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Minimum Life Cycle Cost Heat Losses for Shallow Trench Underground Heat Distribution Systems

Jin B. Fang

U.S. DEPARTMENT OF COMMERCE National Bureau of Standards National Engineering Laboratory Center for Building Technology Building Physics Division Gaithersburg, MD 20899

May 1986

Prepared for: Tri-Service Building Materials Committee

Headquarters, U.S. Army Corps of Engineers Washington, DC 20314-1000

U.S. Navy, Naval Facilities Engineering Command Alexandria, VA 22332-2300

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U.S. DEPARTMENT OF COMMERCE, Malcolm Baldrige, Secretary NATIONAL BUREAU OF STANDARDS, Ernest Ambler, Director NBS Research Information Center

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ABSTRACT

The rates of heat loss from two underground insulated pipes installed in a shallow trench were calculated using a finite element method computer program to solve a two-dimensional steady-state heat conduction problem. The results of pipe heat loss study based on a specified ground temperature condition are summarized for a range of pipe insulation thicknesses, shallow trench sizes, and pipe fluid temperatures. Methods of determining the heat loss associated with the minimum life-cycle cost and the corresponding economic insulation thickness for shallow trench heat distribution systems are presented. Life-cycle costing analysis was performed for two insulated pipes in a concrete trench to determine the cost of construction, annual energy cost associated with pipe heat loss, and yearly operating and maintenance costs. Based on this economic analysis, the least life-cycle cost heat loss and the optimum insulation thickness were determined for specified fluid temperatures in the heat supply and the return lines.

Keywords: finite element method, fuel energy cost, heat loss, life-cycle cost analysis, shallow trench, underground heat distribution system.

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1. Introduction

Underground piping systems distribute hot and chilled water or steam through a network of insulated pipes to serve military installations for space heating and cooling, and industrial purposes. These pipe-lines can be directly buried in the ground, installed in a concrete shallow trench, or placed above the ground. Recent studies [1,2] compared the difference in life cycle costs between direct buried and shallow trench heat distribution systems based on the results of field surveys of existing systems, and estimates of capital cost, maintenance and operating expenses. Based on both studies, it is difficult to draw any conclusions on cost comparison between these two systems, since their findings are discrepant with each other, especially for systems with large diameter pipes. However, continued construction of concrete shallow trench systems as an alternative to the direct buried conduit systems is anticipated due to their reduced maintenance and repair costs although they are minor components of the life-cycle cost.

Engineering projects involving substantial expenditures require economically sound investment decisions based on choosing the best alternative. Economic analysis plays an important role in virtually every engineering project, and can identify and evaluate the economic outcome of a proposed project. Life cycle costing is a useful tool in which the individual costs are determined for the expected life of the project, and the total cost is compared with other alternatives. This approach provides the financial justification for selecting the final design and size for a new underground heat distribution system.

It is well known that the heat loss from piping decreases with an increase in insulation thickness, however a point is reached when further reduction in heat loss becomes uneconomical. In this period of high fuel cost, the conservation of energy through the use of optimal insulation thickness corresponding to the minimum life cycle cost heat loss has obvious benefits to reduce fuel energy and maintenance expenditures in operating the heat distribution system.

This report describes the methodologies and procedures used to estimate the energy cost associated with the pipe heat loss and the minimum life cycle cost heat loss from various underground shallow trench heat distribution systems. It presents cost data for materials and installation for acquisition, and maintenance and repair costs for some selected trench systems. The heat loss data are prepared for various thicknesses of pipe insulation, for different sizes of shallow trench systems, and for a range of pipe fluid temperatures.

2. System Heat Loss Calculations

2.1 The Equations Used for Numerical Solution

Finite element methods have been developed to a high level of refinement and have made a great impact in structural mechanics. Such methods are also applied to steady and transient heat conduction problems [3-5].

The heat loss per unit length from a shallow trench or a loose-fill insulation underground heat distribution system can be calculated using the computer simulation program developed recently by Kusuda [6]. This program applies the finite element method to a steady-state, two-dimensional shallow trench system consisting of a heat supply and a return pipe having different fluid temperatures for the pipe sizes shown in Figure 1. The program contains a predesigned finite-element mesh with 130 triangular elements and 80 nodal points. Figure 2 shows the triangular element mesh for the outer boundary earth region, and for the concrete trench walls and cover. Figure 3 shows the mesh for the air space between the insulated pipes and the trench walls, and Figure 4 illustrates the mesh for the pipe insulation. Prescribed boundary temperatures include the nodal points around the outer surfaces of the circular pipes (nodes 49 through 64 in Figure 4) and the perimeter of the earth zone (nodes 65 through 80 in Figure 2). The remaining nodal point temperatures are calculated by solving the system of simultaneous equations using the Gaussian elimination method.

The undisturbed earth temperature used as the prescribed temperature boundary conditions, is a function of time and depth below the ground surface and can be determined by the following eqution [7]:

$$T = T_{a} + T_{b} \exp(-y \sqrt{\frac{w}{2\alpha}}) \sin[2\pi (t-3)/12 - y \sqrt{\frac{w}{2\alpha}}]$$
(1)

where T = the monthly average earth temperature, oC (oF)

 T_a = the annual average earth temperture of the site, oC (oF) = the annual amplitude of the monthly average temperature cycle, ТЪ OC (OF) y = depth from the ground surface, m (ft)

- w = angular frequency of the annual cycle, rad/h
- α = thermal diffusivity of the soil, m²/h (ft²/h)
- t = the elapsed time from January, in months

The rate of heat flow by natural convection through the airspace bounded by the trench walls and the outer surfaces of the insulated piping can be approximated by an equivalent heat conduction of the form [8,9]:

$$Q/A = k_e (T_h - T_c)/L$$
 (2)

where Q = the average heat flow rate, W (Btu/h)

A = the mean cross-sectional area of the enclosed air layer, m^2 (ft^2)

 k_{ρ} = the effective thermal conductivity of the enclosed air layer, W/m·K (Btu/h·ft·^oF)

 T_{h} = the temperature of the hot surface, ^oC (oF)

 T_{f} = the temperature of the cold surface, ^oC (^oF) L = the thickness of air layer, m (ft)

An effective thermal conductivity is used to modify the simple conduction solution to account for convection. The effect of radiant exchange between the pipes and trench walls may be negligible due to low emissivity of aluminum jacket surface. The ratio of the effective to the actual thermal conductivity of the enclosed airspace, k_e/k , is a function of the Rayleigh number based on the characteristic dimension (the thickness) of the air

layer and on the temperature difference of the hot and cold surfaces [8]. The effective thermal conductivity is also a function of the product of the convective heat transfer coefficient and the characteristic dimension. It is noted that if the natural convection is suppressed and the heat is transferred through the air layer only by heat conduction, the effective to the actual thermal conductivity ratio is unity. No experimental data or correlations are available on natural convection in air confined between a pair of heated pipes and cooler enclosure walls. However, correlations for free convection through plane or cylindrical air layers in enclosures from a hot to a cold surface is the closest configuration available [8,9].

Convective heat transfer between the ground surface in the vicinity of the shallow trench and the ambient air is dependent upon the local climate and surface conditions. The surface resistance to convective transfer may be considered as being contained in an equivalent soil layer. The thickness of this fictitious layer represents the amount by which the earth boundary is extended to account for the effect of the film resistance due to convection, and can be determined by

$$L_{c} = \frac{k_{G}}{h}$$
(3)

where L_c = the thickness of the equivalent soil layer, m (ft) k_G = the thermal conductivity of the soil, W/m K (Btu/h ft $^{\circ}F$) h = surface heat transfer coefficient, W/m² K (Btu/h ft $^{\circ}F$)

The heat loss rate per unit length of piping is obtained using the following equation along with the calculated value of average temperature drop across a circular cylindrical shell of pipe insulation layers:

$$q = \frac{2\pi k_{I}(T_{i}-T_{o})}{\ln(r_{o}/r_{i})}$$
(4)

where q = the heat loss rate per unit length of the insulated underground pipe, W/m (Btu/h·ft)

- k_T = the thermal conductivity of insulation materials, W/m·K (Btu/h ft F)
- $r_0 =$ outside radius of the insulation layer, m (ft)
- $r_i = inside radius of the insulation layer, m (ft)$ $<math>T_i and T_o = the surface temperature of the insulation layer at inner$ $and outer radii <math>r_i$ and r_o , respectively, ^oC (^oF)

For an insulated piping system, the surface film resistance between the hot water and the pipe, and the thermal resistance of the pipe wall can generally be ignored in comparison to the thermal resistance of the insulation.

2.2 Description of the Computer Program

The computer program, which is written in FORTRAN language, consists of a main program (FEUHDS) and seven subroutines called PIPE2, TGO, TWOPIP, SOILK, TGXX, SOLVP and PIPEHL. Input data are read in by the main program, and subroutines PIPE2 and TGO. Output is provided by the main, PIPE2,

TWOPIP, and PIPEHL programs. In addition to handling portions of input data, the main program coordinates and performs most of the calculations. Subroutines PIPE2 is used to calculate rectangular coordinates for each nodal point based on the shallow trench and piping geometry, while subroutine TGO calculates the average undisturbed earth temperatures at various depths for the month of interest. Subprogram TWOPIP called from the main program determines the rate of heat loss from two insulated buried pipes to the underground. Subroutine SOILK provides soil thermal conductivity values for various earth temperatures by linear interpolation of a set of soil thermal conductivity versus temperature data, and TGXX furnishes the external boundary temperatures of the outer earth region surrounding the shallow trench system. Subroutine SOLVP is used to solve system of simultaneous equations by Gauss elimination method, and PIPEHL computes the temperature drops across the pipe insulation layers and the heat loss rates for both underground pipes. A listing of the source code of this computer program is given in Appendix B.

2.3 Preparation of Input Data

Two input data files, DATA1 and DATA2, are created prior to execution of the computer simulation program FEUHDS. The DATA1 file shown in Appendix A.1 contains data for run control parameters, the month of interest, the thermal conductivity and dimensions of the trench walls, convection conditions for the trench enclosed air space, the pipe fluid temperatures, the thermal conductivities and dimensions of the carrier pipes and insulation layers, the thermal properties and dimensions of the earth region surrounding the trench system, and the annual average temperature and amplitude of the monthly temperature cycle for the site involved. The DATA2 file shown in Appendix A.2 consists of data for the finite-elements including the element number, the node numbers for its three vertices, and the surface convection coefficients and ambient temperatures for the sides of each element that experiences convection loss.

Description of Variables:

The computer program requires that the numerical values of all input variables are in engineering units.

MREPT	Number of iterations to be performed for soil temperature and moisture effect analysis					
ITREN	An index, ITREN = 1 A shallow trench system = 0 A loose-fill insulation system					
MONTH	Month for which the heat loss is to be determined					
ICALB	An integer, ICALB = 1 Nodal coordinates will be printed out = 0 No print-out					
КТСТ	Thermal conductivity of trench walls, (Btu'in/h'ft ^{2.0} F)					
TRTK	Thickness of the trench wall, (inches)					
KASP	Equivalent thermal conductivity of air surrounding the underground pipes in the shallow trench, (Btu·in/h·ft ^{2.o} F)					

- T1,T2 Fluid temperatures of pipe numbers 1 and 2, respectively, (°F)
- KII Thermal conductivity of pipe insulation materials, (Btu'in/h'ft^{2.0}F)
- KIG Thermal conductivity of the soil in the vicinity of the shallow trench, (Btu'in/h'ft^{2.0}F)
- PI1, PI2 Outside diameters of pipe numbers 1 and 2, respectively, (inches)
- THI1, Thicknesses of the thermal insulation layers for pipe numbers 1 THI2 and 2, respectively, (inches)
- B1,B2 Depths from ground surface to the centers of pipe numbers 1 and 2, respectively, (ft)
- S Separation distance between the centers of the pipes, (ft)
- TG Monthly average earth temperature, (^oF)
- WW Width of the earth region surrounding the shallow trench system, (ft)
- HH Thickness of the equivalent soil layer to account for the surface convective transfer effect, (ft)
- HY Depth of the earth region underneath the shallow trench system, (ft)
- D Thickness of the trench cover, (ft)
- A Total width of the trench system, (ft)
- B Height of the trench wall, (ft)
- AO Annual average outdoor temperature of the site, (^{OF})
- BO Annual amplitude of the monthly average temperature cycle of the site, (^oF)
- DIFF Thermal diffusivity of the soil in the vicinity of the trench system, (ft²/h)
- N The element number, dimensionless
- I,J,K The node numbers of three vertices of an element
- C Thermal conductivity of an element, (Btu 'in/h 'ft^{2.0}F)
- IXCB An index, IXCB = 1 The element has convection boundary = 0 No convection boundary
- HIJ,HJK, Convective heat transfer coefficients for sides IJ, JK and KI of HKI a triangular element, (Btu/h*ft^{2.0}F)

TIJ,TJK, Ambient temperatures for sides IJ, JK and KI of an element, (^{O}F) TKI

The input format of these aforementioned variables which are stored in data files DATA1 and DATA2, are respectively:

(1) DATA1 File

Record No.	Variables	Format
1	MREPT ITREN	(15 18 14)
2	MONTH, ICALB	(215)
3	KTCT, TRTK	(2F10.4)
4	KASP	(F10.5)
5	T1, T2	(2F10.3)
6	KII, KIG	(2F10.4)
7	PI1, PI2	(2F10.4)
8	THI1, THI2	(2F10.4)
9	B1, B2	(2F10.4)
10	S, TG	(2F10.4)
11	WW, HH, HY	(3F10.4)
12	D, A, B	(3F10.4)
13	AO, BO, DIFF	(3F10.4)

(2) DATA2 File

Record No.	<u>Variables</u>	Format
1	N, I, J, K, C, IXCB	(415,F10.4,I5)
	(Use record No. 2 if IX	CB = 1)
2	HIJ, HJK, HKI, TIJ, TJK, TKI	(6F10.4)
	(Repeat record numbers elements)	l and 2 for a total of

130

2.4 Sample Calculations

Two nominal 150 mm (6-inch) steel pipes are laid side by side in a 1.24 m (4.08 ft) wide by 0.89 m (2.92 ft) high concrete shallow trench having 152 mm (6 in.) thick trench cover and walls. Figure 1 shows a typical high temperature hot water underground system consisting of a heat supply and a return pipe in a concrete trench with the top cover placed flush with ground level. The hot water pipe temperture is $196^{\circ}C$ ($385^{\circ}F$) and the return pipe temperature is $99^{\circ}C$ ($210^{\circ}F$). The centers of both pipes are located 1.30 ft (0.40 m) beneath the ground surface. The pipes are insulated with an aluminum jacket and 89 mm (3.5 in.) thick calcium silicate, allowing 102 mm (4 in.) of separation between surfaces of adjacent pipe insulation. The concrete trench system is surrounded by earth having a thermal conductivity of 2.16 W/mK (15 Btu'in/h'ft^{2.0}F) and an annual average temperature of $13^{\circ}C$ ($56^{\circ}F$). The system heat loss calculations were carried out on an IBM personal computer. The input data

files for the sample case and the outputs from the computer program are listed in Appendix A.

In order to maintain some clearances between the insulated pipes and adjoining trench walls and floor as specified in the design guide [10], the concrete trench has different overall dimensions to accommodate various thicknesses of pipe insulation, as listed in Table 1. The effects of the thickness of pipe insulation on the heat loss of the trench system for two sets of pipe temperatures are shown in Tables 2 and 3. The water temperatures of the heat supply and return lines are 196°C (385°F) and 99°C (210°F), respectively for the heat loss calculations shown in Table 2, and 171°C (340°F) and 143°C (290°F) in Table 3. As expected, the results indicate that the total heat loss from the underground piping system decreases with an increase in insulation thickness due to the increased thermal resistance for radial heat conduction. For a given insulation thickness, the rate of heat loss from a single insulated pipe increases with increasing pipe temperature since the convection loss to the trench air is directly proportional to the temperature difference between the outer surface of the pipe and the surrounding air.

3. Equivalent Energy Cost

The energy cost per unit length of the shallow trench system is calculated using the computed heat losses from the piping system obtained from the computer program, and the equivalent fuel energy cost adjusted for future escalation in energy prices over the life of the system. The equation used for calculating the equivalent energy cost (EEC) or the cost of lost energy per annum, is as follows:

 $EEC = 8760 * Q * EC * UPW^*$

- where Q = system heat loss rate, which is equal to the sum of the heat loss rates of two pipes, (Btu/h'ft)
 - EC = equivalent fuel energy cost, (\$/MBtu)
 - = 100 (FC/EF), in which FC is fuel cost, (\$/MBtu), and EF is plant fuel conversion efficiency (percent)
 - UPW = the modified uniform present worth factor adjusted for future escalation in fuel prices.

In this sample case of life-cycle cost calculations, the fuel to heat conversion efficiency at the plant is assumed to be 100%, and the price of fuel is at \$5.00 per million Btu. In reality, the plant fuel conversion efficiency is dependent upon the type of boiler used, and ranges between 50 and 90 percent.

The energy cost is assumed to escalate in accordance with the data on annual inflation rate of natural gas as projected by the U.S. Army Corps of Engineers [1]:

Type of Fuel	<u>1985-1990</u>	<u>1990–1995</u>	1995-2000	2000-2009
Natural gas	5.13	8.81	3.91	4.69
Distillate fuel	6.02	5.98	3.91	4.69
Residual fuel	6.50	6.06	3.91	4.69
Coal	1.85	3.04	3.04	3.04

(5)

The Modified Uniform Present Worth factor (UPW^{*}) is calculated based on a 10% discount rate for a 25 year life for the underground system, and the escalation rates in natural gas price projected by the U.S. Army Corps of Engineers for the periods covering from 1985 to 2009, using the following equation [11]:

$$UPW^{*} = \sum_{j=1}^{N_{1}} \left(\frac{1+e_{1}}{1+d}\right)^{j} + \left(\frac{1+e_{1}}{1+d}\right)^{N_{1}} \sum_{j=1}^{N_{2}} \left(\frac{1+e_{2}}{1+d}\right)^{j}$$

$$+ \left(\frac{1+e_{1}}{1+d}\right)^{N_{1}} \left(\frac{1+e_{2}}{1+d}\right)^{N_{2}} \sum_{j=1}^{N_{3}} \left(\frac{1+e_{3}}{1+d}\right)^{j}$$

$$+ \left(\frac{1+e_{1}}{1+d}\right)^{N_{1}} \left(\frac{1+e_{2}}{1+d}\right)^{N_{2}} \left(\frac{1+e_{3}}{1+d}\right)^{N_{3}} \sum_{j=1}^{N_{4}} \left(\frac{1+e_{4}}{1+d}\right)^{j}$$
(6)

where $\sum_{j=1}^{N_k} \left(\frac{1+e_k}{1+d}\right)^j = \left(\frac{1+e_k}{d-e_k}\right) \left[1 - \left(\frac{1+e_k}{1+d}\right)^{N_k}\right]$

N_k = the length of the period for a given escalation rate in a given period, (year) d = the discount rate

 e_k = the rate of escalation in each of N_k period

The salvage value of a shallow trench heat distribution system after 25 years of service is assumed to be zero. With these economic assumptions, the modified uniform present worth factor, UPW^{*} for the sample case is found to be 15.8 percent.

4. Material and Installation Costs

The capital cost including material and installation costs for constructing a shallow trench underground heat distribution systems can be calculated using cost data and the estimate procedures given in references [1,12-14]. In order to maintain clearance between the outer surfaces of the pipe insulation and the adjoining trench walls and floor described previously, the trench size varies to accommodate different insulation thicknesses, in accordance with the dimensions listed in Table 1. The costs of materials and installation for constructing concrete shallow trench systems consisting of two 150 mm (6-in.) steel pipes with calcium silicate insulation encased in aluminum jacket are estimated and shown as a function of insulation thickness in Figure 5. As anticipated, both the material and installation costs rise with an increase in insulation thickness.

5. Maintenance and Repair Costs

An annual maintenance cost of \$1,000 per mile for a typical shallow trench system, which was derived from the results of a recent survey of field installations [1], was used for the life-cycle cost analysis. The total present value of these annually recurring costs was calculated by multiplying this maintenance cost by 9.077, the uniform present worth (UPW) factor for a 10% discount rate over a 25 year system life. This was determined to be \$5.64 per meter (\$1.72 per foot) for the shallow trench system.

Based on the field survey results, the calculations for the present value of the repair costs were carried out with an assumption that these expenditures occur during years 12 through 25 [1]. The present value of the nonannually recurring repair costs was approximately \$0.30 per meter (\$0.09 per foot), and was neglected compared to the system installation and the routine maintenance costs.

6. Life-Cycle Cost

Life cycle cost analysis can be used to determine the total expenses associated with various design alternatives. These expenses would include the cost of acquisition, yearly maintenance and repair cost, and yearly fuel energy cost with the adjustment for future escalation in fuel prices. The total life-cycle cost of an underground heat distribution system can be expressed as follows:

LCC = MC + IC + EEC + MRC

(7)

- where MC = material cost, (\$/ft of the underground system)
 IC = installation cost, (\$/ft of the system)
 EEC = equivalent energy cost including the adjustment for future
 escalation in energy prices over the life of the system, (\$/ft
 of the system)
 - MRC = maintenance and repair costs, (\$/ft of the system)

Evaluation of the effect of varying insulation thickness for the underground piping system on the construction and fuel energy costs associated with various insulation thicknesses over the economic life of an underground heat distribution system was performed. Figure 6 shows the effect of pipe insulation thickness on the material and installation costs, equivalent energy cost, and total life-cycle cost for two 150 mm (6-in.) pipes installed in a concrete shallow trench based on a 171°C (340°F) supply and 143°C (290°F) return temperatures. In order to provide the specified clearance between the outer surfaces of pipe insulation and the adjoining trench walls and floor, the overall dimensions of the trench system and the relative location of the underground pipes were varied with the insulation thickness according to dimensions given in Table 1. An equivalent fuel energy cost of \$5.00 per million Btu was assumed for energy cost calculations along with the use of the material and installation cost data shown in Figure 5. Figure 6 shows that the life-cycle cost is minimal at an insulation thickness of 89 mm (3.5 inches). This is the most cost effective thickness for insulating the underground piping.

The calculated economic thickness should be compared to the thickness required to prevent moisture condensation since condensation occurs when the temperature of cold pipe surface is below the dewpoint temperature of the surrounding trench air. However, an insulation system with a protective jacket can minimize moisture penetration if all jacket joints and seams are correctly sealed.

7. Minimum Life-Cycle Heat Loss

A plot of the costs of material and installation, fuel energy cost, and total life-cycle cost of a concrete shallow trench containing two 150 mm (6-in.) insulated pipes versus the heat loss from these pipes is given in Figure 7, which is a replot of Figure 6. At high values of the system heat loss shown in Figure 7, the annual cost of material and installation is low, but the annual equivalent energy cost is high. Decreasing the heat loss by increasing the thickness of pipe insulation reduces the fuel energy cost but adds to the material and installation costs. At a certain value of pipe heat loss, the sum of the costs of material and installation, and the fuel energy cost will be a minimum as indicated by the life-cycle cost curve. Beyond the minimum, the life-cycle cost curve rises because the increased costs of material and installation due to additional insulation thickness is no longer offset by the reduced cost of the system heat loss. As illustrated in the Figure 7, the minimum life-cycle cost was reached at a system heat loss of 146 W/m (152 Btu/h·ft). The insulation thickness corresponding to this minimum life-cycle cost heat loss is 89 mm (3.5 inches).

A plot similar to Figure 7 is shown in Figure 8 where the heat supply pipe transported 196°C (385°F) hot water and the return pipe carrying 99°C (210°F) hot water. In Figure 8, the total life-cycle cost of the trench underground system involved can be represented by a "U" shaped curve. In this sample case, the life-cycle cost curve has a minimum that occurs at the pipe heat loss rate of 151 W/m (157 Btu/h*ft). The thickness of insulation layers at this minimum life-cycle cost heat loss is found to be 76 mm (3 inches).

8. Conclusions

The calculation of heat loss from a pair of insulated piping system installed in a shallow trench underground heat distribution system has been performed using a computer simulation program based on the finite element method. General formulation of the relevant equations of heat flow and boundary conditions for a two dimensional, steady heat conduction problems are presented. The computational scheme and the input data required for executing the simulation program are described, and the output from the computer program presented for a sample case.

Life-cycle costing analysis was performed for a typical concrete shallow trench system containing two nominal 152 mm (6-inch) insulated pipes with various insulation thicknesses. It was demonstrated that the least-cost heat loss from the underground pipes or the most economic insulation thickness can be determined for given fluid temperatures based on the results of the cost analysis. The calculated economic insulation thickness should be sufficient to maintain the surface temperature of the insulation above the dewpoint of the surrounding trench air preventing moisture condensation. The procedures for calculating the equivalent energy cost due to the pipe heat loss, and determining the minimum life-cycle cost heat loss for an underground trench system are presented.

9. Acknowledgment

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Table l

Dimensions of Concrete Shallow Trench Systems for Different Thickness of Pipe Insulation

Pipe Insulation Thickness									
<u>(inch)</u>	<u> </u>	<u> </u>	<u> </u>	<u>D</u>	<u> </u>	F	G	<u>H</u>	<u> I </u>
1.0	28	19	40	31	6	6	8	6	11
1.5	30	20	42	32	6	6	8	6	12
2.0	32	20	44	32	6	6	8	6	12
2.5	34	21	46	33	6	6	9	6	12
3.0	36	22	48	34	6	6	9	6	13
3.5	38	23	50	35	6	6	10	6	13
4.0	40	24	52	36	6	6	10	6	14
5.0	44	26	56	38	6	6	11	6	15
6.0	48	28	60	40	6	6	12	6	16

Note:

1. Refer to Figure 1 for descriptions of symbols A to I.

2. All dimensions in inches, 1 inch = 2.54 cm

Calculated	Results	of Heat L	osses	from Two	150 mm	(6-in.)
Undergro	ound Pipe	es Inside	a Shal	low Trenc	h for P	ipe
Temp	eratures	of 196 an	d 99 C	(385 and	210 F)	

Insulation	Heat Loss Rate (Btu/h [•] ft)				
(inch)	<u>Pipe No. 1</u>	<u>Pipe No. 2</u>	<u>Total</u>		
1.0	238	94	332		
1.5	177	74	251		
2.0	144	62	206		
2.5	124	54	178		
3.0	109	48	157		
3.5	98	44	142		
4.0	90	40	130		
5.0	79	35	114		
6.0	70	32	102		

Table 3

Calculated Results of Heat Losses from Two 150 mm (6-in.) Underground Pipes Inside a Shallow Trench for Pipe Temperatures of 171 and 143 C (340 and 290 F)

Insulation	Heat Los	<u>ss Rate (Btu/h•f</u>	<u>t)</u>
(inch)	<u>Pipe No. 1</u>	Pipe No. 2	<u>Total</u>
1.0	199	157	356
1.5	150	120	270
2.0	122	99	221
2.5	105	85	190
3.0	93	75	168
3.5	84	68	152
4.0	77	63	140
5.0	67	55	122
6.0	60	49	109







Figure 2. Finite Element Design for the Outer Boundary Earth and Concrete Trench Walls and Cover



Figure 3. Finite Element Design for Air Space Surrounding the Pipes in Concrete Trench



Figure 4. Finite Element Design for the Pipe Insulation and the Outer Surfaces of the Pipes















Figure 8. A Plot of the Life-Cycle Cost Versus the Heat Loss from Underground Pipes at 385 and 210 Degree F

APPENDIX A. The Input Data Files and the Outputs from the Computer Program on Sample Calculations

• •

.

A.1 A Listing of DATA1 Input File

!
1
,

1	1	16	2	.06
2	16	17	2	.06
3	2	17	18	.06
4	2	18	3	.06
5	3	18	19	.06
6	3	19	20	.06
7	3	20	4	.06
8	4	20	21	.06
9	4	21	6	.06
10	4	6	5	.06
11	14	13	12	.06
12	14	12	29	.06
13	29	12	28	.06
14	28	12	11	.06
15	28	11	27	.06
16	27	11	26	.06
17	26	11	10	.06
18	26	10	25	.06
19	25	10	8	.06
20	8	10	9	.06
21	16	32	17	.06
22	16	15	32	.06
23	32	15	31	.06
24	31	15	30	.06
25	15	14	30	.06
26	30	14	29	.06
27	6	21	22	.06
28	6	22	7	.06
29	22	23	7	.06
30	23	24	7	.06
31	7	24	8	.06
32	24	25	8	.06
33	17	32	18	.06
34	18	32	34	.06
35	18	34	35	.06
36	18	35	19	.06
37	19	35	20	.06
38	20	35	36	.06
39	20	36	22	.06
40	20	22	21	.06
41	30	29	28	.06
42	30	28	48	.06
43	48	28	47	.06
44	47	28	27	.06
45	47	27	26	.06
46	47	26	46	.06
47	46	26	24	.06
48	24	26	25	.06
49	32	33	34	.06
50	32	31	33	.06
21	31	40	33	.06
52	31	42	40	.06
55	31	41	42	.06
55	20	20	41	.00
22	20	40	41	.00

56	36	37	22	.06
57	22	37	23	.06
58	37	38	23	.06
59	38	44	23	.06
60	23	44	45	.06
61	23	45	24	.06
62	45	46	24	.06
63	40	42	39	.06
64	42	43	39	.06
6.5	39	43	44	.06
66	39	44	38	.06
67	33	49	50	. 06
68	33	50	34	.06
69	34	50	35	.06
70	35	50	51	. 06
71	35	51	52	. 06
72	35	52	36	.06
73	36	52	37	06
75	37	52	53	06
75	27	52	54	.00
75	27	55	20	.00
70	2/	54	20	.00
//	30	54	39	.00
/8	39	54	22	.00
/9	56	39	22	.06
80	40	39	56	06
81	33	40	56	.06
82	33	56	49	.06
83	41	57	58	.06
84	41	58	42	.06
85	42	58	43	.06
86	43	58	59	.06
87	.43	59	60	.06
88	43	60	44	.06
89	44	60	45	.06
90	45	60	61	.06
91	45	61	62	.06
92	45	62	46	.06
93	46	62	47	.06
94	47	62	63	.06
95	47	63	64	.06
96	47	64	48	.06
97	41	48	64	.06
98	41	64	57	.06
99	6.5	80	66	. 06
100	66	80	1	. 06
	66	1	2	06
	66	2	2	06
	66	2	67	.00
10%	67	2		.00.
104	67	5	69	.00
106	607	4	00	.00
107	00	4	70	.00
100	60	70	10	.00
100	0 Ö	70	709	.00
109	/ 8	11	/0	.06
110	/8	/6	13	.06

111	13	76	12	.06
112	12	76	11	.06
113	11	76	75	.06
114	11	75	10	.06
115	10	75	74	.06
116	10	74	9	.06
117	9	74	72	.06
118	72	74	73	.06
119	80	16	1	.06
120	80	79	16	.06
121	16	79	15	.06
122	15	79	14	.06
123	79	78	14	.06
124	14	78	13	.06
125	5	6	70	.06
126	70	6	71	.06
127	6	7	71	.06
128	71	7	8	.06
129	71	8	72	.06
130	8	9	72	.06

A	.3 <u>The</u>	Outputs	from the	Computer	Program						
	TI	T2	КI	KB	D1	50					
3	85.00	210.00	.44	15.00	6.63	D2 5 53					
ľ	TH1	TH2	DP1	DP2	5	0.03 TG					
1	3.50	3.50	1.30	1.30	1.38	56.00					
	WW	HH	HY	MONTH							
	10.00	. 42	20.00	1					-		
	W	Н	D	A	В	WW	нн	нү			
	4.08	2.92	.50	3.08	1.92	10.00	. 42	20.00			
	XC1	YC1	XC2	YC2							
	1.347	1.717	2.733	1.717							
	D1	D2	S	тн к	I KG	T1	T2	Q1	Q2	QT	KP
	6.63	6.63 1	1.38 3.	50.4	4 15.00	385.	210.	95.32	41.26	136.58	. 566
	X(M),M	1=1,NX									
	.00	.00	.00	.00	.00	1.02	2.04	3.06	4.08	4.08	
	4.08	4.08	4.08	3.06	2.04	1.02	: 50	.50	.50	.50	
	.50	1.2/	2.04	2.81	3.58	3.58	3.58	3.58	3.58	2.81	
	2.04	1.2/	1.30	. 75	.78	.95	1.35	1.75	1.92	1.75	
	2.73	2.33	2.10	2.33	2.73	3.13	3.30	3.13	1.35	1.15	
	2.77	2 07	1.35	1.04	1.62	1.54	2.73	2.54	2.46	2.54	
	2.73	2.73	3.01	2.73	-10.00	-10.00	-10.00	-10.00	-10.00	.00	
	2.04 V(M) M		14.08	14.08	14.08	14.08	14.08	4.08	2.04	.00	
	47	1.15	1 88	2 40	र रर	7 77	7 77	7 77		• (•	
	1.88	1.15	. 42	42	42	2.33	3.33	3.33	3.33	2.60	
	2.83	2.83	2.83	2.83	2.83	2.35	1 88	1.40	1.00	2.30	
	.92	. 92	1.15	1.32	1.72	2.12	2.28	2.12	1 72	1 32	
	1.15	1.32	1.72	2.12	2.28	2.12	1.72	1.32	1.44	1.52	
	1.72	1.91	1.99	1.91	1.72	1.52	1.44	1.52	1.72	1.91	
	1.99	1.91	1.72	1.52	.00	. 42	3.33	13.33	23.33	23.33	
-	23.33	23.33	23.33	13.33	3.33	.42	.00	.00	.00	.00	
	MONTH		1								
	TEN	IPERATUR	RE ARRAY	1							
	58.37	62.45	64.72	66.07	66.10	67.21	67.32	66.62	65.21	64.92	
	63.43	61.33	57.96	60.98	62.00	62.30	66.73	70.92	72.83	72.16	
	69.43	73.86	74.17	71.70	67.64	69.14	68.85	67.17	64.36	69.8Ŭ	
	73.82	73.29	75.06	73.41	74.32	75.30	76.77	78.13	79.45	76.92	
	71.34	75.20	78.79	76.59	73.65	71.21	69.56	69.11	385.00	385.00	
	385.00	385.00	385.00	385.00	385.00	385.00	210.00	210.00	210.00	210.00	
	54.00	210.00	210.00	210.00	56.00	56.00	56.00	56.00	56.00	56.00	
	36.00	56.00	56.00	56.00	56.00	56.00	56.00	56.00	56.00	56.00	

AVERAGE TEMPERATURE DROP ACROSS INSULATION

T1= 308.83 T2= 136.82 DEG F HEAT LOSSES FROM UNDERGROUND PIPES :

- -

Q1= 98.67 Q2= 43.71 QT= 142.39 BTU/H-FT

```
PROGRAM FEUHDS
* THIS IS A PROGRAM FOR ANALYSIS OF UNDERGROUND SYSTEM HEAT LOSS
* BASED ON FINITE ELEMENT METHOD
*
      SUBROUTINES CALLED: PIPE2, TWOPIP, TGO, SOILK, TGXX, SOLVP, AND PIPEHL
*
      INPUT DATA FILES: DATA1 AND DATA2
*
      OUTPUT DATA FILE: PLTDTA
      X(I): X-COORDINATE OF NODAL POINT I, IN FT
*
      Y(I): Y-COORDINATE OF NORAL POINT I, IN FT
*
*
      I,J,K
               NODAL POINTS OF ELEMENT M
*
      M
            ELEMENT INDEX
*
      С
             THERMAL CONDUCTIVITY, BTU-IN/HR/FT**2/F
             THICKNESS OF THE ELEMENT, FT
*
      L
С
      WHEN TEMPERATURE (T(I), I=MZ, NX) ARE KNOWN MZ.NE.O.
      DIMENSION Q(130), T(130), X(130), Y(130), KK(130, 130)
      DIMENSION IZ(130), JZ(130), KZ(130), AS(130), B2IZ(130), B3IZ(130), B2JZ
     \&(130), B2KZ(130), B3JZ(130), B3KZ(130)
      DIMENSION CC(130), TGX(12,5), QQ(130)
      COMMON/PP/TP1,TP2,KI,KG,D1,D2,TH1,TH2,DP1,DP2,S,TG,WW,HH,HY,
     & MONTH
      REAL L, KK, KI, KG, KIX, KTCT, KASP
      TRTK=0.
      PI = 4.*ATAN(1.)
71
      CONTINUE
      OPEN (8, FILE='DATA1')
      OPEN (6,FILE='PRN')
      OPEN (9, FILE='DATA2')
      OPEN (10,FILE='PLTDTA',STATUS='NEW')
      READ IN MREPT AND ITREN
С
      READ (8,200, ERR=2000) MREPT, ITREN
200
      FORMAT (15, 1x, 14)
72
      MI1 = 32
      MI2 = 99
      IF(ITREN.EQ.0) GO TO
                              50
      MI1 = 66
      MI2 = 99
50
      CONTINUE
С
      READ MONTH OF INTEREST AND ICALB
      READ (8,250) MONTH, ICALB
250
      FORMAT (215)
      IF(ITREN.NE.1) GO TO 73
С
      READ THE THERMAL CONDUCTIVITY (IN BTU-IN./H-FT**2 - DEG F) AND OF
      THICKNESS (IN INCHES) OF CONCRETE WALL FOR SHALLOW TRENCH SYSTEM.
С
      READ (8,260) KTCT, TRTK
260
      FORMAT (2F10.4)
      READ EQUIVALENT THERMAL CONDUCTIVITY OF AIR SPACE SURROUNDING THE
С.
С
      PIPES IN SHALLOW TRENCH, IN BTU-IN./H-FT**2 - DEG F
      READ (8,265) KASP
      FORMAT (F10.5)
265
73
      CONTINUE
      CALL PIPE2(X,Y,ITREN,TRTK)
      CALL TWOPIP(1, ITREN)
      NX = 80
      NY = 80
      MZ = 49
      MX = 130
```

```
IF(ICALB.NE.2) GO TO 1050
      WRITE(10,15) (X(I),I=1,NX)
15
      FORMAT(10F7.2)
      WRITE(10,15) (Y(I), I=1, NY)
      IF(ICALB.EQ.0) GO TO 80
1050
      WRITE(6, 20)
      FORMAT(X(M), M=1, NX')
20
      WRITE(6,15) (X(I), I=1, NX)
      WRITE(6, 21)
      FORMAT( Y(M), M=1, NY')
21
      WRITE(6,15) (Y(I), I=1, NY)
80
      CONTINUE
      CALL TGO(TGX, PI, Y)
      IF(ICALB.EQ.1) WRITE(6,16)
                                               c1)
      FORMAT(
                    Μ
                           I
                                  J
                                        K
16
      L=1.
      DO 1080 IREPT=1,MREPT
      DO 4 I=1, NX
      Q(I) = 0.
      DO 4 J=1,NY
      KK(I,J)=0.
4
      DO 5 M=1,MX
      IF(IREPT.GT.1) GO TO 1090
      HIJ=0.
      HJK=0.
      HKI = 0.
      TIJ=0.
      TJK=0.
      TKI = 0.
      READING IN THE ELEMENT NUMBER AND ITS NODAL POINTS AND THERMAL
С
      CONDUCTIVITY, AND INDEX OF CONVECTION BOUNDARY
С
      READ(9,500) N,I,J,K,C,IXCB
500
      FORMAT (415, F10.4, 15)
      C = KG/12.
      IF(M.GT.MI1.AND. M.LT.MI2) C=KI/12.
      IF(ITREN.EQ.1.AND.M.LE.32) C=KTCT/12.
      IF(ITREN.EQ.1.AND.M.GT.32.AND.M.LT.67)C=KASP/12.
      IZ(M) = I
      JZ(M) = J
      KZ(M) = K
      CC(M) = C
      GO TO 1070
      I = IZ(M)
1090
      J=JZ(M)
      K = KZ(M)
      C = CC(M)
      IF(M.GT.MI1.AND.M.LT.MI2) GO TO 1070
      TM = (T(I) + T(J) + T(K))/3.
      CALL SOILK(TM,CZ)
      C = CZ * KG / (1.1 * 12)
1070 XI = X(I)
      XJ = X(J)
      XK = X(K)
      YI = Y(I)
      YJ=Y(J)
```

```
YK = Y(K)
CXX = C
CXY=0.
CYX = 0.
CYY = C
B2I = YJ - YK
B3I = XK - XJ
B2J = YK - YI
B3J = XI - XK
B2K=YI-YJ
B3K = XJ - XI
S=0.5*(XJ*B2J+XI*B2I+XK*B2K)
S = ABS(S)
A2=S*2.
AS(M) = A2
B2I = B2I/A2
B3I = B3I/A2
B2J=B2J/A2
B3J=B3J/A2
B2K = B2K/A2
B3K = B3K/A2
B2IZ(M) = B2I
B3IZ(M) = B3I
B2JZ(M) = B2J
B3JZ(M) = B3J
B2KZ(M) = B2K
B3KZ(M) = B3K
BII=S*L*(B2I*B2I*CXX+B2I*B3I*CXY+B3I*B2I*CYX+B3I*B3I*B3I*CYY)
BIJ=S*L*(B2I*B2J*CXX+B2I*B3J*CXY+B3I*B2J*CYX+B3I*B3J*CYY)
BIK = S \times L \times (B2I \times B2K \times CXX + B2I \times B3K \times CXY + B3I \times B2K \times CYX + B3I \times B3K \times CYY)
BJI = S \times L \times (B2J \times B2I \times CXX + B2J \times B3I \times CXY + B3J \times B2I \times CYX + B3J \times B3I \times CYY)
BJJ=S*L*(B2J*B2J*CXX+B2J*B3J*CXY+B3J*B2J*CYX+B3J*B3J*CYY)
BJK=S*L*(B2J*B2K*CXX+B2J*B3K*CXY+B3J*B2K*CYX+B3J*B3K*CYY)
BKI=S*L*(B2K*B2I*CXX+B2K*B3I*CXY+B3K*B2I*CYX+B3K*B3I*CYY)
BKJ=S*L*(B2K*B2J*CXX+B2K*B3J*CXY+B3K*B2J*CYX+B3K*B3J*CYY)
BKK=S*L*(B2K*B2K*CXX+B2K*B3K*CXY+B3K*B2K*CYX+B3K*B3K*CYY)
KK(I,I) = KK(I,I) + BII
KK(I,J) = KK(I,J) + BIJ
KK(I,K) = KK(I,K) + BIK
KK(J,I) = KK(J,I) + BJI
KK(J,J) = KK(J,J) + BJJ
KK(J,K) = KK(J,K) + BJK
KK(K,I) = KK(K,I) + BKI
KK(K, J) = KK(K, J) + BKJ
KK(K,K) = KK(K,K) + BKK
ACCOUNT FOR CONVECTION ON BOUNDARY
IF(IXCB .EQ. 0) GO TO 800
READING IN CONVECTIVE HEAT TRANSFER COEFFICIENTS AND AMBIENT
TEMPERATURES FOR THREE BOUNDARY SEGMENTS
READ(9,700) HIJ, HJK, HKI, TIJ, TJK, TKI
FORMAT(6F10.4)
HHIJ=HIJ*L*SQRT((X(I)-X(J))**2+(Y(I)-Y(J))**2)/6.
HHJK=HJK*L*SQRT((X(J)-X(K))**2+(Y(J)-Y(K))**2)/6.
HHKI = HKI + L + SORT((X(K) - X(I)) + + 2 + (Y(K) - Y(I)) + + 2)/6.
KK(I,I) = HHIJ \star 2 \cdot HHKI \star 2 \cdot KK(I,I)
```

С

С

С

```
KK(I,J) = HHIJ + KK(I,J)
       KK(I,K) = HHKI + KK(I,K)
       KK(J,I) = HHIJ + KK(J,I)
       KK(J,J) = HHIJ * 2 + HHJK * 2 + KK(J,J)
       KK(J,K) = HHJK + KK(J,K)
       KK(K,I) = HHKI + KK(K,I)
       KK(K, J) = HHJK + KK(K, J)
       KK(K,K) = HHJK \star 2. + HHKI \star 2. + KK(K,K)
       HHIJ=TIJ*3.*HHIJ
       HHJK = TJK * 3 . * HHJK
       HHKI = TKI + 3 . + HHKI
       Q(I) = Q(I) + HHIJ + HHKI
       Q(J) = Q(J) + HHIJ + HHJK
       Q(K) = Q(K) + HHJK + HHKI
800
       IF(ICALB.EQ.1) WRITE(6,9) M,I,J,K,C
       IF(ICALB.EQ.2) WRITE(10,9) M,I,J,K,C
       FORMAT(415,F10.4)
9
5·
       CONTINUE
       MZZ = MZ - 1
       MZ8 = MZ + 7
       DO 900 I = MZ, MZ8
       T(I) = TP1
       II = I + 8
       T(II) = TP2
900
       CONTINUE
       MMZ16 = MZ + 16
       DO 1080 MJ=MONTH, MONTH
       CALL TGXX(T,TGX,MJ)
       DO 77 I=1,MZZ
       SUM=0.
       DO 78 J=MZ, NX
78
       SUM = SUM + KK(I, J) * T(J)
77
       QQ(I) = Q(I) - SUM
       WRITE(6,10) MJ
       FORMAT( ' MONTH ....',15)
10
       IF(ICALB.EQ.1) WRITE(6,6)
       FORMAT(6X, QQ
                           ARRAY')
6
       IF(ICALB.EQ.1) WRITE(6,7) (QQ(1), I=1, NX)
7
       FORMAT (10F7.2)
       NXX = NX
       CALL SOLVP(MZZ,MZ,KK,QQ,T,130)
       WRITE(6, 14)
14
       FORMAT(
                      TEMPERATURE ARRAY')
       WRITE(6,7) (T(I), I=1, NX)
       IF(ICALB.EQ.2) WRITE(10,7) (T(I),I=1,NX)
       R1 = D1/24.
       R2 = D2/24.
       TH1X = TH1/12.
       TH2X = TH2 / 12.
       IS1 = 49
       IS2 = 57
       KIX = KI/12.
       CALL
              PIPEHL(T,R1,R2,TH1X,TH2X,IS1,IS2,KIX,8,QTX)
1080
       CONTINUE
       GOTO 2010
```

2000 WRITE (6,2005) 2005 FORMAT (1X, THERE ARE SOME ERRORS IN DATA') 2010 STOP

```
END
```

SUBROUTINE TGO(TGX,PI,Y)

THIS SUBROUTINE CALCULATES THE UNDISTURBED EARTH TEMPERATURES × * AT VARIOUS DEPTHS

1

DIMENSION TGX(12,5),Y(1)

```
READING IN THE ANUAL AVERAGE TEMPERATURE AND AMPLITUDE OF THE
÷
```

```
MONTHLY NORMAL TEMPERATURE CYCLE OF THE SITE, IN DEG F. AND
±
*
```

```
THERMAL DIFFUSIVITY OF SOIL, IN FT**2/H.
```

```
READ (8,300) A0,B0,DIFF
FORMAT (3F10.4)
```

```
300
      W=2.*PI/12.
      WZ=2.*PI/(8760*DIFF*2)
      ZZ = SQRT(WZ)
      DO 1 I=1,12
```

```
DO 1 J=1,5
```

```
Z=ZZ*Y(64+J)
```

```
TGX(I,J)=AO+BO*EXP(-Z)*SIN(W*(I-3)-Z)
1
```

```
RETURN .
END
```

```
SUBROUTINE PIPE2(X,Y,ITREN,TRTK)
* THIS SUBROUTINE GENERATES X AND Y -COORDINATES OF NODAL POINTS FOR
* THE TWO PIPE SYSTEM
      REAL KII, KIG, KI, KG
      DIMENSION X(1), Y(1)
      COMMON /PP/T1,T2,KII,KIG,PI1,PI2,THI1,THI2,B1,B2,S,TG,
     & WW, HH, HY, MONTH
      READ TEMPERATURE OF PIPES 1 AND 2, IN DEG F
С
      READ (8,300) T1,T2
      FORMAT (2F10.3)
300
      READ THERMAL CONDUCTIVITY OF THERMAL INSULATION AND SOIL,
С
C
      RESPECTIVELY, IN BTU-IN./H-FT**2 - DEG F
      READ (8,310) KII,KIG
      FORMAT (2F10.4)
310
С
      READING IN THE OUTSIDE DIAMETERS OF STEEL PIPES 1 AND 2, IN INCHES
      READ (8,310) PI1,PI2
С
      READING IN THE THICKNESS OF THERMAL INSULATION USED FOR PIPES 1
С
      AND 2, RESPECTIVELY, IN INCHES
      READ (8,310) THI1, THI2
С
      READ THE DEPTHS FROM GROUND SURFACE TO THE CENTERS OF PIPES 1 AND
С
      2, RESPECTIVELY, IN FT.
      READ (8,310) B1,B2
С
      READING IN THE SEPARATION DISTANCE (IN FT.) OF THE PIPE CENTERS,
С
      AND THE AVERAGE EARTH TEMPERATURE, IN DEG F.
      READ (8,310) S,TG
      READ IN THE WIDTH AND DEPTHS OF EARTH REGIONS SURROUNDING THE
С
С
      UNDERGROUND SYSTEM, IN FT.
      READ (8,315) WW, HH, HY
      FORMAT (3F10.4)
315
      WRITE(6,44) T1,T2,KII,KIG,PI1,PI2
      FORMAT( 1
                                              D1 D2<sup>(</sup>/6F7.2)
44
                  T1
                        T 2
                                KI
                                       KG
      WRITE(6,55) THI1, THI2, B1, B2, S, TG
55
      FORMAT( 1
                  TH1
                         TH2
                               DP1
                                       DP2
                                                 S
                                                        TG'/6F7.2)
      WRITE(6,66) WW, HH, HY, MONTH
66
      FORMAT(
                  WW HH
                                  ΗY
                                      MONTH (3F7.2,17)
С
      READ IN THE DEPTH OF EARTH COVER, THE WIDTH AND HEIGHT OF TRENCH
      OR LOOSE-FILL.
С
      READ (8,315) D,A,B
      W = 2 \star A
      H=2.*B+D
      P1 = PI1/12.
      R1=P1*0.5
      P2 = PI2/12.
      R2 = P2 * 0.5
      KI = KII / 12.
      KG = KIG/12.
      IF(ITREN.NE.1) GO TO 67
      W=A+2*TRTK/12
      H=B+2*TRTK/12
      WRITE(6,22) W,H,D,A,B,WW,HH,HY
67
      B1 = B1 + HH
      B2=B2+HH
22
      FORMAT( 1
                    W
                          Н
                                  D
                                        Α
                                                 В
                                                        WW
                                                               HH
                                                                       HY
     & (,8F7.2)
      PI=4.*ATAN(1.)
```

```
TH1 = THI1 / 12.
TH2 = THI2 / 12.
DO 1 I = 1, 4
X(I)=0.
Y(I) = H * (I - 1) / 4 + HH
I4 = I + 4
18 = 1 + 8
I12 = I + 12
I16 = I + 16
I20 = I + 20
124 = 1 + 24
128 = 1 + 28
I1 = I - 1
X(I4) = W + I1/4.
Y(I4) = H + HH
X(I8) = W
Y(I8) = H - H + I1/4 + HH
X(112) = W - 11 + W/4.
Y(I12) = HH
X(I16) = (W-A) * 0.5
Y(I16) = D + I1 * B / 4. + HH
X(I20)=0.5*(W-A)+I1*A/4.
Y(I20) = D + B + HH
X(I24) = (W+A) * 0.5
Y(I24) = (D+B) - I1 + B/4 + HH
X(I28)=0.5*(W+A)-I1*A/4.
Y(128) = D + HH
CONTINUE
XC1=W*0.5-S*0.5
YC1=B1
XC2=0.5*(W+S)
\Upsilon C 2 = B 2
WRITE(6,77)XC1,YC1,XC2,YC2
FORMAT (
                                           YC2 / 4F7.3)
                XC1
                         YC1
                                 XC2
DO 2 I=1,8
Q=2.*PI*(I-1)/8.
I1=I
K1 = 32 + I1
K2 = 40 + I1
K3 = 48 + I1
K4 = 56 + I1
X(K3) = XC1 - 0.5 + P1 + SIN(0)
Y(K3) = YC1 - P1 * COS(Q) * 0.5
X(K4) = XC2 - 0.5 + P2 + SIN(Q)
Y(K4) = YC2 - 0.5 + P2 + COS(Q)
X(K1) = XC1 - (TH1 + 0.5 + P1) + SIN(Q)
Y(K1) = YC1 - (TH1 + 0.5 * P1) * COS(Q)
X(K2) = XC2 - (TH2 + 0.5 * P2) * SIN(Q)
Y(K2) = YC2 - (TH2 + 0.5 * P2) * COS(Q)
CONTINUE
X(65) = -WW
X(77) = W + WW
DO 3 I=1, 4
II = 65 + I
JJ = 77 - I
```

```
1
```

77

X(II) = X(65)
X(JJ) = X(77)
CONTINUE
X(70) = X(5)
X(80) = X(70)
X(71) = X(7)
X(72) = X(9)
X(78) = X(72)
x(79) = x(71)
DO 5 I = 77.80
Y(I)=0.
Y(65)=0.
Y(66) = Y(1)
Y(76) = Y(66)
DO 6 $I = 69,73$
Y(I) = HH + H + HY
Y(67) = Y(5)
Y(75) = Y(67)
Y(68) = (Y(5) + Y(70)) * 0.5
Y(74) = Y(68)
RETURN
END

```
SUBROUTINE TWOPIP(IREPT, ITREN)
* THIS SUBROUTINE DETERMINES TWO PIPE HEAT LOSS TO UNDERGROUND
       COMMON /PP/T1,T2,ZKI,ZKSI,DA1,DA2,THI1,THI2,D1,D2,AX,TG
     &, WW, HH, HY, MONTH
      PI=4.*ATAN(1.)
      X1=2.*PI
      R1 = DA1/24.
      R2 = DA2/24.
      THIX=THI1/12.
       TH2 = THI2 / 12.
       ZK1 = ZKI/12.
       ZK2 = ZK1
      D1 = D1 + HH
      D2=D2+HH
       ZKS = ZKSI / 12.
      WRITE(6,6)
      DO 5 I=1, IREPT
2
       TH1 = TH1X + 0.1 * (I - 1)
      TH2=TH1
      A = R1 + R2 + TH1 + TH2 + 0.05
      THI1=TH1*12.
      IF(ITREN.EQ.1) A=AX
      Cl = Xl + ZKl / LOG((Rl + THl)/Rl)
      C2 = X1 \times ZK2 / LOG((R2 + TH2) / R2)
      P11=1.+C1/(X1*ZKS)*LOG((2*D1)/(R1+TH1))
      P12=C2/(X1*ZKS)*LOG((A*A+(D1+D2)**2)/(A*A+(D1-D2)**2))*0.5
      P21=C1/(X1*ZKS)*LOG((A*A+(D1+D2)**2)/(A*A+(D1-D2)**2))*0.5
      P22=1.+C2/(X1*ZKS)*LOG((2*D2)/(R2+TH2))
      DEL=P12*P21-P11*P22
      ZKP1=C1*(P12-P22)/DEL
      ZKP2 = C2 * (P21 - P11) / DEL
      TP1 = (P12 * T2 - P22 * T1) / (P12 - P22)
      TP2 = (P21 * T1 - P11 * T2) / (P21 - P11)
      Q1=ZKP1*(TP1-TG)
      Q2 = ZKP2 * (TP2 - TG)
      0T = Q1 + Q2
       TM = (T1 + T2) * 0.5
      ZK = (QT) / (TM - TG)
      WRITE(6,3) DA1, DA2, A, TH11, ZKI, ZKSI, T1, T2, Q1, Q2, QT, ZK
3
      FORMAT(6F6.2,2F6.0,3F7.2,F6.3)
                                                               T1
                                                                      T 2
                                                                             Q1
      FORMAT(
                     D1
                            D2
                                    S
                                          TH
                                                 KI
                                                        KG
                        KP')
     & Q2
               OT
       CONTINUE
      RETURN
      END
```



```
SUBROUTINE SOILK(T,ZK)
* THIS ROUTINE DETERMINES THERMAL CONDUCTIVITY OF SOIL
* AS A FUNCTION OF TEMPERATURE
       REAL K(14)
       DIMENSION TX(14)
       DATA K/1.1,1.1,1.1,1.0,0.4,0.31,0.25,0.19,0.15,0.11,0.09,0.07,
      & 0.05,0.05/
       DO 1 I=1,14
       TX(I) = 50.+(I-1)*25.
1
       IF(T.GT.TX(1)) GO TO 7
       ZK = 1.1
       GO TO 5
       IF(T.LT.TX(14)) GO TO 6
7
       ZK=0.05
       GO TO 5
       DO 2 I=1,13
6
       Tl = T - TX(I)
       IF(T1.NE.O) GO TO 3
       ZK = K(I)
       GO TO 5
       CONTINUE
3
       T2=T-TX(I+1)
      IF(T2.NE.O.) GO TO 4
       ZK = K(I+1)
       GO TO 5
       CONTINUE
4
      P=T1*T2
       IF(P.GT.0) GO TO 2
       ZK = K(I+1) + T2 * (K(I+1) - K(I)) / 25.
       GO TO 5
2
       CONTINUE
5
       RETURN
      END
      SUBROUTINE TGXX(T,TGX,MONTH)
* THIS SUBROUTINE PROVIDES OUTER BOUNDARY TEMPERATURES OF EARTH REGION
      DIMENSION T(1), TGX(12,5)
      T(65) = TGX(MONTH, 1)
      DO 1 I = 77,80
      T(I) = T(65)
1
      CONTINUE
      T(66)=TGX(MONTH,2)
      T(76) = T(66)
      T(67) = TGX(MONTH, 3)
      T(75) = T(67)
      T(68) = TGX(MONTH, 4)
      T(74) = T(68)
      T(69) = TGX(MONTH, 5)
      DO 2 I=70,73
      T(I) = T(69)
2
      RETURN
      END
```

```
SUBROUTINE SOLVP(M,N,C,D,X,I)
* THIS SUBROUTINE SOLVES THE SIMULTANEOUS EQUATIONS
      DIMENSION A(100,101), C(1,1), D(1), X(1)
      DO 10 IX=1,M
      DO 10 IY=1,M
      A(IX, IY) = C(IX, IY)
10
      DO 20 I2=1, M
20
      A(I2,N)=D(I2)
      L=1
30
      AA=A(L,L)
      DO 40 K=L,N
      A(L,K)=A(L,K)/AA
40
      DO 60 K=1,M
      IF(K.EQ.L) GO TO 60
      AA = -A(K, L)
      DO 50 IA=L,N
      A(K,IA) = A(K,IA) + AA + A(L,IA)
50
      CONTINUE
60
      L=L+1
      IF(L.LE.M) GO TO 30
      DO 70 IP=1,M
      X(IP) = A(IP, N)
70
      RETURN
      END
```

```
SUBROUTINE PIPEHL(T,R1,R2,TH1,TH2,IS1,IS2,ZKS,N1,QT)
* THIS SUBROUTINE CALCULATES THE AVERAGE TEMPERATURE DROPS ACROSS THE
* PIPE INSULATIONS AND THE RATES OF HEAT LOSS FROM THE UNDERGROUND
* PIPES IN TRENCH SYSTEM
       DIMENSION T(1)
       PI=4.*ATAN(1.)
       SUM1=0.
       SUM2=0.
       DO 1 I=1,N1
      K1 = IS1 + (I - 1)
      K2 = IS2 + (I - 1)
      K3 = K1 - 16
      K4 = K2 - 16
       SUM1 = SUM1 + T(K1) - T(K3)
       SUM2 = SUM2 + T(K2) - T(K4)
1
      CONTINUE
      T1 = SUM1/N1
      T2 = SUM2/N1
      WRITE(6,3) T1,T2
FORMAT(// AVERAGE TEMPERATURE DROP ACROSS INSULATION'
3
     & // T1= ',F10.2,' T2= ',F10.2,' DEG F')
      Q1 = ZKS \pm 2 \cdot PI \pm T1 / LOG((R1 + TH1)/R1)
      02 = ZKS \pm 2 \cdot PI \pm T2 / LOG((R2 + TH2) / R2)
      0T = 01 + 02
      WRITE(6,2) Q1,Q2,QT
      FORMAT(3X, 'HEAT LOSSES FROM UNDERGROUND PIPES : '//' Q1=', F10.2,
2
     & Q2=',F10.2, QT=',F10.2, BTU/H-FT')
      RETURN
      END
```

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