	9.559	995.29
23.6	9.552	994.77
24.4	9.55	994.36
14	9.618	993.73
15	9.611	993.21
15.8	9.602	997.48
16.6	9.595	996.96
17.4	9.591	996.34
18.2	9.584	995.82
19.4	9.578	995. 40
20.4	9.571	994.98
21.2	9.566	994.25
22	9.562	993.73
23	9.554	993.10
24	9.55	997.38
24.6	9.544	996.96

**Table C.3 LP1 Liquid Density Measurements**

$T$ (°C)	Lubricant mass (g)	$\rho_l$ (kg/m <sup>3</sup> )
14.2	8.89	927.19
16.4	8.88	926.15
17.2	8.871	925.21
17.6	8.866	924.69
18	8.86	924.06
18.8	8.854	923.43
19.8	8.849	922.91
21.4	8.841	922.08
22	8.838	921.77
22.6	8.832	921.14
23.4	8.828	920.72
24.2	8.822	920.10
15.2	8.875	925.62
16.2	8.87	925.10
16.8	8.866	924.69
17.8	8.862	924.27
19	8.853	923.33
19.6	8.848	922.81
20.4	8.846	922.61
21.6	8.836	921.56
22.6	8.833	921.24
23.4	8.828	920.72
24.2	8.823	920.20
14.4	8.873	925.42
15.6	8.868	924.89
16.6	8.866	924.69
17.6	8.859	923.96
18.4	8.853	923.33
19.4	8.848	922.81
20.4	8.842	922.18
21.6	8.835	921.45
22.4	8.831	921.04
23.4	8.825	920.41
24.2	8.822	920.10

**Table C.4 Density coefficients for eq. C.1**

Lubricant	$a_0$ (kg/m <sup>3</sup> )	$a_1$ (kg m <sup>-3</sup> K <sup>-1</sup> )
Paraffinic Mineral oil (PARA)	1038.76	-0.5431
DGLF118 (PG)	1189.67	-0.6519
LP1	1107.84	-0.6314

## APPENDIX D: CAPILLARY RISE MEASUREMENTS

This appendix presents capillary rise measurements for PARA, PG and LP1 at approximately 25 °C. Tables D.1 through D.3 provide the capillary rise height measurements ( $l$ ) that were used to calculate the surface tension for the lubricant and the additive. The first column for each fluid gives the height of the liquid in the tube after it had been removed from the liquid pool while placing a finger over the tube opening. The second column for each liquid gives the rise height by subtracting off the height of the pool ( $d$ ) from the first column measurements. The standard deviation of the mean measurement for this method was approximately 0.5 % of the measurement. The pool height was kept small so that if a 100 % error had occurred in the measurement of the pool depth it would contribute only approximately 10 % to the measurement of the capillary rise-height.

A force balance on the column of liquid in the capillary tube was used to calculate the surface tension (Adamson and Gast, 1997):

$$\sigma = \frac{r\Delta\rho gl}{2} \approx \frac{r\rho_l gl}{2} \quad (\text{D.1})$$

where the measured radius of the capillary tube ( $r$ ) was 0.97 mm with a B-type estimated uncertainty of  $\pm 0.03$  mm. The liquid densities ( $\rho_l$ ) measurements are given in Appendix C. The uncertainty of the density measurements is approximately  $\pm 1$  kg/m<sup>3</sup>.

The liquid-vapor (air) surface tensions as calculated from eq (D.1) for the PARA, PG and LP1 were 0.0306 N/m  $\pm$  0.0007 N/m, 0.0322 N/m  $\pm$  0.0007 N/m and, 0.0317 N/m  $\pm$  0.0007 N/m, respectively.

**Table D.1 Capillary rise measurements for LP1**

$T$ (°C)	$h + d$ (mm)	$h$ (mm)	$\rho$ (kg m <sup>-3</sup> )	$\sigma$ (N m <sup>-1</sup> )
24.73	29	27	919.75	0.03171
24.73	29	27	919.75	0.03171
24.73	29	27	919.75	0.03171
24.72	29	27	919.75	0.03171
24.71	29	27	919.76	0.03171
24.63	29	27	919.81	0.03171

**Table D.2 Capillary rise measurements for PG**

$T$ (°C)	$h + d$ (mm)	$h$ (mm)	$\rho$ (kg m <sup>-3</sup> )	$\sigma$ (N m <sup>-1</sup> )
24.42	64	26	995.68	0.03306
24.402	63.5	25.5	995.69	0.03242
24.39	63.5	25.5	995.70	0.03242
24.37	63.5	25.5	995.71	0.03242
24.35	63.5	25.5	995.72	0.03242
24.33	63.5	25.5	995.73	0.03242
24.40	63.5	25.5	995.69	0.03242
24.40	63.5	25.5	995.69	0.03242
24.38	60.5	25.5	995.70	0.03242
24.37	60.5	25.5	995.71	0.03242
24.36	60.5	25.5	995.72	0.03242
24.24	60.5	25.5	995.79	0.03242
24.88	60.5	25.5	995.37	0.03241
24.89	48	25	995.37	0.03177
24.89	48.5	25.5	995.37	0.03241
24.83	48	25	995.411	0.03178
24.82	48	25	995.41	0.03178
24.62	48	25	995.54	0.03178
24.08	38	25	995.91	0.03179
24.09	38	25	995.89	0.03179

**Table D.3 Capillary rise measurements for PARA**

$T$ (°C)	$h + d$ (mm)	$h$ (mm)	$\rho$ (kg m <sup>-3</sup> )	$\sigma$ (N m <sup>-1</sup> )
24.63	63.5	27.5	877.03	0.03080
24.62	63.5	27.5	877.04	0.03080
24.61	63.5	27.5	877.04	0.03080
24.60	63.5	27.5	877.05	0.03080
24.60	63.5	27	877.05	0.03024
24.56	63.5	27	877.07	0.03024
24.47	63.5	27	877.12	0.03024
24.46	56	27.5	877.13	0.03080

## APPENDIX E: LUBRICANT VISCOSITY MEASUREMENTS

This appendix presents the liquid kinematic viscosity measurements for PARA, PG and LP1 at approximately 25 °C in Tables E.1 through E.3. Glass viscometers at the appropriate viscosity range were used to measure the viscosities of each lubricant. The B-type uncertainty of the viscometer was quoted by the manufacturer as 0.22 % of the measurement for the PARA lubricant and 0.29 % of the measurement for the PG lubricant for a 95 % confidence level. The average measured viscosity of the PARA, and PG lubricants, for the 95 % confidence level, was  $85 \mu\text{m}^2/\text{s} \pm 1.5 \mu\text{m}^2/\text{s}$ , and  $150.3 \mu\text{m}^2/\text{s} \pm 0.8 \mu\text{m}^2/\text{s}$ , respectively. Only a single viscosity measurement of approximately  $3800 \mu\text{m}^2/\text{s}$  was made for the LP1 lubricant because of its extremely large viscosity requiring over four hours. As a result, the viscosity quoted by the manufacturer on the MSDS of  $2000 \mu\text{m}^2/\text{s}$  was taken as the value for LP1.

A model 300 viscometer with a viscosity range of  $50 \mu\text{m}^2/\text{s}$  to  $250 \mu\text{m}^2/\text{s}$  was used. The calibration constant ( $C_o$ ) for the viscosity was a function of viscometer/liquid temperature:

$$C_o = 0.24317 - 1.6667 \times 10^{-5} T \quad (\text{D.1})$$

where  $T$  is in Celsius and  $C_o$  is in  $\mu\text{m}^2/\text{s}$ .

The kinematic viscosity in  $\mu\text{m}^2/\text{s}$  was obtained by multiplying the measured efflux time in seconds by  $C_o$ .

**Table E.1 Kinematic viscosity measurements for PARA**

$T$ (°C)	Efflux time (s)	$C_o$ ( $\mu\text{m}^2/\text{s}$ )	$\nu$ ( $\mu\text{m}^2/\text{s}$ )
24.648	349.55	0.242755867	84.85
24.655	346.65	0.24275575	84.15
24.645	352.05	0.242755917	85.46
24.637	352.4	0.24275605	85.54
24.635	352.1	0.242756083	85.47
24.629	353.59	0.242756183	85.83

**Table E.2 Kinematic viscosity measurements for PG**

$T$ (°C)	Efflux time (s)	$C_o$ ( $\mu\text{m}^2/\text{s}$ )	$\nu$ ( $\mu\text{m}^2/\text{s}$ )
24.579	617.59	0.242757017	149.92
24.586	599.82	0.2427569	145.61
24.569	618.36	0.242757183	150.11
24.562	620.5	0.2427573	150.63
24.554	620.78	0.242757433	150.69
24.545	619.61	0.242757583	150.41

**Table E.3 Kinematic viscosity measurements for LP1**

$T$ (°C)	Efflux time (s)	$C_o$ ( $\mu\text{m}^2/\text{s}$ )	$\nu$ ( $\mu\text{m}^2/\text{s}$ )
24.81	15633.25	0.242753167	3795.