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# **A Numerical Solution of the Time Dependent Partial Differential Equations which describe a one-dimensional, Laminar, Premixed Flame**

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Institute for Materials Research  
National Bureau of Standards  
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**U. S. DEPARTMENT OF COMMERCE  
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**U.S. DEPARTMENT OF COMMERCE, Elliot L. Richardson, *Secretary***  
**Edward O. Vetter, *Under Secretary***  
**Dr. Betsy Ancker-Johnson, *Assistant Secretary for Science and Technology***  
**NATIONAL BUREAU OF STANDARDS, Ernest Ambler, *Acting Director***



A NUMERICAL SOLUTION OF THE TIME-DEPENDENT PARTIAL DIFFERENTIAL  
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ABSTRACT

The set of time dependent, parabolic differential equations, which describe the physical and chemical processes in a one-dimensional, laminar, premixed flame is solved by adapting a solution procedure originally developed to solve the two-dimensional steady state boundary layer equations. The flame equations are integrated by an implicit method until the steady state is reached. This corresponds to a flame propagating steadily through a mixture of combustible gases. By a suitable choice of boundary conditions, it is also possible to model a flame which is stabilized on a burner. Solution of the flame equations yields the concentration profiles of the different chemical species as well as the temperature profile. From these one can also calculate the production rates of each species, the rate of each chemical reaction, and the heat release rate at each point in the flame. The velocity of the freely propagating flame can be calculated from the integrals over the whole flame zone of any of the species production rates. The model incorporates realistic thermodynamic data and transport property data that are functions of both temperature and concentration. A complete documentation of the computer program which accomplishes the integration is presented.



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## I. INTRODUCTION

The factors which determine how fast a flame will move through a mixture of combustible gases have been known for many years. Flames of practical interest are generated by chemical reactions involving free radicals. The radical reactions usually have high activation energies and thus are rapid only at elevated temperatures. A flame propagates because the cold gases in its path are raised in temperature by thermal conduction and infused with radicals by mass diffusion from the flame front. It is a simple matter to set up equations for the conservation of energy and mass which give the time dependence of the temperature and concentration of chemical species at different positions in the flame. These equations contain transport parameters and chemical rate constants. The transport parameters consist of mass diffusion coefficients for each chemical species in the mixture and the thermal conductivity of the mixture. The kinetic theory of gases provides the functional dependence of these transport properties on the composition and temperature of the mixture. In addition, it also requires a knowledge of binary diffusion coefficients between pairs of species and the thermal conductivities of the pure species. Although the mathematical expressions for the transport properties are very complex, this constitutes no impediment to the solution of the flame equations; their changes with time are slow and do not require calculation at every integration step. Furthermore, the kinetic theory is sufficiently accurate, and our experimental and theoretical knowledge of diffusion coefficients and thermal conductivities sufficiently good that the transport phenomena impose no major barrier to a flame modeling calculation.

The significant barrier arises from a lack of accurate rate constants and until the last few years in efficient numerical techniques for solving the flame equations. While the rate constant problem remains, it is now possible to solve the flame equations on a high speed digital computer in times ranging from a few seconds to several minutes depending on the complexity of the flame. The method used in the present approach was originally developed by Patankar and Spalding<sup>1</sup> to solve the two-dimensional steady-state boundary layer equations. One of the two

spatial variables in that problem becomes time in the flame equations, while the other represents the distance through the flame front. The equations for the two systems are otherwise the same.

In reading this report, it is desirable to begin with Section II. This gives a brief discussion of the method used to solve the flame equations. Appendices A and B should then be read. These give a simple derivation of the one-dimensional flame equations and show how the steady-state flame velocity is calculated from the solutions of the equations. These equations contain the diffusion velocities of the various species. The expressions used for these are derived in Appendix C. We have considered here only the diffusion arising from a concentration gradient. That caused by a thermal gradient has been neglected. Appendix D, which shows how the thermal conductivity of the flame was calculated can be skipped on a first reading. Appendices E and F show how the flame equations are transformed to a new spatial variable. This is done in two steps. The first is the Von Mises transformation which eliminates the equation for overall mass conservation. This removes the explicit dependence on the flame velocity in the remaining equations. Next, a transformation devised by Patankar and Spalding is used to convert the equations into the form of the boundary layer equations. Appendix G discusses the assumptions underlying the unity Lewis number approximation and can be skipped. Appendices H through K show how the equations are written in finite-difference form, how the various source terms are calculated, the way the grid size is controlled during the integration of the equations, and how the boundary conditions are incorporated into the calculation. The last three Appendices can be skipped.

After this, one can begin with Section III, which gives a detailed discussion of how the program works. Throughout this section, references are made to relevant parts of the Appendices for detailed discussions of the various calculations.

A test case is presented in Section IV. For this a  $H_2$ - $Br_2$  flame was used. This shows the form of the input data and the type of output that the program yields.

## II. METHOD USED TO SOLVE THE FLAME EQUATIONS

In this section we present an outline of the method used to solve the flame equations. The details are presented in the Appendices.

The equations which describe a one-dimensional pre-mixed laminar flame can be written in the form of the classical diffusion equation.

$$\frac{\partial C_j}{\partial t} = \frac{\partial}{\partial x} (\tau_j \frac{\partial C_j}{\partial x}) + S_j \quad j = 1, \dots, N \quad (1)$$

$C_j$  is the concentration of the  $j$ 'th species.  $\tau_j$  is a diffusion parameter and  $S_j$  is the chemical source term for  $j$ . Both  $\tau_j$  and  $S_j$  are functions (presumed to be known) of the concentrations of the various species. The independent variables are the time  $t$  and a distance  $x$ .  $x$  is actually a transformed spatial variable which is roughly proportional to the distance  $y$  which would be measured in the laboratory. ( $x = \psi$ , see Appendix E.) There is one of these equations for each species in the flame, and also a similar equation for the temperature or the enthalpy. The derivation of these equations is given in Appendices A and E.

To solve these equations we set up a two-dimensional grid for the independent variables as shown in Fig. 1. The subscript on  $C$  in the figure now denotes a particular grid point on the  $x$  axis and is not a species index. The primes denote the values of the  $C_m$  at the time  $t'$ . The differential equation (1) gives the value of  $\partial C / \partial t$  in terms of the derivatives of the spatial variable. Thus, at a particular grid point  $m$  we have,

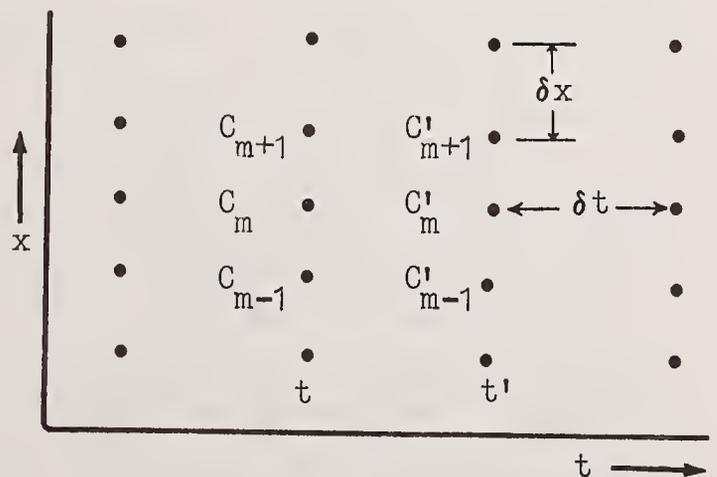


Figure 1. Grid Structure

$$\left\{ \frac{\partial C}{\partial t} \right\}_m = \left\{ \frac{\partial \tau}{\partial x} \right\}_m \left\{ \frac{\partial C}{\partial x} \right\}_m + \tau_m \left\{ \frac{\partial^2 C}{\partial x^2} \right\}_m + S_m \quad (2)$$

To simplify the discussion, let us assume that  $\partial\tau/\partial x = 0$ ; i.e.,  $\tau$  is a constant independent of  $x$ . We also will assume that  $\tau$  is independent of time. This gives

$$\left\{\frac{\partial C}{\partial t}\right\}_m = \tau \left\{\frac{\partial^2 C}{\partial x^2}\right\}_m + S_m \quad (3)$$

To approximate the spatial derivative, assume that  $C$  can be represented by a quadratic function in  $x$  in the region from  $x_{m-1}$  to  $x_{m+1}$ . Thus, let

$$C = a + bx + cx^2 \quad \text{for} \quad x_{m-1} \leq x \leq x_{m+1}$$

From this we get

$$\frac{\partial C}{\partial x} = b + 2cx; \quad \left\{\frac{\partial C}{\partial x}\right\}_m = b + 2cx_m$$

$$\frac{\partial^2 C}{\partial x^2} = 2c = \left\{\frac{\partial^2 C}{\partial x^2}\right\}_m$$

To obtain the parameter  $c$ , we must solve the three equations

$$C_{m+1} = a + bx_{m+1} + cx_{m+1}^2$$

$$C_m = a + bx_m + cx_m^2$$

$$C_{m-1} = a + bx_{m-1} + cx_{m-1}^2$$

A little manipulation of these equations yields

$$c = \frac{1}{2}(C_{m-1} - 2C_m + C_{m+1})/(\delta x)^2$$

where  $\delta x$  is the spacing between the grid points on the  $x$  axis. This gives us an approximate expression for  $\left\{\partial^2 C/\partial x^2\right\}_m$  in terms of the values of  $C$  at the three grid points  $m-1$ ,  $m$ , and  $m+1$ .

$$\left\{\frac{\partial^2 C}{\partial x^2}\right\}_m \approx \frac{1}{\delta x^2} (C_{m-1} - 2C_m + C_{m+1}) \quad (4)$$

From (3) and (4) we have

$$\left\{\frac{\partial C}{\partial t}\right\}_m \approx \frac{\tau}{\delta x^2} (C_{m-1} - 2C_m + C_{m+1}) + S_m \quad (5)$$

$$\left\{\frac{\partial C}{\partial t}\right\}'_m \approx \frac{\tau}{\delta x^2} (C'_{m-1} - 2C'_m + C'_{m+1}) + S'_m \quad (6)$$

Equation (5) gives us an approximate value of  $\left\{ \frac{\partial C}{\partial t} \right\}_m$  which could be used to estimate  $C'_m$  from  $C_m$ ;

$$C'_m \approx C_m + \left\{ \frac{\partial C}{\partial t} \right\}_m \delta t \tag{7}$$

This is not the best estimate for  $C'_m$ . If  $C_m$  varied with time as shown in Fig. 2, one could get a value of  $C'_m$  closer to the true value by using the average of  $\frac{\partial C}{\partial t}_m$  and  $\left\{ \frac{\partial C}{\partial t} \right\}'_m$ ; i.e.,

$$C'_m \approx C_m + \frac{1}{2} \left[ \left\{ \frac{\partial C}{\partial t} \right\}_m + \left\{ \frac{\partial C}{\partial t} \right\}'_m \right] \delta t \tag{8}$$

To be general, let us take a weighted average of the two time derivatives,

$$\left\langle \frac{\partial C}{\partial t} \right\rangle = (1 - \lambda) \left\{ \frac{\partial C}{\partial t} \right\}_m + \lambda \left\{ \frac{\partial C}{\partial t} \right\}'_m$$

Our estimate for  $C'_m$  will now be

$$C'_m \approx C_m + \left\langle \frac{\partial C}{\partial t} \right\rangle \delta t$$

$$= C_m + (1 - \lambda) \frac{\delta t}{\delta x^2} (C_{m-1} - 2C_m + C_{m+1}) + (1 - \lambda) \delta t S_m + \lambda \frac{\delta t}{\delta x^2} (C'_{m-1} - 2C'_m + C'_{m+1}) + \lambda \delta t S'_m \tag{10}$$

We are now confronted with the problem of determining  $S'_m$ , the value of the source term at  $t'$ . For a small change in  $S_m$ , we have

$$S'_m \approx S_m + \frac{\partial S_m}{\partial C_{1m}} (C'_{1m} - C_{1m}) + \dots + \frac{\partial S_m}{\partial C_{Nm}} (C'_{Nm} - C_{Nm})$$

where the first subscript on  $C$  is a species index. The difficulty here is that we don't have values for the  $C'_{jm}$  until an equation like (10) is evaluated for each species  $j$ . To get around this problem, we assume that all of the partial derivatives  $\frac{\partial S_m}{\partial C_{jm}}$  are negligible except in the case of the particular species under consideration in (10).

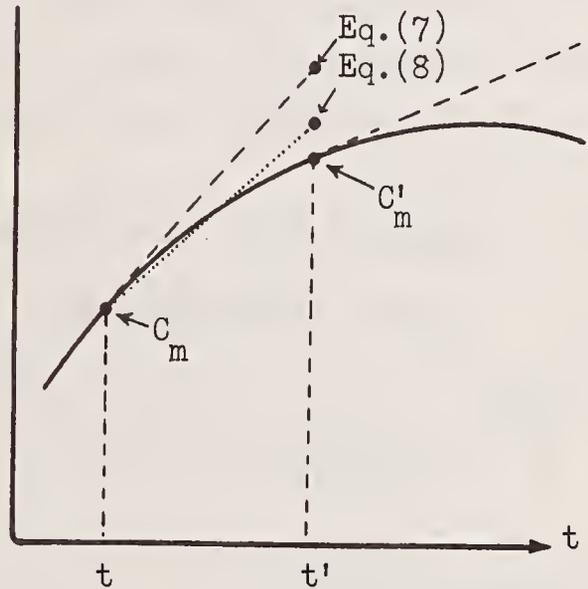


Figure 2.

We then have, (dropping the species index),

$$S'_m \approx S_m + \frac{\partial S_m}{\partial C_m}(C'_m - C_m) \quad (11)$$

This is obviously not a good assumption. Nevertheless, we need only a rough estimate of  $S_m$  at  $t'$ . As  $t$  increases, we eventually reach a steady-state in which  $C'_m = C_m$ . Therefore  $S_m$  also stops changing with time. One could probably use the value of  $S_m$  at time  $t$  for  $S'_m$  and still reach the same steady-state. Using (11) to estimate  $S'_m$ , however, allows one to take somewhat larger time steps and consequently speeds up the integration.

Inserting (11) into (10) and rearranging, we get,

$$C'_m = A_m C'_{m-1} + A_m C'_{m+1} + B_m \quad (12)$$

where

$$A_m = A(\lambda)/D_m(\lambda)$$

$$B_m = B_m(\lambda)/D_m(\lambda)$$

$$A(\lambda) = \lambda \delta t \tau / (\delta x)^2$$

$$B_m(\lambda) = (1 - \lambda) \frac{\delta t \tau}{\delta x^2} (C_{m-1} + C_{m+1}) + \left[ 1 - 2(1 - \lambda) \frac{\delta t \tau}{\delta x^2} - \lambda \delta t \frac{\partial S_m}{\partial C_m} \right] C_m + S_m \delta t$$

$$D_m(\lambda) = 1 + 2\lambda \frac{\delta t \tau}{\delta x^2} - \lambda \delta t \frac{\partial S_m}{\partial C_m}$$

We are free to choose any value between 0 and 1 for the weighting parameter. If we take  $\lambda = 0$ , then

$$C'_m = B_m(0)/D_m(0) = \frac{\delta t \tau}{\delta x^2} (C_{m-1} - 2C_m + C_{m+1}) + S_m \delta t$$

This is an example of a so-called explicit difference equation, in which the value of  $C'_m$  is calculated from the concentrations at time  $t$  only.

When  $\lambda \neq 0$  we have an implicit difference equation. This gives an equation connecting three grid points at time  $t'$ . This results in a set

of simultaneous algebraic equations (12) which must be solved. This set has the form

$$\begin{array}{rcl}
 A_1 C'_0 - C'_1 + A_1 C'_2 & & = -B_1 \\
 & 0 & \\
 A_2 C'_1 - C'_2 + A_2 C'_3 & & = -B_2 \\
 & & \\
 A_3 C'_2 - C'_3 + A_3 C'_4 & & = -B_3 \\
 & \vdots & \\
 & & \\
 & & A_{M-1} C'_{M-2} - C'_{M-1} + A_{M-1} C'_M = -B_{M-1}
 \end{array}$$

where  $M$  is the maximum number of grid points along the  $x$  axis.

For this particular system involving three grid points, the solution of the above system of equations can be obtained without having to invert a matrix. Equation (12) can be written in the simpler form,

$$C'_m = A_m^* C'_{m+1} + B_m^* \quad (13)$$

where

$$\begin{aligned}
 A_m^* &= \frac{A_m}{1 - A_m A_{m-1}^*} \\
 B_m^* &= \frac{A_m B_{m-1}^* + B_m}{1 - A_m A_{m-1}^*}
 \end{aligned} \quad (14)$$

$$A_1^* = A_1$$

$$B_1^* = A_1 C'_0 + B_1$$

The values of  $C'_m$  at the first and last grid points  $C'_0$  and  $C'_M$  are always specified by the boundary conditions for the problem. We can solve (12) by first calculating  $A_m^*$  and  $B_m^*$  for all the grid points starting from  $m = 1$ . It is then easy to obtain the  $C'_m$  values successively with (13) starting from  $C'_{M-1}$  and working down to  $C'_1$ .

This implicit method is what we have used to solve the flame equations. The actual form of the difference equations for the species and enthalpy used in the program is derived in Appendix H. While the methods used there to approximate the spatial derivatives are somewhat different from those used in this section, the basic solution method is the same.

### III. DETAILED DESCRIPTION OF THE PROGRAM

The following is a step-by-step description of the main program and each of its subroutines. Flow diagrams for most of the routines are provided at the end of each section.

#### A. Description of the Main Program SPALD/3

The function of SPALD is to control the calculation. It uses various subroutines to calculate the parameters contained in the finite-difference coefficients in Eq. A40, Appendix H, p. A18. After calculating the coefficients, it then calls a routine CALC which solves the difference equations and yields values of the concentrations and enthalpy a time  $t + \delta t$ . It continues for as many time steps as the user specifies. No test is made for convergence.

SPALD begins by reading the following data:

1) RUNID(M), M=1,12

Format(12A6)

This is the run identification. It consists of 12 words, in A6 format, making 72 characters available for whatever identifying remarks the user wishes to employ.

2) HEADNG(M), M=1,126

Format(63A1)

This provides headings for the output profiles. There are 126 characters available. Two cards must be used. The output profiles are listed in 14 columns. The first 12 are for species concentrations and are in E9.5 format. Column 13 contains the enthalpy profile and is in E10.5 format; this extra space is used for the sign of the enthalpy which can be negative. The last column is for the temperature profile and has E9.5 format. The headings should be spaced within the 126 character range available so that they are aligned at the top of the appropriate column.

3) LH, LO, LOH, LHO2, LH2O2, LX, LHX, LX2, LO2, LH2O, LH2, LN2

Format(14I2)

These are species indices for the  $H_2-O_2-N_2$  flame. They should run from 1 to 12. In earlier versions of the program they were used in the subroutine which calculated the chemical source term. Now, however, they are needed only in a portion of the routine which calculates transport properties. They must be included although they are used only for the  $H_2-O_2$  flame.

4) LD,LE,LC

Format(3I2)

These three numbers determine the number of integration steps for which certain subroutines are called at each step. For example, if  $LD = 20$ , then  $TRANS(1)$  is called for the first 20 integration steps. This means that a complete calculation of the transport properties is made at each of the first 20 steps. The second number controls the calling of  $SCHM1$ , which calculates the part of the chemical source term arising from the diffusion of the mean molecular weight.  $SCHM1$  is called at each integration step for the first  $LE$  steps. The last number controls the reference to  $SENTP$ , the routine which calculates the enthalpy source term. This will be called at every step for the first  $LC$  steps.

5) LDELA,LDELB,LDELC

Format(3I2)

These numbers control the number of time steps for which  $TRANS(1)$ ,  $SCHM1$ , and  $SENTP$  are not called after the first  $LD$ ,  $LE$ , and  $LC$  steps, respectively. For example, after calculating the transport properties for the first  $LD$  steps, the program, from this point on, will calculate them only at intervals of  $LDELA$  steps.

There are two special cases which can arise;

a) Suppose one does not wish to call  $SCHM1$  at any time during the calculation. In most flames, the mean molecular weight changes little throughout the flame so that this part of the chemical source term can be neglected. To do this, let  $LE = 0$  and  $LDELB = 0$ .

b) If one is considering a constant enthalpy flame,  $SENTP$  need not be called. To prevent this, let  $LC = 0$  and  $LDELC = 0$ .

6) N, NII, NEE, OMR

Format(3I2, D5.1)

N is the number of slices into which the  $\omega$  axis from 0 to 1 is divided into to form a grid. It can have a maximum value of 40. The grid points start at  $I = 2$  and run to  $N + 2$ . (See Appendix H, p. A19.) Slices between the grid points NII and NEE have equal widths. There is a non-uniform grid spacing near the boundaries with the spacing increasing from  $I = NII$  to 2 and from NEE to  $N + 2$  so that consecutive slices are in the fixed ratio OMR. This non-uniform spacing is used to increase the number of grid points in the region which has the largest concentration gradients. The spacing of the central grid points is given by the formula,

$$\left[ \frac{\text{OMR}^{**}(\text{NII} - 1) + \text{OMR}^{**}(\text{N} - \text{NEE} + 3) - 2.*\text{OMR}}{\text{OMR} - 1} + \text{NEE} - \text{NII} \right]^{-1}$$

An example of this grid spacing is shown in Fig.3 for the case,  $N = 15$ ,  $NII = 4$ ,  $NEE = 14$ , and  $\text{OMR} = 1.5$ .

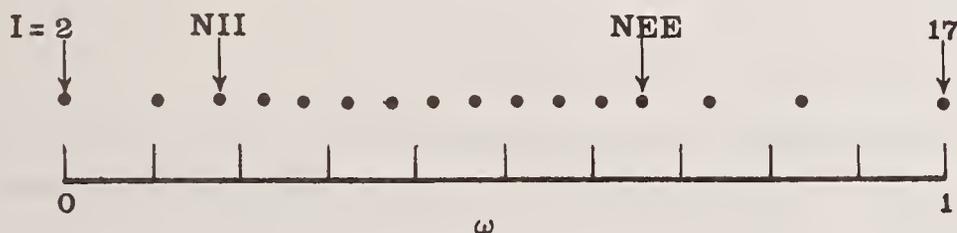


Figure 3

Sample Grid Spacing

The grid points NII and NEE also denote the positions near the hot and cold boundaries, respectively, used in the grid width control procedure. They correspond to the points HH and CC (see Appendix J).

7) JH, JRAD, JM, JMR, JT, JTR, ITER, ITER1, JENTRN, JBODYA, JBODYB

Format(11I2)

JH is the index of the enthalpy variable  $F(\text{JH}, I)$ . It can have a maximum value of 13. It must be larger by one than the total number of species in the flame. The index of the last species will then be  $\text{JHM1} = \text{JH} - 1$ . In the output, the number of the column will correspond

to the first index in  $F(J,I)$ . The enthalpy profile, however, will always appear in column 13.

JRAD is the index of the last minor species. Minor species are always given the lowest index values. Major species start with the index  $JRADP1 = JRAD + 1$ .

JM and JMR are indices for the mean molecular weight and its reciprocal which are contained in the array FS. The mean molecular weight is  $FS(JM,I)$  and its reciprocal is  $FS(JMR,I)$ . We use the values  $JM = 1$ , and  $JMR = 2$ .

JT and JTR are indices for the temperature and its reciprocal which are contained in the array FS. The temperature is  $FS(JT,I)$  and its reciprocal is  $FS(JTR,I)$ . We use  $JT = 3$  and  $JTR = 4$ .

ITER is the number of iterations used in the Newton-Raphson procedure to calculate the temperature from the enthalpy value at a grid point.  $ITER = 2$  has been found to give satisfactory results. (See Appendix M, p. A39.)

ITER1 is not used. Set it equal to zero.

JENTRN is the index of the species which is used to calculate the entrainment rate. A major species is always used for this purpose. (See Appendix J, p. A27.)

JBODYA and JBODYB are indices for third body concentrations which are stored in the array FS.  $FS(JBODYA,I)$  is the third body concentration to be used for the reaction  $H + OH + M \rightleftharpoons H_2O + M$ , and  $FS(JBODYB,I)$  that for the reaction  $H + O_2 + M \rightleftharpoons HO_2 + M$  in the  $H_2-O_2$  flame calculations. These concentrations depend on the mixture composition and are calculated in TRANS. Normally one uses the reciprocal of the mean molecular weight for the third body concentration in the mole/kg concentration units. We let  $JBODYA = 5$  and  $JBODYB = 6$ .

8)  $XI(J), J=JRADP1, JHM1$

$XE(J), J=JRADP1, JHM1$

Format(5D7.3)

These are the mole fractions of all the major species at the hot and cold boundaries, respectively, at the start of the calculation.

For example, consider a  $H_2-O_2-N_2$  flame having on the cold side  $XE(LO_2) = 0.063$ ,  $XE(LH_2O) = 0.0$ ,  $XE(LH_2) = 0.700$ , and  $XE(LN_2) = 0.237$ .

Assuming that the reaction goes to completion, we get hot side values

$$XI(LO_2) = 0.0$$

$$XI(LH_2O) = 2.*XE(LO_2)/(1. - XE(LO_2)) = 0.1345$$

$$XI(LH_2) = (XE(LH_2) - 2.*XE(LO_2))/(1. - XE(LO_2)) = 0.6126$$

$$XI(LN_2) = XE(LN_2)/(1. - XE(LO_2)) = 0.2529$$

9) IPRFL(J), J=JRADP1, JHM1

Format(5I1)

If IPRFL(J) = 0, the initial profile for the major species J decays from the cold to the hot side of the flame. If IPRFL(J) = 1, then we have a growth profile. For the above  $H_2-O_2-N_2$  example,

$$IPRFL(LO_2) = 0$$

$$IPRFL(LH_2O) = 1$$

$$IPRFL(LH_2) = 0$$

$$IPRFL(LN_2) = 1$$

10) TCOLD, PRESS, YWIDTH

Format(D6.0, D6.2, D5.1)

TCOLD is the temperature of the cold boundary in degrees K. PRESS is the pressure of the flame in atmospheres. YWIDTH is the approximate width of the flame front in meters. The program needs a value for this quantity to start the calculation. It uses this to calculate a starting value for the grid width parameter  $PEI = \eta = \psi_C - \psi_H$ . (See Appendix F, p. A11.) Although the grid width is automatically adjusted at each step in the calculation, it is best to start with a value as close as possible to the final value. We have not experimented with the effect of the choice of YWIDTH (and thus the initial PEI) on the course of the calculation.

11) HREF(J),DCP(J),ECP(J),FCP(J),WT(J),J=1,JHM1

Format(4D12.0,D10.2)

HREF(J) is the reference enthalpy value  $h_j^0$  for the species J in joules/kg. (See Appendix M, Eqs. (A78a) and (A78b), p. A37 for their definition.) DCP(J), ECP(J), and FCP(J) are the parameters  $d_j$ ,  $e_j$ , and  $f_j$  in the quadratic expression used for the heat capacities of the individual species. The units are joules/kg-deg. (See Appendix M, Eq. (A76), p. A37.)

It is worth noting here that one needs values of these parameters only for the major species. The enthalpy,  $h = \sum_j h_j Y_j$ , so that contributions from species having small  $Y_j$  are not important in its calculation. The  $h_j$  (actually  $h_j^* = M_j h_j$ ) also appear in the source term for the enthalpy as the products  $h_j^* \partial \varphi_j / \partial \omega$ . Here again the contributions from the trace species will be negligible compared to that from the major species. This is so because the values of  $\partial \varphi_j / \partial \omega$  will be small when  $\varphi_j$  is small even though these gradients could be large on a percentage basis. Calculation time could be saved by not summing over the trace species in the enthalpy source term. The effect of neglecting these terms should be examined in future work.

WT(J) is the molecular weight of each species in kg/mole.

12) DXMIN,DXMAX,STEPS

Format(2D5.0,D6.0)

The size of the time step DX is determined from the formula  $DX = DXMIN + ((DXMAX - DXMIN)/STEPS)*L$ , where L is the number of the step and STEPS is a number comparable to the maximum number of steps to be used. The step size increases linearly from DXMIN to DXMAX when  $L = STEPS$ . We normally have  $DXMIN \ll DXMAX$ . Thus DX is small at the beginning of the calculation and increases as the steady-state is approached. Other step size control formulas could be investigated. Perhaps the step size could be made inversely proportional to the rate of change of the grid width PEI. As the steady-state is reached, PEI changes little from step-to-step justifying larger values of DX. At the beginning of the calculation, when the profiles are changing rapidly,

PEI will also be changing and the step size would be kept small.

13) LMAX, LPRINT

Format(2I3)

LMAX is the maximum number of steps to be used in a given calculation. LPRINT specifies the step interval at which profiles are to be printed. One page of profiles is printed every LPRINT steps. When L reaches LMAX, the complete output is printed.

14) ALPHA

Format(F3.2)

ALPHA is the parameter  $\nu$  which appears in the grid control formulas Eqs. A58-60. (See Appendix J, p. A26.) We have been setting ALPHA equal to 0.1.

15) KRAT

Format(I2)

This is the number of chemical reactions. For a particular reaction, the forward and reverse reactions count as one reaction. (Note that KRATE = KRAT.) KRAT can have a maximum value of 30.

16) FRQ(K), BETA(K), EACT(K), BFLAG(K), EFLAG(K), K=1, KRATE

Format(D6.2, D7.2, D8.2, 2L1)

These are the Arrhenius parameters for the forward rate constant of reaction K. If we express the temperature dependence of a rate constant  $k$  as  $k = AT^\beta \exp(-E/RT)$  then

$$FRQ(K) = A$$

$$BETA(K) = \beta$$

$$EACT(K) = E/R \text{ (in } ^\circ K)$$

$k$  must be in units of  $m^3/mole$  and  $m^6/mole^2$  for two and three body reactions, respectively.

BFLAG(K) = .TRUE. if BETA(K)  $\neq$  0, and EFLAG(K) = .TRUE. if EACT(K)  $\neq$  0.

17) EQA(K),EQB(K),EQC(K),BOD1(K),BOD2(K),K=1,KRATE  
 Format(3D11.5,2L1)

The first three quantities are the coefficients in the expansion of the free energy  $\Delta F^{\circ}$  of each reaction as a power series in the temperature. We use

$$-\Delta F^{\circ} = \text{EQA}(K) + \text{EQB}(K)*T + \text{EQC}(K)*T^2$$

The values of these coefficients must be such that  $\Delta F^{\circ}$  will be in units of kcal/mole.

If BOD1(K) = .TRUE., then either the forward or reverse reaction is third order. If BOD2(K) = .TRUE., then the reverse reaction is third order.

18) ITEST(M),M=1,10  
 Format(10I1)

When set to unity these flags produce various outputs used for checking the operation of the program. They are normally set to zero. An examination of the listing of SPALD will show what quantities are printed. They should be used only for flames with  $JH \leq 6$ . With some modification of SPALD they could be used for larger values of JH.

19) INPUT  
 Format(I2)

When INPUT = 1, input profiles are supplied from a data file. This file must contain the concentration and enthalpy profiles and the grid width PEI. This file is taken from logical unit 45 by the statement READ(45) ((F(J,I),J=1,JH),I=1,NP3),PEI. These can be output profiles from a previous calculation. The output profiles are placed in logical unit 46 by the statement WRITE(46) ((F(J,I),J=1,JH),I=1,NP3),PEI. (NP3 equals N + 3) If the user is generating his own input profiles, they must be stored in the above order and be double precision numbers. Note that F(J,1) and F(J,NP3) are not actually used and should be given zero values. Remember that F(JH,I) is the enthalpy, and that small values of I refer to the hot side of the flame.

When INPUT = 0, the initial profiles are generated by INITL.

20) `LORDER(J), J=1, JHM2`

`Format(14I2)`

These numbers specify the order in which one wishes the species equations to be solved. Spalding has found that the efficiency of the calculation is improved by solving the species equations in order of increasing species concentration. The trace species equations are solved first, followed by the major species equations. (Note that  $JHM2 = JH - 2$ ; this is the index of the next to the last major species.) The species  $JHM1$  is calculated from the relation  $\sum \varphi_j M_j = 1$ , and so `LORDER(JHM1)` need not be specified.

As an example, consider the  $H_2$ - $Br_2$  flame ( $JH = 6$ ) where the species have been given the indices  $Br = 1$ ,  $H = 2$ ,  $H_2 = 3$ ,  $Br_2 = 4$ , and  $HBr = 5$ . If we wanted to solve the species equations in the order  $H$ ,  $Br$ ,  $Br_2$ ,  $H_2$ ,  $HBr$ , we would have

`LORDER(1) = 2`

`LORDER(2) = 1`

`LORDER(3) = 4`

`LORDER(4) = 3`

The equation for  $J = 5$  will automatically be solved last.

(Note that in the test case to be presented later for which this flame was used, we solved the species equations in the order of species index, i.e., we used `LORDER(J) = 1, 2, 3, 4.`)

21) `SPECIE(J), J=1, JHM2`

`Format(14L1)`

If `SPECIE(J) = .FALSE.`, then the equation for that  $J$  is not solved.

22) `REACT(K), K=1, KRATE`

`Format(30L1)`

If `REACT(K) = .FALSE.`, then both the forward and reverse rate constants for reaction  $K$  are set to zero.

23) `FFLAG(K), K=1, KRATE`

`RFLAG(K), K=1, KRATE`

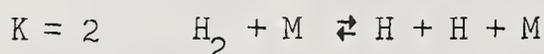
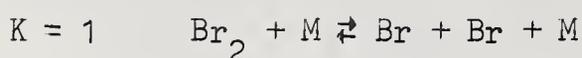
`Format(30L1)`

If  $\text{FFLAG}(K) = \text{.FALSE.}$ , then the forward rate constant for reaction  $K$  is set to zero; if  $\text{RFLAG}(K) = \text{.FALSE.}$ , then the reverse rate constant is set to zero.

24)  $\text{LA}(K), \text{LB}(K), \text{LRA}(K), \text{LRB}(K), K=1, \text{KRATE}$

Format(4I2)

These numbers specify the reaction mechanism to be used. Consider the  $\text{H}_2\text{-Br}_2$  flame as an example. We use the reactions,



For a species index assignment  $\text{Br} = 1, \text{H} = 2, \text{H}_2 = 3, \text{Br}_2 = 4, \text{HBr} = 5$ , we have the following assignment for  $\text{LA}, \text{LB}, \text{LRA}$ , and  $\text{LRB}$ :

K	LA(K)	LB(K)	LRA(K)	LRB(K)
1	4	0	1	1
2	3	0	2	2
3	1	3	5	2
4	2	4	5	1

We see that the indices  $\text{LA}(K)$  and  $\text{LB}(K)$  refer to the species entering into the forward reaction, while  $\text{LRA}(K)$  and  $\text{LRB}(K)$  pertain to those contained in the reverse reaction. There is space for two species on each side of a reaction. The third body  $\text{M}$  is not counted as a species. In cases where there is only one species aside from  $\text{M}$  involved in the reaction, one must always set either  $\text{LB}(K)$  or  $\text{LRB}(K)$  equal to zero. The index of the one species present should be assigned to  $\text{LA}(K)$  or  $\text{LRA}(K)$ . For example, if we had written reaction 1 as



then the assignment would have been,

$$\text{LA}(1) = 1, \text{LB}(1) = 1, \text{LRA}(1) = 4, \text{LRB}(1) = 0.$$

25) FLAG(I), I=1,20

Format(30L1)

These flags provide a series of options regarding the calculation of transport properties, source terms, and the nature of the output.

FLAG(1) = .FALSE. The complete calculation of the transport parameters is made.

= .TRUE. Constant and equal transport parameters are used.

These are contained in the array PREF and are assigned values in a DATA statement in SPALD, line SPDO0350. Constant, but non-equal values could be used by changing this data statement.

FLAG(2) = .FALSE. If the flame contains a buffer gas like  $N_2$  in a low temperature  $H_2-O_2-N_2$  flame, its mass fraction can be considered to remain constant throughout the flame. This buffer will be assigned the species index JHM1. The relation between the mass fractions gives

$$\sum_{J=1}^{JHM3} Y_J + Y_{JHM2} + Y_{JHM1} = 1.$$

When  $Y_{JHM1}$  is the buffer mass fraction and is kept constant,  $Y_{JHM2}$  can be calculated from this equation and its species equation need not be solved. Therefore when this option is used one must also have SPECIE(JHM2) = .FALSE..

= .TRUE. The buffer concentration is allowed to vary. This option must be used for flames containing no buffer species.

FLAG(3) = .FALSE. The pressure equals 1 atmosphere.

= .TRUE. The pressure is different from 1 atmosphere.

FLAG(4) (Not used)

FLAG(5) = .FALSE. The enthalpy source term is not calculated. This option can also be achieved by using LC = 0 and LDELC = 0.

= .TRUE. The enthalpy source term is calculated.

FLAG(6) (Not used)

FLAG(7) = .FALSE. The transport properties for the ozone decomposition flame are not calculated.

= .TRUE. The transport properties for the ozone flame are calculated.

FLAG(8) = .FALSE. The transport properties for the hydrogen-bromine flame are not calculated.

= .TRUE. The transport properties for the hydrogen-bromine flame are calculated.

FLAG(9) = .FALSE. The rate constants as a function of distance through the flame are not printed in the final output.

= .TRUE. The rate constants are printed.

FLAG(10) = .FALSE. The rates of each reaction at each point in the flame are not printed in the final output.

= .TRUE. The reaction rates are printed.

FLAG(11) = .FALSE. Heat release rates for individual reactions at each point in the flame are not printed in the final output.

= .TRUE. Heat release rates for the reactions are printed.

FLAG(12 through 20) (Not used)

This completes the input data requirements for SPALD.

After reading this data SPALD calls the subroutine SWITCH. Its function is to set three flags, G1, G2, and G3, which describe the nature of each reaction. These flags are then used in the calculation of the chemical source term by subroutine SCHM2.

FLAG(1) is then tested. If false the routine DIFUSE is called. This calculates binary diffusion coefficients for the  $H_2-O_2-N_2$  flame from Lennard-Jones parameters. The values of these parameters are contained in a DATA statement in DIFUSE.

The index JSTOP is set equal to JHM3. FLAG(2) is then tested; if true, then the buffer gas will vary and JSTOP is equated to JHM2.

The flag INPUT is tested. If INPUT = 1, profiles are read from a file located in logical unit 45.

The quantities L, LCOUNT, X, and DX are initialized. L is the number of the step, LCOUNT is the number of steps taken since the last profile print-out, X is the time variable, and DX is the time step size.

The grid structure is calculated by a call to OMEGA.

The flag INPUT is again tested and if equal to zero INITL is called to calculate the initial profiles.

After these preliminaries, the point 706 is reached to which the program returns after every integration step. This therefore represents the beginning of each step. TRANS(1) is then called if a complete transport calculation is to be made, otherwise TRANS(0) is called. TRANS(0) calculates the temperature at each grid point from the enthalpy and stores it in FS(JT,I). TRANS(1) in addition calculates the transport parameter array PREF(J,I).

The flag INPUT is tested once again and, if zero, an initial value of PEI is determined by calling ZCALC with PEI = 1.0. This routine produces a value of Y(I), the laboratory spatial variable, at the cold boundary grid point NP2. PEI is then taken to be the ratio YWIDTH/Y(NP2). To see the rationale behind this, refer to Eq. (A70), Appendix L. This formula gives for Y(NP2)

$$Y(NP2) = 2\eta \sum_{i=2}^{N+2} \frac{\omega_{i+1} - \omega_i}{\rho_{i+1} - \rho_i}$$

The quantities  $\omega_i$  were calculated in OMEGA, and the  $\rho_i$  were calculated in TRANS(1). We want to have a value of  $\eta$  (=PEI) such that  $Y(NP2) = YWIDTH$ . By calling ZCALC with PEI = 1.0, we get the quantity

$$2 \sum_{i=2}^{N+2} \frac{\omega_{i+1} - \omega_i}{\rho_{i+1} - \rho_i}$$

Dividing YWIDTH by this gives a value of PEI which makes  $Y(NP2) = YWIDTH$ .

SPALD next calculates the quantities  $PEID2 = \frac{1}{2}\eta$ ,  $PEIOMD(I) = \eta(\omega_{i+1} - \omega_i)$ , and  $PEIOM2(I) = \frac{1}{2}\eta(\omega_{i+1} - \omega_{i-1})$ .

RATCN is then called to calculate the values of the rate constants at each point in the flame.

To calculate the entrainment rates  $\dot{m}_C$  and  $\dot{m}_H$  (see Appendix J, p. A26), it is necessary to have a value of  $R(JENTRN, I)$ ; this is the production rate of species JENTRN in moles/kg-s. Since this is normally calculated by SCHM2 later in the program, on the first integration step it is necessary to call  $SCHM2(JENTRN)$  to calculate the production rate for this species.

SPALD then evaluates the finite-difference coefficients  $A(J, I)$ ,  $B(J, I)$ , and  $D(J, I)$  (minus the source term). These are the quantities  $A'$ ,  $B'$ , and  $D + 2S_F$  of Eq. (A40), Appendix H, p. A18.

If desired, the chemical source term arising from the diffusion of the mean molecular weight is then calculated by a call to SCHM1.

The finite-difference coefficient  $C(J, I)$  (minus the source term) is then calculated. This is the quantity  $C' - 2S_P$  of Eq. (A40).

The chemical source terms  $S_P$  and  $S_F$  are then calculated for each species by a call to  $SCHM2(J)$ . Also calculated by this routine are the production rates  $R(J, I)$ . Following this,  $C(J, I) = C'$  and  $D(J, I) = D$  are calculated.

New species concentrations for time  $t + \delta t$  (i.e.,  $X + DX$ ) are then calculated by calling  $CALC(J)$  in the order specified by the numbers  $LORDER(J)$ .

Next, the hot boundary condition  $\{\partial\varphi/\partial\omega\}_H = 0$  is set by equating the concentration at the hot boundary point to the value at the adjacent grid point; i.e., we set  $F(J, 2) = F(J, 3)$ .

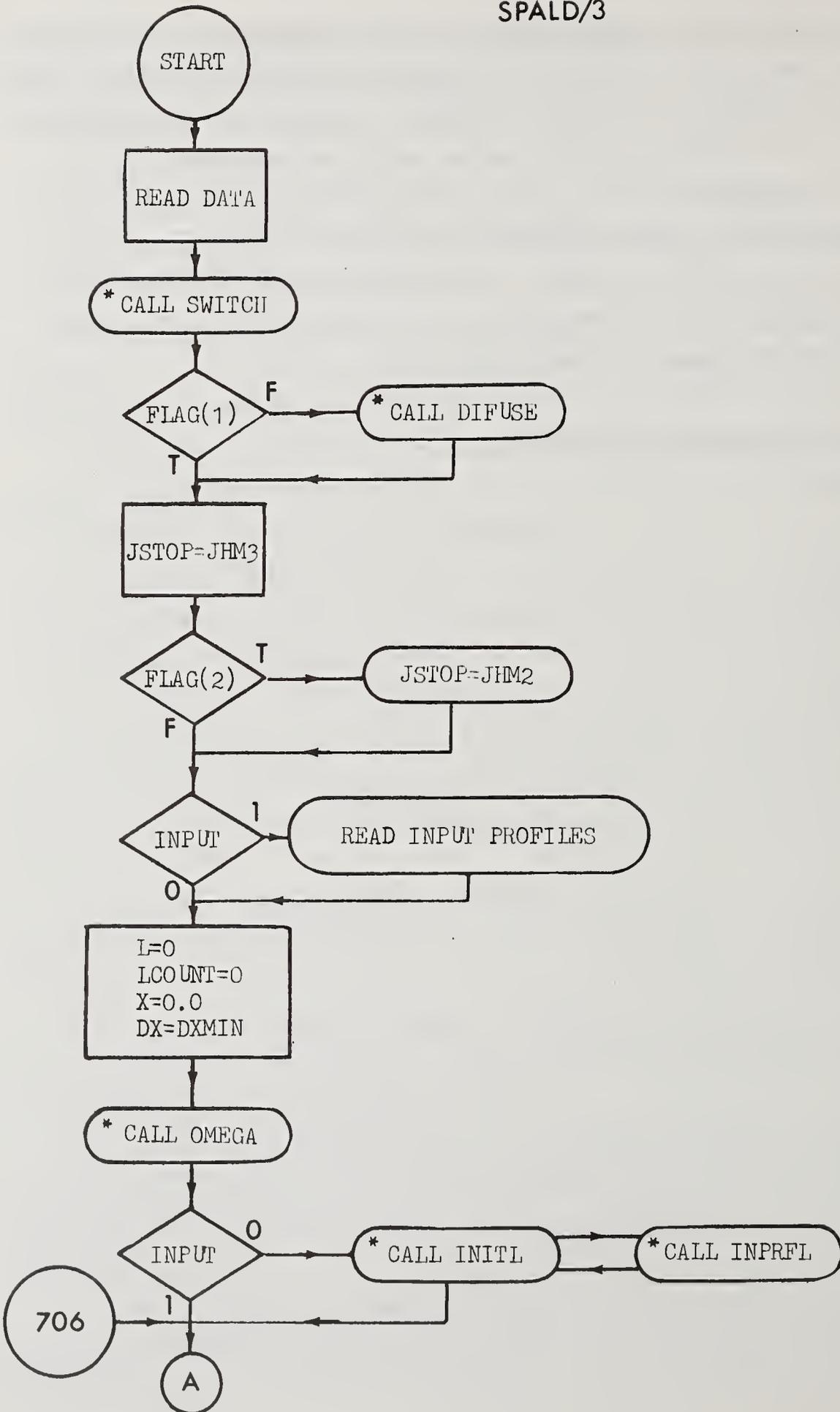
The value of the concentration of the JHM1 species is then calculated from the relation  $\sum_j \varphi_j M_j = 1$ ; if the buffer concentration is being kept constant, then the concentration of the JHM2 species is calculated from this relation.

If the enthalpy is allowed to vary, its source term is calculated by a call to  $SENTP$  and its new value at  $t + \delta t$  is calculated by calling  $CALC(JH)$ .

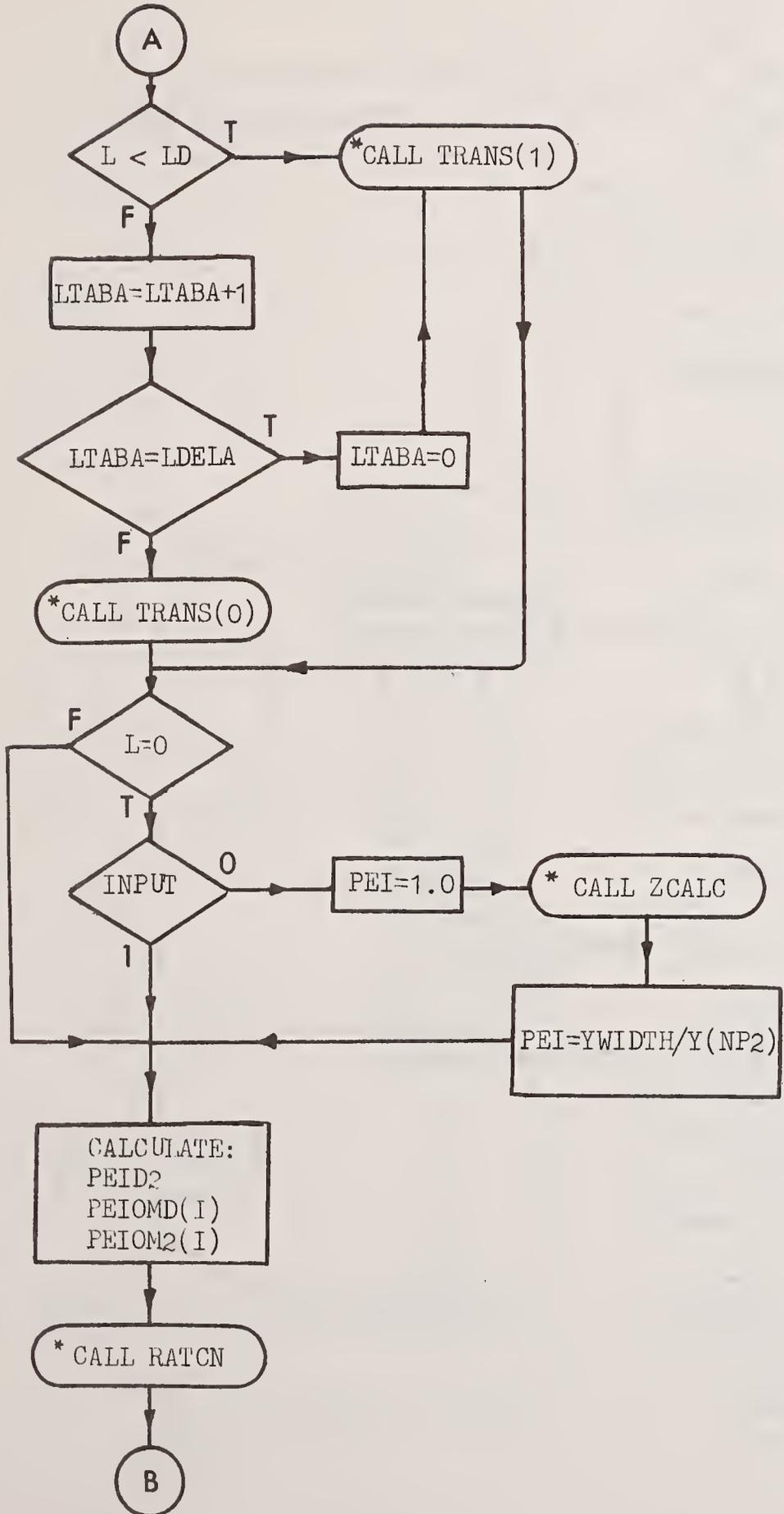
The step counters L and LCOUNT are then incremented. If an output point has been reached, flame velocities are calculated for each species by a call to ENTRN(J,1). These velocities should be the same for each species. In practice, this will happen only for the major species (see Eq. (A25) Appendix F, p. A12). The laboratory distances Y(I) are then calculated by calling ZCALC and the partial results are printed out by a call to OUTPUT. When L = LMAX, the call to OUTPUT produces a complete print out. In addition, the profiles for concentrations and enthalpy are stored in logical unit 46.

If L is not equal to LMAX, then the program returns to point 706 and the calculations are repeated.

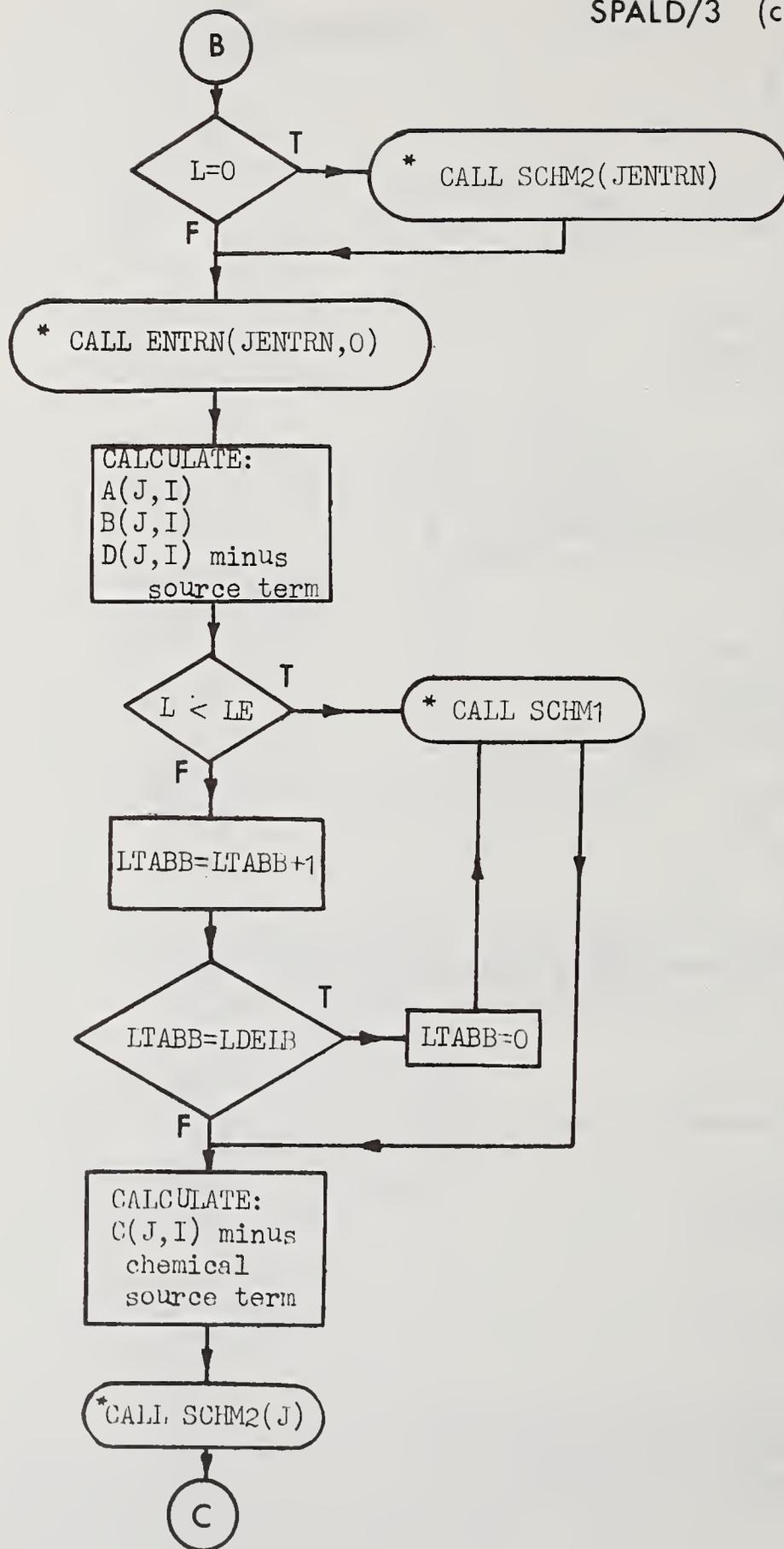
SPALD/3



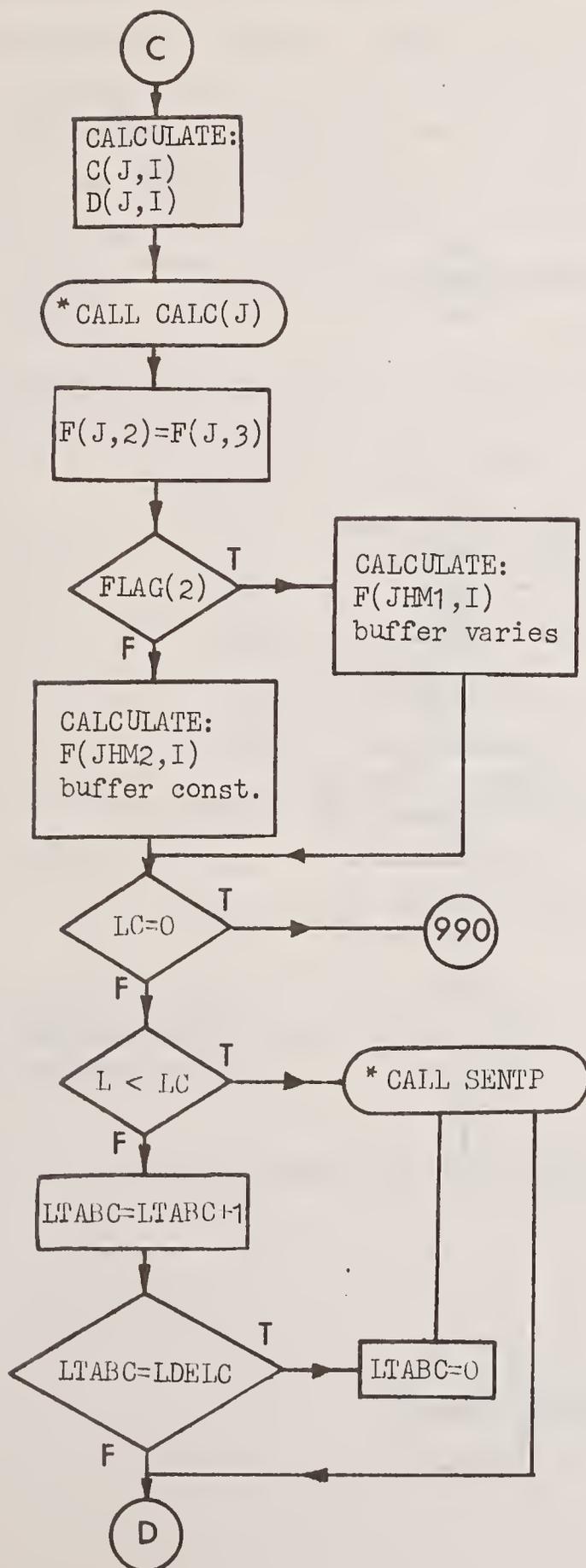
SPALD/3 (cont.)



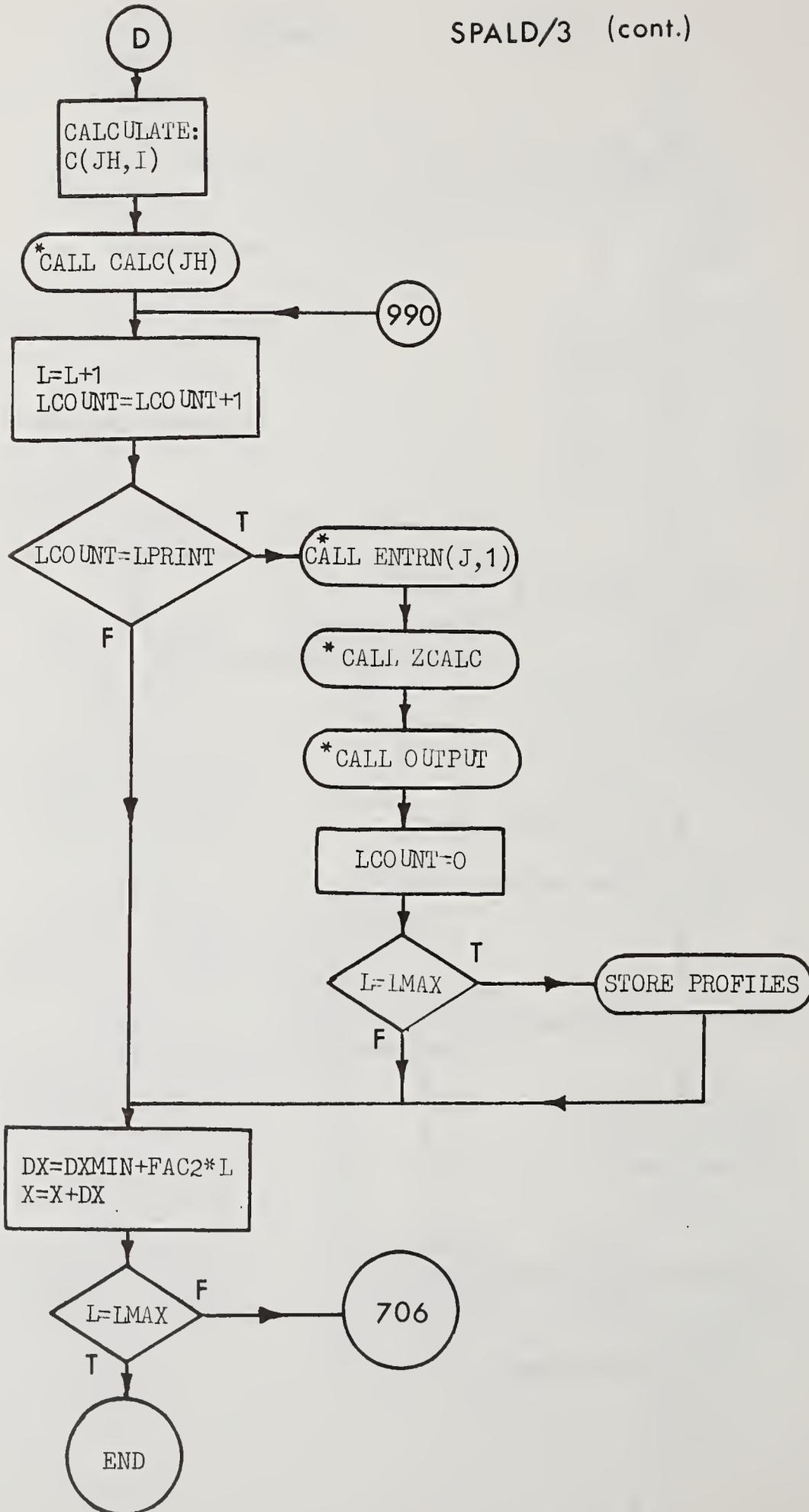
## SPALD/3 (cont.)



## SPALD/3 (cont.)



## SPALD/3 (cont.)



## B. Description of Subroutine SWITCH(KRATE)

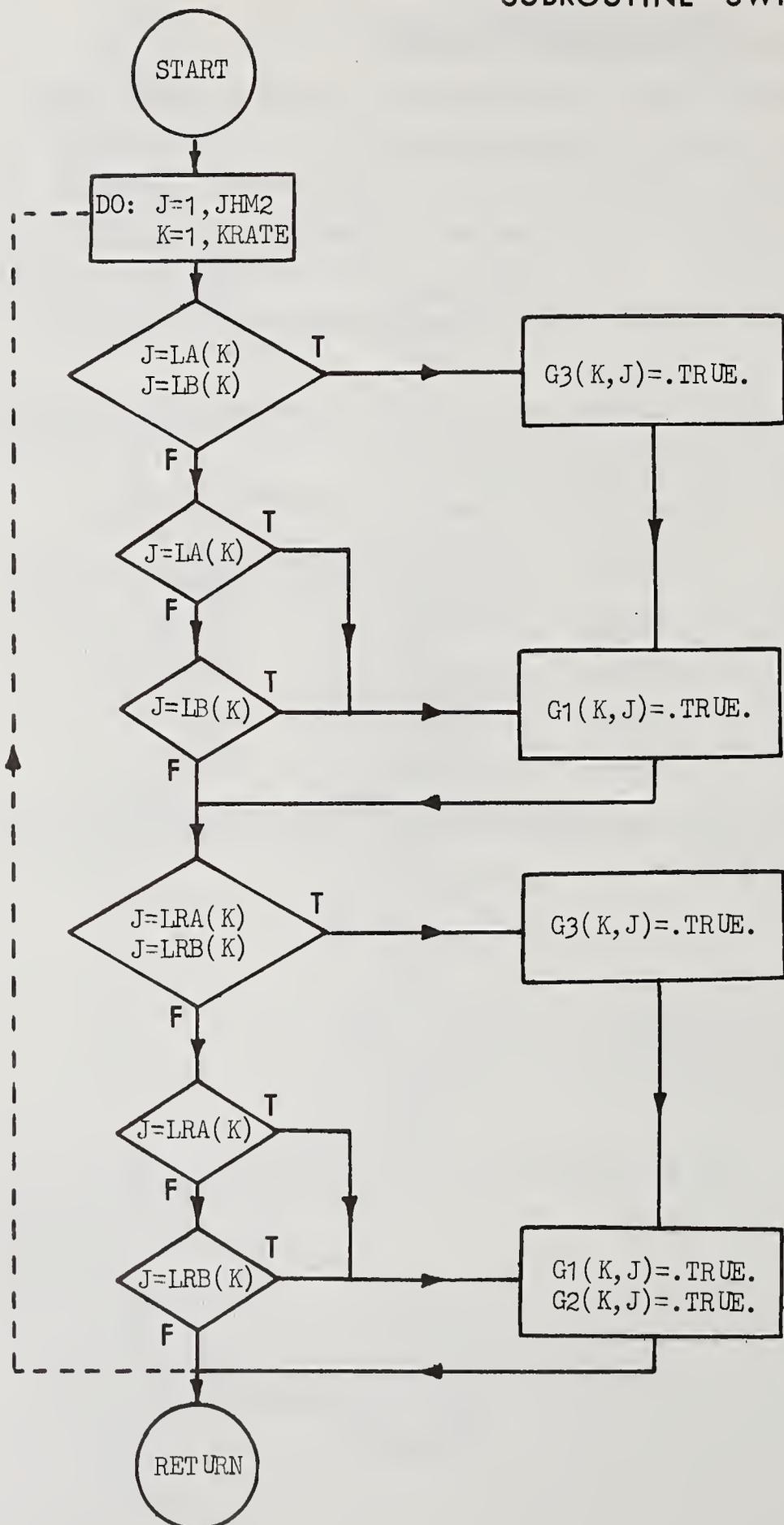
This subroutine sets three flags G1, G2, and G3 which denote the presence and location of a particular species J in a particular reaction K. These flags are initially set to the .FALSE. position. SWITCH sets G3(K,J) to .TRUE. if the species J appears twice on either side of the reaction equation. If it appears on the left side of the equation, G1(K,J) is set to .TRUE.; if on the right side, both G1(K,J) and G2(J,K) are made .TRUE..

As an example, consider the following reactions which are tested for the presence and location of species A.

Reaction	Values of the Flags after Calling SWITCH		
	G1	G2	G3
A + B ⇌ C + D	.TRUE.	.FALSE.	.FALSE.
C + D ⇌ A + B	.TRUE.	.TRUE.	.FALSE.
A + A ⇌ C + D	.TRUE.	.FALSE.	.TRUE.
C + D ⇌ A + A	.TRUE.	.TRUE.	.TRUE.
B + D ⇌ E + F	.FALSE.	.FALSE.	.FALSE.

These flag values are used in SCHM2 in the calculation of the chemical source terms.

## SUBROUTINE SWITCH(KRATE)

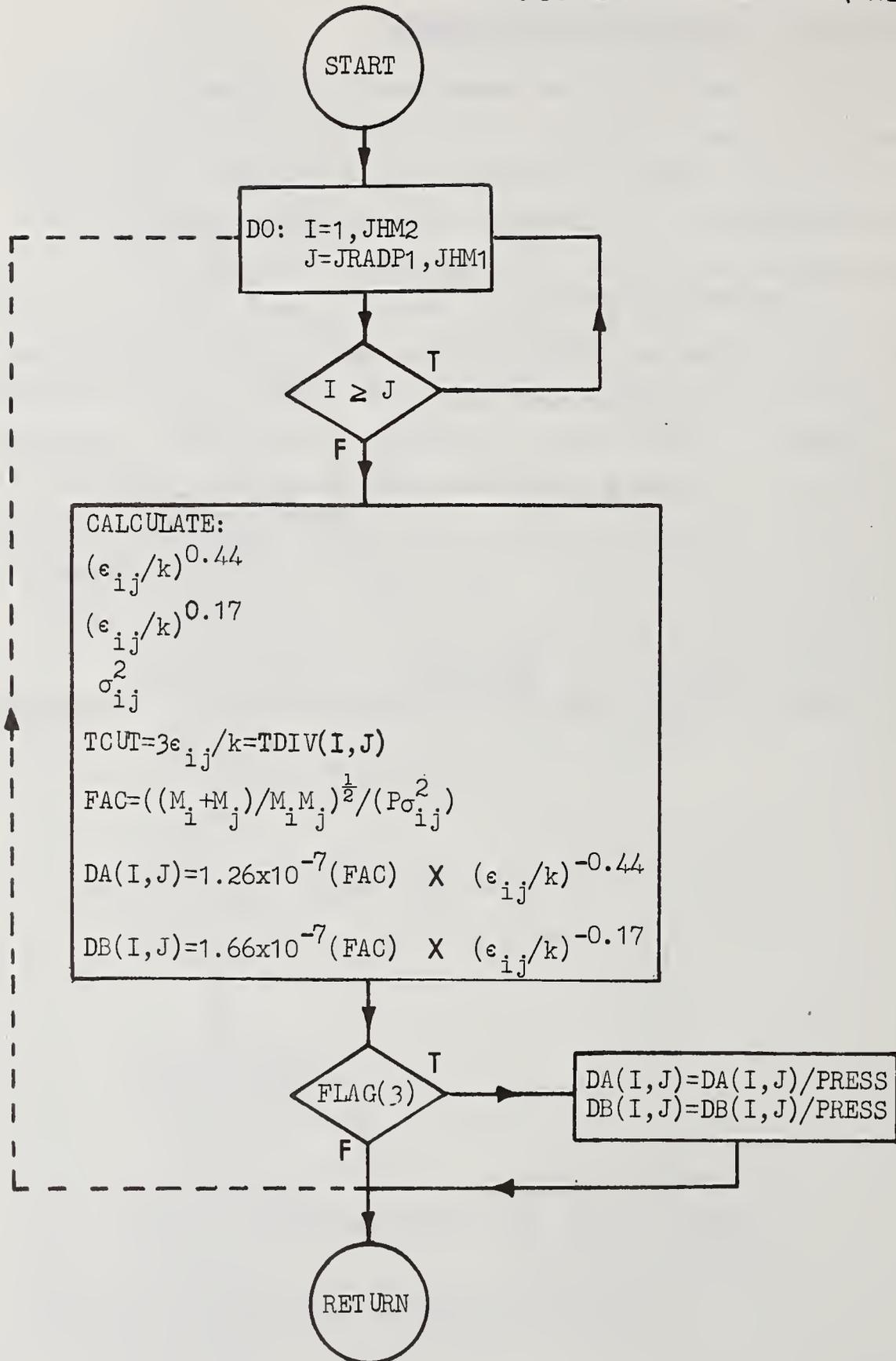


### C. Description of Subroutine DIFUSE(PRESS)

This routine calculates the temperature independent portion of the binary diffusion coefficients for the  $H_2-O_2-N_2$  flame from Lennard-Jones parameters. These parameters are assigned values in the DATA statement starting at line DIF00100. M is an array containing the molecular weights in gm/mole; EK is the Lennard-Jones potential parameter  $\epsilon_j/k$  in  $^{\circ}K$ ; and S is the parameter  $\sigma_j$  in  $\text{\AA}$ . These are listed in order of species index. The quantities calculated are DA(I,J) and DB(I,J) which are the temperature independent portion of Eqs. (A72) and (A73), Appendix M, p. A36. Also calculated is the cut-off temperature below which DA(I,J) is used rather than DB(I,J). This is TDIV(I,J) =  $3\epsilon_{ij}/k$ . I and J are species indices. The index I runs from 1 to JHM2, while J goes from JRADP1 to JHM1. This means that DIFUSE calculates these quantities for interactions between minor and major species, major and major, but not minor and minor.

If FLAG(3) = .TRUE., then DIFUSE divides DA and DB by the pressure PRESS.

## SUBROUTINE DIFUSE(PRESS)



#### D. Description of Subroutine OMEGA

This routine takes the values of N, NII, NEE, and OMR and calculates the grid structure. (See Section III-A, p.11.) The quantities determined are

$$\text{OM}(I) = \omega_i$$

$$\text{OMD}(I) = \omega_{i+1} - \omega_i$$

$$\text{ROMD}(I) = 1./\text{OMD}(I)$$

$$\text{OMP}(I) = \omega_i + \omega_{i+1}$$

$$\text{BOM}(I) = \omega_{i+1} - \omega_{i-1}$$

$$\text{BOMT3}(I) = 3.*\text{BOM}(I)$$

Note that lines OMG00300 and 310 can be removed since the variables OMI and OME which are calculated there are not used.

### E. Description of Subroutine INITL/1

This program begins by defining the function  $10\omega^3 - 15\omega^4 + 6\omega^5$  which is used to generate S-shaped profiles. It then tests the variable INPRFL(J), where J is the index of a major species. If .FALSE., then the initial profile for J is a decay profile and is calculated from the formula

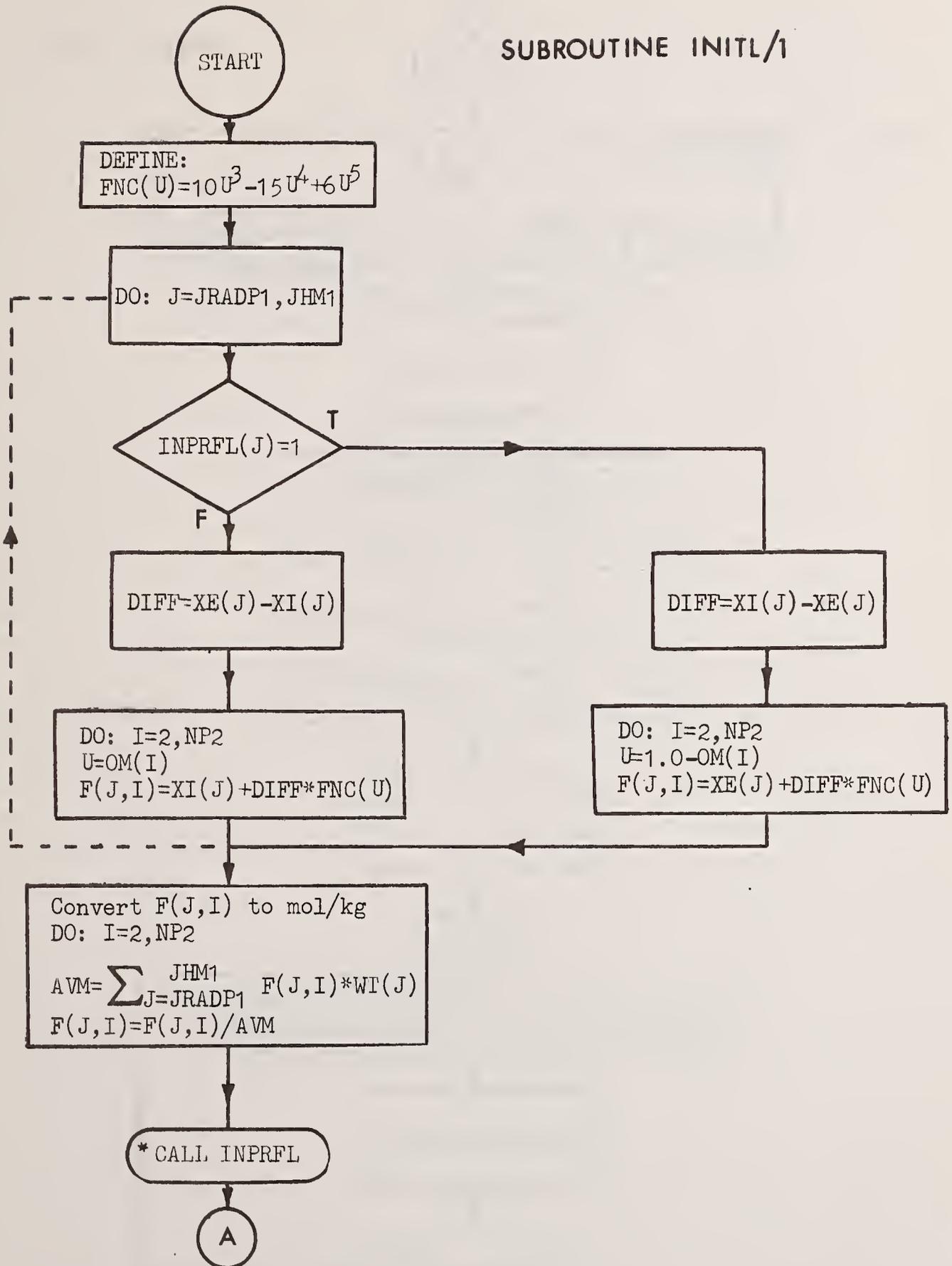
$$XI(J) + (XE(J) - XI(J))(10\omega^3 - 15\omega^4 + 6\omega^5)$$

If .TRUE., then J has a growth profile which is calculated by interchanging XI and XE and inserting  $1 - \omega$  for  $\omega$  in the above formula. This yields mole fraction profiles. To convert them to units of moles/kg, INITL calculates the average molecular weight at a particular grid point from the formula  $\langle M \rangle = \sum_j x_j M_j$ . The desired profiles are gotten by dividing the mole fraction profiles by  $\langle M \rangle$ .

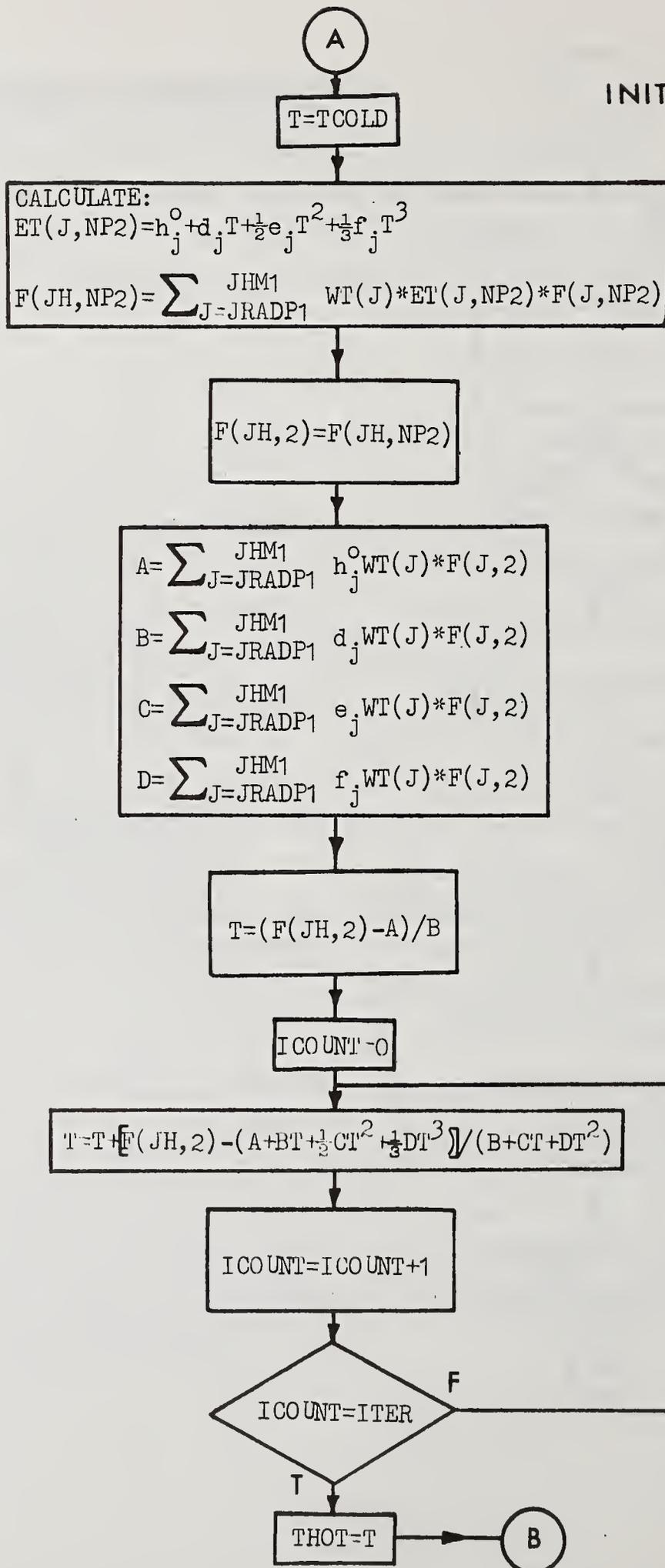
At this point a small subroutine INPRFL is called to set the initial profiles for the trace species. Normally, the trace species are assumed to have negligible concentrations at the cold boundary grid points. In setting up INPRFL, do not use equal concentrations at the two boundary grid points, 2 and NP2. This would result in a division by zero on the first call to ENTRN. (See ENTRN, lines ENTO0190, 210, and 230.)

INITL next calculates ET(J,JP2), the enthalpy at the cold boundary grid point for all the major species. This will be in units of joules/kg. From these quantities and the composition at the cold boundary, the total enthalpy value  $h_c = F(JH, NP2)$  is calculated. The enthalpy at the hot boundary,  $F(JH, 2)$ , is then set equal to  $h_c$  and the temperature there is calculated from this and the initial hot side composition by the Newton-Raphson method discussed in Appendix N, p. A39. From the two temperatures TCOLD and THOT an S-shaped temperature profile is calculated. Finally, the enthalpy at each grid point is given the cold boundary grid point value.

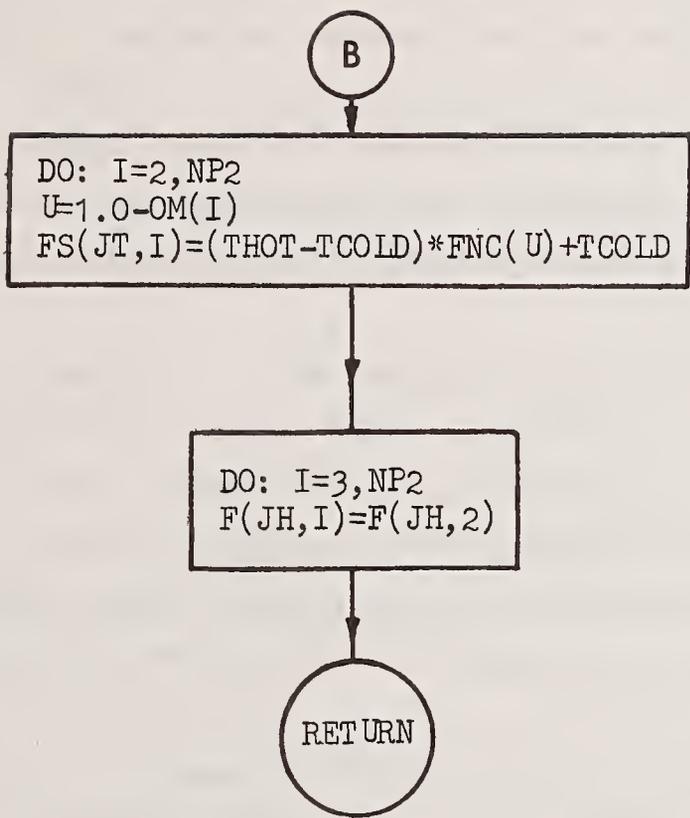
## SUBROUTINE INITL/1



INITL/1 (cont.)



INITL/1 (cont.)



### F. Description of Subroutine TRANS(ICALC)/6

This subroutine calculates the temperature, the mean molecular weight, the density, and the transport parameters for three flames; ozone decomposition,  $H_2-Br_2$ , and  $H_2-O_2-N_2$ . It begins by calculating the temperature and its reciprocal from the enthalpy by the Newton-Raphson procedure described in Appendix N, p. A39. At the same time it also calculates the mean molecular weight and its reciprocal at each grid point.

The quantity ICALC is then tested; if it equals zero, then TRANS jumps to the next grid point. Under this condition the transport properties are not calculated. If ICALC equals one, then TRANS continues by calculating the density  $RHO(I)$  at each grid point. FLAG(3) is tested, and if .TRUE., the density is multiplied by PRESS which, in this case, will be different from unity; if .FALSE., then this multiplication is not made.

FLAG(7) is tested and if .TRUE., transport properties are calculated for the ozone decomposition flame. The desired parameters are  $\Delta_j \rho^2$  for diffusion, and  $\lambda \rho / C_p$  for thermal conductivity. The average heat capacity  $C_p$  (=CPMIX) is first calculated and from this and the density,  $\lambda \rho / C_p$  (=PREF(JH,I)) is calculated from the expression  $2.2026 \times 10^{-3} T^{1/2} \rho / C_p$ . For this flame a Lewis number of 0.94 was used so that the quantities  $\Delta_j \rho^2$  (=PREF(J,I)) are all given the values  $0.94 * PREF(JH,I)$ . The program then jumps to statement 57.

If FLAG(7) was .FALSE., FLAG(8) is tested; if .TRUE., transport properties are calculated for the  $H_2-Br_2$  flame. We used the following expressions to calculate the diffusion parameters;

$$\Delta_{Br} \rho^2 = 0.155 \times 10^{-4} (T/TCOLD)^{1.67} \rho^2$$

$$\Delta_H \rho^2 = 1.05 \times 10^{-4} (T/TCOLD)^{1.67} \rho^2$$

$$\Delta_{H_2} \rho^2, \Delta_{Br_2} \rho^2, \Delta_{HBr} \rho^2 = 0.101 \times 10^{-4} (T/TCOLD)^{1.67} \rho^2$$

The thermal conductivity of the mixture was taken to be

$$\lambda \rho / C_p = 3.34944 \times 10^{-2} (T/TCOLD)^{0.67} \rho / CPMIX.$$

The program then jumps to statement 57.

If FLAG(8) was .FALSE., FLAG(1) is tested; if .TRUE., the program jumps to the next grid point. This is the situation where we are using constant values for the transport parameters. If FLAG(1) = .FALSE., then TRANS proceeds to calculate the transport properties for the  $H_2-O_2-N_2$  flame. It begins by calculating the binary diffusion coefficients using the values of DA(I,J) and DB(I,J) determined in DIFUSE. Since each species at a particular grid point uses either  $T^{1.94}$  or  $T^{1.67}$  as the factor multiplying DA or DB, it is desirable to avoid doing these exponentiations more than once per grid point. This feat is accomplished by assigning .TRUE. values to the variables TAGA and TAGB (initially .FALSE.) if either  $T^{1.94}$  or  $T^{1.67}$  has already been calculated for a particular species. Testing these flags on subsequent passes through the DO loop allows one to avoid repeated evaluation of  $T^{1.94}$  or  $T^{1.67}$ . The temperature dependent binary diffusion coefficients are stored in the array DD(KI,KJ), where KI and KJ are species indices.

The diffusion coefficients  $\Delta_j$  are then calculated from the DD(KI,KJ) values and the concentrations  $F(J,I)$  by means of Eq. (A9), Appendix C, p. A7. Diffusion parameters  $PREF(J,I) = \Delta_j \rho^2$  are then calculated.

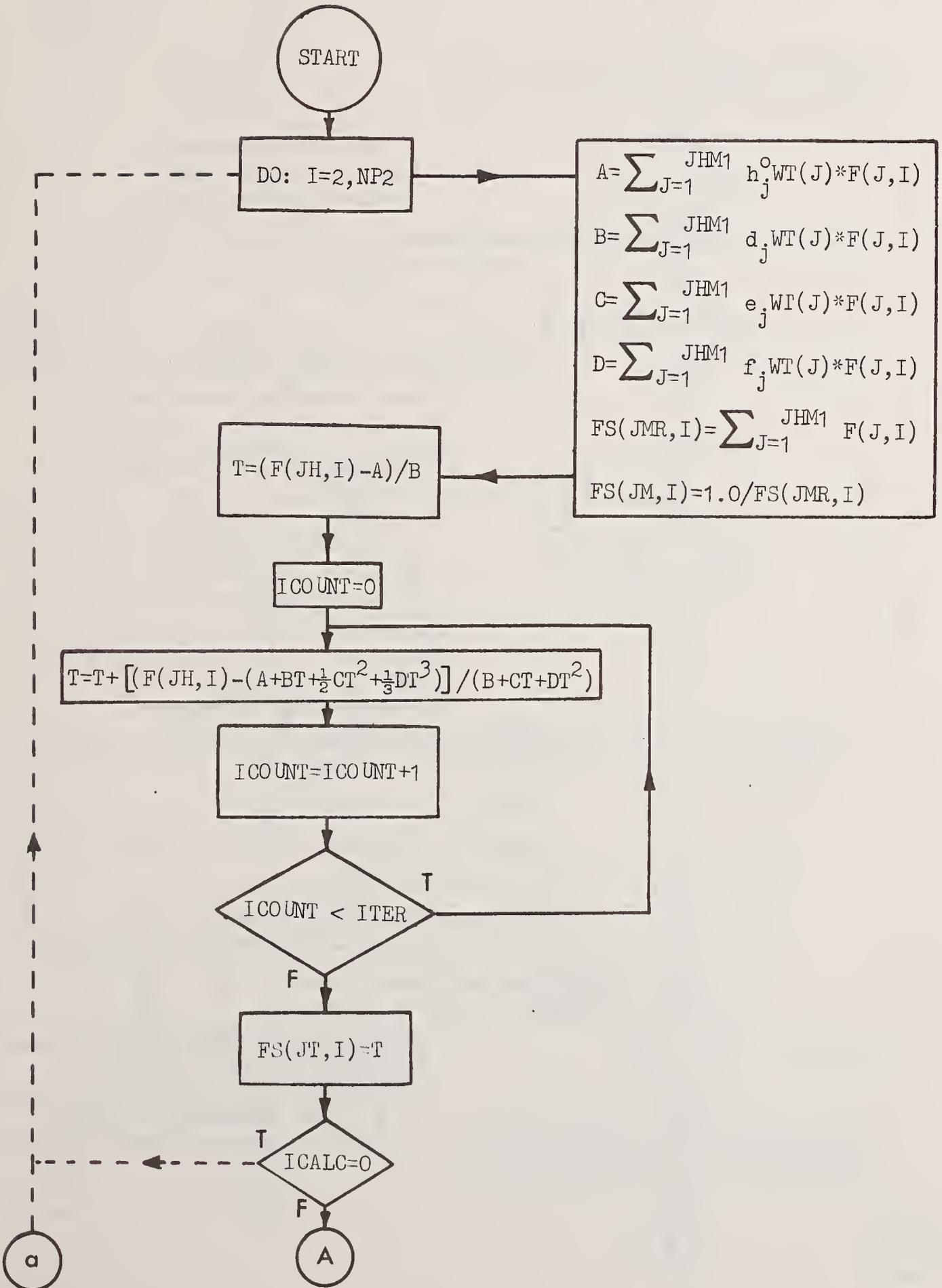
TRANS next calculates the thermal conductivity of the mixture from the formula given in Appendix D, p. A8. For this, it is necessary to have values of the temperature dependent coefficients  $A_{ij}$  defined by Eq. (A11), Appendix D, p. A8. These were calculated from this formula by a separate program, and the results fit by a least squares procedure to a power series in the temperature. These expressions for the  $A_{ij}$  are given by lines TRNO1510 through 01620. Note that the values are stored in the DD array since the binary diffusion coefficients previously stored there are no longer needed. Also given just before these are the power series expansions of the thermal conductivities of the pure major species; lines TRNO1420 through 01450. These are placed in CON(J). Note that it is for these arrays that the indices LO2, LH2O, LH2, and LN2 are required. From the  $A_{ij}$  and CON(J) values, the mixture thermal conductivity is calculated. This is then converted to the thermal conductivity parameter  $PREF(JH,I) = \lambda \rho / C_p$ .

FLAG(5) is then tested. If .FALSE., TRANS goes to the next grid point. If .TRUE., the enthalpy source term is being calculated and values of ENT(J,I), the enthalpy of each species at each grid point are determined.

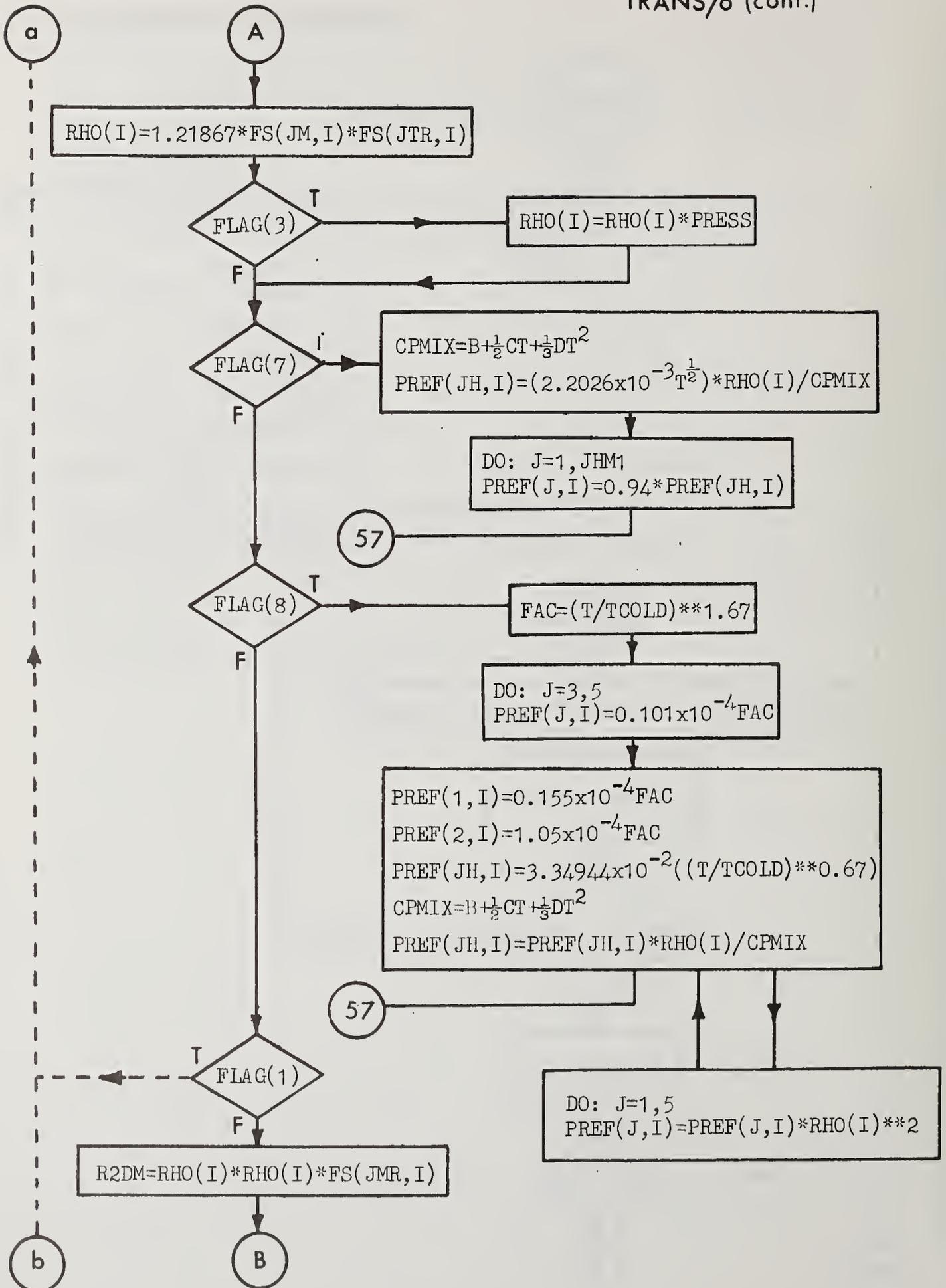
After the calculations have been completed for all the grid points, ICALC is tested. If equal to zero, a return to SPALD occurs. If equal to one, TRANS proceeds to calculate the values of the reciprocal of the mean molecular weight at the control volume boundaries. This is taken to be the average value of FS(JMR,I) at two adjacent grid points, and is stored in RMB(I). The same thing is also done for the diffusion and thermal conductivity parameters, and the results placed in PREF(J,I) and PREF(JH,I). Thus, these arrays will contain the control volume boundary values and not the grid point values.

If FLAG(5) = .TRUE., several quantities appearing in the enthalpy source term are evaluated. Note that both parts of the enthalpy source term are calculated together and not separately as in the case of the chemical source term.

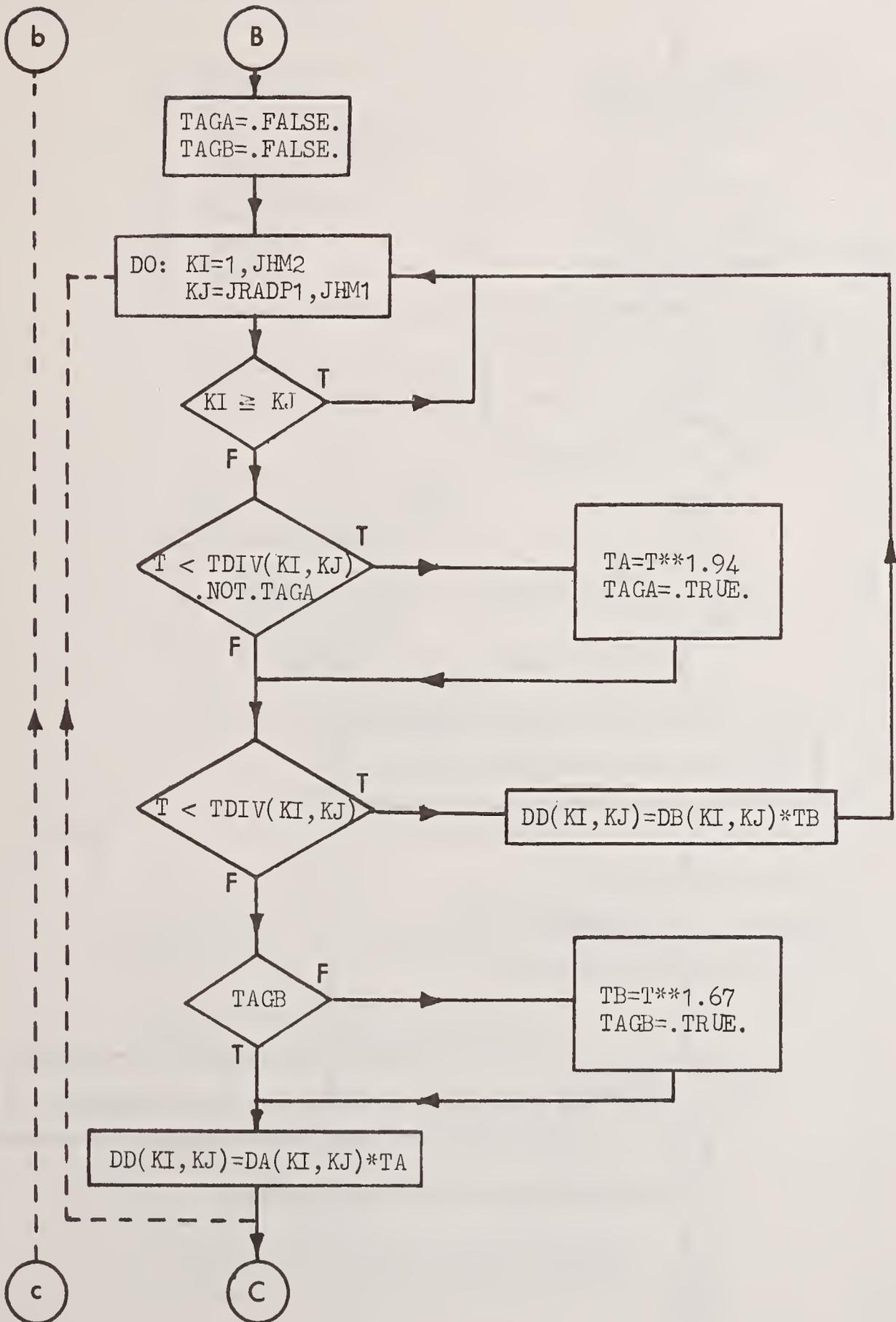
SUBROUTINE TRANS(ICALC) / 6



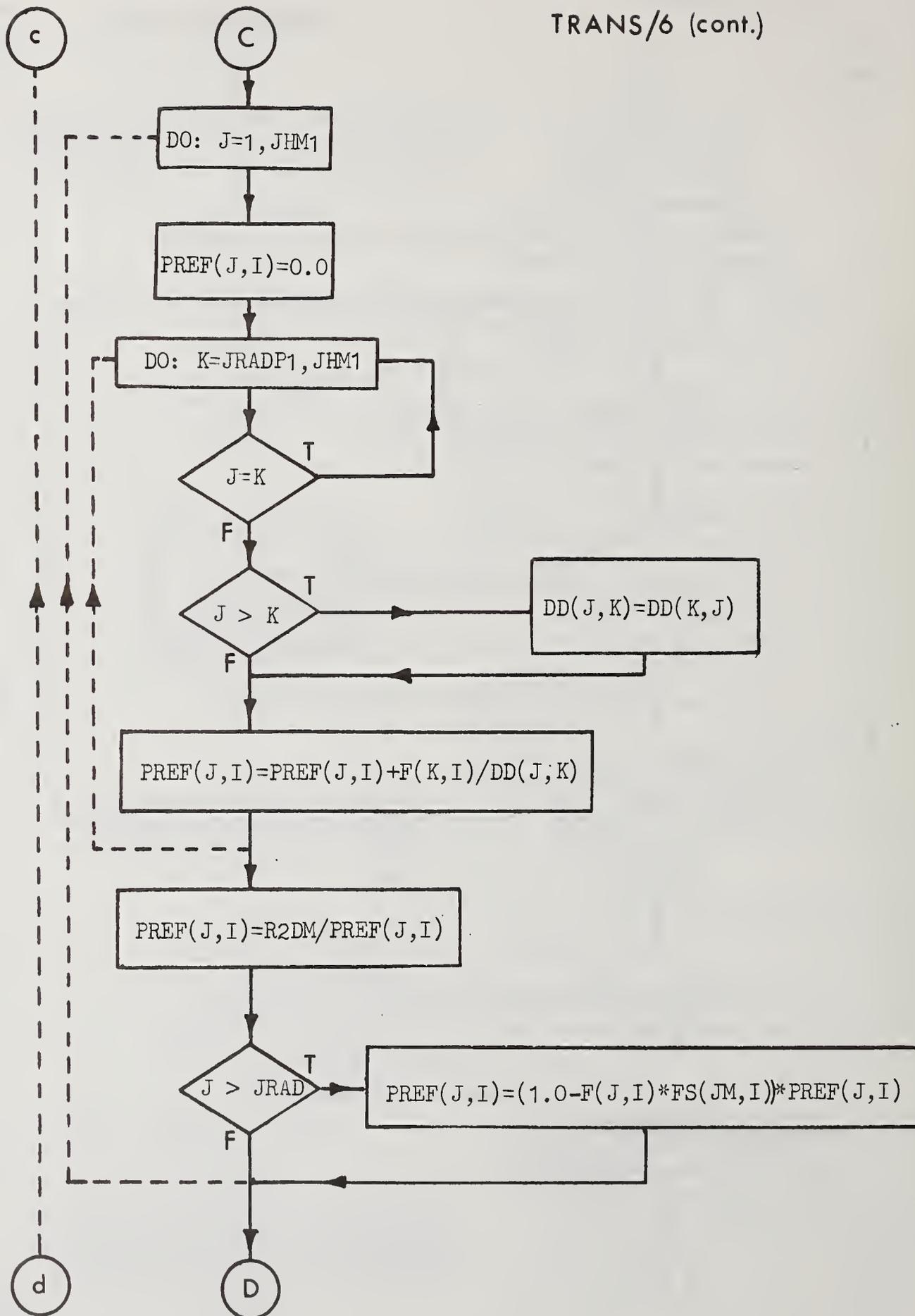
## TRANS/6 (cont.)



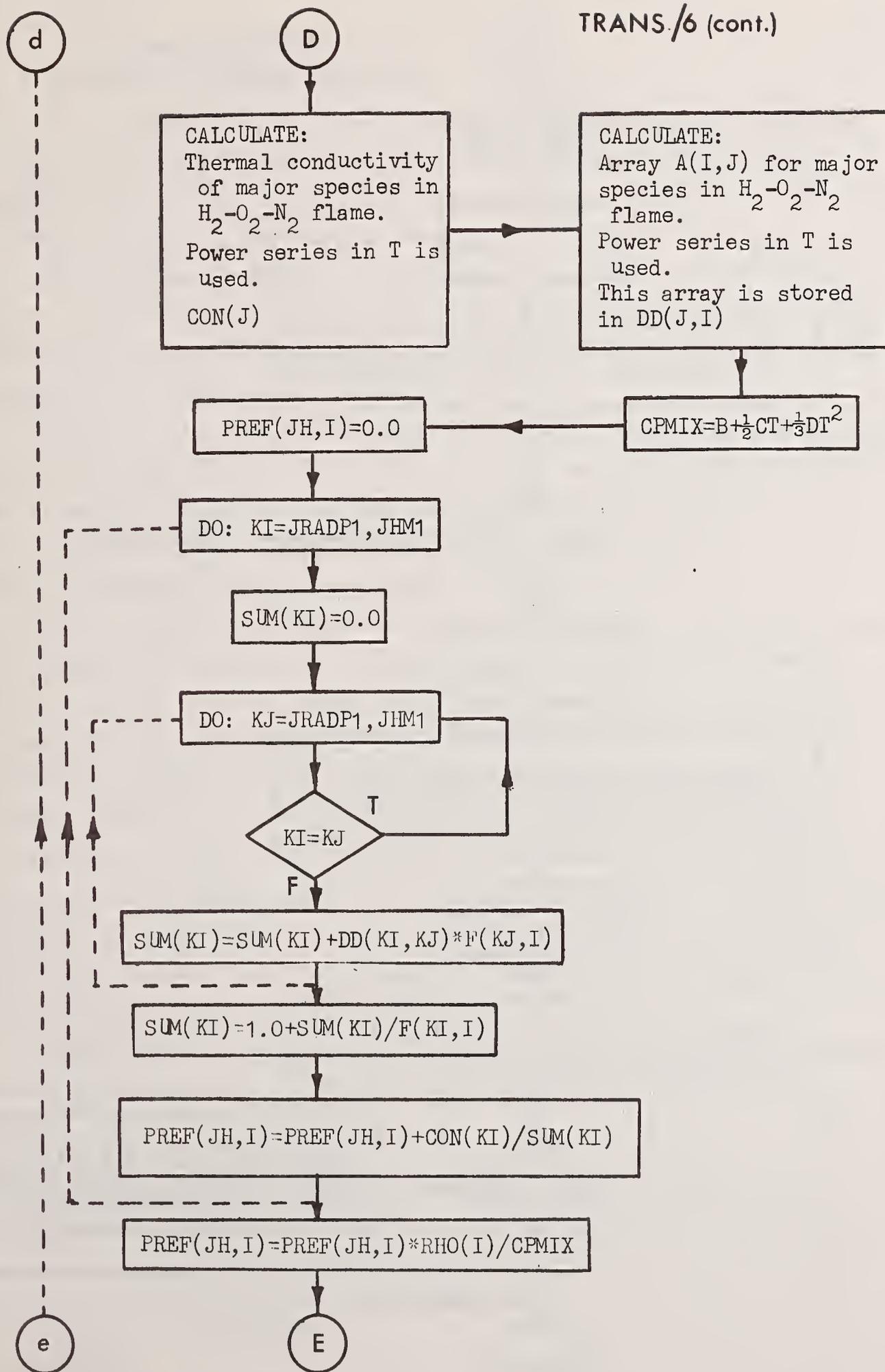
TRANS/6 (cont.)



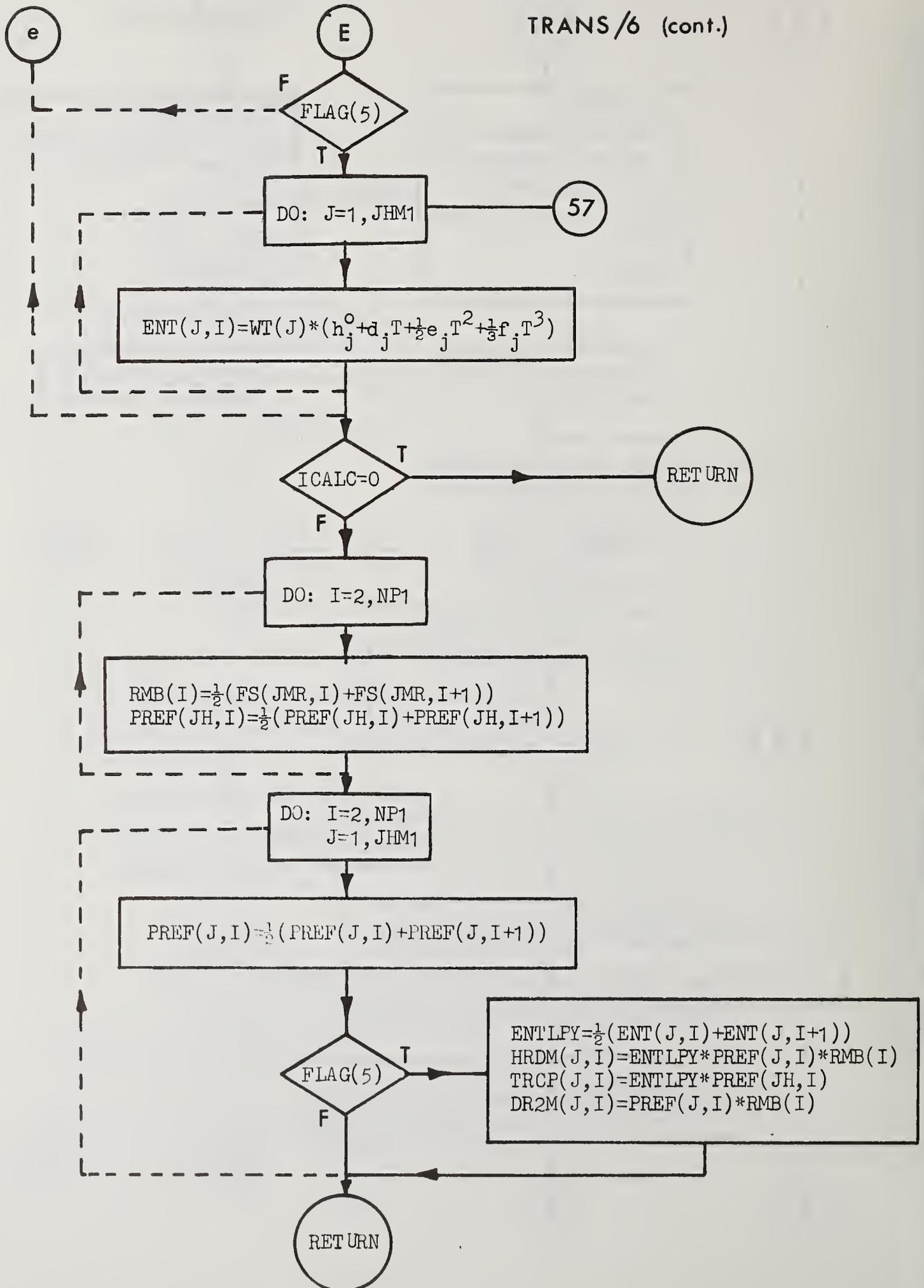
## TRANS/6 (cont.)



## TRANS./6 (cont.)



TRANS/6 (cont.)



### G. Description of Subroutine RATCN/3

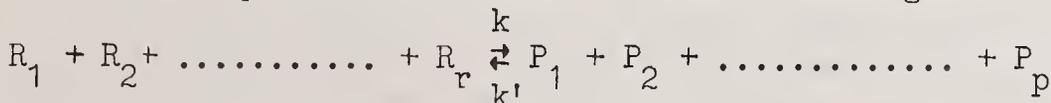
This subroutine takes the Arrhenius parameters and calculates the forward rate constants for each reaction; from these, and the free energies of reaction, it then calculates the rate constants for the reverse reactions.

RATCN starts by calculating the quantities  $(RT)^{-1}$  at each grid point.  $R$  is the gas constant and has the value  $8.2057 \times 10^{-5} \text{ atm-m}^3/\text{mole-deg}$ . These are stored in  $RGT(I)$  and are used later in the calculation of the reverse rate constants.

The calculation of the forward rate constants starts by the testing of  $FFLAG(K)$ ; if  $.FALSE.$ , RATCN sets  $RATE(K,I)$  to zero and goes to the next  $K$  value. This option allows one to examine the effect of neglecting the forward rate without having to change the Arrhenius parameters which may have been used in a previous calculation.

Next,  $BFLAG(K)$  and  $EFLAG(K)$  are tested together; if both are  $.FALSE.$ , then both  $\beta$  and  $E/R$  are zero for this reaction and so exponentiation can be avoided. In this case  $RATE(K,I)$  is given the value  $FRQ(K)$ , the Arrhenius frequency factor, and RATCN moves to the next  $K$ . If either  $BFLAG(K)$  or  $EFLAG(K)$  or both are  $.TRUE.$ ,  $BFLAG(K)$  is tested and if  $.FALSE.$ ,  $RATE(K,I)$  is given the value  $FRQ(K) * \text{DEXP}(-EACT(K)/FS(JT,I))$ . If  $BFLAG(K) = .TRUE.$ , then  $EFLAG(K)$  is tested and if  $.FALSE.$ ,  $RATE(K,I)$  takes the value  $FRQ(K) * (FS(JT,I))^{**BETA(K)}$ . If  $EFLAG(K) = .TRUE.$ , then  $RATE(K,I)$  is given the value  $FRQ(K) * FS(JT,I)^{**BETA(K)} * \text{DEXP}(-EACT(K)/FS(JT,I))$ . Only in this last case is it necessary to calculate the complete Arrhenius expression.

After calculating all of the forward rate constants, RATCN evaluates all of the equilibrium constants. Consider the general reaction,



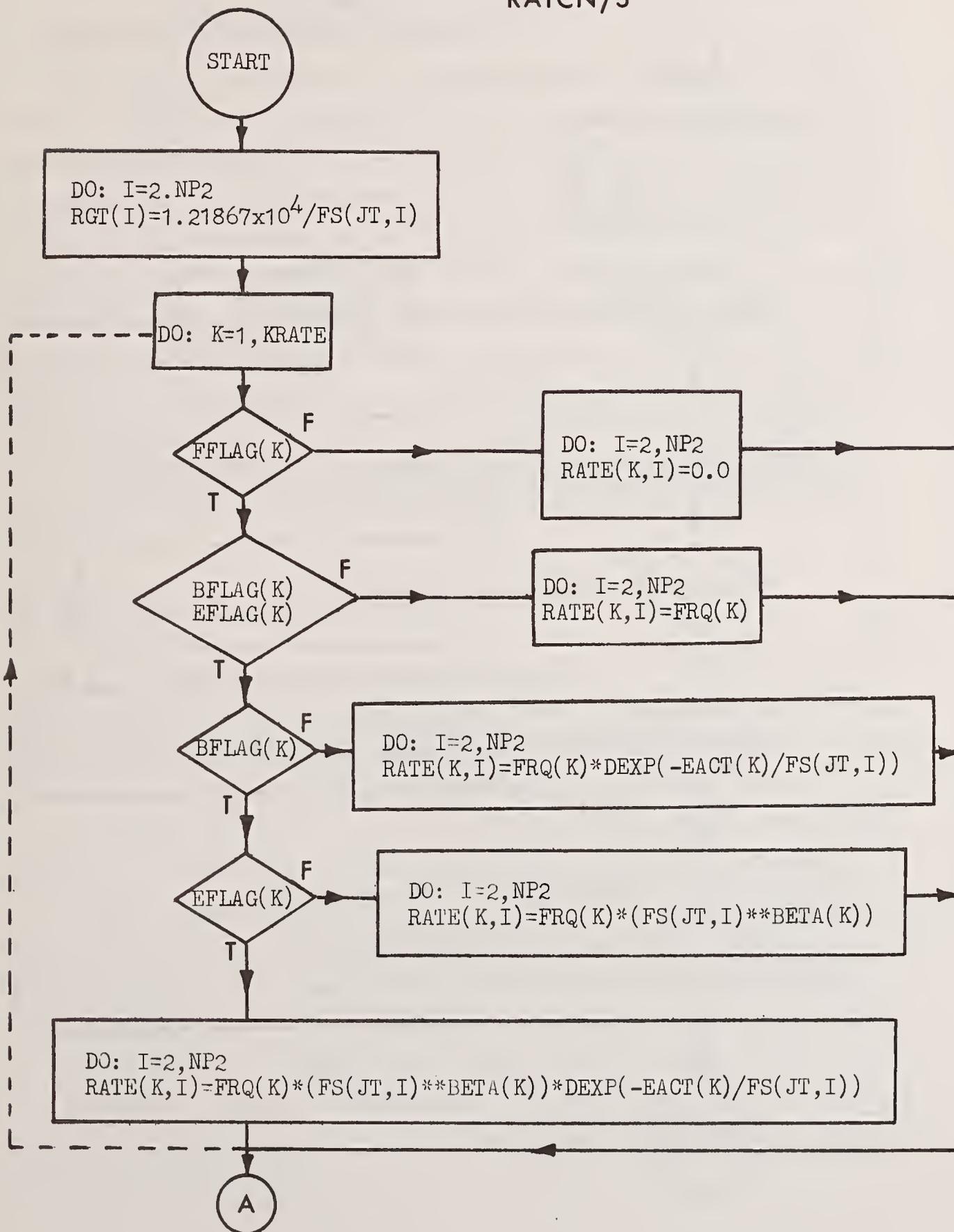
$R_i$  and  $P_i$  represent one molecule of a species, so two symbols may represent the same molecule. The equilibrium constant in terms of the partial pressures of the various species is

$$K_p = \frac{\prod_{i=1}^p P_{P_i}}{\prod_{i=1}^r P_{R_i}} = \exp(-\Delta F^0/RT)$$

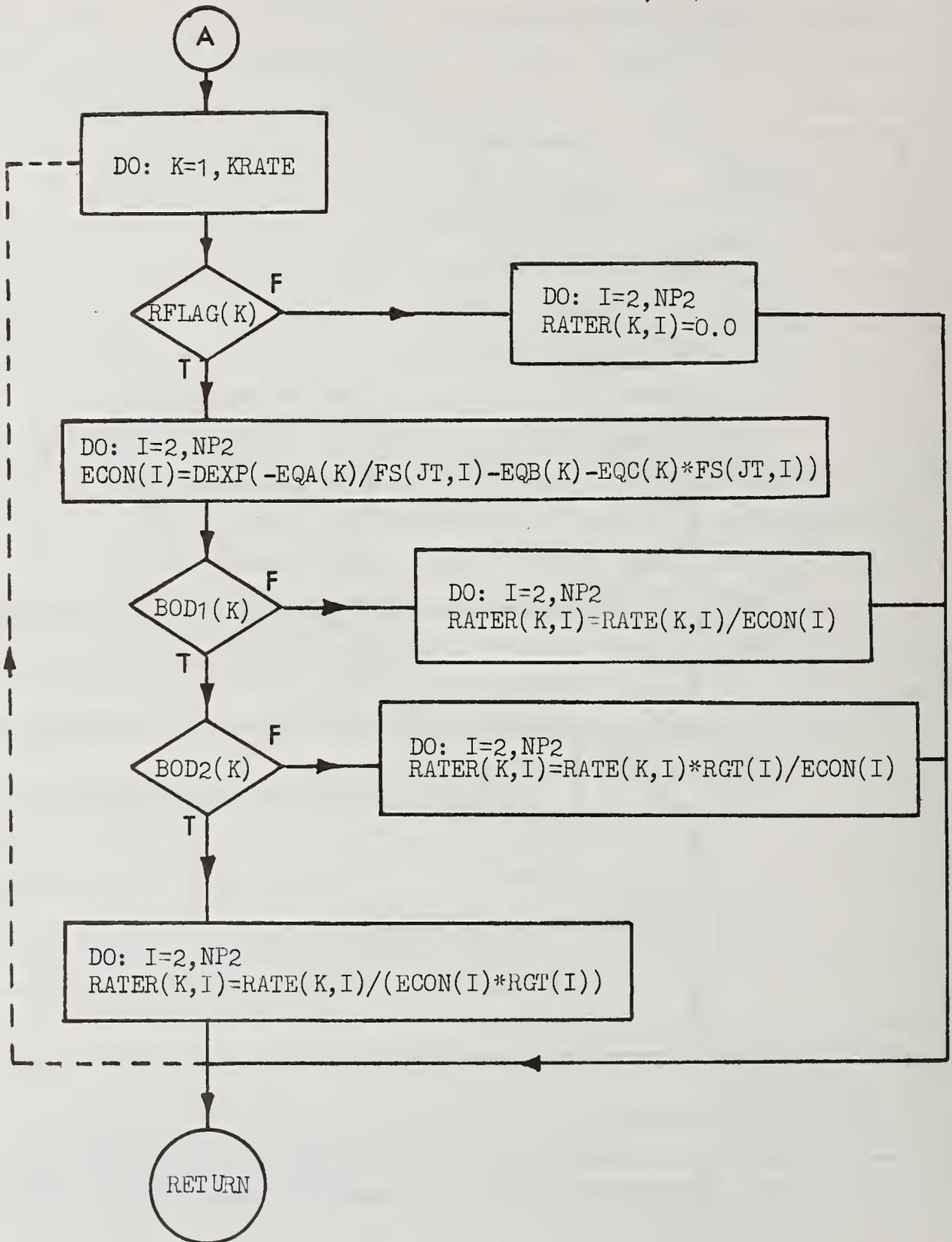
We want the equilibrium constant in terms of concentrations; this is related to  $K_p$  by the expression  $K_c = K_p (RT)^{-n}$ , where  $n = p - r$  and  $R$  is the gas constant in  $\text{atm}\cdot\text{m}^3/\text{mole}\cdot\text{deg}$ . Since  $K_c = k/k'$ , we have for the reverse rate constant  $k' = k(RT)^n/K_p$ .

To calculate  $k'$  RATCN first tests RFLAG(K); if .FALSE., it sets the reverse rate constant  $k'$  (=RATER(K,I)) to zero and goes to the next K value. If this flag is .TRUE., then it calculates  $-\Delta F^0/RT$  from the expression  $-EQA(K)/T - EQB(K) - EQC(K)*T$ , evaluates  $\exp(-\Delta F^0/RT)$ , and stores the results in ECON(I). (Note that EQA(K), EQB(K), and EQC(K) as read by SPALD give  $\Delta F^0$  in kcal/mole. SPALD immediately multiplies them by 1000/R, where  $R = 1.9869 \text{ cal/mole}\cdot\text{deg}$  to give  $\Delta F^0/R$ .) BOD1(K) is tested next; if .FALSE., then the number of molecules does not change in the reaction;  $n = 0$  and RATER(K,I) is given the value  $\text{RATE}(K,I)/\text{ECON}(I)$ . If BOD1(K) is .TRUE., then we have a 3-body reaction with a unity change in the number of molecules. BOD2(K) is tested to find out the sign of  $n$ . If BOD2(K) = .FALSE., then  $n = -1$  and RATER(K,I) is given the value  $\text{RATE}(K,I)*\text{RGT}(I)/\text{ECON}(I)$ ; if .TRUE., then  $n = +1$  and it has the value  $\text{RATE}(K,I)/(\text{ECON}(I)*\text{RGT}(I))$ . (Remember that  $\text{RGT}(I) = (RT)^{-1}$ .)

SUBROUTINE  
RATCN/3



## RATCN/3 (cont.)



#### H. Description of Subroutine ENTRN(J,IFOW)

The purpose of ENTRN is to calculate the so-called entrainment rates  $\dot{m}_C$  (=RME) and  $\dot{m}_H$  (=RMI). RME is the mass flow rate in  $\text{kg}/\text{m}^2\text{-s}$  across the cold boundary and RMI that across the hot boundary. When a steady-state is attained, RME and RMI become equal. Their values depend on the concentrations of the species JENTRN at the hot and cold boundaries and at the points NEE and NII. The grid width PEI is changed if RME and RMI have different values. By this means the grid can be kept centered about the region containing the largest concentration gradients. (See Appendix J, p. A26.) This routine also calculates the flame velocity.

During the course of the integration ENTRN is called with  $J = \text{JENTRN}$  and  $\text{IFOW} = 0$ . The routine begins by evaluating the integral

$$\eta \int_0^1 \frac{R_j}{\rho M_j} d\omega,$$

where  $R_j$  is the mass production rate of species  $j$  in  $\text{kg}/\text{m}^3\text{-s}$ . The quantity  $R_j/\rho M_j$  has units of moles/kg-s and is the quantity stored in the array  $R(J,I)$ . An approximation with three grid points is used for this integral. This is given by Eq. (A61), Appendix J, p. A27.

The flame velocity is gotten by dividing the value of the above integral by the density at the cold boundary grid point,  $\text{RHO}(\text{NP2})$ .

IFOW is then tested; if equal to zero, ENTRN continues by calculating RME and RMI from Eqs. (A58), (A59), and (A60), Appendix J, p. A26. The grid width PEI is then incremented by the amount  $(\text{RMI} - \text{RME}) * \text{DX}$ .

The condition  $\text{IFOW} = 1$  is used at an output point. Then ENTRN is called for  $J = 1$  through  $\text{JHM1}$  to calculate the flame velocity from the production rates of each species. In principle, these velocities should be the same in the steady-state flame. When  $\text{IFOW} = 1$ , RMI and RME are not calculated.

### I. Description of the Subroutine SCHM1/1

This short routine calculates the chemical source term arising from the diffusion of mean molecular weight. (See Eq. (A57), Appendix I, p. A24.) Concentration values at the control volume boundaries are required; these are calculated from the average of two adjacent grid points and stored in BF(J,I). The other quantities required are all evaluated elsewhere. Values of this source term are placed in the array SC1(J,I).

## J. Description of Subroutine SCHM2(J)/7

This subroutine calculates the part of the chemical source term arising from the chemical reactions.

SCHM2 begins by testing  $G1(K,J)$ ; if `.FALSE.`, then species J does not appear in reaction K and the program moves to the next K value. If `.TRUE.`, then K is tested to see if it equals 5 or 10. These are the reactions  $H + OH + M \rightleftharpoons H_2O + M$  and  $H + O_2 + M \rightleftharpoons HO_2 + M$  in the  $H_2-O_2-N_2$  flame which have the special third-body concentrations  $FS(JBODYA,I)$  and  $FS(JBODYB,I)$ . This routine can also be used for the ozone decomposition and the  $H_2-Br_2$  flames because both require fewer than 5 reactions. To use SCHM2 for other flames, it will be necessary to remove the following lines:

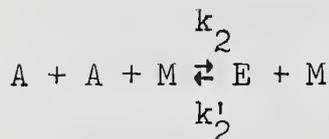
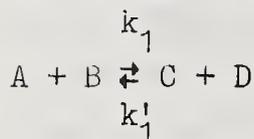
```
SC200260
SC200270
SC200420
  ⋮
  ⋮
  ⋮
SC200480
```

If K is not equal to 5 or 10, then  $BOD1(K)$  is tested; if `.FALSE.`, K does not contain a three-body reaction and SCHM2 proceeds to calculate the forward and reverse rates of the reaction. These are put into  $FOR(I)$  and  $REV(I)$ . At this point, these rates are in units of  $\text{moles}\cdot\text{m}^3/\text{kg}^2\cdot\text{s}$  and will later be multiplied by the density to give them in units of  $\text{moles}/\text{kg}\cdot\text{s}$ . If  $BOD1(K) = \text{.TRUE.}$ , then K contains a three-body reaction and  $BOD2(K)$  is tested to find out whether it is the forward or the reverse reaction. If  $BOD2(K) = \text{.FALSE.}$ , then it is the forward reaction and  $FOR(I)$  and  $REV(I)$  are calculated accordingly. If  $BOD2(K) = \text{.TRUE.}$ , then the opposite situation holds.  $FOR(I)$  and  $REV(I)$  are now combined in various ways depending on the values of the flags  $G1$ ,  $G2$ , and  $G3$ . The results are placed in  $SU(J,I)$  and  $SD(J,I)$ . This process is repeated for all the K values, and the results added to  $SU(J,I)$  and  $SD(J,I)$ .

After multiplying  $SU(J,I)$  and  $SD(J,I)$  by the density  $RHO(I)$ , the production rate of species J,  $R(J,I)$ , is obtained by adding them together. The final desired quantities are gotten by dividing the value in

SD(J,I) by the concentration of J, F(J,I), and multiplying both SD(J,I) and SU(J,I) by  $\frac{1}{2}\eta\Omega$ .

For an example of how this routine functions consider the reaction mechanism discussed in Appendix I, p. A21. There are two reactions



We want to calculate the source term parameters  $S_P(A)$  and  $S_F(A)$  for the species A defined in Eq. (A53), p. A23.

For K = 1, we have BOD1(1) = .FALSE.

$$BOD2(1) = .FALSE.$$

$$G1(1, LA) = .TRUE.$$

$$G2(1, LA) = .FALSE.$$

$$G3(1, LA) = .FALSE.$$

For K = 2, we have BOD1(2) = .TRUE.

$$BOD2(2) = .FALSE.$$

$$G1(2, LA) = .TRUE.$$

$$G2(2, LA) = .FALSE.$$

$$G3(2, LA) = .TRUE.$$

When K = 1, SCHM2 places in SU(LA,I) and SD(LA,I), respectively, the quantities  $k_1'\rho\varphi_C\varphi_D$  and  $-k_1\varphi_A\varphi_B$ . Moving to K = 2, it adds to SU(LA,I) and SD(LA,I), the quantities  $2(k_2\varphi_A\varphi_A\varphi_M\rho + k_2'\varphi_E\varphi_M)$  and  $-4k_2\varphi_A\varphi_A\varphi_M\rho$ . Note that  $\varphi_M$ , the concentration of the buffer species, is taken to be the reciprocal of the mean molecular weight, FS(JMR,I). SU(LA,I) and SD(LA,I) are then both multiplied by the density. At this point they have the values

$$SU(LA,I) = k_1'\rho\varphi_C\varphi_D + 2k_2\rho^2\varphi_A\varphi_A\varphi_M + 2k_2'\rho\varphi_E\varphi_M$$

$$SD(LA,I) = -k_1\rho\varphi_A\varphi_B - 4k_2\rho^2\varphi_A\varphi_A\varphi_M$$

The production rate R(LA,I) of species A is the sum of SU(LA,I) and SD(LA,I).

$SU(LA, I)$  and  $SD(LA, I)$  are not yet equal to the parameters  $S_P(A)$  and  $S_F(A)$ . These are obtained by dividing  $SD(LA, I)$  by  $\varphi_A$  and multiplying both  $SD(LA, I)$  and  $SU(LA, I)$  by  $\frac{1}{2}\eta\Omega = \frac{1}{2}\eta(\omega_{i+1} - \omega_{i-1}) = PEIOM2(I)$ .

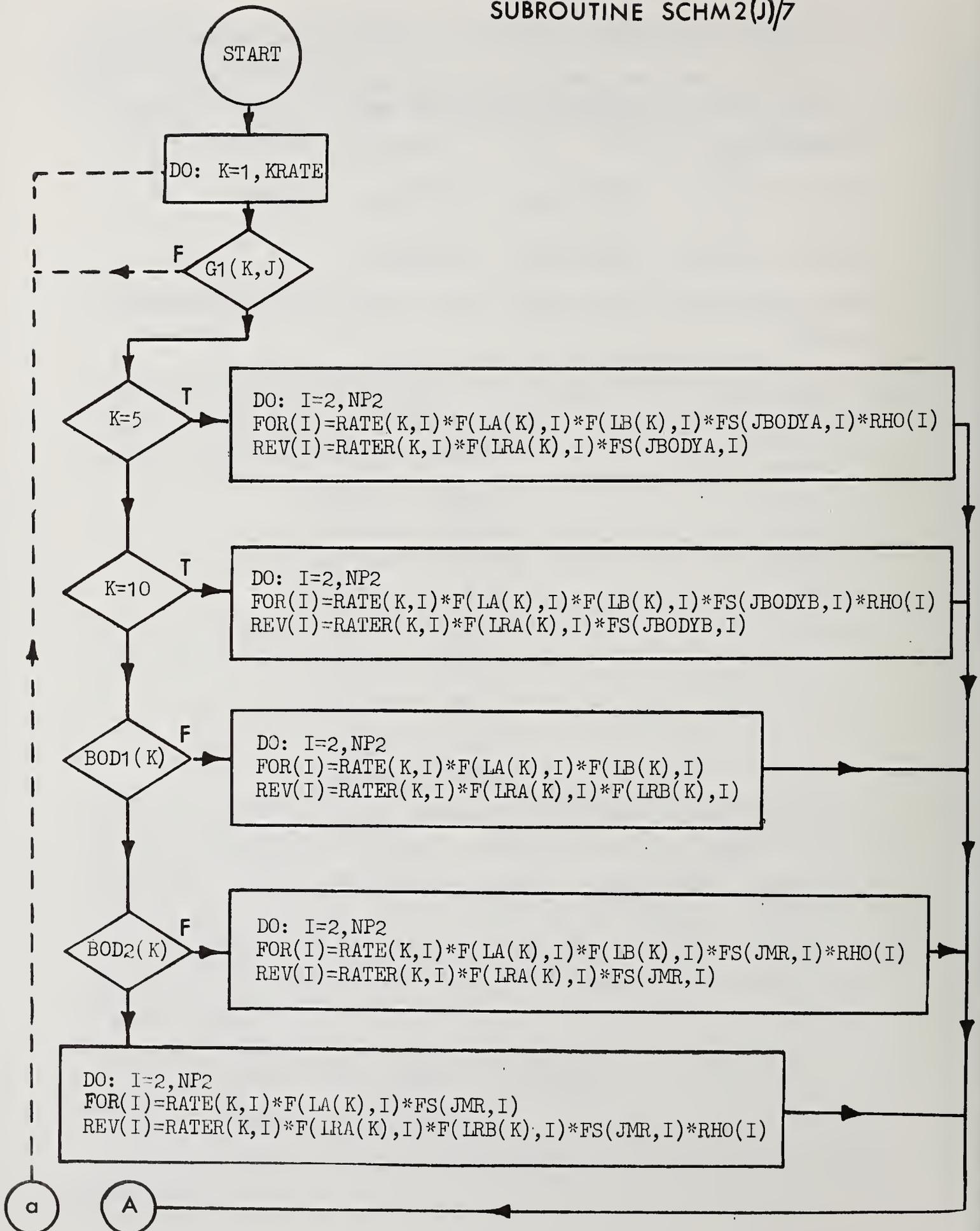
The result is

$$SU(LA, I) = S_P(A) = \frac{1}{2}\eta\Omega(k_1^1 \rho \varphi_C \varphi_D + 2k_2 \rho^2 \varphi_A^2 \varphi_M + 2k_2^1 \rho \varphi_E \varphi_M)$$

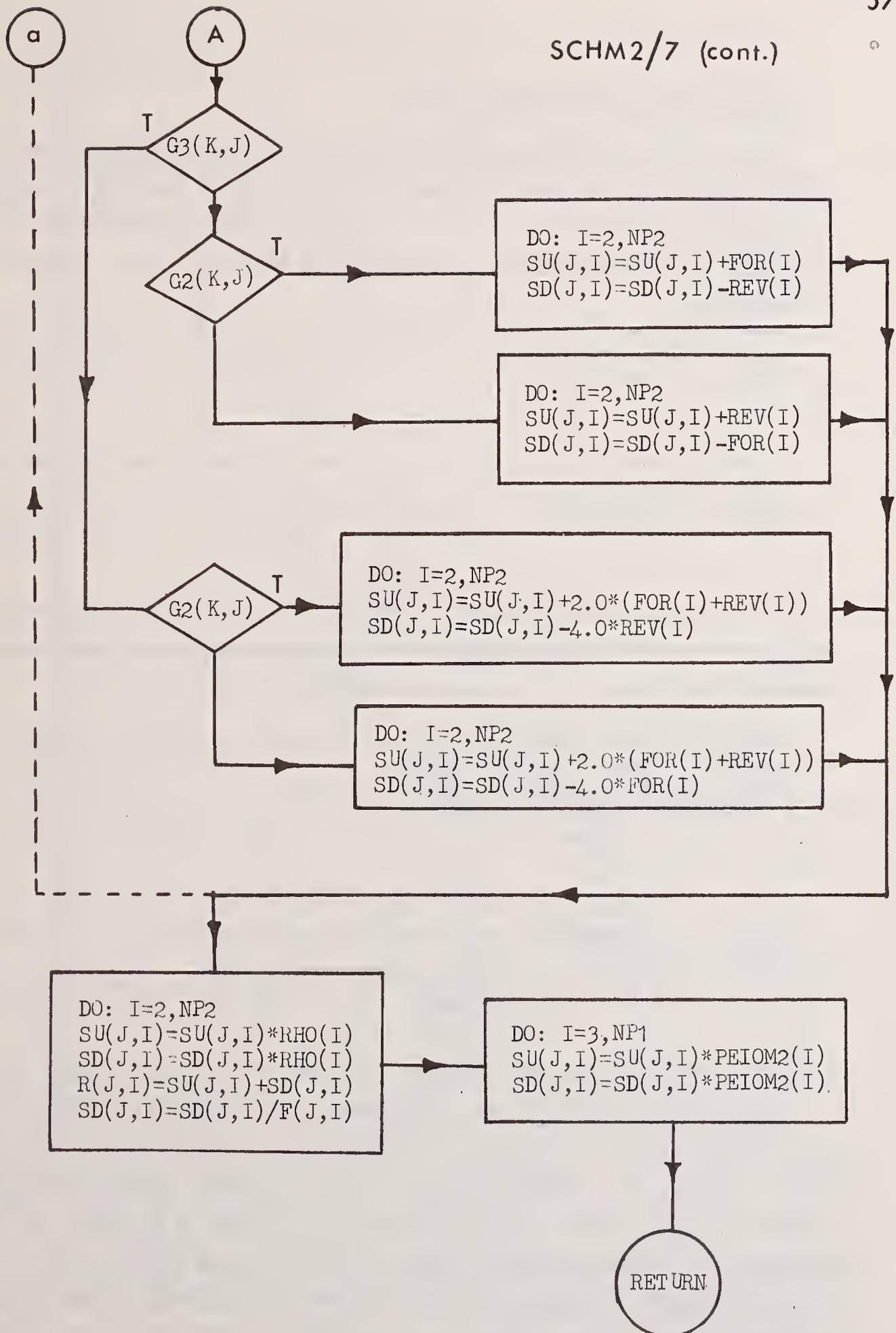
$$SD(LA, I) = S_F(A) = \frac{1}{2}\eta\Omega(-k_1 \rho \varphi_B - 4k_2 \rho^2 \varphi_A \varphi_M)$$

This is the same as that shown in Eqs. (A51) and (A52), Appendix I, p. A23.

## SUBROUTINE SCHM2(J)/7



## SCHM2/7 (cont.)



### K. Description of Subroutine CALC(J)

This is the subroutine which solves the finite-difference equations (A42), Appendix H, p. A19. To see how this program functions, consider a 4 point grid system ( $N = 3$ ), with the grid index running from 2 to 5. There will be two unknowns,  $\Phi_3$  and  $\Phi_4$ , for which Eq. (A42) yields two equations,

$$\Phi_3 = \frac{A'_3}{D_3} \Phi_4 + \frac{B'_3}{D_3} \Phi_2 + \frac{C'_3}{D_3}$$

$$\Phi_4 = \frac{A'_4}{D_4} \Phi_5 + \frac{B'_4}{D_4} \Phi_3 + \frac{C'_4}{D_4}$$

$A'$ ,  $B'$ ,  $C'$ , and  $D$  are given by Eqs. (A40), p. A13 and the values  $\Phi_2$  and  $\Phi_5$  at the hot and cold boundaries, respectively, are known from the boundary conditions. By simple elimination, we obtain

$$\Phi_3 = \frac{\frac{A'_3 A'_4}{D_3 D_4} \Phi_5 + \frac{A'_3 C'_4}{D_3 D_4} + \frac{B'_3}{D_3} \Phi_2 + \frac{C'_3}{D_3}}{1 - \frac{A'_3 B'_4}{D_3 D_4}} \quad (15)$$

$$\Phi_4 = \frac{A'_4}{D_4} \Phi_5 + \frac{B'_4}{D_4} \left\{ \frac{\frac{A'_3 A'_4}{D_3 D_4} \Phi_5 + \frac{A'_3 C'_4}{D_3 D_4} + \frac{B'_3}{D_3} \Phi_2 + \frac{C'_3}{D_3}}{1 - \frac{A'_3 B'_4}{D_3 D_4}} \right\} + \frac{C'_4}{D_4} \quad (16)$$

Let us go through CALC step-by-step and see how it solves these equations.

There are 5 arrays to be considered;  $A(J,I)$ ,  $B(J,I)$ ,  $C(J,I)$ ,  $D(J,I)$ ,  $F(J,I)$ , and also a variable  $T$ . Initially,  $A(J,I)$ ,  $B(J,I)$ ,  $C(J,I)$ , and  $D(J,I)$  contain the values of  $A'$ ,  $B'$ ,  $C'$ , and  $D$  which were evaluated by SPALD with Eqs. (A40).  $F(J,I)$  contains the values of  $\Phi_i$  from the previous step. The contents of  $A(J,I)$ ,  $B(J,I)$ ,  $T$ , and  $F(J,I)$  as CALC is executed change as follows: (Note that  $NP1 = 4$ , and the number in parentheses is the number of the step executed.)

A(J,3)	$(0)_{A'_3}$	$(1) \frac{A'_3}{D_3}$
A(J,4)	$(0)_{A'_4}$	$(4) \frac{A'_4}{D_4 - \frac{B'_4 A'_3}{D_3}}$
B(J,3)	$(0)_{B'_3}$	$(2) \frac{B'_3 \Phi_2 + C'_3}{D_3}$
B(J,4)	$(0)_{B'_4}$	$(5) \frac{B'_4 \left( \frac{B'_3 \Phi_2 + C'_3}{D_3} \right) + C'_4}{D_4 - \frac{B'_4 A'_3}{D_3}}$
T	$(0)_{0.0}$	$(3) D_4 - \frac{B'_4 A'_3}{D_3}$
F(J,3)	$(0)_{\Phi_3(\text{old})}$	$(7) \frac{A'_3}{D_3} \left\{ \frac{A'_3 \Phi_5 + B'_4 \left( \frac{B'_3 \Phi_2 + C'_3}{D_3} \right) + C'_4}{D_4 - \frac{B'_4 A'_3}{D_3}} \right\} + \frac{B'_3 \Phi_2 + C'_3}{D_3}$
F(J,4)	$(0)_{\Phi_4(\text{old})}$	$(6) \frac{A'_4 \Phi_5 + B'_4 \left( \frac{B'_3 \Phi_2 + C'_3}{D_3} \right) + C'_4}{D_4 - \frac{B'_4 A'_3}{D_3}}$

A little algebra will show that the contents of F(J,3) and F(J,4) after these 7 steps is the same as that given by Eqs. (15) and (16).

After solving (A42) for all the grid points, CALC tests for negative species concentrations. If the test is positive, it puts a very small positive number in F(J,I).

### L. Description of Subroutine SENTP/1

This routine calculates the total enthalpy source term which is given by the sum of Eqs. (A55) and (A56), Appendix I, p. A24. To see how it works consider the enthalpy equation (A23), Appendix F, p. A11. The complete source term is

$$\mathcal{S} = \frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left\{ \sum_j h_j^* \left( \frac{\Delta_j \rho^2}{\langle M \rangle} \frac{\partial(\varphi_j \langle M \rangle)}{\partial \omega} - \frac{\lambda \rho}{C_p} \frac{\partial \varphi_j}{\partial \omega} \right) \right\}$$

SENTP calculates

$$S = \eta \int_{-}^{+} \mathcal{S} d\omega = \frac{1}{\eta} \sum_j \left[ \left\{ \frac{h_j^* \Delta_j \rho^2}{\langle M \rangle} \right\}_{+} \left\{ \frac{\partial(\varphi_j \langle M \rangle)}{\partial \omega} \right\}_{+} - \left\{ \frac{h_j^* \Delta_j \rho^2}{\langle M \rangle} \right\}_{-} \left\{ \frac{\partial(\varphi_j \langle M \rangle)}{\partial \omega} \right\}_{-} \right. \\ \left. - \left\{ \frac{h_j^* \lambda \rho}{C_p} \right\}_{+} \left\{ \frac{\partial \varphi_j}{\partial \omega} \right\}_{+} + \left\{ \frac{h_j^* \lambda \rho}{C_p} \right\}_{-} \left\{ \frac{\partial \varphi_j}{\partial \omega} \right\}_{-} \right]$$

The transport parameters were calculated in TRANS and represent values at the control volume boundaries. They are

$$\text{HDRM}(J, I) = \left\{ \frac{h_j^* \Delta_j \rho^2}{\langle M \rangle} \right\}_{+}$$

$$\text{HDRM}(J, I-1) = \left\{ \frac{h_j^* \Delta_j \rho^2}{\langle M \rangle} \right\}_{-}$$

$$\text{TRCP}(J, I) = \left\{ \frac{h_j^* \lambda \rho}{C_p} \right\}_{+}$$

$$\text{TRCP}(J, I-1) = \left\{ \frac{h_j^* \lambda \rho}{C_p} \right\}_{-}$$

The derivatives are approximated by

$$\left\{ \frac{\partial(\varphi_j \langle M \rangle)}{\partial \omega} \right\}_{+} \approx (F(J, I+1) * \text{FS}(JM, I+1) - F(J, I) * \text{FS}(JM, I)) * \text{ROMD}(I)$$

etc.,

where  $ROMD(I) = 1/(\omega_{i+1} - \omega_i)$

It would be desirable to rewrite SENTIP so that the part arising from the diffusion of the mean molecular weight was calculated separately. Thus, it could easily be neglected if desired.

M. Description of Subroutine ZCALC/1

This short routine calculates the value of the laboratory spatial variable  $y_i$  ( $=Y(I)$ ) which corresponds to a particular value of the spatial variable  $\omega_i$  in the Spalding coordinate system. It does this by evaluating the integral

$$y = \eta \int_0^{\omega} \frac{d\omega}{\rho}$$

by means of the approximation given by Eq. (A70), Appendix L, p. A34.

N. Description of Subroutine OUTPUT(L,HEADNG,RUNID,X,INPUT,LMAX)

This subroutine generates and controls the printing of a variety of output information. At intervals of LPRINT steps during the integration a single page of output is generated. This contains the values at each grid point of the species concentrations, the enthalpy, and the temperature. It also prints the velocities calculated from the production rates of each species, the grid width PEI, and the entrainment rates RME and RMI. When L = LMAX, the integration is stopped and 11 pages of additional information are printed:

Page 1) Values at each grid point of  $\omega$ , the laboratory spatial variable y in meters, the density in  $\text{kg/m}^3$ , the mean molecular weight  $\langle M \rangle$  in  $\text{kg/mole}$ , and the total heat release rate in  $\text{J/kg-s}$ .

2) The production rates  $R(J,I)$  in  $\text{moles/kg-s}$  for each species at each grid point.

3) The transport parameters at each grid point; i.e., the array  $\text{PREF}(J,I)$ ,  $\Delta_j \rho^2$  and  $\lambda \rho / C_p$  in  $\text{kg}^2/\text{m}^4\text{-s}$ .

4,5) Values of each rate constant, both forward and reverse rates, at each grid point.

6,7) The rates, forward and reverse, of each reaction at each grid point in  $\text{moles/kg-s}$ .

8) The maximum rates, forward and reverse, of each reaction and the grid point index at which they occur.

9) A list of the maximum rates of each reaction ranked in descending values. For each maximum rate, its K value and direction are specified.

10,11) Heat release rates, both forward and reverse, of each reaction at each grid point in  $\text{J/kg-s}$ .

OUTPUT begins by printing the step number L, the step size DX, and the integration time X. It then prints the heading for the profile columns and follows with  $F(J,I)$ ,  $F(JH,I)$ ,  $FS(JI,I)$ ,  $VEL(J)$ ,  $PEI$ ,  $RMI$ , and  $RME$ .

It tests for  $L = LMAX$ ; if `.TRUE.`, it proceeds by printing the values of the grid parameters,  $N$ ,  $NII$ ,  $NEE$ , and  $OMR$ ; the index parameters  $JH$ ,  $JRAD$ ,  $JM$ ,  $JMR$ ,  $JT$ ,  $JTR$ ,  $JBODYA$ ,  $JBODYB$ ; the temperature on the cold side of the flame  $TCOLD$ , the pressure  $PRESS$ , and the value of  $INPUT$ .

$FLAG(2)$  is tested; if `.FALSE.`, then the buffer concentration is being kept constant. This means that its production rate  $R(JHM1, I)$  is zero. In this case, the production rate for the next major species  $R(JIM2, I)$  was not calculated by `SCHM2`. `OUTPUT` calculates it from the relation  $\sum_J^{JHM1} R(J, I) WT(J) = 0.0$ . This section is skipped if  $FLAG(2) = .TRUE.$ .

`OUTPUT` next calculates the quantities  $PUT(J) = h_j M_j = h_j^*$ , and  $FOR(I) = \sum_J^{JHM1} R(J, I) * PUT(J)$ . This latter quantity is the total heat release rate in  $J/kg-s$ . It then prints  $OM(I)$ ,  $Y(I)$ ,  $RHO(I)$ ,  $FS(JM, I)$ , and  $FOR(I)$ , followed by the arrays  $R(J, I)$  and  $PREF(J, I)$ .

It then tests  $FLAG(9)$ ; if `.TRUE.`, it prints the rate constants. This requires two pages. Each column represents a grid point whose index value appears as the heading. There are 16 columns per page giving a total of 32 grid positions. This may not be sufficient to cover all the grid points. Since points near the boundaries are not very interesting, we start printing at  $I = ISTART$ , where  $ISTART = 4$  in the present version of `OUTPUT`. Its value can, of course, be changed to suit the user's requirements. The next page starts printing at  $ISTART = ISTART + 16$  which equals 20 in the present case.

$FLAG(10)$  is then tested; if `.TRUE.`, the reactions rates are calculated at each point in the flame. Note that  $FOR(I)$  which contained the total heat release rate is now used for the forward reaction rate. The rates are calculated from scratch; i.e., from the values of the rate constants  $RATE(K, I)$ ,  $RATER(K, I)$  and the concentrations  $F(J, I)$ . Also calculated at this time are the heat release rates for the forward and reverse reactions. These are stored in the arrays  $HV(K, I)$ , and  $HW(K, I)$ , respectively. Note that we have treated the cases  $K = 5$  and 10 in the same manner as discussed in `SCHM2`. To use `OUTPUT` for flames (other than  $H_2-O_2-N_2$ ) which have more than 4 reactions, it is necessary

to remove the following lines:

```

OUTO1680
OUTO1690
OUTO1960
  :
  :
OJT 02100

```

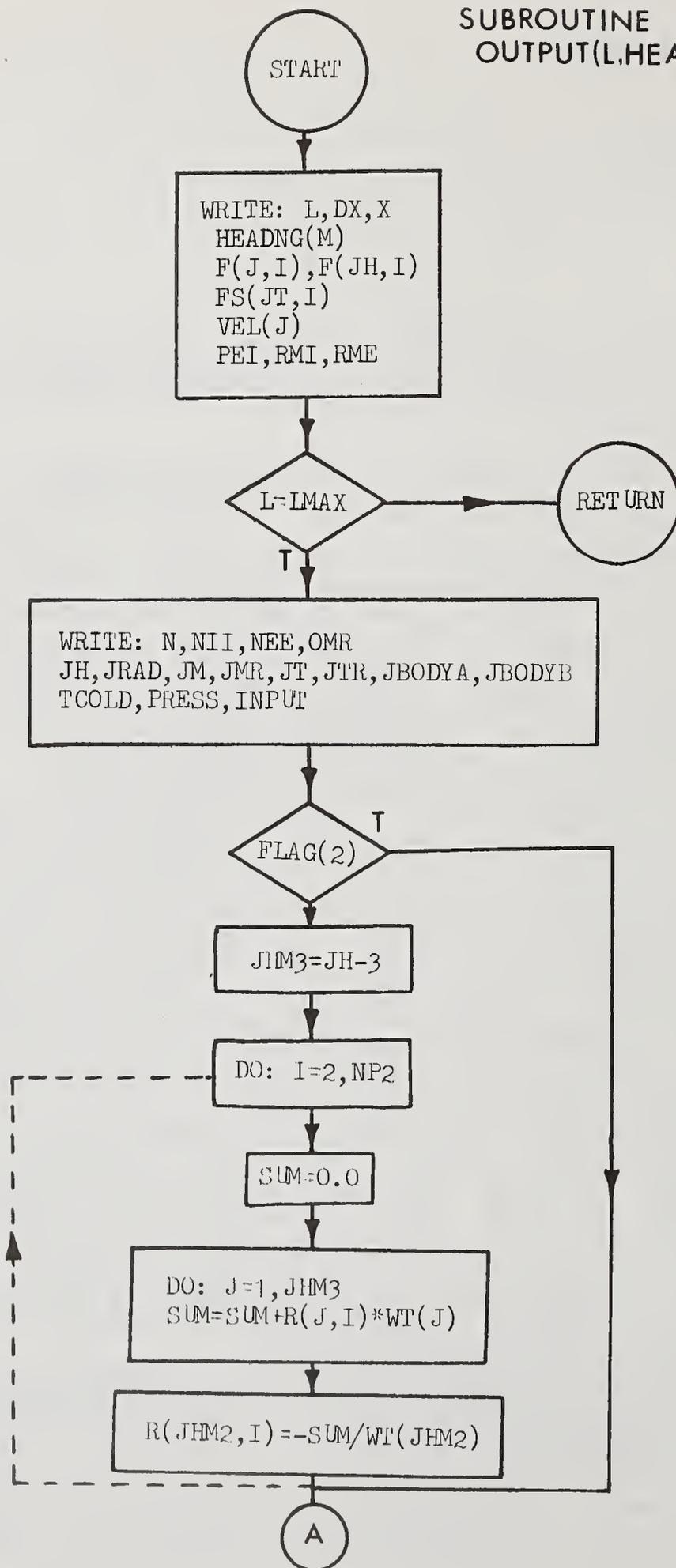
The reaction rates are then printed on two pages in the same format as the rate constants were printed. Note that ISTART also appears in this portion of the routine.

The maximum rate of each reaction is now determined. This is accomplished by running through the grid index I for each reaction K and setting the variable XMAX(K) equal to V(K,I) if V(K,I) is greater than the previous value of XMAX(K). V(K,I) is the rate of the forward reaction and is in single precision. The same thing is done for the reverse reaction and yields YMAX(K). XMAX(K) and YMAX(K) are then printed along with the grid index value to which they correspond.

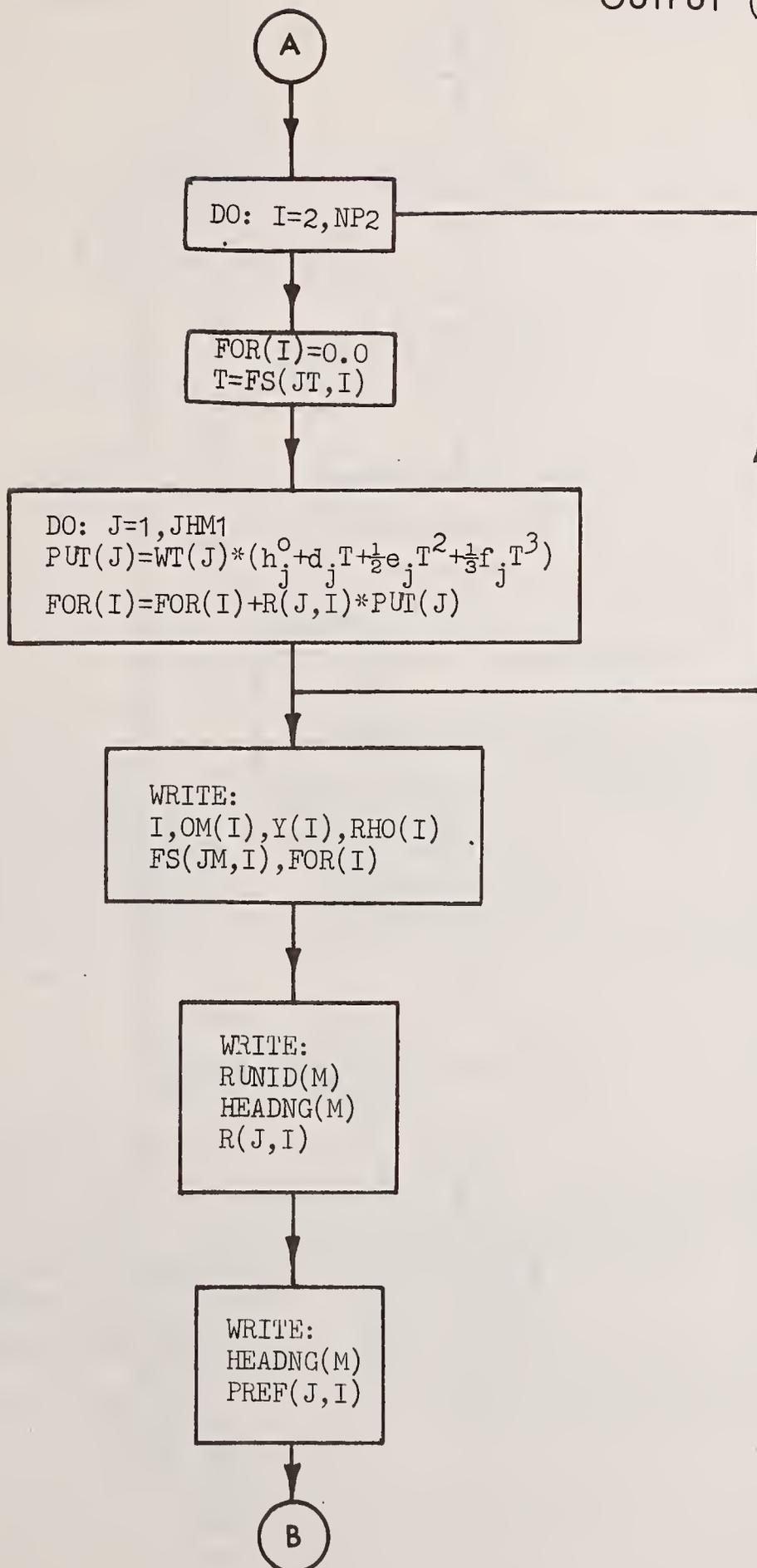
A new single precision variable Z(K) is then defined;  $Z(K) = XMAX(K)$  and  $Z(K+KRATE) = YMAX(K)$  for  $K = 1, KRATE$ . Thus Z(K) runs from  $K = 1$  to  $2*KRATE$ . The ranking of the maximum values of the rates is accomplished by running through Z(K) and setting ZTAB = Z(K) if Z(K) is greater than the previous value of ZTAB. KTAB is set equal to the value of K for this maximum - maximum rate. Also defined is a logical variable SKIP(K),  $K = 1, 2*KRATE$ , set initially to the .FALSE. position. At this point OUTPUT sets  $SKIP(KTAB) = .TRUE.$ . Then the Z(K) are re-examined for the maximum value, but now Z(KTAB) is omitted. This is accomplished by testing SKIP(K); if .TRUE., then KTAB is skipped and the next K value is compared with ZTAB. The list from which the maximum value is selected thus becomes smaller as each maximum is removed. In this way the ranking is accomplished. The maximum values of each reaction are then printed in order of decreasing magnitude.

Finally, OUTPUT prints two pages of heat release rates for each reaction. Note that ISTART appears here also.

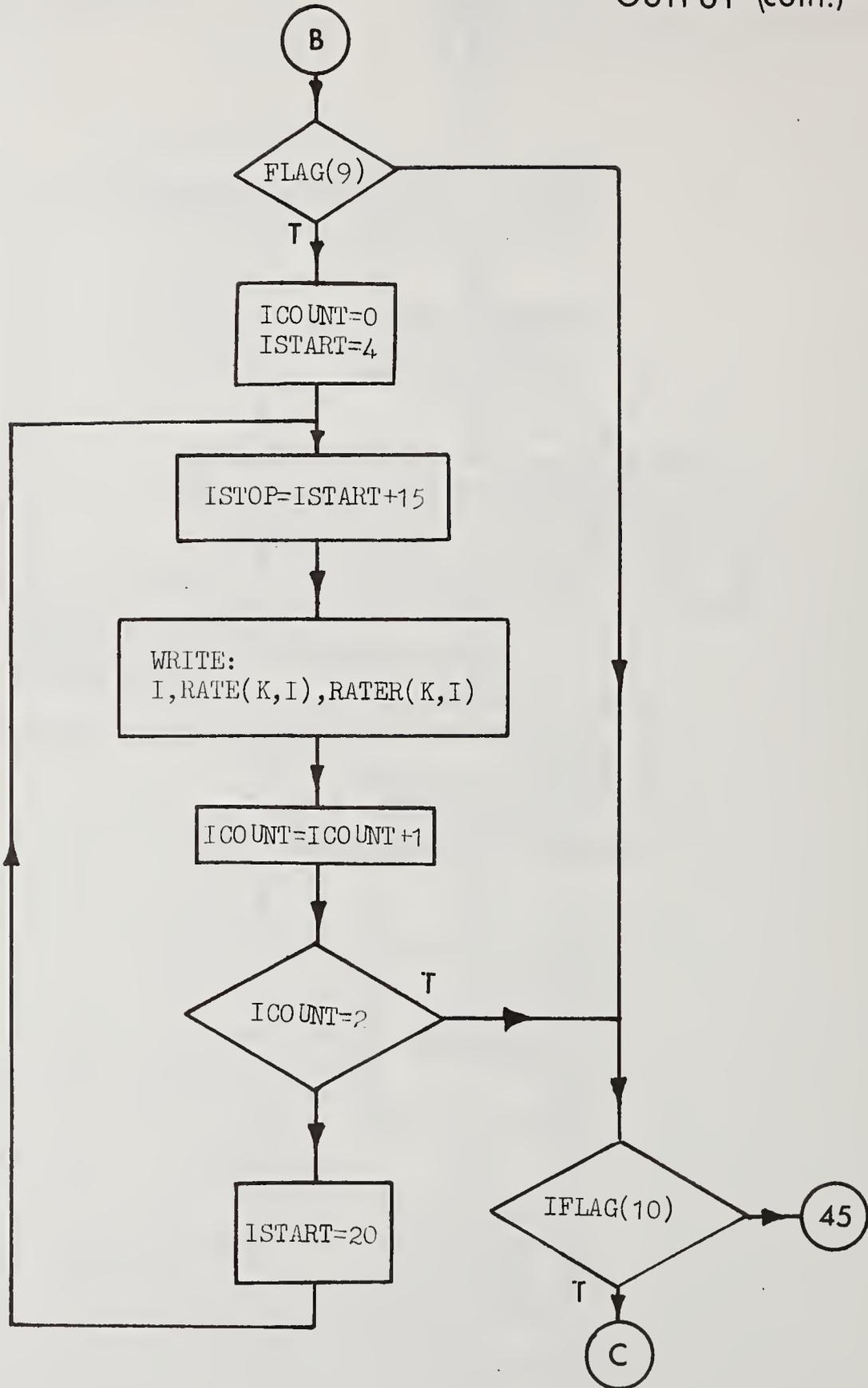
SUBROUTINE  
OUTPUT(L,HEADNG,RUNID,X,INPUT,LMAX)



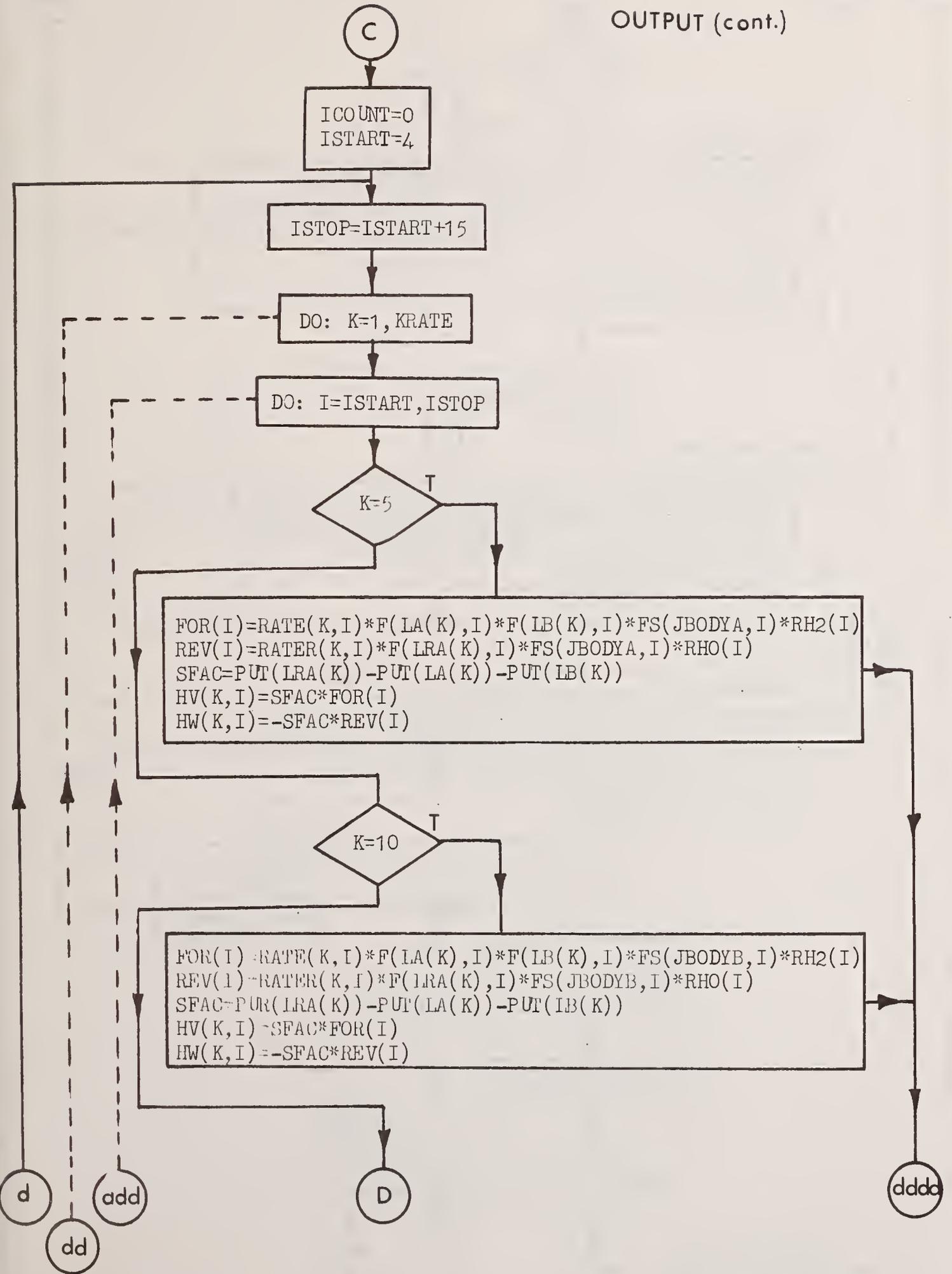
OUTPUT (cont.)



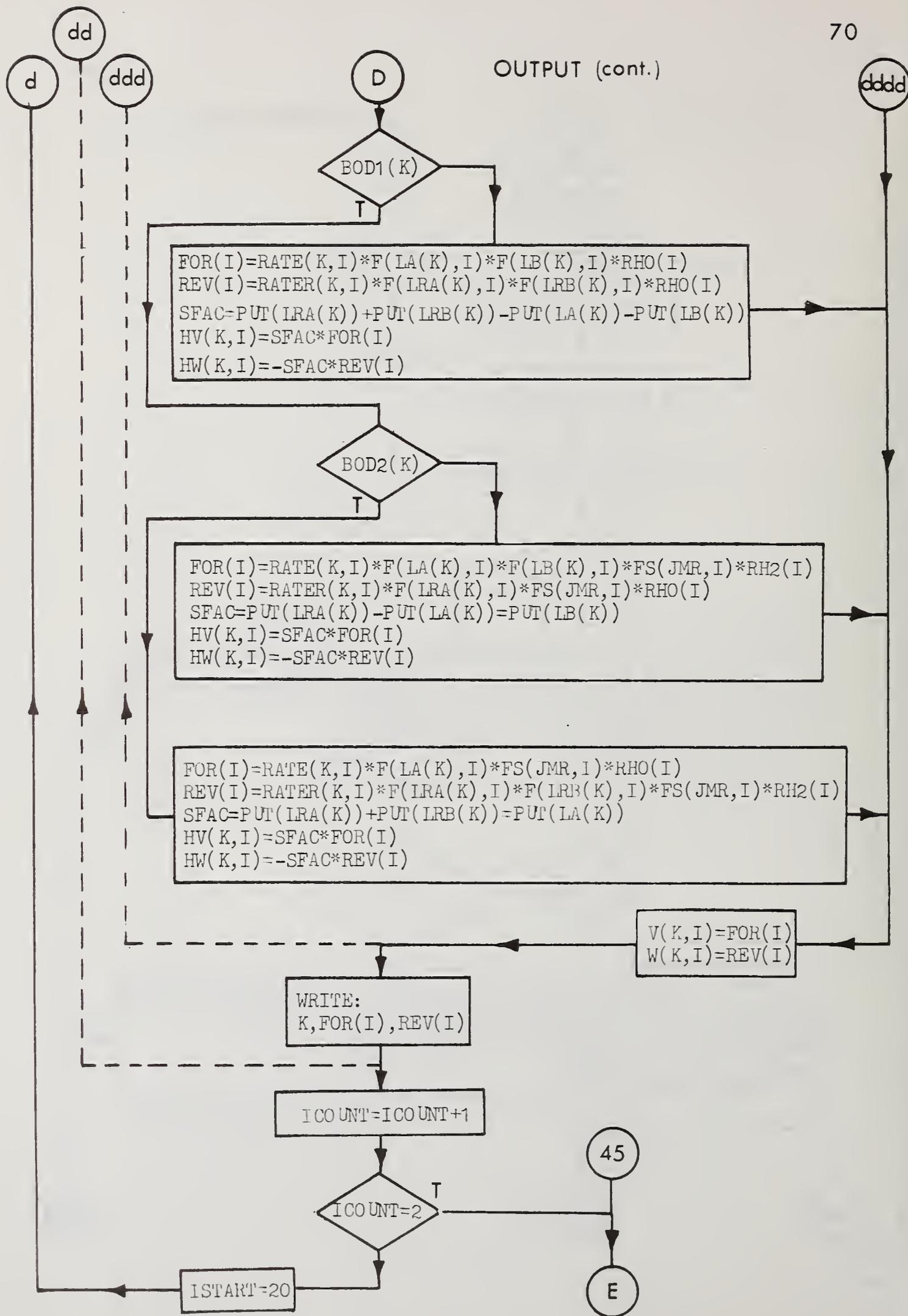
OUTPUT (cont.)



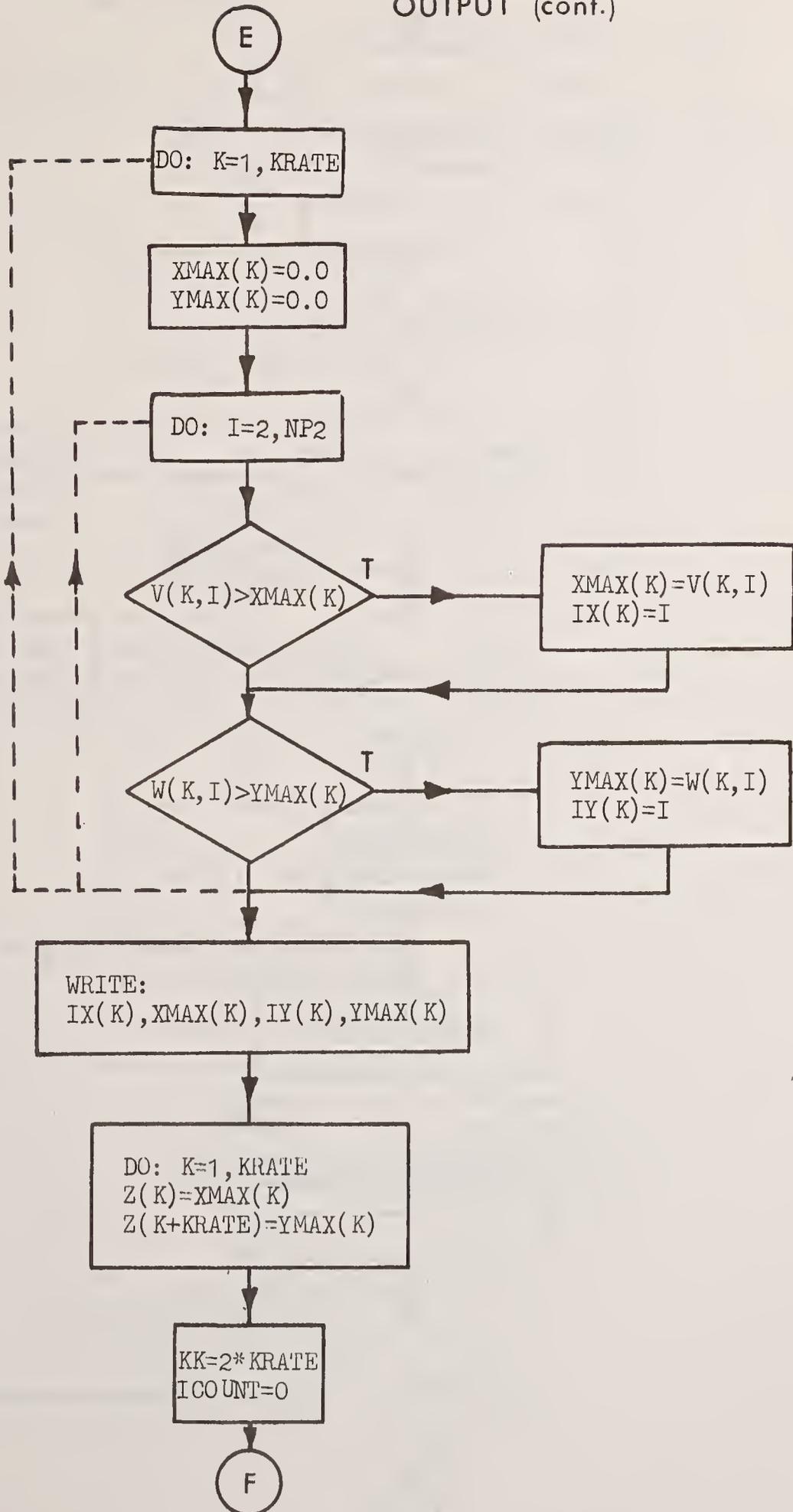
OUTPUT (cont.)



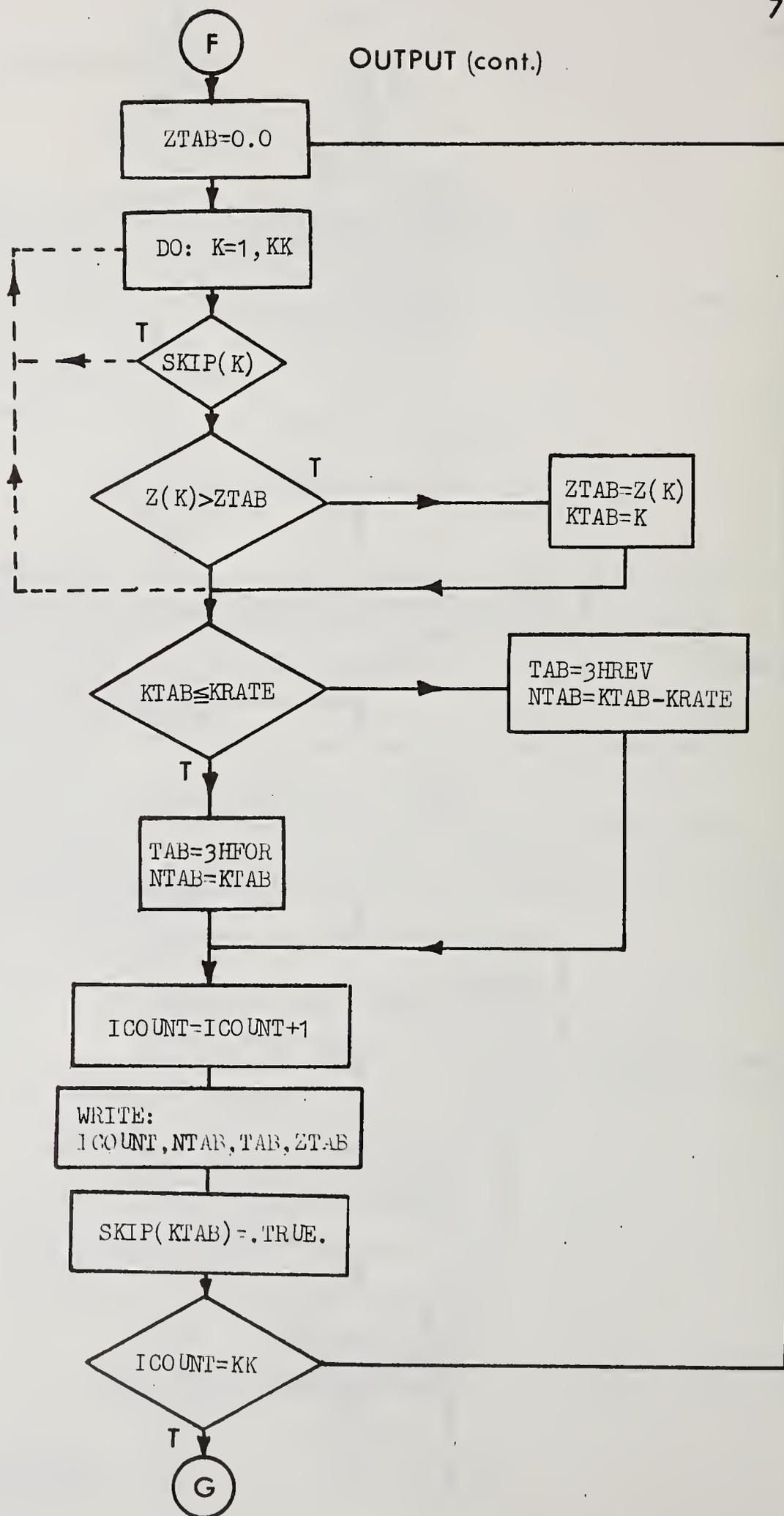
## OUTPUT (cont.)

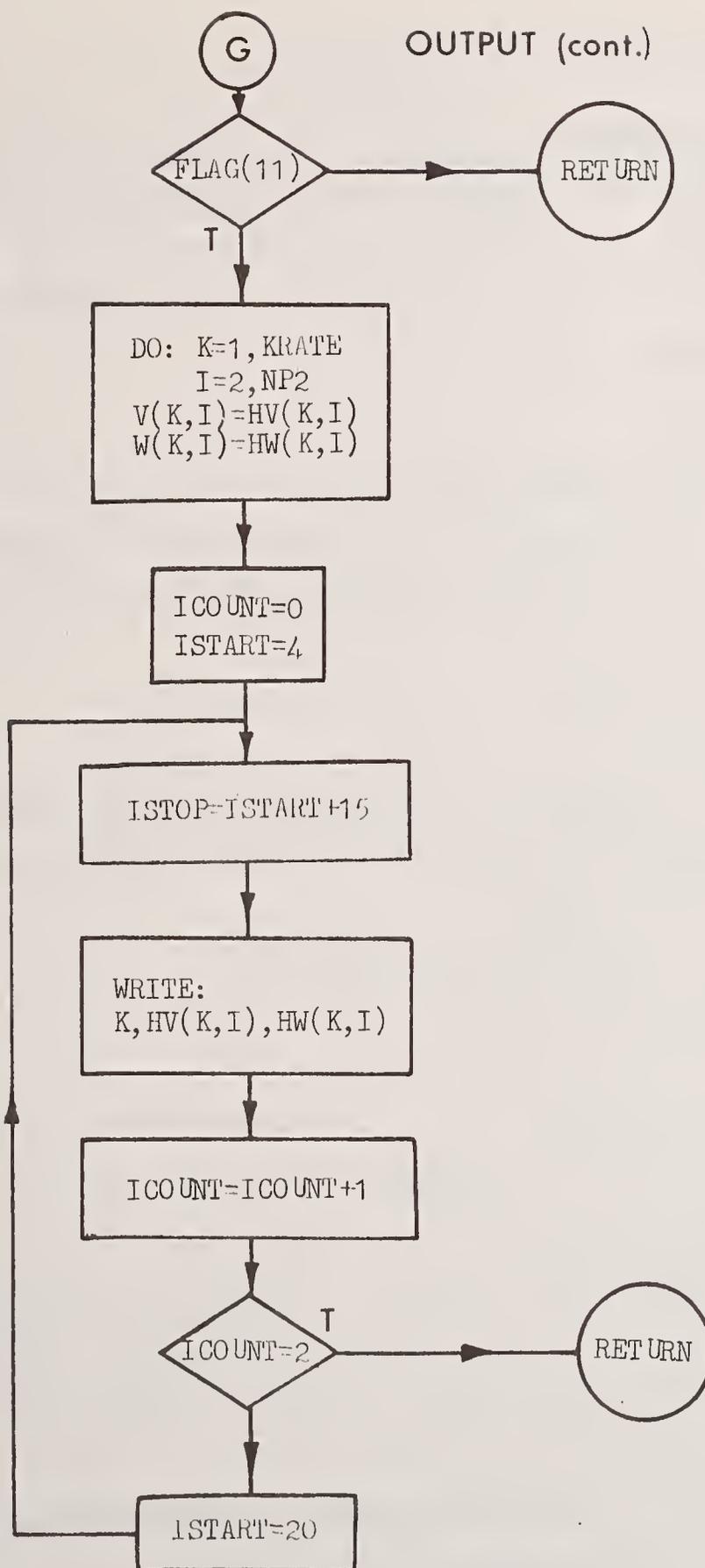


OUTPUT (cont.)



OUTPUT (cont.)





## IV. TEST CASE

As an example of the use of this program we have utilized the  $H_2$ - $Br_2$  flame. The input data were taken from a paper by Lovachev and Kaganova<sup>2</sup>. Spalding and Stephenson<sup>3</sup> have also used the Spalding method for this flame. The following input information was used:

Thermal conductivity:

$$\lambda = \lambda_u (T/T_u)^{0.67}$$

where  $\lambda_u$  is the thermal conductivity at the temperature  $T_u$  of the unburnt gas. It is assumed to be independent of concentration. For  $T_u = 323^\circ K$ ,  $\lambda_u$  is taken to have the value  $3.349 \times 10^{-2} J/m \cdot ^\circ K \cdot s$ .

Diffusion Coefficients:

$$\Delta_j = \Delta_{ju} (T/T_u)^{1.67} / P$$

where  $P$  is the pressure in atmospheres and the  $\Delta_{ju}$  are considered to be independent of concentration and are the values at temperature  $T_u$ . For  $T_u = 323^\circ K$ , we use

$$\Delta_{H_2u} = \Delta_{Br_2u} = \Delta_{HBr_u} = 0.101 \times 10^{-4} m^2/s$$

$$\Delta_{Hu} = 1.05 \times 10^{-4} m^2/s$$

$$\Delta_{Bru} = 0.155 \times 10^{-4} m^2/s$$

Thermodynamic properties:

The specific heat of the mixture is assumed to be independent of temperature and concentration. The value used was  $C_p = 532 J/kg \cdot ^\circ K = d_j$ . In this case, the coefficients  $e_j$  and  $f_j = 0$  and all species have the same value of  $d_j$ .

The enthalpies of formation as taken from the JANAF tables are,

$$h_{Br}^R(298^\circ) = 1.4009 \times 10^6 J/kg \quad h_{Br_2}^R(298^\circ) = 1.93505 \times 10^5 J/kg$$

$$h_H^R(298^\circ) = 2.1641 \times 10^8 \quad h_{HBr}^R(298^\circ) = -4.4807 \times 10^5$$

$$h_{H_2}^R(298^\circ) = 0.0$$

Note that the value for HBr is slightly different from the value of  $-4.5304 \times 10^5$  J/kg used by Lovachev and Kaganova.

For the program we want values for  $h_j^0 = \text{HREF}(J) = h_j^R - d_j T_R$ , where  $T_R$  is the reference temperature for  $h_j^R$ . We have

$$h_{\text{Br}}^0 = 1.242540 \times 10^6 \text{ J/kg}$$

$$h_{\text{H}}^0 = 2.162510 \times 10^8$$

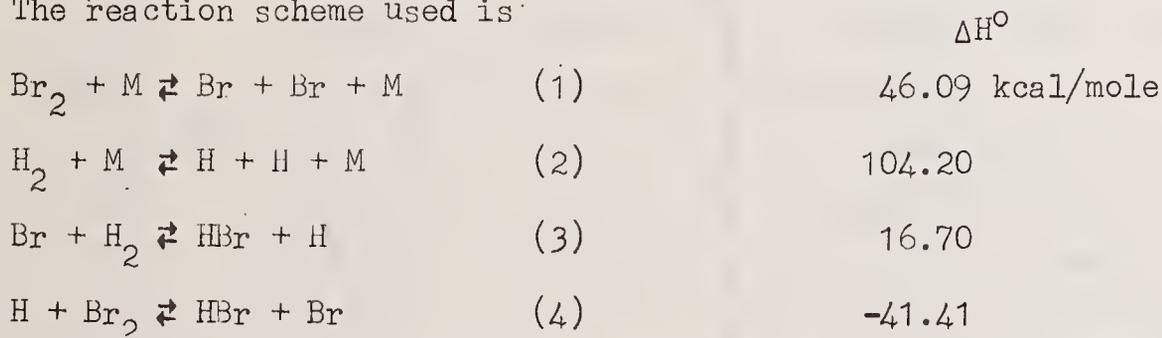
$$h_{\text{H}_2}^0 = -1.583691 \times 10^5$$

$$h_{\text{Br}_2}^0 = 3.513587 \times 10^4$$

$$h_{\text{HBr}}^0 = -6.064374 \times 10^5$$

Rate Constants:

The reaction scheme used is:



The forward rate constants used were

$$k_1 = 7.03 \times 10^{11} T^{-\frac{1}{2}} \exp(-2.3655 \times 10^4 / T) \quad \text{m}^3/\text{mole-s}$$

$$k_2 = 7.63 \times 10^{13} T^{-1} \exp(-5.4356 \times 10^4 / T)$$

$$k_3 = 3.46 \times 10^4 T \exp(-8.3547 \times 10^3 / T)$$

$$k_4 = 6.42 \times 10^6 T^{\frac{1}{2}} \exp(-5.5866 \times 10^2 / T)$$

Free energies of reaction in kcal/mole were calculated from the formulas,

$$\Delta F_1^0 = 46.1756 - 2.49467 \times 10^{-2} T - 6.5671 \times 10^{-7} T^2$$

$$\Delta F_2^0 = 104.426 - 2.38954 \times 10^{-2} T - 1.417614 \times 10^{-6} T^2$$

$$\Delta F_3^0 = 16.532 - 1.49615 \times 10^{-3} T - 1.91112 \times 10^{-7} T^2$$

$$\Delta F_4^0 = -41.7184 - 2.54835 \times 10^{-3} T + 5.69792 \times 10^{-7} T^2$$

For the reverse reactions Lovachev and Kaganova use

$$k'_1 = 3.63 \times 10^3 \text{ m}^6/\text{mole}^2\text{-s}$$

$$k'_2 = 3.63 \times 10^3$$

$$k'_3 = 9.06 \times 10^5 T^{\frac{1}{2}} \exp(-8.6064 \times 10^2/Y) \text{ m}^3/\text{mole-s}$$

$$k'_4 = 6.52 \times 10^4 T \exp(-2.1440 \times 10^4/T)$$

These formulas yield values of the reverse rate constants which are slightly different from those calculated with the  $\Delta F^0$  values via the equilibrium constants. For comparison, Table I gives the forward rates, equilibrium constants  $K_p$ , reverse rates calculated from  $K_p$ , and the reverse rates of Lovachev and Kaganova.

The other conditions for this test case were, pressure = 1 atm, cold boundary temperature = 323°K, cold gas composition,  $\text{Br}_2 = 40\text{mole}\%$ , balance,  $\text{H}_2$ .

The format for this input data is shown in Table II. Cards containing the symbol @ are control statements for the NBS UNIVAC 1108.

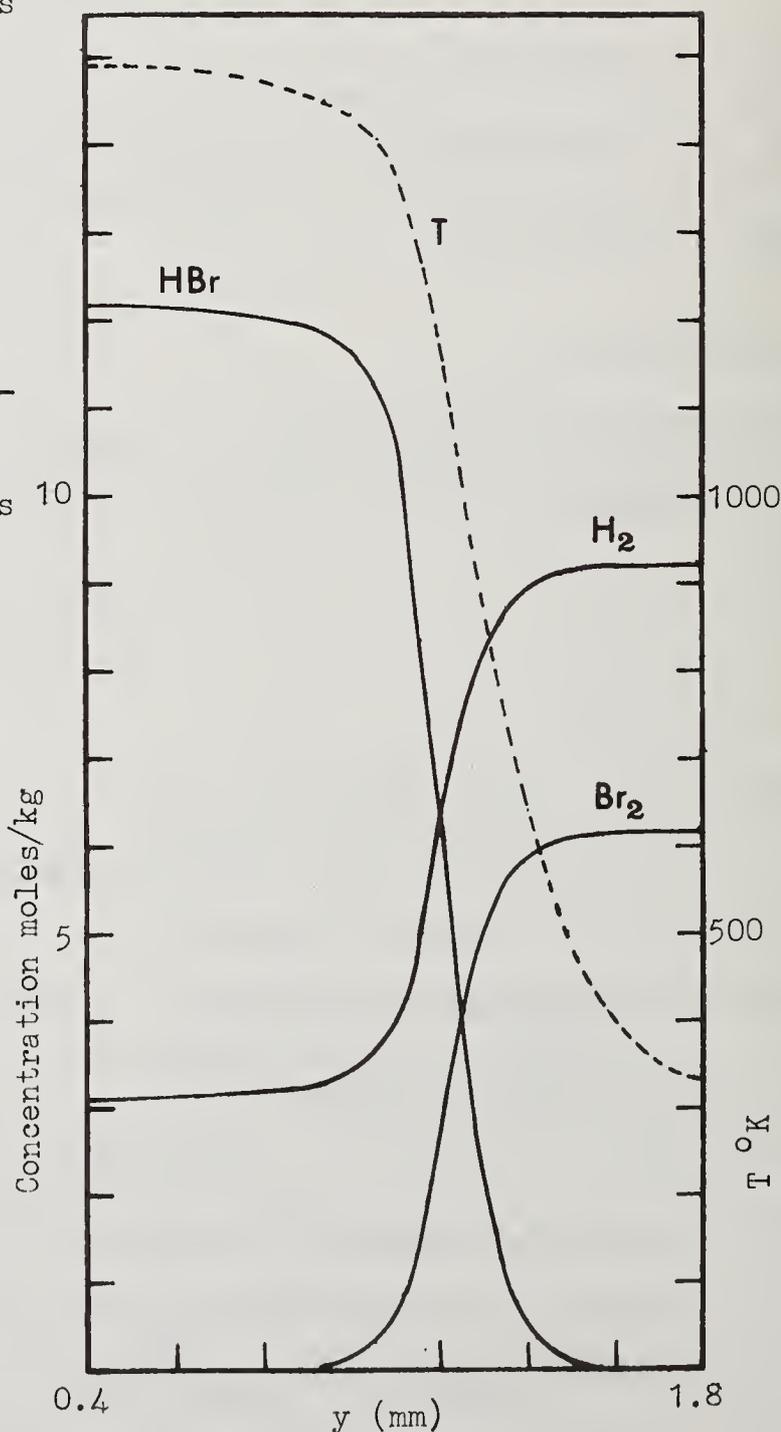


Figure 4  
Concentration Profiles for the Major Species

TABLE I. Rate constants for the  $H_2$ - $Br_2$  flame as a function of temperature.

Temperature	$k_1$	$K_p$	$k_1'$	$k_1'^a$
400	$7.2918 \times 10^{-16}$	$1.8963 \times 10^{-20}$	$1.2621 \times 10^3$	$3.6300 \times 10^3$
600	$2.1668 \times 10^{-7}$	$5.2174 \times 10^{-12}$	$2.0447 \times 10^3$	"
800	$3.5799 \times 10^{-3}$	$8.9451 \times 10^{-8}$	$2.6272 \times 10^3$	"
1000	1.1850	$3.1882 \times 10^{-5}$	$3.0499 \times 10^3$	"
1200	$5.5762 \times 10^1$	$1.6384 \times 10^{-3}$	$3.3513 \times 10^3$	"
1400	$8.6271 \times 10^2$	$2.7841 \times 10^{-2}$	$3.5598 \times 10^3$	"
1600	$6.6700 \times 10^3$	$2.3689 \times 10^{-1}$	$3.6967 \times 10^3$	"
Temperature	$k_2$	$K_p$	$k_2'$	$k_2'^a$
400	$1.8373 \times 10^{-48}$	$1.9216 \times 10^{-52}$	$3.1383 \times 10^2$	$3.6300 \times 10^3$
600	$5.7569 \times 10^{-29}$	$2.3267 \times 10^{-33}$	$1.2182 \times 10^3$	"
800	$2.9600 \times 10^{-19}$	$8.6949 \times 10^{-24}$	$2.2348 \times 10^3$	"
1000	$1.8881 \times 10^{-13}$	$5.1000 \times 10^{-18}$	$3.0379 \times 10^3$	"
1200	$1.3529 \times 10^{-9}$	$3.7477 \times 10^{-14}$	$3.5547 \times 10^3$	"
1400	$7.4921 \times 10^{-7}$	$2.2544 \times 10^{-11}$	$3.8178 \times 10^3$	"
1600	$8.4011 \times 10^{-5}$	$2.8378 \times 10^{-9}$	$3.8868 \times 10^3$	"
Temperature	$k_3$	$K_p$	$k_3'$	$k_3'^a$
400	$1.1753 \times 10^{-2}$	$2.0411 \times 10^{-9}$	$5.7582 \times 10^6$	$2.1073 \times 10^6$
600	$1.8616 \times 10^1$	$2.1356 \times 10^{-6}$	$8.7170 \times 10^6$	$5.2875 \times 10^6$
800	$8.0661 \times 10^2$	$6.9745 \times 10^{-5}$	$1.1565 \times 10^7$	$8.7390 \times 10^6$
1000	$8.1410 \times 10^3$	$5.6918 \times 10^{-4}$	$1.4303 \times 10^7$	$1.2116 \times 10^7$
1200	$3.9318 \times 10^4$	$2.3220 \times 10^{-3}$	$1.6933 \times 10^7$	$1.5319 \times 10^7$
1400	$1.2402 \times 10^5$	$6.3739 \times 10^{-3}$	$1.9457 \times 10^7$	$1.8332 \times 10^7$
1600	$2.9885 \times 10^5$	$1.3658 \times 10^{-2}$	$2.1881 \times 10^7$	$2.1163 \times 10^7$
Temperature	$k_4$	$K_p$	$k_4'$	$k_4'^a$
400	$3.1769 \times 10^7$	$2.0142 \times 10^{23}$	$1.5773 \times 10^{-16}$	$1.3729 \times 10^{-16}$
600	$6.1978 \times 10^7$	$4.7888 \times 10^{15}$	$1.2942 \times 10^{-8}$	$1.1838 \times 10^{-8}$
800	$9.0324 \times 10^7$	$7.1752 \times 10^{11}$	$1.2588 \times 10^{-4}$	$1.1968 \times 10^{-4}$

TABLE V. Continued.

Temperature	$k_4$	$K_p$	$k_4'$	$k_4'^a$
1000	$1.1612 \times 10^8$	$3.5582 \times 10^9$	$3.2634 \times 10^{-2}$	$3.1826 \times 10^{-2}$
1200	$1.3962 \times 10^8$	$1.0152 \times 10^8$	1.3752	1.3611
1400	$1.6117 \times 10^8$	$7.8715 \times 10^6$	$2.0475 \times 10^1$	$2.0386 \times 10^1$
1600	$1.8112 \times 10^8$	$1.1402 \times 10^6$	$1.5885 \times 10^2$	$1.5802 \times 10^2$

<sup>a</sup>Reverse rate constants calculated from the expressions given by Lovachev and Kaganova.

TABLE II. Format of input data for the H<sub>2</sub>-Br<sub>2</sub> flame.

```

@RUN,M/R BROWN2,36242-BROWN,ABCDE,15,75
@ASG,A FLAME.
@PRT,T FLAME.
@MAP,IS TEST,TEST
LIB FLAME.
IN SPALD/3,INITI/1,TRANS/6,RATCH/3,SCHM2/7,CALC/1
IN ENIRN/3,SENTP/1,SCH1/1,ZCALC/1
END
@XQT TEST

5 DEC 74 R1 H2-BR2 FLAME. IOVACHEV & KAGNOVA DATA.
BR      H      H2      BR2      HBR      BLANK      BLANK      BLANK
BLANK   BLANK  BLANK  BLANK  BLANK  ENTHALPY  TEMP
1 2 3 4 5 6 7 8 9101112
301530
051001
3008231.3D0
6 2 1 2 3 4 2 0 4 5 6
.200D0 .000D0 .800D0
.600D0 .400D0 .000D0
001
323.D01.00D0.2D-2
1.242540D+6 0.531724D+3 0.000000D+0 0.000000D+0 79.916D-3
2.162510D+8 0.531724D+3 0.000000D+0 0.000000D+0 1.008D-3
-1.583691D+5 0.531724D+3 0.000000D+0 0.000000D+0 2.016D-3
3.513587D+4 0.531724D+3 0.000000D+0 0.000000D+0 159.83D-3
-6.064374D+5 0.531724D+3 0.000000D+0 0.000000D+0 80.92D-3
5.D-65.D-4600.D0
300030
.10
04
7.0D11-.500D02.3655D4TT
7.6D13-1.00D05.4356D4TT
3.46D4 1.00D08.3547D3TT
6.42D6 .500D05.5866D2TT

```

```

RUNID(M),M=1,12
HEADNG(M),M=1,126
IH,LO,LOH,LH02,LH202,IX,LHX,IX2,LO2,
LD,LE,LC
LDELA,LDEIB,LDELIC
N,NII,NEE,OMR
JH,JRAD,JM,JR,JT,JTR,ITER,ITER1,
XI(J),J=JRADP1,JHM1
XF(J),J=JRADP1,JHM1
INPRFL(J),J=JRADP1,JHM1
TCOLD,PRESS,YWIDTH
HREF(J),DCP(J),ECP(J),FCP(J),WT(J),
J=1,JHM1

```

```

DXMIN,DXMAX,STEPS
IMAX,LPRINT
ALPHA
KRAT
FRQ(K),BETA(K),EACT(K),BFLAG(K),
EFFLAG(K),K=1,KRATE

```



Since the format of the output makes it unsuitable for reproduction in this report, we shall illustrate some of it in graphic form. The steady-state velocity of this flame is 0.241 m/s and the hot side flame temperature is 1518°K for a complete reaction. This velocity is 1.2% lower than that obtained by Spalding and Stephenson; the difference probably arises from the slightly different values used here for the reverse rate constants. (See Table I.)

Concentration profiles for the major species are shown in Fig. 4. These are plotted as a function of the laboratory spatial variable  $y$  in mm from the hot boundary. Also shown here is the temperature profile.

Profiles for the two trace species, H and Br, are shown in Fig. 5 along with the enthalpy profile. This latter profile has units of J/kg. The rates of the major reactions are shown in Fig. 6. The + sign in the figure denotes a forward rate. Figure 7 gives the heat release rates for the major reactions and also the total heat release rate as a function of distance through the flame.

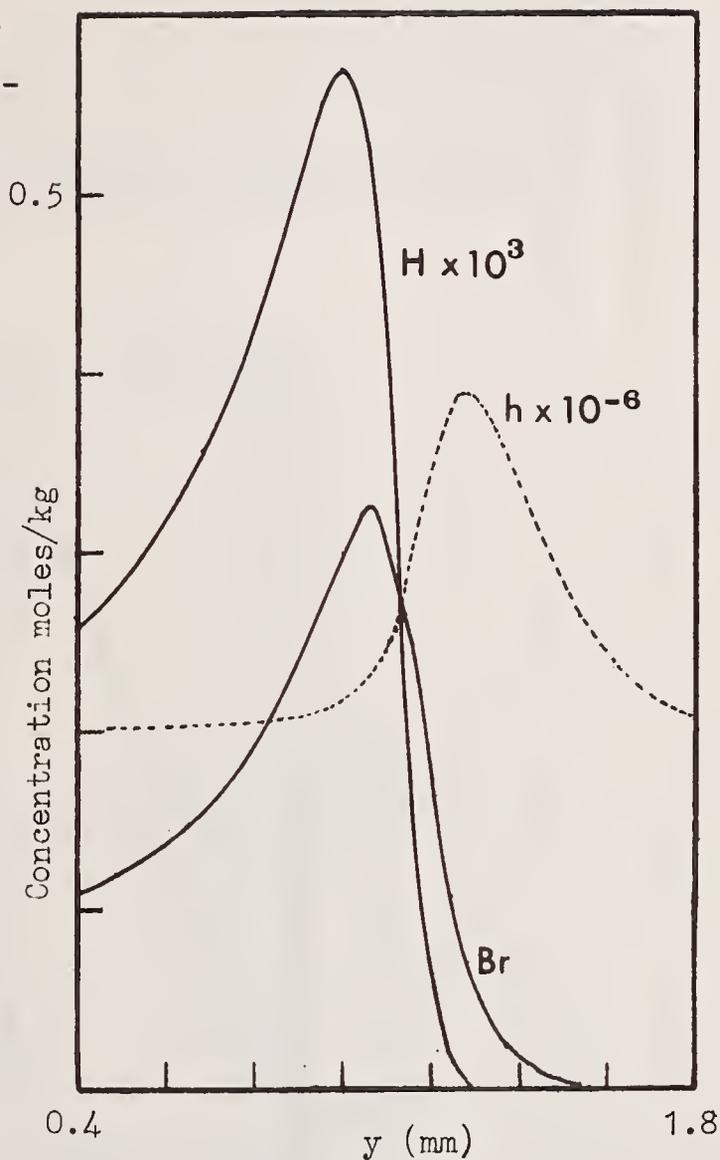


Figure 5  
Concentration Profiles for the Minor Species

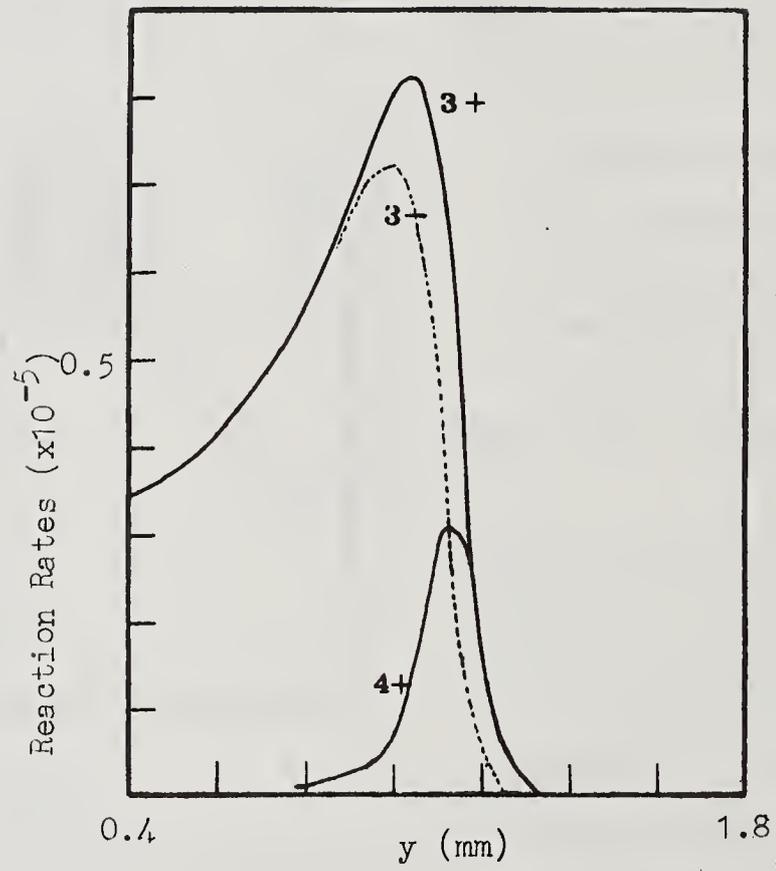


Figure 6  
Rates of Major Reactions

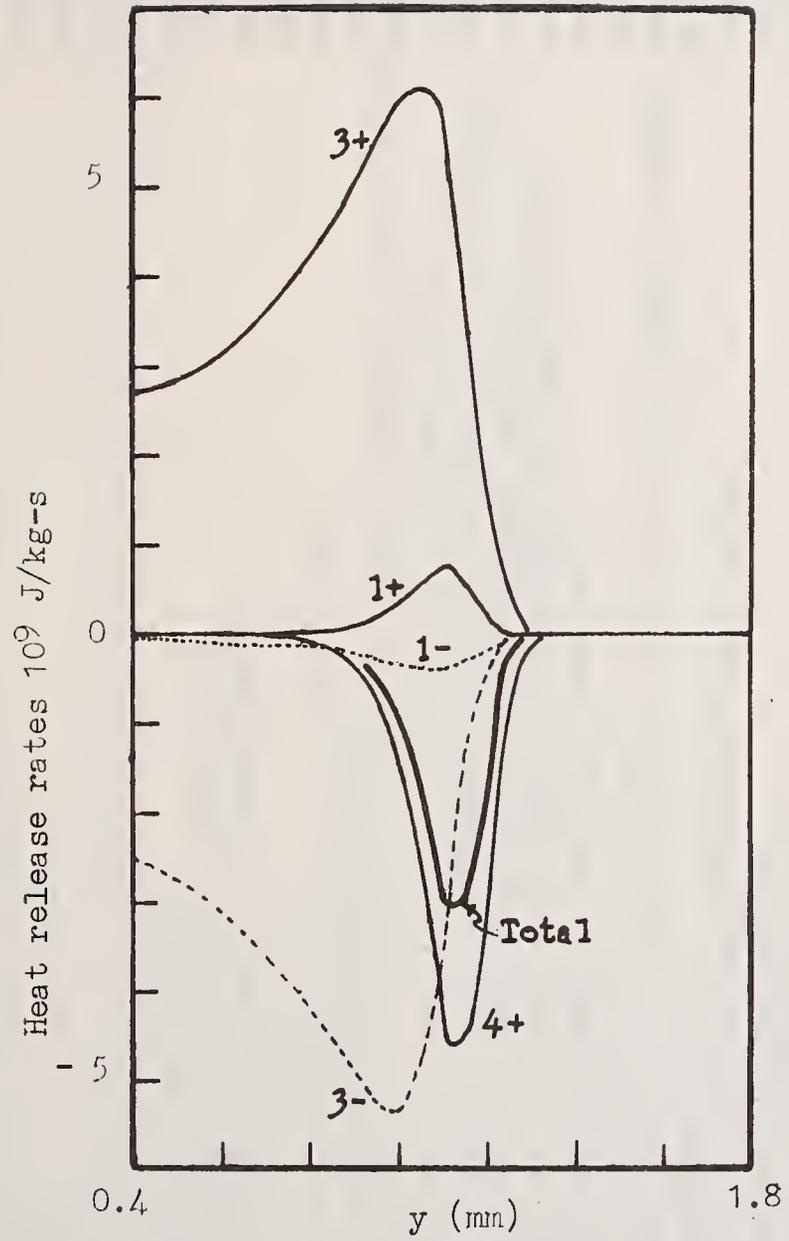


Figure 7  
Heat Release Rates

@ELT, SIB ABCDE\*FLAME(2), SPALD/3,.., 173314133212  
 IMPLICIT REAL\*8 (A-H, O-Z)  
 REAL ALPHA, RUNID(12), HEADNG(126)  
 LOGICAL BØD1, BØD2, FLAG, G1, G2, G3  
 LOGICAL SPECIE(15), REACT, BFLAG, EFLAG, FFLAG, RFLAG  
 CØMMØN/IVALSN, NP1, NP2, NP3  
 CØMMØN/JVALS/JH, JHM1, JHM2, JRAD, JRADP1, JM, JMR, JT, JTR, JBØDYA, JBØDYB  
 CØMMØN/ØMGA1/NI1, NEE, ØMR  
 CØMMØN/ØMGA2/ØM(43), ØMI, ØME, RØMD(43)  
 CØMMØN/ØMGA3/BØM(43), BØMT3(43), ØMP(43), PEIØMD(43), PEIØM2(43)  
 CØMMØN/INIT/XI(15), XE(15), IPRFL(15), TCØLD, PRESS, YWIDTH  
 CØMMØN/TRANS1/HREF(15), DCP(15), ECP(15), FCP(15), WT(15), ITER, ITER1  
 CØMMØN/TRANS2/RMB(43), PREF(15,43), HDRM(15,43), TRCP(15,43)  
 \*, DR2M(15,43)  
 CØMMØN/TRANS3/RHØ(43), RU(43), RH2(43)  
 CØMMØN/TRANS4/DA(14,14), DB(14,14), TDIV(14,14)  
 CØMMØN/VAR1/F(15,43), FS(15,43)  
 CØMMØN/YCALC/PEI, Y(43), YD(43), YAV(43), RPEI  
 CØMMØN/RTPARM/FRQ(30), BETA(30), EACT(30)  
 \*, BØD1(30), BØD2(30), REACT(30)  
 CØMMØN/RATC/RATE(30,43), RATER(30,43), KRATE, EQA(30), EQB(30), EQC(30)  
 CØMMØN/SCM1/SC1(15,43)  
 CØMMØN/SCM2/SU(15,43), SD(15,43)  
 CØMMØN/SENT/SEN(43)  
 CØMMØN/CØEF/A(15,43), B(15,43), C(15,43), D(15,43)  
 CØMMØN/ENTR/VEL(15), RMI, RME, DX, ALPHA  
 CØMMØN/MASPRØ/R(15,43)  
 CØMMØN/CMINDX/LH, LØ, LØH, LHØ2, LH2Ø2, LX, LHX, LX2, LØ2, LH2Ø, LH2, LN2  
 CØMMØN/LVALS/LCØUNT, LPRINT  
 CØMMØN/REACT/LA(30), LB(30), LRA(30), LRB(30)  
 CØMMØN/FLAGS/FLAG(20)  
 CØMMØN/FLAGR/BFLAG(30), EFLAG(30), FFLAG(30), RFLAG(30)  
 CØMMØN/GSWITC/G1(30,15), G2(30,15), G3(30,15)  
 DIMENSION ITEST(10), PGØMP(43), PGØMM(43), PBØM(43)  
 DIMENSION LØRDER(15)

DATA SC1/645\*.0D0/, SEN/43\*.0D0/, PREF/645\*.7D-5/, RMB/43\*.0D0/,  
\* HDRM/645\*.0D0/, TRCP/645\*.0D0/, DR2M/645\*.0D0/

C

C A. PARAMETERS AND CONTROL INDICES.

READ(5,2000) (RUNID(M),M=1,12)

2000 F0RMA T(12A6)

READ(5,2001) (HEADNG(M),M=1,126)

2001 F0RMA T(63A1)

READ(5,2002) LH,L0,L0H,LH02,LH202,LX,LHX,IX2,L02,LH20,LH2,LN2

2002 F0RMA T(14I2)

READ(5,450) LD,LE,LC

READ(5,450) LDELA,LDELB,LDEL C

450 F0RMA T(3I2)

LTABA=0

LTABB=0

LTABC=0

READ(5,500) N,NII,NEE,0MR

500 F0RMA T(3I2,D5.1)

NP1=N+1

NP2=N+2

NP3=N+3

READ(5,503) JH,JRAD,JM,JMR,JT,JTR,ITER,ITER1,JENTRN,JB0DYA,JB0DYB

503 F0RMA T(11I2)

JHM1=JH-1

JHM2=JH-2

JHM3=JH-3

JRADP1=JRAD+1

READ(5,502) (XI(J),J=JRADP1,JHM1)

READ(5,502) (XE(J),J=JRADP1,JHM1)

502 F0RMA T(5D7.3)

READ(5,103) (IPRFL(J),J=JRADP1,JHM1)

103 F0RMA T(5I1)

READ(5,101) TC0LD,PRESS,YWIDTH

101 F0RMA T(D6.0,D6.2,D5.1)

READ(5,504) (HREF(J),DCP(J),ECP(J),WT(J),J=1,JHM1)

SPD000350  
SPD000360  
SPD000370  
SPD000380  
SPD000390  
SPD000400  
SPD000410  
SPD000420  
SPD000430  
SPD000440  
SPD000450  
SPD000460  
SPD000470  
SPD000480  
SPD000490  
SPD000500  
SPD000510  
SPD000520  
SPD000530  
SPD000540  
SPD000550  
SPD000560  
SPD000570  
SPD000580  
SPD000590  
SPD000600  
SPD000610  
SPD000620  
SPD000630  
SPD000640  
SPD000650  
SPD000660  
SPD000670  
SPD000680  
SPD000690

```

504 F0RMA1(4D12.6,D10.2)
    READ(5,506) DXMIN,DXMAX,STEPS
506 F0RMA1(2D5.0,D6.0)
    FAC2=(DXMAX-DXMIN)/STEPS
    READ(5,508) LMAX,LPRINT
508 F0RMA1(2I3)
    READ(5,510) ALPHA
510 F0RMA1(F3.2)
    READ(5,512) KRAT
512 F0RMA1(I2)
    KRATE=KRAT
    READ(5,514) (FRQ(K),BETA(K),EACT(K),BFLAG(K),EFLAG(K),K-1,KRATE)
514 F0RMA1(D6.2,D7.2,D8.2,2L1)
    READ(5,515) (EQA(K),EQB(K),EQC(K),B0D1(K),B0D2(K),K-1,KRATE)
515 F0RMA1(3D11.5,2L1)
    D0 517 K-1,KRATE
    EQA(K)=EQA(K)*5.03297D2
    EQB(K)=EQB(K)*5.03297D2
517 EQC(K)=EQC(K)*5.03297D2
    READ(5,516) (ITEST(M),M-1,10)
516 F0RMA1(10I1)
    READ(5,512) INPUT
    READ(5,2002) (L0RDER(J),J-1,JHM2)
    READ(5,2003) (SPECIE(J),J-1,JHM2)
2003 F0RMA1(14L1)
    READ(5,2005) (REACT(K),K-1,KRATE)
2005 F0RMA1(30L1)
    READ(5,2005) (FFLAG(K),K-1,KRATE)
    READ(5,2005) (RFLAG(K),K-1,KRATE)
    READ(5,354) (LA(K),LB(K),LRA(K),LRB(K),K-1,KRATE)
354 F0RMA1(4I2)
    READ(5,2005) (FLAG(I),I-1,20)
    CALL SWITCH(KRATE)
    IF(.NOT.FLAG(1)) CALL DIFUSE(PRESS)
    JSTEP=JHM3
SPD000700
SPD000710
SPD000720
SPD000730
SPD000740
SPD000750
SPD000760
SPD000770
SPD000780
SPD000790
SPD000800
SPD000810
SPD000820
SPD000830
SPD000840
SPD000850
SPD000860
SPD000870
SPD000880
SPD000890
SPD000900
SPD000910
SPD000920
SPD000930
SPD000940
SPD000950
SPD000960
SPD000970
SPD000980
SPD000990
SPD01000
SPD01010
SPD01020
SPD01030
SPD01040

```



```

702 IF( INPUT.EQ.0) CALL INITL
C ----- TEST 2 -----
  IF( ITEST(2).EQ.1) GØ TØ 704
  GØ TØ 706
704 WRITE(6,604)
  WRITE(6,604)
  WRITE(6,653)
653 FØRMAT(1X,34H((F(J,I),J=1,JH),FS(JT,I),I=1,NP3))
  WRITE(6,606) ((F(J,I),J=1,JH),FS(JT,I),I=1,NP3)
606 FØRMAT(1X,7D14.5)
C -----
C
C TRANSPORT PRØPERTIES, DENSITY, AND TEMPERATURE.
706 IF(L.LT.LD) GØ TØ 400
  LTABA=LTABA+1
  IF(LTABA.EQ.LDELA) GØ TØ 401
  CALL TRANS(0)
  GØ TØ 402
401 LTABA=0
400 CALL TRANS(1)
402 CØNTINUE
  IF(L.GT.0) GØ TØ 31
  IF( INPUT.EQ.0) GØ TØ 30
  GØ TØ 31
30 PEI=1.D0
  CALL ZCALC
  PEI=YWIDTH/Y(NP2)
31 CØNTINUE
  PEID2=PEI*.5D0
  PEIØMD(2)=PEI#ØMD(2)
  DØ 701 I=3,NP1
  PEIØMD(I)=PEI#ØMD(I)
701 PEIØMD2(I)=PEID2*BØM(I)
C -----
SPD01400
-----SPD01410
SPD01420
SPD01430
SPD01440
SPD01450
SPD01460
SPD01470
SPD01480
SPD01490
-----SPD01500
SPD01510
SPD01520
SPD01530
SPD01540
SPD01550
SPD01560
SPD01570
SPD01580
SPD01590
SPD01600
SPD01610
SPD01620
SPD01630
SPD01640
SPD01650
SPD01660
SPD01670
SPD01680
SPD01690
SPD01700
SPD01710
SPD01720
SPD01730

```

```

C ----- TEST 3 -----SPD01740
IF( ITEST(3).EQ.1) GO TO 708 SPD01750
GO TO 710 SPD01760
708 WRITE(6,604) SPD01770
WRITE(6,604) SPD01780
WRITE(6,654) SPD01790
654 FORMAT(1X,28H((PREF(J,I),J=1,JH),I=1,NP3)) SPD01800
WRITE(6,608) ((PREF(J,I),J=1,JH),I=1,NP3) SPD01810
608 FORMAT(1X,6D14.5) SPD01820
WRITE(6,604) SPD01830
WRITE(6,655) SPD01840
655 FORMAT(1X,30H((HDRM(J,I),J=1,JHM1),I=1,NP3)) SPD01850
WRITE(6,610) ((HDRM(J,I),J=1,JHM1),I=1,NP3) SPD01860
610 FORMAT(1X,5D14.5) SPD01870
WRITE(6,604) SPD01880
WRITE(6,656) SPD01890
656 FORMAT(1X,30H((TRCP(J,I),J=1,JHM1),I=1,NP3)) SPD01900
WRITE(6,610) ((TRCP(J,I),J=1,JHM1),I=1,NP3) SPD01910
WRITE(6,604) SPD01920
WRITE(6,657) SPD01930
657 FORMAT(1X,30H((DR2M(J,I),J=1,JHM1),I=1,NP3)) SPD01940
WRITE(6,610) ((DR2M(J,I),J=1,JHM1),I=1,NP3) SPD01950
WRITE(6,604) SPD01960
WRITE(6,658) SPD01970
658 FORMAT(1X,29H(RH0(I),RU(I),RMB(I),I=1,NP3)) SPD01980
WRITE(6,612) (RH0(I),RU(I),RMB(I),I=1,NP3) SPD01990
612 FORMAT(1X,3D14.5) SPD02000
WRITE(6,604) SPD02010
WRITE(6,603) SPD02020
603 FORMAT(1X,27H(FS(JM,I),FS(JT,I),I=1,NP3)) SPD02030
WRITE(6,614) (FS(JM,I),FS(JT,I),I=1,NP3) SPD02040
614 FORMAT(1X,2D12.6) SPD02050
C ----- SPD02060
C ----- SPD02070

```

```

C      CALCULATION OF RATE CONSTANTS.
710 CALL RATCN
C
C      ----- TEST 4 -----
IF( ITEST(4).EQ.1 ) GO TO 712
GO TO 714
712 WRITE(6,604)
   WRITE(6,604)
   WRITE(6,659)
659 FORMAT(1X,42H((RATE(K,I),RATER(K,I),K-1,KRATE),I-2,NP2))
   WRITE(6,618) ((RATE(K,I),RATER(K,I),K-1,KRATE),I-2,NP2)
618 FORMAT(1X,8D14.5)
C
C      ----- TEST 5 -----
C      CALCULATION OF RMI AND RME. PEI IS ALSO CALCULATED FOR THE
C      COMING STEP.
714 IF(L.EQ.0) CALL SCHM2(JENTRN)
   CALL ENTRN(JENTRN,0)
   RPEI=1.D0/PEI
C
C      ----- TEST 5 -----
IF( ITEST(5).EQ.1 ) GO TO 716
GO TO 727
716 WRITE(6,604)
   WRITE(6,604)
   WRITE(6,660)
660 FORMAT(1X,26HVEL(JENTRN),RMI,RME,PEI,DX)
   WRITE(6,620) VEL(JENTRN),RMI,RME,PEI,DX
620 FORMAT(1X,5D14.5)
C
C      C. PRELIMINARIES FOR COEFFICIENTS BEFORE SUCCESSIVE CALCULATION.
C      QUANTITIES WHICH ARE THE SAME FOR ALL I AND J VALUES.
727 PX=PEI/DX
   PD4=.25D0*PX

```

```

SPD02080
SPD02090
SPD02100
-----SPD02110
SPD02120
SPD02130
SPD02140
SPD02150
SPD02160
SPD02170
SPD02180
SPD02190
-----SPD02200
SPD02210
SPD02220
SPD02230
SPD02240
SPD02250
SPD02260
-----SPD02270
SPD02280
SPD02290
SPD02300
SPD02310
SPD02320
SPD02330
SPD02340
SPD02350
-----SPD02360
SPD02370
SPD02380
SPD02390
SPD02400
SPD02410

```

G=RMI-RME  
 GD4=.25D0\*G  
 PG=PX\*G  
 PGD4=.25D0\*PG  
 RMID2=.5D0\*RMI

C  
 C QUANTITIES WHICH ARE THE SAME FOR ALL J VALUES.

----- MID SECTION -----

DØ 3004 I=3,NP1  
 PGØMP(I)=PGD4\*ØMD(I)  
 PGØMM(I)=PGD4\*ØMD(I-1)  
 3004 PBØM(I)=PX\*PBØM(I)

C  
 C QUANTITIES WHICH DEPEND ON I AND J BUT NOT ON F(J,I).

----- MID SECTION -----

DØ 3005 I=3,NP1  
 FLP=RMID2-GD4\*ØMP(I)  
 FLM=RMID2-GD4\*ØMP(I-1)  
 AFLP=DABS(FLP)  
 AFLM=DABS(FLM)  
 TFLP=FLP\*FLP  
 TFLM=FLM\*FLM  
 DØ 20 J=1,JSTØP  
 TP=PREF(J,I)/PEIØMD(I)  
 TM=PREF(J,I-1)/PEIØMD(I-1)  
 TP\*TP\*AFLP\*DABS(TP-AFLP)  
 TM\*TM\*AFLM\*DABS(TM-AFLM)  
 A(J,I)=TP-TFLP-PGØMP(I)  
 B(J,I)=TM-TFLM-PGØMM(I)  
 20 D(J,I)=A(J,I)\*B(J,I)\*PBØM(I)  
 IF(.NOT.FLAG(5)) GØ TØ 3005  
 TP=PREF(JH,I)/PEIØMD(I)  
 TM=PREF(JH,I-1)/PEIØMD(I-1)

SPD02420  
 SPD02430  
 SPD02440  
 SPD02450  
 SPD02460  
 SPD02470  
 SPD02480  
 SPD02490  
 SPD02500  
 SPD02510  
 SPD02520  
 SPD02530  
 SPD02540  
 SPD02550  
 SPD02560  
 SPD02570  
 SPD02580  
 SPD02590  
 SPD02600  
 SPD02610  
 SPD02620  
 SPD02630  
 SPD02640  
 SPD02650  
 SPD02660  
 SPD02670  
 SPD02680  
 SPD02690  
 SPD02700  
 SPD02710  
 SPD02720  
 SPD02730  
 SPD02740  
 SPD02750  
 SPD02760





```

D6 352 K=1, KHM3
SUM=SUM*F(K, I)*WT(K)
SUM=SUM*F(JHM1, I)*WT(JHM1)
F(JHM2, I)=(1.D0-SUM)/WT(JHM2)
351 CONTINUE
G6 T6 300
353 D6 340 I=2, NP2
SUM=.0D0
D6 350 K=1, KHM2
SUM=SUM*F(K, I)*WT(K)
F(JHM1, I)=(1.D0-SUM)/WT(JHM1)
340 CONTINUE
300 CONTINUE
C
C ----- TEST 9 -----
IF( ITEST(9).EQ.1) G6 T6 731
G6 T6 732
731 WRITE(6,604)
WRITE(6,604)
WRITE(6,665)
665 FORMAT(1X,15HSU(J, I),SD(J, I))
WRITE(6,618) ((SU(J, I),J=1, JHM2),(SD(J, I),J=1, JHM2), I=2, NP2)
WRITE(6,604)
WRITE(6,666)
666 FORMAT(1X,27HA, B, C, AND D COEFFICIENTS)
WRITE(6,624) ((A(J, I),J=1, JHM2), I=3, NP1)
WRITE(6,604)
WRITE(6,624) ((B(J, I),J=1, JHM2), I=3, NP1)
WRITE(6,604)
WRITE(6,624) ((C(J, I),J=1, JHM2), I=3, NP1)
WRITE(6,604)
WRITE(6,624) ((D(J, I),J=1, JHM2), I=3, NP1)
C
C -----
SPD0 3480
SPD0 3490
SPD0 3500
SPD0 3510
SPD0 3520
SPD0 3530
SPD0 3540
SPD0 3550
SPD0 3560
SPD0 3570
SPD0 3580
SPD0 3590
SPD0 3600
SPD0 3610
SPD0 3620
SPD0 3630
SPD0 3640
SPD0 3650
SPD0 3660
SPD0 3670
SPD0 3680
SPD0 3690
SPD0 3700
SPD0 3710
SPD0 3720
SPD0 3730
SPD0 3740
SPD0 3750
SPD0 3760
SPD0 3770
SPD0 3780
SPD0 3790
SPD0 3800
SPD0 3810

```

```

C SOLVE FOR DOWNSTREAM ENTHALPIES.
C COEFFICIENTS FOR ENTHALPY EQUATION.
732 IF(LC.EQ.0) GO TO 599
IF(L.LT.LC) GO TO 406
LTABC=LTABC+1
IF(LTABC.EQ.LDELCL) GO TO 407
GO TO 408
407 LTABC=0
406 CALL SENTP
408 CONTINUE
DE 100 I=3, NP1
100 C(JH, I) = PD4 * (BEMT3(I) * F(JH, I+1) * OMD(I) + F(JH, I-1) * OMD(I-1))
SPD03820
SPD03830
SPD03840
SPD03850
SPD03860
SPD03870
SPD03880
SPD03890
SPD03900
SPD03910
SPD03920
SPD03930
SPD03940
SPD03950
SPD03960
SPD03970
SPD03980
SPD03990
SPD04000
SPD04010
SPD04020
SPD04030
SPD04040
SPD04050
C -----TEST 10 -----
C IF(ITEST(10).EQ.1) GO TO 733
GO TO 734
733 WRITE(6,604)
WRITE(6,604)
WRITE(6,667)
667 FORMAT(1X, 27HC COEFFICIENTS FOR ENTHALPY)
WRITE(6,626) (C(JH, I), I=3, NP1)
626 FORMAT(1X, D14.5)
C -----
C COEFFICIENTS FOR THE ENTHALPY HAVE NOW BEEN CALCULATED.
734 CALL CALC(JH)
C
999 L=L+1
LCOUNT=LCOUNT+1
IF(LCOUNT.EQ.LPRINT) GO TO 735
GO TO 661
735 DE 800 J=1, JHM2
IF(.NOT.SPECIE(J)) GO TO 800
CALL ENTRN(J, 1)
SPD04060
SPD04070
SPD04080
SPD04090
SPD04100
SPD04110
SPD04120
SPD04130
SPD04140
SPD04150
SPD04160

```

```

800 CONTINUE
    CALL ZCALC
    RESULTS PRINTED OUT.
    CALL OUTPUT(L, HEADNG, RUNID, X, INPUT, LMAX)
    LCOUNT=0
661 IF(L.EQ.LMAX) WRITE(46) ((F(J, I), J=1, JH), I=1, NP3), PEI
    Q=L
    DX=DXMIN*FAC2*Q
    X=X+DX
    IF(L.EQ.LMAX) GO TO 1000
    GE TO 706
1000 CONTINUE
    END

```

```

SPD04170
SPD04180
SPD04190
SPD04200
SPD04210
SPD04220
SPD04230
SPD04240
SPD04250
SPD04260
SPD04270
SPD04280
SPD04290

```

```

DELT, SIB ABCDE#FLAME(2). SWITCH,, 141371132312
SUBROUTINE SWITCH(KRATE)
LOGICAL G1, G2, G3
COMMON/JVALS/JH, JHM1, JHM2, JRAD, JRADP1, JM, JMR, JT, JTR, JBODYA, JBODYB
COMMON/REACT/LA(30), LB(30), LRA(30), LRB(30)
COMMON/GSWIIC /G1(30,15), G2(30,15), G3(30,15)
DATA G1/450*.FALSE./, G2/450*.FALSE./, G3/450*.FALSE./
D0 15 J-1, JHM2
D0 15 K-1, KRATE
IF((J.EQ.LA(K)).AND.(J.EQ.LB(K))) G0 T0 10
IF(J.EQ.LA(K)) G0 T0 11
IF(J.EQ.LB(K)) G0 T0 11
G0 T0 12
10 G3(K,J)=.TRUE.
11 G1(K,J)=.TRUE.
12 CONTINUE
IF((J.EQ.LRA(K)).AND.(J.EQ.LRB(K))) G0 T0 13
IF(J.EQ.LRA(K)) G0 T0 14
IF(J.EQ.LRB(K)) G0 T0 14
G0 T0 15
13 G3(K,J)=.TRUE.
14 G1(K,J)=.TRUE.
15 G2(K,J)=.TRUE.
CONTINUE
RETURN
END

```

```

SWH00010
SWH00020
SWH00030
SWH00040
SWH00050
SWH00060
SWH00070
SWH00080
SWH00090
SWH00100
SWH00110
SWH00120
SWH00130
SWH00140
SWH00150
SWH00160
SWH00170
SWH00180
SWH00190
SWH00200
SWH00210
SWH00220
SWH00230
SWH00240
SWH00250

```

ØELT, SIB ABCDE\*FLAME(2). DIFUSE,.. 141343132312

SUBROUTINE DIFUSE(PRESS)

REAL M(12)

DOUBLE PRECISION TDIV, DA, DB, PRESS

LØGICAL FLAG

COMMON/JVALS/JH, JHM1, JHM2, JRAD, JRADP1, JM, JMR, JT, JTR, JBØDYA, JBØDYB

COMMON/TRANS4/DA(14, 14), DB(14, 14), TDIV(14, 14)

COMMON/FLAGS/FLAG(20)

DIMENSION EK(12), S(12)

DATA TDIV/196\*.0D0/, DA/196\*.0D0/, DB/196\*.0D0/

DATA M/1.008, 16., 17.008, 33.008, 34.016, 79.916, 80.92, 159.83

\* , 32., 18.016, 2.016, 28.02/, EK/51.6, 555., 682., 298., 298., 342.

\* , 342., 403.5, 88., 809., 38., 79.8 /, S/2.1, 2.3, 2.47, 4.2, 4.2

\* , 3.71, 3.71, 4.69, 3.541, 2.640, 2.915, 3.749/

DØ 43 I=1, JHM2

DØ 43 J=JRADP1, JHM1

IF(I.GE.J) GE TØ 43

FMASS=(M(I)\*M(J))/(M(I)\*M(J))

FMASS=SQRT(FMASS)

EKIJ=SQRT(EK(I)\*EK(J))

EKA=EKIJ\*\* .44

EKB=EKIJ\*\* .17

S2=.5\*(S(I)\*S(J))

S2=S2\*S2

TCUT=3.\*EKIJ

TDIV(I, J)=TCUT

D=1.26E-7\*FMASS/(S2\*EKA)

DA(I, J)=D

IF(FLAG(3)) DA(I, J)=DA(I, J)/PRESS

D=1.66E-7\*FMASS/(S2\*EKB)

DB(I, J)=D

IF(FLAG(3)) DB(I, J)=DB(I, J)/PRESS

43 CONTINUE

RETURN

END

DIF00010  
DIF00020  
DIF00030  
DIF00040  
DIF00050  
DIF00060  
DIF00070  
DIF00080  
DIF00090  
DIF00100  
DIF00110  
DIF00120  
DIF00130  
DIF00140  
DIF00150  
DIF00160  
DIF00170  
DIF00180  
DIF00190  
DIF00200  
DIF00210  
DIF00220  
DIF00230  
DIF00240  
DIF00250  
DIF00260  
DIF00270  
DIF00280  
DIF00290  
DIF00300  
DIF00310  
DIF00320  
DIF00330  
DIF00340

DEL1, S18 ABCDE\*FLAME(2). OMEGA.,.,141377132312

OMG000010  
OMG000020  
OMG000030  
OMG000040  
OMG000050  
OMG000060  
OMG000070  
OMG000080  
OMG000090  
OMG000100  
OMG000110  
OMG000120  
OMG000130  
OMG000140  
OMG000150  
OMG000160  
OMG000170  
OMG000180  
OMG000190  
OMG000200  
OMG000210  
OMG000220  
OMG000230  
OMG000240  
OMG000250  
OMG000260  
OMG000270  
OMG000280  
OMG000290  
OMG000300

```

SUBROUTINE OMEGA
IMPLICIT REAL*8 (A-H, O-Z)
COMMON/IVALS/N, NP1, NP2, NP3
COMMON/OMEGA1/NII, NEE, OMR
COMMON/OMEGA2/OM(43), OMD(43), OMI, OME, OMD(43)
COMMON/OMEGA3/BOM(43), BOMT3(43), OMP(43), PEIOMD(43), PEIOM2(43)
COMMON/YCALC/PEI, Y(43),
      YD(43), YAV(43), RPEI
NIIPI=NII+1
NEEM1=NEE-1
ROMR=1. DO/OMR
SUM=1. DO
OMD(2)=1. DO
DO 10 I=3, NII
OMD(I)=ROMR*OMD(I-1)
10 SUM=SUM+OMD(I)
DO 11 I=NIIPI, NEEM1
OMD(I)=OMD(NII)
11 SUM=SUM+OMD(I)
DO 12 I=NEE, NP1
OMD(I)=OMR*OMD(I-1)
12 SUM=SUM+OMD(I)
SUM=1. DO/SUM
DO 13 I=2, NP1
OMD(I)=OMD(I)*SUM
OM(1)=. ODO
OM(2)=. ODO
DO 14 I=3, NP2
OM(I)=OM(I-1)*OMD(I-1)
OM(NP3)=1. DO
OMI=.5DO*OM(3)

```

```
ØME=.5DØ*(1.DØ-ØX(NP1))
DØ 4Ø I=2,NP1
ØMD(I)=ØM(I+1)-ØM(I)
RØMD(I)=1.DØ/ØMD(I)
4Ø ØMP(I)=ØM(I)*ØM(I+1)
DØ 5Ø I=3,NP1
BØM(I)=ØM(I+1)-ØM(I-1)
5Ø BØMT3(I)=3.DØ*BØM(I)
RETURN
END
```

```
ØMGØØ31Ø
ØMGØØ32Ø
ØMGØØ33Ø
ØMGØØ34Ø
ØMGØØ35Ø
ØMGØØ36Ø
ØMGØØ37Ø
ØMGØØ38Ø
ØMGØØ39Ø
ØMGØØ40Ø
```

DEL1,SIH ABCDE\*FLAME(2).INITL/1,..,107650032113

SUBROUTINE INITL

IMPLICIT REAL\*8 (A-H,θ-Z)

COMMON/IVALS/N,NP1,NP2,NP3

COMMON/OMEGA2/OM(43),θMD(43),θMI,θME,θMD(43)

COMMON/YCALC/PEI,Y(43),

YD(43),YAV(43),RPEI

COMMON/TRANS1/HREF(15),DCP(15),ECP(15),FCP(15),WT(15),ITER,ITER1

COMMON/JVALS/JH,JHM1,JHM2,JRAD,JRADP1,JM,JMR,JT,JTR,JBODYA,JBODYB

COMMON/INIT/XI(15),XE(15),IPRFL(15),ICOLD,PRESS,YWIDTH

COMMON/VAR1/F(15,43),FS(15,43)

DIMENSION ET(15,43),Z(15),XN2(43)

FNC(U)=U\*U\*(10.D0-15.D0\*U+6.D0\*U\*U)

INITIAL PROFILES FOR NON-RADICALS.

XI(I) AND XE(I) ARE HOT AND COLD SIDE MOLE FRACTIONS OF THE NON-

RADICAL SPECIES. WHEN PRFL(J)=0, WE HAVE A DECAY PROFILE, AND

WHEN IPRFL(J)=1, A FORMATION PROFILE.

C

THIS SECTION CALCULATES MOLE FRACTION PROFILES FOR ALL NON-RADICAL SPECIES EXCEPT ONE.

C

DO 100 J=JRADP1,JHM1

IF(IPRFL(J).EQ.1) GO TO 120

DIFF=XE(J)-XI(J)

DO 130 I=2,NP2

U=θM(I)

130 F(J,I)=XI(J)+DIFF\*FNC(U)

GO TO 100

120 DIFF=XI(J)-XE(J)

DO 140 I=2,NP2

U=1.D0-θM(I)

140 F(J,I)=XE(J)+DIFF\*FNC(U)

100 CONTINUE

C

AT THIS POINT, THE F(J,I) ARE MOLE FRACTIONS.

C

THIS SECTION CALCULATES THE MOLE FRACTION PROFILE FOR THE BUFFER

C

ITL00010  
ITL00020  
ITL00030  
ITL00040  
ITL00050  
ITL00060  
ITL00070  
ITL00080  
ITL00090  
ITL00100  
ITL00110  
ITL00120  
ITL00130  
ITL00140  
ITL00150  
ITL00160  
ITL00170  
ITL00180  
ITL00190  
ITL00200  
ITL00210  
ITL00220  
ITL00230  
ITL00240  
ITL00250  
ITL00260  
ITL00270  
ITL00280  
ITL00290  
ITL00300  
ITL00310  
ITL00320  
ITL00330  
ITL00340  
ITL00350

```

C SPECIES.
D0 150 I=2,NP2
XX=0.00
D0 160 J=JRADP1,JHM1
160 XX=XX*F(J,I)
150 XN2(I)=1.00-XX
C HERE, THE MOLE FRACTION PROFILES ARE CONVERTED TO MOL/KG PROFILES.
D0 170 I=2,NP2
AVM=.000
D0 180 J=JRADP1,JHM1
180 AVM=AVM*F(J,I)*WT(J)
RAVM=1.00/AVM
D0 190 J=JRADP1,JHM1
190 F(J,I)=F(J,I)*RAVM
170 CONTINUE
C
C INITIAL PROFILES FOR RADICALS AND TRACE SPECIES.
CALL INPREF(F,JRAD,N)
C
C GIVEN THE TEMPERATURE ON THE COLD SIDE, WE CALCULATE THE COLD SIDE
ENTHALPY.
T=TCOLD
T2=T*T
D0 20 J=1,JHM1
20 ET(J,NP2)=HREF(J)*T*(DCP(J)+ECP(J))*T*.500*FCP(J)*T2*.333333D0)
F(JH,NP2)=.000
D0 30 J=JRADP1,JHM1
30 F(JH,NP2)=F(JH,NP2)*WT(J)*F(J,NP2)*ET(J,NP2)
C
C THE ENTHALPY ON THE HOT SIDE IS SET EQUAL TO THAT ON THE COLD SIDE
F(JH,2)=F(JH,NP2)
NOW, THE HOT SIDE TEMPERATURE IS CALCULATED FROM THE HOT SIDE
ENTHALPY AND COMPOSITION. NOTE THAT Y(J) IS THE MASS FRACTION
ON THE HOT SIDE.
A=.000
ITL00360
ITL00370
ITL00380
ITL00390
ITL00400
ITL00410
ITL00420
ITL00430
ITL00440
ITL00450
ITL00460
ITL00470
ITL00480
ITL00490
ITL00500
ITL00510
ITL00520
ITL00530
ITL00540
ITL00550
ITL00560
ITL00570
ITL00580
ITL00590
ITL00600
ITL00610
ITL00620
ITL00630
ITL00640
ITL00650
ITL00660
ITL00670
ITL00680
ITL00690
ITL00700

```

```

      H=.0D0
      C2=.0D0
      D3=.0D0
      D6 40 J=JRADP1,JHM1
      Z(J)=F(J,2)*WT(J)
      A=A+BREF(J)*Z(J)
      B=B+DCP(J)*Z(J)
      C2=C2+ECF(J)*Z(J)
      40 D3=D3+FCP(J)*Z(J)
      C=C2*.5D0
      D=D3*.333333D0
      T=(F(JH,2)-A)/B
      ICOUNT=0
      23 FAC=F(JH,2)-(A+T*(B+T*(C+D*T)))
      FAC=FAC/(B+T*(C+D3*T))
      T=T+FAC
      ICOUNT=ICOUNT+1
      IF(ICOUNT.LI.ITER) GO TO 23
      THET=T
      C AN S-SHAPED TEMPERATURE PROFILE IS CALCULATED, AND FROM THIS, THE
      C ENTHALPY PROFILE.
      D6 50 I=2,NP2
      U=1.D0-EM(I)
      50 FS(JT,I)=(THET-TCOLD)*FNC(U)+TCOLD
      C NOTE THAT CONSTANT ENTHALPY IS USED AS A STARTING PROFILE.
      D6 70 I=3,NP2
      70 F(JH,I)=F(JH,2)
      C
      RETURN
      END
ITL00710
ITL00720
ITL00730
ITL00740
ITL00750
ITL00760
ITL00770
ITL00780
ITL00790
ITL00800
ITL00810
ITL00820
ITL00830
ITL00840
ITL00850
ITL00860
ITL00870
ITL00880
ITL00890
ITL00900
ITL00910
ITL00920
ITL00930
ITL00940
ITL00950
ITL00960
ITL00970
ITL00980
ITL00990
ITL01000

```

```

@ELT,SIH ABCDE*FLAME(2).INPRFL,..103730063613
SUBROUTINE INPRFL(F,JRAD,N)
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION F(15,43)
NP2=N+2
DO 10 I=2,NP2
DO 20 J=1,4
20 F(J,I)=.1D-4
DO 21 J=5,8
21 F(J,I)=1.D-50
10 CONTINUE
DO 22 J=1,8
22 F(J,NP2)=1.D-50
RETURN
END

```

```

INP00010
INP00020
INP00030
INP00040
INP00050
INP00060
INP00070
INP00080
INP00090
INP00100
INP00110
INP00120
INP00130
INP00140

```

```

OELT, SIB ABCDE*FLAME(2), TRANS/6.,.,111236132413
SUBROUTINE TRANS(ICALC)
IMPLICIT REAL*8 (A-H, O-Z)
LOGICAL FLAG, TAGA, TAGB
COMMON/IVALS/N, NP1, NP2, NP3
COMMON/JVALS/JH, JHM1, JHM2, JRAD, JRADP1, JM, JMR, JT, JTR, JBODYA, JBODYB
COMMON/VARI/F(15,43), FS(15,43)
COMMON/TRANS1/HREF(15), DCP(15), ECP(15), FCP(15), WT(15), ITER, ITER1
COMMON/TRANS2/RMB(43), PREF(15,43), HDRM(15,43), TRCP(15,43)
*, DR2M(15,43)
COMMON/TRANS3/RH0(43), RU(43), RH2(43)
COMMON/INIT/XI(15), XE(15), IPRFL(15), TC0LD, PRESS, YWIDTH
COMMON/TRANS4/DA(14,14), DB(14,14), TDIV(14,14)
COMMON/FLAGS/FLAG(20)
COMMON/CMIDX/LH, L0, L0H, LH02, LH202, LX, LHX, LX2, L02, LH2, LN2, LN2
DIMENSION DD(14,14), C0N(14), ENT(15,43), SUM(14)
DATA AH2/1.D0/, A02/1.D0/, AH20/12.D0/
*, BH2/2.28D0/, B02/.8D0/, BH20/14.8D0/
C AVERAGE MOLECULAR WEIGHT AND MASS FRACTIONS AT GRID POINTS.
DE 10 I=2, NP2
WTAV=.0D0
A=.0D0
B=.0D0
C2=.0D0
D3=.0D0
DE 14 J=1, JHM1
Y=WT(J)*F(J, I)
WTAV=WTAV+F(J, I)
A=A+HREF(J)*Y
B=B+DCP(J)*Y
C2=C2+ECP(J)*Y
14 D3=D3+FCP(J)*Y
C=C2*.5D0
D=D3*.333333D0
FS(JMR, I)=WTAV
TRN00010
TRN00020
TRN00030
TRN00040
TRN00050
TRN00060
TRN00070
TRN00080
TRN00090
TRN00100
TRN00110
TRN00120
TRN00130
TRN00140
TRN00150
TRN00160
TRN00170
TRN00180
TRN00190
TRN00200
TRN00210
TRN00220
TRN00230
TRN00240
TRN00250
TRN00260
TRN00270
TRN00280
TRN00290
TRN00300
TRN00310
TRN00320
TRN00330
TRN00340

```

```

C
C
FS(JM,I)=1.D0/WIAV
C
C
TEMPERATURE AT GRID POINTS.
T=(F(JH,I)-A)/B.
ICOUNT=0
23 FAC=F(JH,I)-(A*T*(B*T*(C*D*T)))
FAC=FAC/(B*T*(C2*D3*T))
T=T+FAC
ICOUNT=ICOUNT+1
IF(ICOUNT.LT.ITER) GO TO 23
FS(JT,I)=T
FS(JTR,I)=1.D0/T
FS(JB0DYA,I)=AH2*F(LH2,I)+A02*F(L02,I)+AH20*F(LH20,I)+F(LN2,I)
FS(JB0DYB,I)=BH2*F(LH2,I)+B02*F(L02,I)+BH20*F(LH20,I)+F(LN2,I)
IF(ICALC.EQ.0) GO TO 10
C
C
DENSITY AT GRID POINTS.
RH0(I)=1.21867D4*FS(JM,I)*FS(JTR,I)
IF(FLAG(3)) RH0(I)=RH0(I)*PRESS
RH2(I)=RH0(I)*RH0(I)
C
C
EFFECTIVE DIFFUSION COEFFICIENTS AT GRID POINTS.
C
C
THIS SECTION CALCULATES PREF AT GRID POINTS FOR 03 FLAME.
LEWIS NUMBER =.94.
IF(.NOT.FLAG(7)) GO TO 211
CPMIX=B*T*(C2*D3*T)
PREF(JH,I)=2.2026D-3*DSQRT(T)
PREF(JH,I)=PREF(JH,I)*RH0(I)/CPMIX
XLEWIS=.94D0
FAC=XLEWIS*PREF(JH,I)
D0 212 J=1,JHM1
212 PREF(J,I)=FAC
GO TO 57
211 CONTINUE

```

```

TRN00350
TRN00360
TRN00370
TRN00380
TRN00390
TRN00400
TRN00410
TRN00420
TRN00430
TRN00440
TRN00450
TRN00460
TRN00470
TRN00480
TRN00490
TRN00500
TRN00510
TRN00520
TRN00530
TRN00540
TRN00550
TRN00560
TRN00570
TRN00580
TRN00590
TRN00600
TRN00610
TRN00620
TRN00630
TRN00640
TRN00650
TRN00660
TRN00670
TRN00680
TRN00690

```

```

C
C THIS SECTION CALCULATES PREF AT GRID POINTS FOR H2-BR2 FLAME.
C LEVACHEV AND KAGANOVA TRANSPORT PROPERTIES ARE USED.
C IF(.NOT.FLAG(8)) GO TO 214
  FAC=(I/TCOLD)**1.67
  DO 213 J=3,5
    213 PREF(J,I)=.101D-4*FAC
  PREF(1,I)=.155D-4*FAC
  PREF(2,I)=1.05D-4*FAC
  DO 215 J=1,5
    215 PREF(J,I)=PREF(J,I)*RH2(I)
  PREF(JH,I)=3.34944D-2*((T/TCOLD)**.67)
  CPMIX=B+T*(C2+D3*T)
  PREF(JH,I)=PREF(JH,I)*RH0(I)/CPMIX
  GO TO 57
214 CONTINUE
C
C IF FLAG(1)=T, CONSTANT TRANSPORT PROPERTIES ARE USED.
C (VALUES OF PREF APPEAR IN A DATA STATEMENT IN SPALD.)
C IF(FLAG(1)) GO TO 10
  R2DM=RH2(I)*FS(JMR,I)
C
C COMPOSITION DEPENDENT TRANSPORT PROPERTIES ARE CALCULATED IN THE
C SECTION THAT FOLLOWS.
C
C BINARY DIFFUSION COEFFICIENTS CALCULATED AT TEMPERATURE T.
  TAGA=.FALSE.
  TAGB=.FALSE.
  DO 37 KI=1,JHM2
  DO 37 KJ=JRADP1,JHM1
  IF(KI.GE.KJ) GO TO 37
  IF((T.LT.TDIV(KI,KJ)).AND.(.NOT.TAGA)) GO TO 32
  GO TO 33
32 TA=T**1.94
  TAGA=.TRUE.

```

```

TRN00700
TRN00710
TRN00720
TRN00730
TRN00740
TRN00750
TRN00760
TRN00770
TRN00780
TRN00790
TRN00800
TRN00810
TRN00820
TRN00830
TRN00840
TRN00850
TRN00860
TRN00870
TRN00880
TRN00890
TRN00900
TRN00910
TRN00920
TRN00930
TRN00940
TRN00950
TRN00960
TRN00970
TRN00980
TRN00990
TRN01000
TRN01010
TRN01020
TRN01030
TRN01040
TRN01050

```

```

33 IF(T.LT.TDIV(KI,KJ)) G0 T0 34
   G0 T0 35
34 DD(KI,KJ)=DA(KI,KJ)*TA
   G0 T0 30
35 IF(TAGB) G0 T0 36
   TB=T*1.67
   IAGB=.TRUE.
36 DD(KI,KJ)=DB(KI,KJ)*TB
30 CONTINUE
37 CONTINUE
C
C THE ABOVE DO-LOOP CALCULATES THE BINARY DIFFUSION COEFFICIENTS AT TRN01170
C THE TEMPERATURE T. TAGA AND TAGB ARE USED SO THAT THE EXPONENTIA-TRN01180
C TION STEPS T*1.94 AND T*1.67 ARE CALCULATED ONLY ONCE AT EACH T. TRN01190
C TRN01200
C IN THE NEXT SECTION, PREF(J,I) IS CALCULATED FOR THE FULL MODEL.
  D0 38 J=1,JHM1
  PREF(J,I)=.0D0
  D0 39 K=JRADP1,JHM1
  IF(J.EQ.K) G0 T0 39
  IF(J.GT.K) DD(J,K)=DD(K,J)
  PREF(J,I)=PREF(J,I)+F(K,I)/DD(J,K)
39 CONTINUE
  PREF(J,I)=R2DM/PREF(J,I)
  IF(J.GT.JRAD ) PREF(J,I)=(1.D0-F(J,I))*FS(JM,I)*PREF(J,I)
38 CONTINUE
C
C THIS SECTION CALCULATES THE THERMAL CONDUCTIVITY FOR THE MIXTURE
C ACCORDING TO LINDSAY AND BRØMLEY, IND. AND ENG. CHEM. VOL. 42,
C P. 1508. THE A(I,J) VALUES HAVE BEEN CALCULATED SEPARATELY FOR
C THE FOUR MAJOR SPECIES, O2,H2O,H2,N2.
C
C THE THERMAL CONDUCTIVITIES FOR THE FOUR PURE SPECIES ARE
  CALCULATED FIRST.
  T2=1*1

```

```

TRN01060
TRN01070
TRN01080
TRN01090
TRN01100
TRN01110
TRN01120
TRN01130
TRN01140
TRN01150
TRN01160
TRN01170
TRN01180
TRN01190
TRN01200
TRN01210
TRN01220
TRN01230
TRN01240
TRN01250
TRN01260
TRN01270
TRN01280
TRN01290
TRN01300
TRN01310
TRN01320
TRN01330
TRN01340
TRN01350
TRN01360
TRN01370
TRN01380
TRN01390
TRN01400

```

T3=I\*T2  
 CEN(L02)=-1.455D0\*1.73669D-2\*T-1.87901D-6\*T2  
 CEN(LH20)=-2.51153D0\*1.7143D-2\*T+1.15567D-5\*T2-3.15212D-9\*T3  
 CEN(LH2)=-6.8173D0\*1.33568D0\*T-3.89395D-5\*T2+9.66224D-9\*T3  
 CEN(LN2)=-9.85661D-2\*2.2693D-2\*T-9.4717D-6\*T2+2.25781D-9\*T3  
 D0 59 J=JRADP1, JHM1  
 59 CEN(J)=4.1868D-3\*CEN(J)

C  
 C NEXT, THE A(I, J) COEFFICIENTS ARE CALCULATED.  
 C THE DD ARRAY IS USED FOR THESE QUANTITIES.  
 C DD(L02, LH20)=-135.4D0\*FS(JTR, I)\*.7654D0  
 DD(L02, LH2)=-.576D0  
 DD(L02, LN2)=-1.03D0  
 DD(LH20, L02)=-.232322D0\*1.2517D-3\*T-6.94629D-7\*T2+1.39243D-10\*T3  
 DD(LH20, LH2)=-.160533D0\*5.95658D-4\*T-3.17638D-7\*T2+6.16877D-11\*T3  
 DD(LH20, LN2)=-.280067D0\*1.08447D-3\*T-4.78746D-7\*T2+7.53136D-11\*T3  
 DD(LH2, L02)=-1.98336D0\*7.98D-5\*T  
 DD(LH2, LH20)=-309.29D0\*FS(JTR, I)\*1.4557D0  
 DD(LH2, LN2)=-2.09875D0\*1.105D-4\*T  
 DD(LN2, L02)=-.96429D0  
 DD(LN2, LH20)=-138.68D0\*FS(JTR, I)\*.7244D0  
 DD(LN2, LH2)=-.5636D0

C  
 C NEW, PREF(JH, I), WHICH IS THE THERMAL TRANSPORT PARAMETER FOR THE  
 C MIXTURE, IS CALCULATED.  
 C FIRST, HOWEVER, THE MIXTURE HEAT CAPACITY AT THE GRID POINTS IS  
 C CALCULATED. ALSO CALCULATED, IS THE SPECIES MOLAR ENTHALPY WHICH  
 C WILL BE NEEDED FOR THE ENTHALPY SOURCE TERM. THIS LATTER  
 C CALCULATION WILL BE DONE, HOWEVER, ONLY IF THE ENTHALPY SOURCE  
 C TERM IS TO BE CALCULATED. THIS OPTION IS TRIGGERED BY FLAG(5)-T.  
 C  
 C MIXTURE HEAT CAPACITY.  
 C CPMIX=B\*T\*(C2+D3\*I)  
 C MIXTURE THERMAL CONDUCTIVITY AT GRID POINTS.  
 C PREF(JH, I)=-.0D0

TRN01410  
 TRN01420  
 TRN01430  
 TRN01440  
 TRN01450  
 TRN01460  
 TRN01470  
 TRN01480  
 TRN01490  
 TRN01500  
 TRN01510  
 TRN01520  
 TRN01530  
 TRN01540  
 TRN01550  
 TRN01560  
 TRN01570  
 TRN01580  
 TRN01590  
 TRN01600  
 TRN01610  
 TRN01620  
 TRN01630  
 TRN01640  
 TRN01650  
 TRN01660  
 TRN01670  
 TRN01680  
 TRN01690  
 TRN01700  
 TRN01710  
 TRN01720  
 TRN01730  
 TRN01740  
 TRN01750

TRN01760  
 TRN01770  
 TRN01780  
 TRN01790  
 TRN01800  
 TRN01810  
 TRN01820  
 TRN01830  
 TRN01840  
 TRN01850  
 TRN01860  
 TRN01870  
 TRN01880  
 TRN01890  
 TRN01900  
 TRN01910  
 TRN01920  
 TRN01930  
 TRN01940  
 TRN01950  
 TRN01960  
 TRN01970  
 TRN01980  
 TRN01990  
 TRN02000  
 TRN02010  
 TRN02020  
 TRN02030  
 TRN02040  
 TRN02050  
 TRN02060  
 TRN02070  
 TRN02080  
 TRN02090

```

DE 50 KI=JRADP1, JHM1
SUM(KI)=-.0D0
DE 52 KJ=JRADP1, JHM1
IF(KI.EQ.KJ) GO TO 52
52 SUM(KI)=SUM(KI)*DD(KI, KJ)*F(KJ, I)
SUM(KI)=1.D0*SUM(KI)/F(KI, I)
50 PREF(JH, I)=PREF(JH, I)*CEN(KI)/SUM(KI)
PREF(JH, I)=PREF(JH, I)*RH0(I)/CPMIX
IF(FLAG(5)) GO TO 57
GO TO 10
C MELAR ENTHALPY AT GRID POINTS.
57 DE 70 J=1, JHM1
FAC=HREF(J)*T*(DCP(J)*ECP(J)*T*.5D0*FCP(J)*T2*.333333D0)
70 ENT(J, I)=FAC*WT(J)
10 CONTINUE
C
IF(ICALC.EQ.0) GO TO 300
VALUES AT CENTREL VOLUME BOUNDARIES.
DE 200 I=2, NP1
RMB(I)=(FS(JMR, I)*FS(JMR, I+1))*5D0
200 PREF(JH, I)=(PREF(JH, I)*PREF(JH, I+1))*5D0
DE 210 I=2, NP1
DE 210 J=1, JHM1
PREF(J, I)=(PREF(J, I)*PREF(J, I+1))*5D0
C THE NEXT QUANTITIES APPEAR ONLY IN THE ENTHALPY SOURCE TERM.
IF(FLAG(5)) GO TO 58
GO TO 210
58 ENTLPY=.5*(ENT(J, I)*ENT(J, I+1))
HDRM(J, I)=ENTLPY*PREF(J, I)*RMB(I)
TRCP(J, I)=ENTLPY*PREF(JH, I)
DR2M(J, I)=PREF(J, I)*RMB(I)
210 CONTINUE
300 RETURN
END
  
```

QELT,SIH ABCDE\*FLAME(2).RATCN/3.,.132060133512

SUBROUTINE RATCN

IMPLICIT REAL\*8 (A-H,θ-Z)

LOGICAL BθD1,BθD2,REACT,BFLAG,EFLAG,FFLAG,RFLAG

COMMON/IVALS/N,NP1,NP2,NP3

COMMON/JVALS/JH,JHM1,JHM2,JRAD,JRADP1,JM,JMR,JT,JTR,JBθDYA,JBθDYB

COMMON/VAR1/F(15,43),FS(15,43)

COMMON/RTPARM/FRQ(30),BETA(30),EACT(30)

\*,BθD1(30),BθD2(30),REACT(30)

COMMON/RATC/RATE(30,43),RATER(30,43),KRATE,EOA(30),EQB(30),EQC(30),EQD(30)

COMMON/FLAGR/BFLAG(30),EFLAG(30),FFLAG(30),RFLAG(30)

DIMENSION ECθN(43),RGI(43)

Dθ 68 I=2,NP2

68 RGI(I)=1.21867D4\*FS(JTR,I)

Dθ 40 K=1,KRATE

IF(.NOT.FFLAG(K)) Gθ Tθ 50

IF(.NOT.BFLAG(K)).AND.(.NOT.EFLAG(K))) Gθ Tθ 10

Gθ Tθ 12

10 Dθ 11 I=2,NP2

11 RATE(K,I)=FRQ(K)

Gθ Tθ 40

12 IF(.NOT.BFLAG(K)) Gθ Tθ 13

Gθ Tθ 15

13 Dθ 14 I=2,NP2

E=-EACT(K)\*FS(JTR,I)

14 RATE(K,I)=FRQ(K)\*DEXP(E)

Gθ Tθ 40

15 IF(.NOT.EFLAG(K)) Gθ Tθ 16

Gθ Tθ 17

16 Dθ 18 I=2,NP2

18 RATE(K,I)=FRQ(K)\*(FS(JT,I)\*\*BETA(K))

Gθ Tθ 40

17 Dθ 19 I=2,NP2

E=-EACT(K)\*FS(JTR,I)

19 RATE(K,I)=FRQ(K)\*(FS(JT,I)\*\*BETA(K))\*DEXP(E)

RCN00010  
RCN00020  
RCN00030  
RCN00040  
RCN00050  
RCN00060  
RCN00070  
RCN00080  
RCN00090  
RCN00100  
RCN00110  
RCN00120  
RCN00130  
RCN00140  
RCN00150  
RCN00160  
RCN00170  
RCN00180  
RCN00190  
RCN00200  
RCN00210  
RCN00220  
RCN00230  
RCN00240  
RCN00250  
RCN00260  
RCN00270  
RCN00280  
RCN00290  
RCN00300  
RCN00310  
RCN00320  
RCN00330  
RCN00340

```

RCN00350
RCN00360
RCN00370
RCN00380
RCN00390
RCN00400
RCN00410
RCN00420
RCN00430
RCN00440
RCN00450
RCN00460
RCN00470
RCN00480
RCN00490
RCN00500
RCN00510
RCN00520
RCN00530
RCN00540
RCN00550
RCN00560
RCN00570
RCN00580
RCN00590
RCN00600
RCN00610

GE TE 40
50 DØ 67 I=2, NP2
67 RATE(K, I)=-.0D0
40 CONTINUE
DE 60 K=1, KRATE
IF(.NET.RFLAG(K)) GØ TE 61
DØ 62 I=2, NP2
ECØN(I)=-EQA(K)*FS(JTR, I)-EQB(K)-EQC(K)*FS(JT, I)
62 ECØN(I)=DEXP(ECØN(I))
IF(BØD1(K)) GØ TE 128
GE TE 124
128 IF(BØD2(K)) GØ TE 126
GE TE 122
124 DØ 63 I=2, NP2
63 RATER(K, I)=RATE(K, I)/ECØN(I)
GE TE 60
122 DØ 64 I=2, NP2
64 RATER(K, I)=RATE(K, I)*RGT(I)/ECØN(I)
GE TE 60
126 DØ 65 I=2, NP2
65 RATER(K, I)=RATE(K, I)/(ECØN(I)*RGT(I))
GE TE 60
61 DØ 66 I=2, NP2
66 RATER(K, I)=-.0D0
60 CONTINUE
RETURN
END

```

```

0ELI,SIH ABCDE*FLAME(2). ENTRN/3.,. 141333132312
SUBROUTINE ENTRN(J,IF0W)
IMPLICIT REAL*8 (A-H,0-Z)
REAL ALPHA
COMMON/IVALS/N,NP1,NP2,NP3
COMMON/JVALS/JH,JHM1,JHM2,JRAD,JRADP1,JM,JMR,JT,JTR,JB0DYA,JB0DYB
COMMON/EMEGA1/NI1,NEE,0MR
COMMON/EMEGA2/0M(43),0MD(43),0MI,0ME,R0MD(43)
COMMON/TRANS1/HREF(15),DCP(15),ECP(15),FCP(15),WT(15),ITER,ITER1
COMMON/TRANS3/RH0(43),RU(43),RH2(43)
COMMON/EMEGA3/B0M(43),B0MT3(43),0MP(43),PEI0MD(43),PEI0M2(43)
COMMON/VAR1/F(15,43),FS(15,43)
COMMON/ENTR/VEL(15),RMI,RME,DX,ALPHA
COMMON/YCALC/PEI,Y(43),YD(43),YAV(43),RPEI
COMMON/MASPR0/R(15,43)
12 SUM=.0D0
DE 30 I=3,NP1
30 SUM=SUM+R(J,I)*B0MT3(I)+R(J,I+1)*0MD(I)+R(J,I-1)*0MD(I-1)
SUM=.125D0*PEI*DABS(SUM)
H=F(J,2)
HH=F(J,NI1)
C=F(J,NP2)
CC=F(J,NEE)
FAC=1.D0/DABS(H-C)
SUM=-SUM*FAC
VEL(J)=SUM/RH0(NP2)

IF(IF0W.EQ.1) G0 T0 500
RMI=(H-HH)*FAC*100.D0
RMI=DABS(RMI)
RMI=RMI**ALPHA
RMI=2.D0-RMI
RMI=SUM*RMI
RME=(CC-C)*FAC*100.D0
RME=DABS(RME)
ENT00010
ENT00020
ENT00030
ENT00040
ENT00050
ENT00060
ENT00070
ENT00080
ENT00090
ENT00100
ENT00110
ENT00120
ENT00130
ENT00140
ENT00150
ENT00160
ENT00170
ENT00180
ENT00190
ENT00200
ENT00210
ENT00220
ENT00230
ENT00240
ENT00250
ENT00260
ENT00270
ENT00280
ENT00290
ENT00300
ENT00310
ENT00320
ENT00330
ENT00340

```

C

ENT00350  
ENT00360  
ENT00370  
ENT00380  
ENT00390

RME=RME\*\*ALPHA  
RME=SUM\*RME  
PEI=PEI\*(RMI-RME)\*DX  
500 RETURN  
END

DEL1, SIH ABCDE\*FLAME(2).SCHM1/1.,.,141373132312

```

SUBROUTINE SCHM1
  IMPLICIT REAL*8 (A-H, O-Z)
  COMMON/IVALS/N, NP1, NP2, NP3
  COMMON/JVALS/JH, JHM1, JHM2, JRAD, JRADP1, JM, JMR, JT, JTR, JBODYA, JBODYB
  COMMON/CEGA2/OM(43), OMD(43), OMI, OME, RØMD(43)
  COMMON/TRANS2/RMB(43), PREF(15,43), HDRM(15,43), TRCP(15,43)
  *, DR2M(15,43)
  COMMON/TRANS3/RHØ(43), RU(43), RH2(43)
  COMMON/VAR1/F(15,43), FS(15,43)
  COMMON/YCALC/PEI, Y(43), YD(43), YAV(43), RPEI
  COMMON/SCM1/SC1(15,43)
  DIMENSION BF(15,43)
  DO 10 I=2, NP2
  DE 10 J=1, JHM2
  BF(J, I)=(F(J, I)+F(J, I+1))*SDO
10 CONTINUE
  DO 20 I=3, NP1
  DE 20 J=1, JHM2
  FAC=PREF(J, I) *RMB(I)*BF(J, I)+(FS(JM, I+1)-FS(JM, I))*RØMD(I)
  FAC=FAC-PREF(J, I-1) *RMB(I-1)*BF(J, I-1)+(FS(JM, I)
1-FS(JM, I-1))*RØMD(I-1)
  SC1(J, I)=FAC*RPEI
20 CONTINUE
  RETURN
  END
SC100010
SC100020
SC100030
SC100040
SC100050
SC100060
SC100070
SC100080
SC100090
SC100100
SC100110
SC100120
SC100130
SC100140
SC100150
SC100160
SC100170
SC100180
SC100190
SC100200
SC100210
SC100220
SC100230
SC100240
SC100250
```

```

DEL1,SIH ABCDE*FLAME(2).SCHM2/7,..141405132312
SUBROUTINE SCHM2(J)
  IMPLICIT REAL*8 (A-H,θ-Z)
  LOGICAL G1,G2,G3,BØD1,BØD2
  CØMMØN/IVALS/N,NP1,NP2,NP3
  CØMMØN/JVALS/JH,JHM1,JHM2,JRAD,JRADP1,JM,JMR,JT,JTR,JBØDYA,JBØDYB
  CØMMØN/TRANS3/RHØ(43),RU(43),RH2(43)
  CØMMØN/VAR1/F(15,43),FS(15,43)
  CØMMØN/EMEGA3/BØM(43),BØMT3(43),ØMP(43),PEIØMD(43),PEIØM2(43)
  CØMMØN/RATC/RATE(30,43),RATER(30,43),KRATE,EQA(30),EQB(30),EQC(30)
  CØMMØN/SCM2/SU(15,43),SD(15,43)
  CØMMØN/MASPRØ/R(15,43)
  CØMMØN/GSWITC /G1(30,15),G2(30,15),G3(30,15)
  CØMMØN/REACT/LA(30),LB(30),LRA(30),LRB(30)
  CØMMØN/RTPARM/FRQ(30),BETA(30),EACT(30)
  * , BØD1(30),BØD2(30),REACT(30)
  DIMENSION FØR(43),REV(43)
  DØ 60 I=2,NP2
  SU(J,I)=.ØDØ
  60 SD(J,I)=.ØDØ
  DØ 18 K=1,KRATE
  IF(.NØT.G1(K,J)) GØ TØ 18
  LAKKK=LA(K)
  LBKKK=LB(K)
  LRAKKK=LRA(K)
  LRBKKK=LRB(K)
  IF(K.EQ.5) GØ TØ 47
  IF(K.EQ.10) GØ TØ 48
  IF(BØD1(K)) GØ TØ 41
  DØ 50 I=2,NP2
  FØR(I)=RATE(K,I)*F(LAKKK,I)*F(LBKKK,I)
  50 REV(I)=RATER(K,I)*F(LRAKKK,I)*F(LRBKKK,I)
  GØ TØ 40
  41 IF(BØD2(K)) GØ TØ 42
  DØ 51 I=2,NP2
  SC200010
  SC200020
  SC200030
  SC200040
  SC200050
  SC200060
  SC200070
  SC200080
  SC200090
  SC200100
  SC200110
  SC200120
  SC200130
  SC200140
  SC200150
  SC200160
  SC200170
  SC200180
  SC200190
  SC200200
  SC200210
  SC200220
  SC200230
  SC200240
  SC200250
  SC200260
  SC200270
  SC200280
  SC200290
  SC200300
  SC200310
  SC200320
  SC200330
  SC200340

```

FØR(I)=RATE(K,I)\*F(LAKK,I)\*F(LBKK,I)\*FS(JMR,I)\*RHØ(I)  
 51 REV(I)=RATER(K,I)\*F(LRAKK,I)\*FS(JMR,I)  
 GE TØ 40  
 42 DØ 53 I=2, NP2  
 FØR(I)=RATE(K,I)\*F(LAKK,I)\*F(LBKK,I)\*FS(JMR,I)  
 53 REV(I)=RATER(K,I)\*F(LRAKK,I)\*F(LRBKK,I)\*FS(JMR,I)\*RHØ(I)  
 GE TØ 40  
 47 DØ 54 I=2, NP2  
 FØR(I)=RATE(K,I)\*F(LAKK,I)\*F(LBKK,I)\*FS(JBØDYA,I)\*RHØ(I)  
 54 REV(I)=RATER(K,I)\*F(LRAKK,I)\*FS(JBØDYA,I)  
 GE TØ 40  
 48 DØ 55 I=2, NP2  
 FØR(I)=RATE(K,I)\*F(LAKK,I)\*F(LBKK,I)\*FS(JBØDYB,I)\*RHØ(I)  
 55 REV(I)=RATER(K,I)\*F(LRAKK,I)\*FS(JBØDYB,I)  
 40 CØNTINUE  
 IF(G3(K,J)) GØ TØ 19  
 IF(G2(K,J)) GØ TØ 20  
 DØ 56 I=2, NP2  
 SU(J,I)=SU(J,I)\*REV(I)  
 56 SD(J,I)=SD(J,I)-FØR(I)  
 GE TØ 18  
 20 DØ 57 I=2, NP2  
 SU(J,I)=SU(J,I)\*FØR(I)  
 57 SD(J,I)=SD(J,I)-REV(I)  
 GE TØ 18  
 19 IF(G2(K,J)) GØ TØ 21  
 DØ 58 I=2, NP2  
 SU(J,I)=SU(J,I)\*2. DØ\*(FØR(I)\*REV(I))  
 58 SD(J,I)=SD(J,I)-4. DØ\*FØR(I)  
 GE TØ 18  
 21 DØ 59 I=2, NP2  
 SU(J,I)=SU(J,I)\*2. DØ\*(FØR(I)\*REV(I))  
 59 SD(J,I)=SD(J,I)-4. DØ\*REV(I)  
 18 CØNTINUE  
 DØ 30 I=2, NP2  
 SC200350  
 SC200360  
 SC200370  
 SC200380  
 SC200390  
 SC200400  
 SC200410  
 SC200420  
 SC200430  
 SC200440  
 SC200450  
 SC200460  
 SC200470  
 SC200480  
 SC200490  
 SC200500  
 SC200510  
 SC200520  
 SC200530  
 SC200540  
 SC200550  
 SC200560  
 SC200570  
 SC200580  
 SC200590  
 SC200600  
 SC200610  
 SC200620  
 SC200630  
 SC200640  
 SC200650  
 SC200660  
 SC200670  
 SC200680  
 SC200690

SC200700  
SC200710  
SC200720  
SC200730  
SC200740  
SC200750  
SC200760  
SC200770  
SC200780

```
SU(J,I)=SU(J,I)*RH0(I)
SD(J,I)=SD(J,I)*RH0(I)
R(J,I)=SU(J,I)+SD(J,I)
30 SD(J,I)=SD(J,I)/F(J,I)
   DE 31 I=3,NP1
SU(J,I)=SU(J,I)*PEI0M2(I)
31 SD(J,I)=SD(J,I)*PEI0M2(I)
   RETURN
   END
```

```

@EL1,SIH ABCDE*FLAME(2).CALC/1.,.,137751133112
SUBROUTINE CALC(J)
IMPLICIT REAL*8 (A-H,Ø-Z)
COMMON/IVALS/N,NP1,NP2,NP3
COMMON/JVALS/JH,JHM1,JHM2,JRAD,JRADP1,JM,JMR,JT,JTR,JBØDYA,JBØDYB
COMMON/VAR1/F(15,43),FS(15,43)
COMMON/CØEF/A(15,43),B(15,43),C(15,43),D(15,43)
A(J,3)=A(J,3)/D(J,3)
B(J,3)=(B(J,3)*F(J,2)*C(J,3))/D(J,3)
DØ 20 I=4,NP1
T=D(J,I)-B(J,I)*A(J,I-1)
A(J,I)=A(J,I)/T
20 B(J,I)=(B(J,I)*B(J,I-1)*C(J,I))/T
DØ 30 IDASH=3,NP1
I=N+4-IDASH
F(J,I)=A(J,I)*F(J,I+1)*B(J,I)
IF(J.EQ.JH) GØ TØ 30
IF(F(J,I).LT..ØDØ) F(J,I)=1.D-3Ø
30 CONTINUE
RETURN
END
CLCØØØ1Ø
CLCØØØ2Ø
CLCØØØ3Ø
CLCØØØ4Ø
CLCØØØ5Ø
CLCØØØ6Ø
CLCØØØ7Ø
CLCØØØ8Ø
CLCØØØ9Ø
CLCØØ1ØØ
CLCØØ11Ø
CLCØØ12Ø
CLCØØ13Ø
CLCØØ14Ø
CLCØØ15Ø
CLCØØ16Ø
CLCØØ17Ø
CLCØØ18Ø
CLCØØ19Ø
CLCØØ2ØØ

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@ELT,SIH ABCDE*FLAME(2).SENTP/1,..,141375132312
SUBROUTINE SENTP
IMPLICIT REAL*8 (A-H,Ø-Z)
COMMON/IVALS/N,NP1,NP2,NP3
COMMON/JVALS/JH,JHM1,JHM2,JRAD,JRADP1,JM,JMR,JT,JTR,JBØDYA,JBØDYB
COMMON/EMEGA2/ØM(43),ØMD(43),ØMI,ØME,RØMD(43)
COMMON/TRANS2/RMB(43),PREF(15,43),HDRM(15,43),TRCP(15,43)
*,DR2M(15,43)
COMMON/TRANS3/RBØ(43),RU(43),RH2(43)
COMMON/VAR1/F(15,43),FS(15,43)
COMMON/YCALC/PEI,Y(43),
COMMON/SENT /SEN(43)
DØ 20 I=3,NP1
SEN(I)=-.ØDØ
DØ 10 J=1,JHM1
FAC=HDRM(J,I)*(F(J,I+1)*FS(JM,I+1)-F(J,I)*FS(JM,I))*RØMD(I)
FAC=FAC-HDRM(J,I-1)*(F(J,I)*FS(JM,I)-F(J,I-1)*FS(JM,I-1))
1*RØMD(I-1)
FAC=FAC-TRCP(J,I)*(F(J,I+1)-F(J,I))*RØMD(I)
FAC=FAC+TRCP(J,I-1)*(F(J,I)-F(J,I-1))*RØMD(I-1)
SEN(I)=SEN(I)+FAC
10 CONTINUE
SEN(I)=SEN(I)*RPEI
20 CONTINUE
RETURN
END
SNP00010
SNP00020
SNP00030
SNP00040
SNP00050
SNP00060
SNP00070
SNP00080
SNP00090
SNP00100
SNP00110
SNP00120
SNP00130
SNP00140
SNP00150
SNP00160
SNP00170
SNP00180
SNP00190
SNP00200
SNP00210
SNP00220
SNP00230
SNP00240
SNP00250

```

```

OELT,SIH ABCDE*FLAME(2).ZCALC/1.,.,141401132312
SUBROUTINE ZCALC
IMPLICIT REAL*8 (A-H,θ-Z)
COMMON/IVALS/N,NP1,NP2,NP3
COMMON/CMEGA2/θM(43),θMD(43),θM1,θME,θMD(43)
COMMON/TRANS3/RHθ(43),RU(43),RH2(43)
COMMON/YCALC/PEI,Y(43),YD(43),YAV(43),RPEI
Dθ 50 I=2,NP1
50 RU(I)=(RHθ(I)+RHθ(I+1))*θMD
Y(1)=θDθ
Y(2)=θDθ
Dθ 20 I=3,NP2
20 Y(I)=Y(I-1)+PEI*θMD(I-1)/RU(I-1)
Y(NP3)=Y(NP2)
Dθ 30 I=2,NP1
30 YD(I)=Y(I+1)-Y(I)
Dθ 40 I=3,NP1
40 YAV(I)=(Y(I+1)-Y(I-1))*θMD
RETURN
END

```

```

ZCL00010
ZCL00020
ZCL00030
ZCL00040
ZCL00050
ZCL00060
ZCL00070
ZCL00080
ZCL00090
ZCL00100
ZCL00110
ZCL00120
ZCL00130
ZCL00140
ZCL00150
ZCL00160
ZCL00170
ZCL00180
ZCL00190

```

```

@ELT,SIH ABCDE*FLAME(2).@OUTPUT,.,.142032031413
SUBROUTINE @OUTPUT(L,HEADNG,RUNID,X,INPUT,LMAX)
IMPLICIT REAL*8 (A-H,@-Z)
REAL V(30,43),W(30,43),XMAX(30),YMAX(30),Z(60)
REAL ZTAB,TAB,PUT(15),HV(30,43),HW(30,43),SFAC
LOGICAL SKIP(60),FLAG
DIMENSION IX(30),IY(30)
COMMON/IVALS/N,NP1,NP2,NP3
COMMON/JVALS/JH,JHM1,JHM2,JRAD,JRADP1,JM,JMR,JT,JTR,JBODYA,JBODYB
COMMON/EMEGA1/NII,NEE,@MR
COMMON/CEGA2/@M(43),@MI,@ME,@MD(43)
COMMON/INIT/XI(15),XE(15),IPFL(15),TC@LD,PRESS,YWIDTH
COMMON/TRANS1/HREF(15),DCP(15),ECP(15),FCP(15),WT(15),ITER,ITER1
COMMON/TRANS2/RMB(43),PREF(15,43),HDRM(15,43),TRCP(15,43)
*,DR2M(15,43)
COMMON/TRANS3/RH@(43),RU(43),RH2(43)
COMMON/VAR1/F(15,43),FS(15,43)
COMMON/YCALC/PEI,Y(43),YD(43),YAV(43),RPEI
COMMON/ENTR/VEL(15),RMI,RME,DX,ALPHA
COMMON/MASPR@/R(15,43)
COMMON/FLAGS/FLAG(20)
REAL FFER(43),RREV(43)
LOGICAL B@D1,B@D2
COMMON/RTPARM/FRQ(30),BETA(30),EACT(30)
*,BED1(30),B@D2(30),REACT(30)
COMMON/RATC/RATE(30,43),RATER(30,43),KRATE,EQA(30),EQB(30),EQC(30)
COMMON/REACT/LA(30),LB(30),LRA(30),LRB(30)
REAL HEADNG(126),@UT(14),@UTH,@UTT,@UTM,RUNID(12)
DIMENSION F@R(43),REV(43)
DATA @UT/14*.0/,SKIP/60*.FALSE./
69 F@RMAT(1H )
WRITE(6,10) L,DX,X
10 F@RMAT(1H1,'NUMBER OF STEPS TAKEN=',I3,3X,'STEP SIZE FOR LAST STEP@UT00320
*,I,D12.5,3X,'ELAPSED TIME=',I,D12.5 )
WRITE(6,69)
@UT00010
@UT00020
@UT00030
@UT00040
@UT00050
@UT00060
@UT00070
@UT00080
@UT00090
@UT00100
@UT00110
@UT00120
@UT00130
@UT00140
@UT00150
@UT00160
@UT00170
@UT00180
@UT00190
@UT00200
@UT00210
@UT00220
@UT00230
@UT00240
@UT00250
@UT00260
@UT00270
@UT00280
@UT00290
@UT00300
@UT00310
@UT00320
@UT00330
@UT00340

```

```

WRITE(6,11) (HEADNG(M),M=1,126)
11 FORMAT(3X,126A1,1X,'I')
DO 12 I=2,NP2
DO 13 J=1,JHM1
13 OUT(J)=F(J,I)
OUTH=F(JH,I)
OUTI=FS(JT,I)
OUTM=FS(JM,I)
WRITE(6,15) ((OUT(J),J=1,12),OUTH,OUTT,I)
15 FORMAT(2X,12E9.5,E10.5,E9.5,I3)
12 CONTINUE
WRITE(6,69)
WRITE(6,25)
25 FORMAT(50X,'FLAME VELOCITIES ')
DO 26 J=1,JHM1
26 OUT(J)=VEL(J)
WRITE(6,27) (OUT(J),J=1,JHM2)
27 FORMAT(3X,14E9.4)
WRITE(6,69)
WRITE(6,28) PEI,RMI,RME
28 FORMAT(' GRID WIDTH; PEI=',D12.5,5X,'ENTRAINMENT RATES:',
*1X,'RMI=',D12.5,2X,'RME=',D12.5)
IF(L.EQ.LMAX) GO TO 16
GO TO 99
16 WRITE(6,17)
17 FORMAT(1H1,50X,'PARAMETERS FOR THIS RUN')
WRITE(6,69)
WRITE(6,18) N,NII,NEE,OMR
18 FORMAT(1X,'GRID STRUCTURE:',3X,'N=',I2,2X,'NII=',I2,2X,'NEE=',
*I2,2X,'OMR=',D8.2)
WRITE(6,19) JH,JRAD,JM,JMR,JT,JTR,JBODYA,JBODYB
19 FORMAT(1X,'J INDICES:',3X,'JH=',I2,2X,'JRAD=',I2,2X,'JM=',I2,
*2X,'JMR=',I2,2X,'JT=',I2,2X,'JTR=',I2,2X,'JBODY=',I2,2X,'JBODYB=',
*I2)
WRITE(6,20) TCOLD,PRESS,INPUT

```

```

OUT00350
OUT00360
OUT00370
OUT00380
OUT00390
OUT00400
OUT00410
OUT00420
OUT00430
OUT00440
OUT00450
OUT00460
OUT00470
OUT00480
OUT00490
OUT00500
OUT00510
OUT00520
OUT00530
OUT00540
OUT00550
OUT00560
OUT00570
OUT00580
OUT00590
OUT00600
OUT00610
OUT00620
OUT00630
OUT00640
OUT00650
OUT00660
OUT00670
OUT00680
OUT00690

```

```

20 FORMAT(1X,TEMPERATURE AT COLD SIDE=1,D12.5,3X,PRESSURE=1,
#D12.5,3X,INPUT=0,11/1X,(IF INPUT=0, PROFILES WERE CALCULATED BY
*INITL; IF INPUT=1, THEY WERE TAKEN FROM 45. THE OUTPUT WENT INTO 4
*6.))
WRITE(6,69)
C FOR(I) ARRAY IS USED FOR Q(I), THE HEAT RELEASE RATE.
C FIRST, THE PRODUCTION RATE OF THE JHM2 SPECIES IS CALCULATED
C IN THE CASE THAT THE CONSTANT N2 APPROXIMATION IS BEING USED.
IF(FLAG(2)) GO TO 90
JHM3=JH-3
DO 91 I=2, NP2
SUM=.0D0
DO 92 J=1, JHM3
SUM=SUM+R(J,I)*WT(J)
92 SUM=SUM+R(J,I)*WT(J)
91 R(JHM2,I)=-SUM/WT(JHM2)
90 DO 70 I=2, NP2
FOR(I)=.0D0
T=FS(JT,I)
T2=T*T
C NOTE THAT R(J,I)=0 FOR THE JHM1 SPECIES IF IT IS A BUFFER.
DO 71 J=1, JHM1
FAC=HREF(J)*T*(DCP(J)*ECP(J)*T*.5D0+FCP(J)*T2*.333333D0)
PUT(J)=FAC*WT(J)
71 FOR(I)=FOR(I)+R(J,I)*FAC*WT(J)
70 CONTINUE
WRITE(6,69)
WRITE(6,23)
23 FORMAT(1X,SPACE VARIABLES)
WRITE(6,21)
21 FORMAT(2X,I,7X,OMEGA,9X,Y,9X,DENSITY,4X,AV MOL WT,
*.4X,HEAT REL.)
WRITE(6,22)(I,GM(I),Y(I),RH(I),FS(JM,I),FOR(I),I=2, NP2)
22 FORMAT(1X,13,2X,5D12.5)
WRITE(6,69)
WRITE(6,69)

```

```

@UT00700
@UT00710
@UT00720
@UT00730
@UT00740
@UT00750
@UT00760
@UT00770
@UT00780
@UT00790
@UT00800
@UT00810
@UT00820
@UT00830
@UT00840
@UT00850
@UT00860
@UT00870
@UT00880
@UT00890
@UT00900
@UT00910
@UT00920
@UT00930
@UT00940
@UT00950
@UT00960
@UT00970
@UT00980
@UT00990
@UT01000
@UT01010
@UT01020
@UT01030
@UT01040

```

```

WRITE(6,24) (RUNID(M),M=1,12)
24 FORMAT( 1X,'RUN IDENTIFICATION:',3X,12A6)
WRITE(6,29)
29 FORMAT(1H1,10X,'MOLE PRODUCTION RATES OF THE VARIOUS CHEMICAL SPEC
*IES' )
WRITE(6,69)
WRITE(6,11) (HEADNG(M),M=1,118)
D0 30 I=2, NP2
D0 31 J=1, JHM1
31 OUT(J)=R(J,I)
D0 32 J=JH,14
32 OUT(J)=.0
WRITE(6,33) ((OUT(J),J=1,14),I)
33 FORMAT(3X,14E9.4,I3)
30 CONTINUE
WRITE(6,34)
34 FORMAT(1H1,'TRANSPORT PARAMETERS; I.E., THE ARRAY PREF' )
WRITE(6,69)
WRITE(6,11) (HEADNG(M),M=1,118)
JHP1=JH+1
D0 35 I=2, NP2
D0 36 J=1, JH
36 OUT(J)=PREF(J,I)
D0 37 J=JHP1,14
37 OUT(J)=.0
WRITE(6,33) ((OUT(J),J=1,14),I)
35 CONTINUE
IF(.NOT.FLAG(9)) G0 T0 100
ICOUNT=0
ISTART=4
104 ISTEP=ISTART+15
WRITE(6,101)
101 FORMAT(1H1,'VALUES OF RATE CONSTANTS AT EACH POINT IN THE FLAME' )
WRITE(6,69)
WRITE(6,44) (I,I=ISTART,ISTOP)
OUT01050
OUT01060
OUT01070
OUT01080
OUT01090
OUT01100
OUT01110
OUT01120
OUT01130
OUT01140
OUT01150
OUT01160
OUT01170
OUT01180
OUT01190
OUT01200
OUT01210
OUT01220
OUT01230
OUT01240
OUT01250
OUT01260
OUT01270
OUT01280
OUT01290
OUT01300
OUT01310
OUT01320
OUT01330
OUT01340
OUT01350
OUT01360
OUT01370
OUT01380
OUT01390

```

```

C      NOTE THAT FFØR AND RREV ARRAYS ARE USED TØ HOLD RATE AND RATER.
DØ 102 K=1, KRATE
DØ 103 I=ISTART, ISTOP
      FFØR(I)=RATE(K,I)
103 RREV(I)=RATER(K,I)
      WRITE(6,43) (K,(FFØR(I), I-ISTART, ISTOP))
      WRITE(6,43) (K,(RREV(I), I-ISTART, ISTOP))
102 CØNTINUE
      ICØUNT=ICØUNT+1
      IF(ICØUNT.EQ.2) GØ TØ 100
      ISTART=20
      GØ TØ 104
100 CØNTINUE
      IF(.NOT.FLAG(10)) GØ TØ 45
      ICØUNT=0
      ISTART=4
      ISTEP=ISTART+15
      WRITE(6,38)
38 FØRMAT(1H1, 'RATES ØF EACH REACTION AT EACH PØINT IN THE FLAME')
      WRITE(6,69)
      WRITE(6,44) (I, I-ISTART, ISTOP)
44 FØRMAT(1X, 16(3X, 12, 3X))
      DØ 39 K=1, KRATE
      LAKKK=LA(K)
      LBKKK=LE(K)
      LRAKKK=LRA(K)
      LRBKKK=LRB(K)
      DØ 50 I=ISTART, ISTOP
      IF(K.EQ.5) GØ TØ 47
      IF(K.EQ.10) GØ TØ 48
      IF(BØD1(K)) GØ TØ 41
      FØR(I)=RATE(K, I)*F(LAKKK, I)*F(LBKKK, I)*RHØ(I)
      REV(I)=RATER(K, I)*F(LRAKKK, I)*F(LRBKKK, I)*RHØ(I)
      SFAC=PUT(LRAKKK)*PUT(LRBKKK)-PUT(LAKKK)-PUT(LBKKK)
      FFØR(I)=FØR(I)
      ØUTØ1400
      ØUTØ1410
      ØUTØ1420
      ØUTØ1430
      ØUTØ1440
      ØUTØ1450
      ØUTØ1460
      ØUTØ1470
      ØUTØ1480
      ØUTØ1490
      ØUTØ1500
      ØUTØ1510
      ØUTØ1520
      ØUTØ1530
      ØUTØ1540
      ØUTØ1550
      ØUTØ1560
      ØUTØ1570
      ØUTØ1580
      ØUTØ1590
      ØUTØ1600
      ØUTØ1610
      ØUTØ1620
      ØUTØ1630
      ØUTØ1640
      ØUTØ1650
      ØUTØ1660
      ØUTØ1670
      ØUTØ1680
      ØUTØ1690
      ØUTØ1700
      ØUTØ1710
      ØUTØ1720
      ØUTØ1730
      ØUTØ1740

```

RREV(I)-REV(I)  
 HV(K,I)-SFAC\*FFØR(I)  
 HW(K,I)--SFAC\*RRREV(I)  
 GØ TØ 40  
 41 IF(BØD2(K)) GØ TØ 42  
 FØR(I)-RATE(K,I)\*F(LAKKK,I)\*F(LBKKK,I)\*FS(JMR,I)\*RH2(I)  
 REV(I)-RATER(K,I)\*F(LRAKKK,I)\*FS(JMR,I)\*RHØ(I)  
 SFAC=PUT(LRAKKK)-PUT(LAKKK)-PUT(LBKKK)  
 FFØR(I)-FØR(I)  
 RREV(I)-REV(I)  
 HV(K,I)-SFAC\*FFØR(I)  
 HW(K,I)--SFAC\*RRREV(I)  
 GØ TØ 40  
 42 FØR(I)-RATE(K,I)\*F(LAKKK,I)\*FS(JMR,I)\*RHØ(I)  
 REV(I)-RATER(K,I)\*F(LRAKKK,I)\*F(LRBKKK,I)\*FS(JMR,I)\*RH2(I)  
 SFAC=PUT(LRAKKK)+PUT(LRBKKK)-PUT(LAKKK)  
 FFØR(I)-FØR(I)  
 RREV(I)-REV(I)  
 HV(K,I)-SFAC\*FFØR(I)  
 HW(K,I)--SFAC\*RRREV(I)  
 GØ TØ 40  
 47 FØR(I)-RATE(K,I)\*F(LAKKK,I)\*F(LBKKK,I)\*FS(JBØDYA,I)\*RH2(I)  
 REV(I)-RATER(K,I)\*F(LRAKKK,I)\*FS(JBØDYA,I)\*RHØ(I)  
 SFAC=PUT(LRAKKK)-PUT(LAKKK)-PUT(LBKKK)  
 FFØR(I)-FØR(I)  
 RREV(I)-REV(I)  
 HV(K,I)-SFAC\*FFØR(I)  
 HW(K,I)--SFAC\*RRREV(I)  
 GØ TØ 40  
 48 FØR(I)-RATE(K,I)\*F(LAKKK,I)\*F(LBKKK,I)\*FS(JBØDYB,I)\*RH2(I)  
 REV(I)-RATER(K,I)\*F(LRAKKK,I)\*FS(JBØDYB,I)\*RHØ(I)  
 SFAC=PUT(LRAKKK)-PUT(LAKKK)-PUT(LBKKK)  
 FFØR(I)-FØR(I)  
 RREV(I)-REV(I)  
 HV(K,I)-SFAC\*FFØR(I)

ØUT01750  
 ØUT01760  
 ØUT01770  
 ØUT01780  
 ØUT01790  
 ØUT01800  
 ØUT01810  
 ØUT01820  
 ØUT01830  
 ØUT01840  
 ØUT01850  
 ØUT01860  
 ØUT01870  
 ØUT01880  
 ØUT01890  
 ØUT01900  
 ØUT01910  
 ØUT01920  
 ØUT01930  
 ØUT01940  
 ØUT01950  
 ØUT01960  
 ØUT01970  
 ØUT01980  
 ØUT01990  
 ØUT02000  
 ØUT02010  
 ØUT02020  
 ØUT02030  
 ØUT02040  
 ØUT02050  
 ØUT02060  
 ØUT02070  
 ØUT02080  
 ØUT02090

```

HW(K,I)=-SFAC*RREV(I)
40 CONTINUE
V(K,I)=FFOR(I)
W(K,I)=RREV(I)
50 CONTINUE
WRITE(6,43) (K,(FFOR(I),I=ISTART,ISTOP))
43 FORMAT(1X,12,1X,16E8.4)
WRITE(6,43) (K,(RREV(I),I=ISTART,ISTOP))
39 CONTINUE
ICOUNT=ICOUNT+1
IF(ICOUNT.EQ.2) GO TO 45
ISTART=20
GO TO 46
45 CONTINUE
DO 85 K=1,KRATE
XMAX(K)=.0
YMAX(K)=.0
DO 86 I=2,NP2
IF(V(K,I).GT.XMAX(K)) GO TO 72
GE TO 73
72 XMAX(K)=V(K,I)
IX(K)=I
73 IF(W(K,I).GT.YMAX(K)) GO TO 74
GE TO 86
74 YMAX(K)=W(K,I)
IY(K)=I
86 CONTINUE
85 CONTINUE
IF(IFINK.EQ.0) GO TO 204
WRITE(6,205)
205 FORMAT(1H1,'MAXIMUM HEAT RELEASE RATES FOR EACH REACTION')
GO TO 206
204 CONTINUE
WRITE(6,75)
75 FORMAT(1H1,'MAXIMUM RATES OF FORWARD AND REVERSE REACTIONS')

```

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@UT02100
@UT02110
@UT02120
@UT02130
@UT02140
@UT02150
@UT02160
@UT02170
@UT02180
@UT02190
@UT02200
@UT02210
@UT02220
@UT02230
@UT02240
@UT02250
@UT02260
@UT02270
@UT02280
@UT02290
@UT02300
@UT02310
@UT02320
@UT02330
@UT02340
@UT02350
@UT02360
@UT02370
@UT02380
@UT02390
@UT02400
@UT02410
@UT02420
@UT02430
@UT02440

```

```

206 CONTINUE
    WRITE(6,69)
    WRITE(6,76)
76  FORMAT(1X, 'POSITION, FORWARD MAX POSITION, REVERSE MAX REACTION' )
77  FORMAT(4X, I2, 5X, E8.4, 7X, I2, 5X, E8.4, 7X, I2)
84  FORMAT(1H1, 'RANK', 2X, 'REACTION', 2X, 'DIRECTION', 2X, 'MAXIMUM' )
    DC 78 K=1, KRATE
    Z(K)=XMAX(K)
78  Z(K+KRATE)=YMAX(K)
    KK=2+KRATE
    ICEUNT=0
83  ZTAB=.0
    DO 79 K=1, KK
    IF(SKIP(K)) GO TO 79
    IF(Z(K).GT.ZTAB) GO TO 80
    GO TO 79
80  ZTAB=Z(K)
    KTAB=K
79  CONTINUE
    IF(KTAB.LE.KRATE) GO TO 87
    TAB=3BREV
    NTAB=KTAB-KRATE
    GO TO 88
87  TAB=3HFCE
    NTAB=KTAB
88  CONTINUE
    ICEUNT=ICEUNT+1
    WRITE(6,81) ICEUNT, NTAB, TAB, ZTAB
81  FORMAT(2X, I2, 6X, I2, 7X, A3, 6X, E8.4)
    SKIP(KTAB)=.TRUE.
    IF(ICEUNT.EQ.KK) GO TO 82
    GO TO 83
82  CONTINUE

```

0UT02450

0UT02460

0UT02470

0UT02480

0UT02490

0UT02500

0UT02510

0UT02520

0UT02530

0UT02540

0UT02550

0UT02560

0UT02570

0UT02580

0UT02590

0UT02600

0UT02610

0UT02620

0UT02630

0UT02640

0UT02650

0UT02660

0UT02670

0UT02680

0UT02690

0UT02700

0UT02710

0UT02720

0UT02730

0UT02740

0UT02750

0UT02760

0UT02770

0UT02780

0UT02790

```

IF(.NOT.FLAG(11)) GO TO 99
DO 200 K=1,KRATE
DO 200 I=2,NP2
V(K,I)=HV(K,I)
200 W(K,I)=HW(K,I)
ICOUNT=0
ISTART=4
209 ISTEP=ISTART+15
WRITE(6,201)
201 FORMAT(1H1, 'HEAT RELEASE RATES FOR EACH REACTION')
WRITE(6,69)
WRITE(6,44) (I,I=ISTART,ISTOP)
DO 202 K=1,KRATE
WRITE(6,250) (K,(HV(K,I),I=ISTART,ISTOP))
WRITE(6,250) (K,(HW(K,I),I=ISTART,ISTOP))
250 FORMAT(1X,I2,1X,16E8.3)
202 CONTINUE
ICOUNT=ICOUNT+1
IF(ICOUNT.EQ.2) GO TO 99
ISTART=20
GO TO 209
99 RETURN
END

```

```

@UT02800
@UT02810
@UT02820
@UT02830
@UT02840
@UT02850
@UT02860
@UT02870
@UT02880
@UT02890
@UT02900
@UT02910
@UT02920
@UT02930
@UT02940
@UT02950
@UT02960
@UT02970
@UT02980
@UT02990
@UT03000
@UT03010
@UT03020

```

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## APPENDIX A.

## DERIVATION OF THE ONE-DIMENSIONAL LAMINAR FLAME EQUATIONS

Consider a control volume  $\delta V$  oriented as shown in Fig. A1. Since the flame is one-dimensional, there will be gradients only in the  $y$ -direction. The gas mixture flows from right to left through the  $+$  face of the control volume and out through the  $-$  face. The area of each face is  $\delta A$ , and the length of the volume is  $\delta y$ . The velocity of the mixture  $v$  is the mass average flow velocity defined by the equation

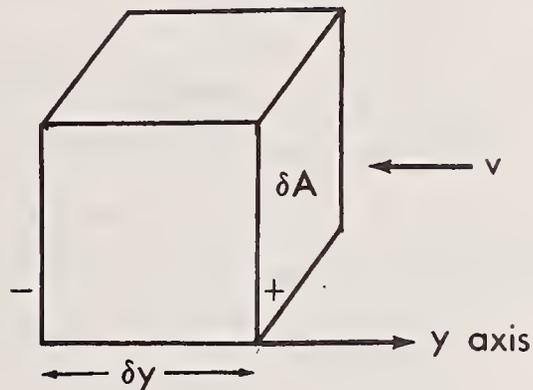


Figure A1.  
Orientation of control volume

$$v = \rho^{-1} \sum_j C_j M_j \bar{v}_j$$

where  $\bar{v}_j$  is the average velocity,  $C_j$  the concentration, and  $M_j$  the molecular weight of each chemical species;  $\rho$  is the density of the mixture.

There are three conservation equations to be derived;

- a) the conservation of total mass;
- b) the conservation of each chemical species;
- c) the conservation of energy.

a) Conservation of total mass

Let  $\rho_+$  be the density of the gas mixture entering the  $+$  face of  $\delta V$ , and let  $\rho_-$  be the density of the gas leaving at the  $-$  face. Let  $v_+$  and  $v_-$  be the mass average velocities of the gas at the  $+$  and  $-$  faces. If  $\rho$  is in units of  $\text{kg}/\text{m}^3$  and  $v$  in  $\text{m}/\text{s}$ , then the number of  $\text{kg}$  of gas entering  $\delta V$  per second is  $\delta A v_+ \rho_+$ , and the amount leaving is

$\delta A v_{\rho_-}$ . The difference between these two quantities gives the rate at which the mass of gas in  $\delta V$  changes. This is  $\delta \rho \delta V / \delta t$ , where  $\delta \rho$  is the change in the average density of the gas in  $\delta V$  in the time interval  $\delta t$ . Thus we get the equation,

$$\delta \rho \delta V / \delta t = - \delta A (v_+ \rho_+ - v_- \rho_-).$$

Note that  $v_+$  and  $v_-$  are negative numbers because the flow is from right to left. Dividing this equation by  $\delta V$ , we obtain the expression

$\delta \rho / \delta t = - (v_+ \rho_+ - v_- \rho_-) / \delta y$ , since  $\delta V = \delta A \delta y$ . If  $\delta y$  is sufficiently small, this equation can be written in derivative notation;

$$\partial \rho / \partial t = - \partial (\rho v) / \partial y. \quad (A1)$$

This is the mass conservation equation.

#### b) Species conservation equations

Let  $Y_j$  be the mass fraction of the chemical species  $j$ . Then  $Y_j \rho$  is the density of species  $j$  in the mixture. We can derive the same type of equation for the rate at which the mass of  $j$  changes in  $\delta V$  as we did for the change of total mass. We get,

$$\delta (\rho Y_j) \delta V / \delta t = - \delta A \left[ (v + V_j)_+ (\rho Y_j)_+ - (v + V_j)_- (\rho Y_j)_- \right] + R_j \delta V.$$

Here,  $V_j$  is the diffusion velocity of species  $j$ , and  $R_j$  is the mass production rate of species  $j$  arising from chemical reactions in  $\delta V$ .

Dividing this equation by  $\delta V$  and passing over to derivative notation, we get the species conservation equations,

$$\partial (\rho Y_j) / \partial t = - \frac{\partial}{\partial y} \left[ (v + V_j) (\rho Y_j) \right] + R_j, \quad j=1, n-1. \quad (A2)$$

There will be one of these equations for each species except one.

The mass fraction of the  $n$ 'th species is related to the mass fractions of the remaining species by the requirement that the sum of all the mass fractions equal unity; i.e.  $\sum_j^n Y_j = 1$ .

c) Energy conservation equation

Let  $h_j$  be the enthalpy per unit mass of the pure species  $j$  at the temperature  $T$ . The enthalpy of the mixture is then  $h = \sum_j Y_j h_j$ . Let  $\lambda$  be the thermal conductivity of the mixture ( $\text{J m}^{-1} \text{s}^{-1} \text{K}^{-1}$ ).

The rate at which energy enters the + face of the control volume due to heat conduction arising from a temperature gradient  $(\partial T / \partial y)_+$  at the + face is  $(\lambda \partial T / \partial y)_+ \delta A$ . In addition, energy is carried into  $\delta V$  through the + face by species  $j$ . This occurs at a rate  $(v + V_j)_+ (\rho Y_j h_j)_+ \delta A$ . Equating the rate of change in the total energy in  $\delta V$  to the difference between the amount entering and leaving by heat conduction and the amounts carried in and out by each species, we get,

$$\delta(\rho h) \delta V / \delta t = \delta A \left[ (\lambda \partial T / \partial y)_+ - (\lambda \partial T / \partial y)_- \right] - \delta A \left\{ \sum_j (v + V_j)_+ (\rho Y_j h_j)_+ - \sum_j (v + V_j)_- (\rho Y_j h_j)_- \right\}.$$

Dividing this equation by  $\delta V$  and going to derivative notation, we obtain the energy conservation equation,

$$\partial(\rho h) / \partial t = \frac{\partial}{\partial y} (\lambda \partial T / \partial y) - \frac{\partial}{\partial y} \left( \sum_j (v + V_j) Y_j \rho h_j \right). \quad (\text{A3})$$

## APPENDIX B.

## DETERMINATION OF THE VELOCITY OF A STEADILY PROPAGATING FLAME

The equations derived in Appendix A give the time dependence per unit volume of the total mass, the masses of each chemical species, and the total energy in the system. For a steadily propagating flame, the time derivatives will be zero in a coordinate system which moves with the velocity of the flame front. Let us consider the consequences of allowing the time derivatives in Eqs. A1, A2, and A3 to vanish. From A1 we get  $\partial(\rho v)/\partial y = 0$ ; thus  $\rho v = M$  is constant throughout the flame.  $M$  is the mass flow rate in the steady flame ( $\text{kg m}^{-2} \text{s}^{-1}$ ).

From A2, we get,

$$\frac{\partial}{\partial y} (v + V_j)(\rho Y_j) = R_j.$$

Integrating this equation from the hot to the cold side of the flame gives,

$$\int_H^C \frac{\partial}{\partial y} (v + V_j)(\rho Y_j) dy = \int_H^C R_j dy = (v + V_j)_C (\rho Y_j)_C - (v + V_j)_H (\rho Y_j)_H.$$

If the integration extends sufficiently far into the hot and cold regions, the diffusion velocities  $(V_j)_C$  and  $(V_j)_H$  become negligible compared to  $v_C$  and  $v_H$  since the concentration gradients become small. Thus, we can write this equation as,

$$\int_H^C R_j dy = (\rho v Y_j)_C - (\rho v Y_j)_H = M(Y_{jC} - Y_{jH}),$$

since  $(\rho v)_C = (\rho v)_H = M$  from the steady-state mass conservation equation A1. Any of the integrals  $\int R_j dy$  can be used to calculate  $M$  or  $v_C$ , which is the steady flame velocity referred to the cold side of the flame.

Thus, the equation for  $v_C$  is,

$$v_C = \int_H^C R_j dy / [\rho_C (Y_{jC} - Y_{jH})]. \quad (A4)$$

The solution of the flame equations in the steady-state limit yields the mass fractions  $Y_j$  of each chemical species as a function of  $y$ , the distance through the flame. From these values the mass production rates

$R_j$  can be calculated as a function of  $y$ . If the flame equations have been solved correctly, each of the integrals  $\int R_j dy$  should yield the same value for  $v_C$ .

Consider now an adiabatic steadily propagating flame. In this situation there is no heat conducted through the cold or hot boundaries.

Thus the temperature gradients vanish at the boundaries; i.e.,

$(\partial T / \partial y)_C = (\partial T / \partial y)_H = 0$ . From this, together with the condition

$(V_j)_C = (V_j)_H = 0$ , we get from the steady-state form of the energy

equation A3, the relation,

$$\sum_j (\rho v Y_j h_j)_C = \sum_j (\rho v Y_j h_j)_H.$$

Since  $\rho v$  is a constant this becomes,

$$\sum_j (Y_j h_j)_C = \sum_j (Y_j h_j)_H, \quad \text{or simply, } h_C = h_H.$$

Therefore, in an adiabatic flame, the enthalpy of the gas mixture is the same at the hot and cold boundaries. Inside the flame, however, the enthalpy will generally be different from  $h_C$  because  $\partial T / \partial y$  and the diffusion velocities  $V_j$  are not zero there. There exists, however, a condition in which  $h$  remains constant throughout the flame. If the energy flux from heat conduction  $\lambda \partial T / \partial y$  equals the energy flux from diffusion  $\rho \sum_j V_j Y_j h_j$  at each point in the flame, then we see from A3 that the quantity  $\rho v h$  is constant throughout the flame. Since we know from A1 that  $\rho v$  is invariant, then  $h$  must also be invariant. This condition is equivalent to assuming that the Lewis number  $D_p C_p / \lambda$  for the mixture is equal to unity at all points in the flame. The reason for this equivalence will be discussed in Appendix G. Although this condition is only approximately satisfied by real flames, it is often invoked to avoid having to solve the energy equation.

## APPENDIX C.

## CALCULATION OF DIFFUSION VELOCITIES

To solve the species conservation equations it is necessary to assume that Fick's law adequately represents the mass diffusion processes in the flame. This law assumes that a particular diffusion velocity  $V_j$  is proportional to the concentration gradient of species  $j$  only. If we consider each species in the mixture in turn to be one component of a binary mixture, with all the other species lumped together as the other component, then it is possible to derive an expression for the proportionality factor in Fick's law from the kinetic theory of gases. This factor will be a function of all the possible binary diffusion coefficients in the mixture and the concentrations of all the species.

According to Fick's law, the diffusion velocity of species  $j$  is given by the equation,

$$V_j = - \Delta_j x_j^{-1} \partial x_j / \partial y, \quad (A5)$$

where  $\Delta_j$  is an effective diffusion coefficient for species  $j$  in the mixture, and  $x_j$  is its mole fraction. Our task is to find an expression for  $\Delta_j$  in terms of the mole fractions  $x_j$  of all the species and the binary diffusion coefficients  $D_{ji}$  for all possible pairs of species in the mixture.

The kinetic theory of gases<sup>4</sup> gives the following expression for  $\partial x_j / \partial y$  in terms of the binary diffusion coefficients and the diffusion velocities of the different species  $j$ ;

$$\frac{\partial x_j}{\partial y} = x_j \sum_{i \neq j} \frac{x_i}{D_{ji}} (V_i - V_j) = x_j \left[ \sum_{i \neq j} \frac{x_i V_i}{D_{ji}} - V_j \sum_{i \neq j} \frac{x_i}{D_{ji}} \right]. \quad j=1, n \quad (A6)$$

These equations are subject to the condition,

$$\sum_j x_j V_j = 0. \quad (A7)$$

We now assume that all of the diffusion velocities except  $V_j$  have the same value  $V^*$ . Then A6 becomes,

$$\partial x_j / \partial y = x_j (V^* - V_j) \sum_{i \neq j} x_i / D_{ji}. \quad (A8)$$

From A7 we get the relation,

$$V^* = -x_j V_j (1 - x_j)^{-1}.$$

Substituting this expression for  $V^*$  into A8 gives,

$$\frac{\partial x_j}{\partial y} = - \left( \frac{x_j}{1 - x_j} \right) V_j \sum_{i \neq j} \frac{x_i}{D_{ji}}.$$

Solving for  $V_j$ , we get

$$V_j = - \frac{(1 - x_j)}{\sum_{i \neq j} \frac{x_i}{D_{ji}}} \frac{1}{x_j} \frac{\partial x_j}{\partial y}.$$

The proportionality factor  $\Delta_j$  in Fick's law is thus,

$$\Delta_j = \frac{(1 - x_j)}{\sum_{i \neq j} \frac{x_i}{D_{ji}}}. \quad (A9)$$

## APPENDIX D.

## CALCULATION OF THE THERMAL CONDUCTIVITY OF THE FLAME

An empirical equation was used to calculate the thermal conductivity of the mixture of gases in the flame. This was developed by Lindsay and Bromley<sup>5</sup>. It requires a knowledge of the pure component conductivities, heat capacities, boiling points, and molecular weights.

The expression for  $\lambda$  is,

$$\lambda = \sum_i \lambda_i \left[ x_i \sum_j A_{ij} x_j \right]^{-1}, \quad (\text{A10})$$

where  $\lambda_i$  is the thermal conductivity of species  $i$ , and the quantities  $A_{ij}$  are given by the formula,

$$A_{ij} = \frac{1}{4} \left\{ 1 + \left[ \frac{\mu_i \left( \frac{M_j}{M_i} \right)^{\frac{3}{4}}}{\mu_j \left( \frac{M_i}{M_j} \right)^{\frac{3}{4}}} \frac{\left( 1 + \frac{S_i}{T} \right)^{\frac{1}{2}}}{\left( 1 + \frac{S_j}{T} \right)} \right]^2 \frac{\left( 1 + \frac{S_{ij}}{T} \right)}{\left( 1 + \frac{S_i}{T} \right)} \right\}. \quad (\text{A11})$$

The viscosity ratio  $\mu_i/\mu_j$  is given by the expression,

$$\mu_i/\mu_j = (\lambda_i/\lambda_j)(C_{pj} + 1.25R/M_j)(C_{pi} + 1.25R/M_i)^{-1},$$

where  $C_{pi}$  and  $C_{pj}$  are the heat capacities, and  $M_i$  and  $M_j$  the molecular weights of species  $i$  and  $j$ .  $R$  is the gas constant. The quantities  $S_i$  and  $S_j$  are the Sutherland constants of  $i$  and  $j$ . For hydrogen, the value is  $79^{\circ}\text{K}$ . For other pure gases it is taken to be  $1.5T_B$  where  $T_B$  is the boiling point at 1 atmosphere pressure. The constant  $S_{ij}$  is the geometric mean of  $S_i$  and  $S_j$  except when one gas is strongly polar; then  $S_{ij} = 0.733(S_i S_j)^{\frac{1}{2}}$ .

## APPENDIX E.

## VON MISES TRANSFORMATION OF THE FLAME EQUATIONS

Preliminaries

Before applying the von Mises transformation to the flame equations let us insert Fick's diffusion formula into the species conservation equation and eliminate temperature from the energy equation.

Substitution of A5, the expression for  $V_j$ , into the species conservation equations A2 yields the equation,

$$\rho \frac{\partial Y_j}{\partial t} + \rho v \frac{\partial Y_j}{\partial y} = \frac{\partial}{\partial y} \left( \Delta_j \rho \frac{Y_j}{x_j} \frac{\partial x_j}{\partial y} \right) + R_j.$$

Using the identity,

$$\frac{Y_j}{x_j} \frac{\partial x_j}{\partial y} = \frac{\partial Y_j}{\partial y} + \frac{Y_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial y},$$

where  $\langle M \rangle = \sum_j x_j M_j$  is the average molecular weight of the mixture we obtain,

$$\rho \frac{\partial Y_j}{\partial t} + \rho v \frac{\partial Y_j}{\partial y} = \frac{\partial}{\partial y} \left( \Delta_j \rho \frac{\partial Y_j}{\partial y} \right) + \frac{\partial}{\partial y} \left( \Delta_j \rho \frac{Y_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial y} \right) + R_j. \quad (A12)$$

Consider next the energy equation (A3). The enthalpy  $h$  is a function of the mass fractions  $Y_j$  and the temperature  $T$ ; i.e.,  $h = f(Y_j, T)$ .

The partial derivative of  $h$  with respect to  $y$  is,

$$\frac{\partial h}{\partial y} = \sum_j \frac{\partial h}{\partial Y_j} \frac{\partial Y_j}{\partial y} + \frac{\partial h}{\partial T} \frac{\partial T}{\partial y} = \sum_j \frac{\partial Y_j h_j}{\partial y} + C_p \frac{\partial T}{\partial y}, \quad (A13)$$

where we used the relations  $\partial h / \partial Y_j = h_j$  and  $\partial h / \partial T = C_p$ , the heat capacity of the mixture at constant pressure. Solving A13 for  $\partial T / \partial y$ , and substituting this as well as A5 into the energy conservation equation A3, we get after some algebraic manipulation,

$$\rho \frac{\partial h}{\partial t} + \rho v \frac{\partial h}{\partial y} = \frac{\partial}{\partial y} \left( \frac{\lambda}{C_p} \frac{\partial h}{\partial y} \right) + \frac{\partial}{\partial y} \left\{ \sum_j h_j \left[ \frac{\Delta_j \rho}{\langle M \rangle} \frac{\partial (Y_j \langle M \rangle)}{\partial y} - \frac{\lambda}{C_p} \frac{\partial Y_j}{\partial y} \right] \right\}. \quad (A14)$$

We now change from mass fractions to a different concentration variable  $\varphi_j$  defined as

$$\varphi_j \equiv Y_j/M_j. \quad (\text{A15})$$

This has units of moles/kg. The species and energy conservation equations then take the form,

$$\rho \frac{\partial \varphi_j}{\partial t} + \rho v \frac{\partial \varphi_j}{\partial y} = \frac{\partial}{\partial y} \left( \Delta_j \rho \frac{\partial \varphi_j}{\partial y} \right) + \frac{\partial}{\partial y} \left( \Delta_j \rho \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial y} \right) + \frac{R_j}{M_j} \quad (\text{A16})$$

$$\rho \frac{\partial h}{\partial t} + \rho v \frac{\partial h}{\partial y} = \frac{\partial}{\partial y} \left( \frac{\lambda}{C_p} \frac{\partial h}{\partial y} \right) + \frac{\partial}{\partial y} \left\{ \sum_j h_j^* \left[ \frac{\Delta_j \rho}{\langle M \rangle} \frac{\partial (\varphi_j \langle M \rangle)}{\partial y} - \frac{\lambda}{C_p} \frac{\partial \varphi_j}{\partial y} \right] \right\} \quad (\text{A17})$$

In A17 the quantities  $h_j^* = h_j M_j$  are the enthalpies of the pure species in J/mole.

#### Von Mises Transformation

We introduce a new spatial variable  $\psi$  in place of  $y$ . It is defined by the equations,

$$\partial \psi / \partial y = \rho, \quad \text{and} \quad \partial \psi / \partial t = -\rho v. \quad (\text{A18})$$

Applying this transformation to A16 and A17 eliminates the convection terms  $\rho v \partial \varphi_j / \partial y$  and  $\rho v \partial h / \partial y$ . The mass conservation equation A1 is satisfied automatically because of the relations,

$$\partial \rho / \partial t = \partial^2 \psi / \partial t \partial y = \partial^2 \psi / \partial y \partial t = -\partial(\rho v) / \partial y.$$

We are left with only the species and energy equations in the form,

$$\frac{\partial \varphi_j}{\partial t} = \frac{\partial}{\partial \psi} \left( \Delta_j \rho^2 \frac{\partial \varphi_j}{\partial \psi} \right) + \frac{\partial}{\partial \psi} \left( \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \psi} \right) + \frac{R_j}{\rho M_j} \quad (\text{A19})$$

$$\frac{\partial h}{\partial t} = \frac{\partial}{\partial \psi} \left( \frac{\lambda \rho}{C_p} \frac{\partial h}{\partial \psi} \right) + \frac{\partial}{\partial \psi} \left\{ \sum_j h_j^* \frac{\Delta_j \rho^2}{\langle M \rangle} \frac{\partial (\varphi_j \langle M \rangle)}{\partial \psi} - \frac{\lambda \rho}{C_p} \sum_j h_j^* \frac{\partial \varphi_j}{\partial \psi} \right\} \quad (\text{A20})$$

## APPENDIX F.

## SPALDING'S TRANSFORMATION OF THE FLAME EQUATIONS

The technique developed by Patankar and Spalding for solving the two-dimensional boundary-layer equations can be applied to the solution of the flame equations. To apply this method, a new spatial variable  $\omega$  is introduced. This is defined by the equation,

$$\omega = (\psi - \psi_H) / (\psi_C - \psi_H). \quad (A21)$$

The quantities  $\psi_C$  &  $\psi_H$  are the values of  $\psi$  at the cold and hot boundaries of the flame. They are functions of time only. The advantage of using  $\omega$  is that if the calculation procedure is limited to the range  $0 \leq \omega \leq 1$ , then it is automatically limited to that part of space where the important changes in the dependent variables  $\varphi_j$  and  $h$  occur.

After the transformation, the species and energy equations A19 and A20 become,

$$\frac{\partial \varphi_j}{\partial t} + \left( \frac{a + b\omega}{\eta} \right) \frac{\partial \varphi_j}{\partial \omega} = \frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left( \Delta_j \rho^2 \frac{\partial \varphi_j}{\partial \omega} \right) + \frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left( \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \omega} \right) + \frac{R_j}{\rho M_j} \quad (A22)$$

$$\begin{aligned} \frac{\partial h}{\partial t} + \left( \frac{a + b\omega}{\eta} \right) \frac{\partial h}{\partial \omega} &= \frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left( \frac{\lambda \rho}{C_p} \frac{\partial h}{\partial \omega} \right) \\ &+ \frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left\{ \sum_j h_j^* \left[ \frac{\Delta_j \rho^2}{\langle M \rangle} \frac{\partial (\varphi_j \langle M \rangle)}{\partial \omega} - \frac{\lambda \rho}{C_p} \frac{\partial \varphi_j}{\partial \omega} \right] \right\} \end{aligned} \quad (A23)$$

where  $\eta = \psi_C - \psi_H$ . In making the transformation we have used the relation for  $\partial \omega / \partial t$  in the  $\psi, t$  coordinate system,  $\partial \omega / \partial t = (a + b\omega) / \eta$ .

The quantities  $a$  and  $b$  are defined by,

$$a = -d\psi_H/dt, \quad b = -d\psi_C/dt + d\psi_H/dt.$$

The derivatives  $d\psi_C/dt$  and  $d\psi_H/dt$  are the mass flow rates across the cold and hot boundaries. In the steady-state flame they have the same constant

value. When this limit is reached  $\partial\varphi_j/\partial t \rightarrow 0$ ,  $b \rightarrow 0$ , and A22, integrated from  $\omega = 0$  to 1, becomes,

$$\rho_C v_C (\varphi_C - \varphi_H) = \eta \int_0^1 \frac{R_j}{\rho M_j} d\omega. \quad (\text{A24})$$

This relation arises because  $a = \rho_H v_H = \rho_C v_C$  in the steady-state limit (see A18); also the gradients of  $\varphi_j$  and  $\langle M \rangle$  are assumed to vanish at the hot and cold boundaries. This gives an equation for  $v_C$  in terms of  $R_j$  expressed in the  $\omega, t$  coordinate system which is analogous to A4 derived for  $R_j$  in the  $y, t$  system. For  $v_C$  we have,

$$v_C = \left( \eta \int_0^1 \frac{R_j}{\rho} d\omega \right) \left[ \rho_C M_j (\varphi_C - \varphi_H) \right]^{-1}. \quad (\text{A25})$$

## APPENDIX G.

## UNITY LEWIS NUMBER APPROXIMATION

In Appendix B we demonstrated that in the adiabatic flame, if the energy flux from heat conduction equals that from species diffusion at each point in the flame, then the enthalpy remains constant throughout the flame. We stated that this assumption was equivalent to the assumption that the Lewis number was unity at all points. We shall now demonstrate how this equivalence arises.

The Lewis number is defined for all species pairs as,

$$Le_{ij} \equiv \rho C_p D_{ij} / \lambda,$$

where  $D_{ij}$  is the binary diffusion coefficient for species pair  $i,j$  and  $C_p$  is the average specific heat at constant pressure for the mixture.

If we assume that  $Le = 1$  for all species pairs, then we must assume that all the binary diffusion coefficients  $D_{ij}$  are equal. Suppose that  $D_{ij} = D$ . When this is substituted into A9, the expression for  $\Delta_j$ , we get,

$$\Delta_j = \frac{1 - x_j}{D \sum_{i \neq j} x_i} = D, \quad \text{since } 1 - x_j = \sum_{i \neq j} x_i.$$

Thus, the assumption  $Le = 1$  gives,

$$\rho C_p D / \lambda = \rho C_p \Delta_j / \lambda = 1; \quad \text{or} \quad D \rho^2 = \Delta_j \rho^2 = \lambda \rho / C_p.$$

Consider the energy equation A23 in the  $\omega, t$  coordinate system under steady-state conditions. Here,  $\partial h / \partial t = 0$ ,  $b = 0$ , and  $h$  is a function of  $\omega$  only. We must make one further assumption, namely that the change in the average molecular weight  $\langle M \rangle$  is negligible throughout the flame. Then the last term in A23 vanishes when  $\Delta_j \rho^2 = \lambda \rho / C_p$  and in the steady-state A23 becomes,

$$a \frac{dh}{d\omega} = \frac{1}{\eta} \frac{d}{d\omega} \left( \frac{\lambda \rho}{C_p} \frac{dh}{d\omega} \right). \quad (\text{A26})$$

Next, insert into A26 the variable  $U \equiv dh/d\omega$ . We get,

$$aU = \frac{1}{\eta} \frac{d}{d\omega} \left( \frac{\lambda\rho}{C_p} \right) U + \frac{1}{\eta} \frac{\lambda\rho}{C_p} \frac{dU}{d\omega}.$$

Solving for  $dU/d\omega$  we obtain,

$$\frac{1}{U} \frac{dU}{d\omega} = \frac{\eta C_p}{\lambda\rho} \left[ a - \frac{1}{\eta} \frac{d}{d\omega} \left( \frac{\lambda\rho}{C_p} \right) \right] \equiv f(\omega).$$

Integrating this expression starting from the cold side of the flame we get,

$$\ln(U/U_0) = \int_0^\omega f(\omega) d\omega, \quad \text{or } U = U_0 \exp \left[ \int_0^\omega f(\omega) d\omega \right], \quad (\text{A27})$$

where  $U_0$  is the value of  $dh/d\omega$  at the cold boundary. For the adiabatic flame  $U_0 = 0$  at the cold side; from A27 we see that if  $U_0 = 0$ , then  $U = dh/d\omega$  is zero for all values of  $\omega$ . Thus  $h$  remains unchanged throughout the flame.

Therefore, if the enthalpy is to remain constant in the adiabatic flame we must assume that  $\Delta_j \rho^2 = \lambda\rho/C_p$  and that  $\frac{d\langle M \rangle}{d\omega} = 0$ ; it is not necessary to assume that  $\lambda\rho/C_p$  remains constant.

## APPENDIX H.

## FINITE-DIFFERENCE FORMS OF THE FLAME EQUATIONS

The species and energy conservation equations A22 and A23 have the same general form;

$$\frac{\partial \Phi}{\partial t} + (\alpha + \beta \omega) \frac{\partial \Phi}{\partial \omega} = \frac{\partial}{\partial \omega} \left( \gamma \frac{\partial \Phi}{\partial \omega} \right) + \mathcal{S}_m + \mathcal{S}, \quad (\text{A28})$$

where  $\alpha = a/\eta$ ,  $\beta = b/\eta$ ,  $\gamma$  is the transport coefficient,  $\mathcal{S}_m$  is a source term arising from the diffusion of average molecular weight, and  $\mathcal{S}$  is a source term arising from chemical reaction in the species equation, and from the difference between conductive and diffusional transport of enthalpy in the energy equation.  $\gamma$ ,  $\mathcal{S}_m$ , and  $\mathcal{S}$  are shown explicitly below;

	Energy	Species
$\gamma$	$\frac{1}{\eta^2} \frac{\lambda \rho}{c_p}$	$\frac{1}{\eta^2} \Delta_j \rho^2$
$\mathcal{S}_m$	$\frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left( \sum_j h_j^* \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \omega} \right)$	$\frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left( \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \omega} \right)$
$\mathcal{S}$	$\frac{1}{\eta^2} \frac{\partial}{\partial \omega} \left[ \sum_j h_j^* \left( \Delta_j \rho^2 - \frac{\lambda \rho}{c_p} \right) \frac{\partial \varphi_j}{\partial \omega} \right]$	$\frac{R_j}{\rho M_j}$

To set up a finite-difference equation for A28, we divide the  $\omega$  axis between 0 and 1 into  $N$  strips which need not have the same widths. The profile of the dependent variable  $\Phi$  is assumed to be linear between grid points. We consider control volumes whose boundaries lie midway between the grid points. A portion of the grid is shown in Fig. A2. This shows the control volume and illustrates the linear-profile assumption. Equation A28 is now integrated over the control volume shown in Fig. A2. This leads to the following equation;

$$\begin{aligned}
& \left[ \int_{-}^{+} \Phi \, d\omega - \int_{-}^{+} \Phi_P \, d\omega \right] / \delta t + \left[ \{(\alpha + \beta\omega)\Phi\}_{+} - \{(\alpha + \beta\omega)\Phi\}_{-} \right] - \beta \int_{-}^{+} \Phi \, d\omega \\
& = \left[ \left( \gamma \frac{\partial \Phi}{\partial \omega} \right)_{+} - \left( \gamma \frac{\partial \Phi}{\partial \omega} \right)_{-} \right] + \int_{-}^{+} \mathcal{S}_m \, d\omega + \int_{-}^{+} \mathcal{S} \, d\omega \quad (A29)
\end{aligned}$$

The subscript P on  $\Phi$  is used to denote its value before the time step  $\delta t$  is taken. An unscripted  $\Phi$  refers to its value after the time step.

It can be shown after some surprisingly lengthly algebra that the integral over the control volume of any function like  $\Phi$ , which is assumed to be linear between grid points has the value,

$$\int_{-}^{+} \Phi \, d\omega = \frac{1}{8} (3\Omega\Phi + \Omega_+\Phi_{++} + \Omega_-\Phi_{--}), \quad (A30)$$

where  $\Omega = \omega_{i+1} - \omega_{i-1}$ ,

$$\Omega_+ = \omega_{i+1} - \omega_i$$

$$\Omega_- = \omega_i - \omega_{i-1}.$$

Note that  $\omega_+ = \frac{1}{2}(\omega_{i+1} + \omega_i)$

and  $\omega_- = \frac{1}{2}(\omega_i + \omega_{i-1})$ . We introduce some additional symbols

$$P \equiv \eta / \delta t$$

$$G \equiv -\eta\beta = b$$

$$L_+ \equiv \eta(\alpha + \beta\omega_+) = a - \omega_+G$$

$$L_- \equiv a - \omega_-G \quad (A31)$$

The derivatives of  $\Phi$  at the control volume boundaries are approximated by

$$\left( \frac{\partial \Phi}{\partial \omega} \right)_{+} = \frac{\Phi_{++} - \Phi}{\omega_{i+1} - \omega_i}, \quad \text{and} \quad \left( \frac{\partial \Phi}{\partial \omega} \right)_{-} = \frac{\Phi - \Phi_{--}}{\omega_i - \omega_{i-1}} \quad (A32)$$

Next, we define  $T_+$  and  $T_-$  by the equations

$$\eta \left( \gamma \frac{\partial \Phi}{\partial \omega} \right)_{+} = T_+(\Phi_{++} - \Phi), \quad \text{and} \quad \eta \left( \gamma \frac{\partial \Phi}{\partial \omega} \right)_{-} = T_-(\Phi - \Phi_{--}). \quad (A33)$$

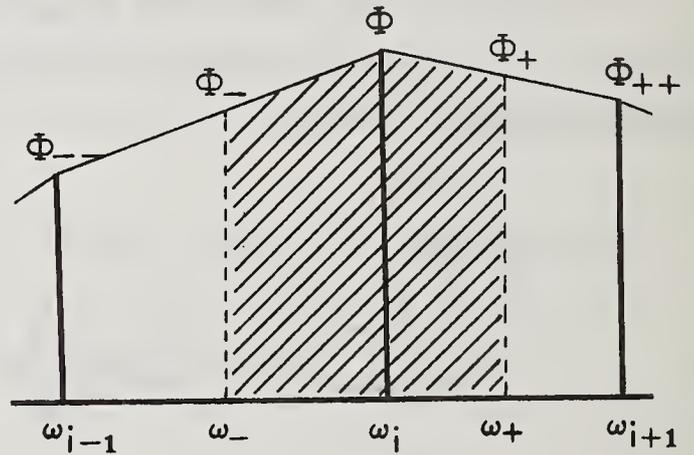


Figure A2.

Thus we have

$$T_+ = \eta\gamma/(\omega_{i+1} - \omega_i) \quad \text{and} \quad T_- = \eta\gamma/(\omega_i - \omega_{i-1}). \quad (\text{A34})$$

We further define

$$S_m \equiv \eta \int_{-}^{+} \mathcal{S}_m d\omega \quad \text{and} \quad S \equiv \eta \int_{-}^{+} \mathcal{S} d\omega. \quad (\text{A35})$$

For the species chemical source term  $\mathcal{S}$  only, we take  $S$  to have the form

$S(\text{species}) = S_P + S_F \Phi$ , where  $\Phi$  is the value after the time step  $\delta t$  is taken.  $S$ , and  $S_F$  will be functions of all the  $\Phi$  for the different species, but the values will be taken as the prestep ones. For all of the other source terms, only the prestep values of the variables are used. Thus for these,  $S_F = 0$ .

Let us then generalize  $S_P$  somewhat, and consider it to be compounded as follows;

$$S_P = S_m(\text{species}) + S_m(\text{energy}) + S(\text{energy}) + S_P(\text{species})$$

$$S_F = S_F(\text{species}) \quad (\text{A36})$$

Thus,

$$S_m + S = S_P + S_F \Phi, \quad (\text{A37})$$

where  $S_P$  and  $S_F$  are defined by A36.

We now multiply A29 by  $\eta$  and insert into it the quantities defined by A30, A31, A34, A35, and A37. The resulting equation is linear in the poststep values of  $\Phi$ ; it can be solved for  $\Phi$  at a particular grid point in terms of the values of  $\Phi$  at the grid points on each side of it. The equation one obtains has the form

$$\Phi = A \Phi_{++} + B \Phi_{--} + C. \quad (\text{A38})$$

The coefficients  $A$ ,  $B$ , and  $C$  contain only prestep values of  $\Phi$ , and are given by

$$A \equiv A'/D$$

$$B \equiv B'/D$$

$$C \equiv C'/D$$

$$A' \equiv 2T_+ - L_+ - \frac{1}{4}(P + G)\Omega_+ \quad (A40)$$

$$B' \equiv 2T_- + L_- - \frac{1}{4}(P + G)\Omega_-$$

$$C' \equiv \frac{1}{4}P(3\Phi_P\Omega + \Phi_{P++}\Omega_+ + \Phi_{P--}\Omega_-) + 2S_P$$

$$D \equiv 2(T_+ + T_-) + L_+ - L_- + \frac{3}{4}(P + G)\Omega - 2S_F$$

$$= A' + B' + P\Omega - 2S_F$$

According to Spalding, these finite difference coefficients have a defect. This can appear if  $\frac{1}{2}|L_{\pm}|$ , (the convective flux terms), become greater than  $T_{\pm}$ , (the diffusive flux terms). It is not clear to me at the present time why this problem exists. Spalding does discuss the problem at length, so the reader can look there for an explanation. The defect is remedied by replacing  $T_{\pm}$  in the formulas A40 with the quantity  $T_{\pm}^*$  defined by,

$$T_{\pm}^* \equiv \frac{1}{2} \left[ T_{\pm} + \left| \frac{1}{2}L_{\pm} \right| + \left| T_{\pm} - \left| \frac{1}{2}L_{\pm} \right| \right] \right]. \quad (A41)$$

This has the following consequences;

$$i) \text{ If } \left| \frac{1}{2}L_{\pm} \right| \leq T_{\pm}, \text{ then } 2T_{\pm}^* \mp L_{\pm} = 2T_{\pm} \mp L_{\pm}.$$

Thus  $T_{\pm}^* = T_{\pm}$  and the replacement of  $T_{\pm}$  by  $T_{\pm}^*$  has no effect.

$$ii) \text{ If } \frac{1}{2}L_{\pm} > T_{\pm}, \text{ then } 2T_{\pm}^* - L_{\pm} = 0 \text{ and } 2T_{\pm}^* + L_{\pm} = 2L_{\pm}.$$

$$iii) \text{ If } \frac{1}{2}L_{\pm} < -T_{\pm}, \text{ then } 2T_{\pm}^* - L_{\pm} = -2L_{\pm} \text{ and } 2T_{\pm}^* + L_{\pm} = 0.$$

This modification of the formulas A40 has the effect of neglecting the diffusion term in the flame equations whenever it becomes somewhat smaller than the convection term.

Let us now see how the difference equations A38 are solved.

Consider the equation for grid point  $i$ ;

$$\Phi_i = A_i \Phi_{i+1} + B_i \Phi_{i-1} + C_i. \quad (A42)$$

We consider  $N+1$  grid points. (Remember that  $N$  is the number of strips into which the  $\omega$  axis is divided.) The hot boundary is arbitrarily set a grid point  $i = 2$ ; thus the cold boundary will fall at  $i = N+2$ .

The grid points  $i = 1$  and  $N+3$  will not be used in the present calculations.<sup>@</sup>

Equations A42 can be written in the form

$$\Phi_i = A_i^* \Phi_{i+1} + B_i^*, \quad i = 3, N+1 \quad (A43)$$

where

$$A_3^* \equiv A_3$$

$$B_3^* \equiv B_3 \Phi_2 + C_3$$

$$A_i^* \equiv A_i / (1 - B_i A_{i-1}^*)$$

$$B_i^* \equiv (B_i B_{i-1}^* + C_i) / (1 - B_i A_{i-1}^*).$$

After calculating  $A_i^*$  and  $B_i^*$  for  $i = 3, N+1$ , one can easily calculate the

$\Phi_i$  from A43 starting from  $\Phi_{N+1} = A_{N+1}^* \Phi_{N+2} + B_{N+1}^*$  and working down to  $\Phi_3$ . Note that the values of  $\Phi_2$  and  $\Phi_{N+2}$  will be determined by the boundary conditions as discussed in Appendix K.

<sup>@</sup>Spalding uses the grid points  $i=1$  and  $i=N+3$  in the boundary-layer problem in a special way to cope with large gradients in  $\Phi$  at the boundaries. Because such gradients do not arise in the flame equations, this particular treatment is unnecessary. Although we could have started the grid at  $i=1$ , this fact was not realized until later and so the computer program was written with the hot boundary starting at  $i=2$ .

## APPENDIX I.

## CALCULATION OF THE SOURCE TERMS

There are four source terms to be calculated;  $\mathcal{S}_m$  for energy and species arises from the diffusion of average molecular weight; (these are normally quite small and can usually be neglected;) and  $\mathcal{S}$  for energy and species, which arises from the difference between diffusion and conduction in the energy case, and from chemical reaction in the species case. By far the most important source term is that for the species. As described in Appendix H, this term is given special treatment. Because this term can undergo very large changes during a time step it is necessary to estimate at the beginning of a step its value at the end of the time step. The chemical source term for a particular species  $j$  is a function of the concentrations of the various species and the temperature;  $\mathcal{S}_j = \mathcal{S}_j(\varphi_j, \varphi_k, \dots, T)$ . Thus we could approximate a change in  $\mathcal{S}_j$  by a linear expression,

$$\delta \mathcal{S}_j \approx \frac{\partial \mathcal{S}_j}{\partial \varphi_j} \delta \varphi_j + \frac{\partial \mathcal{S}_j}{\partial \varphi_k} \delta \varphi_k + \dots + \frac{\partial \mathcal{S}_j}{\partial T} \delta T.$$

$$\text{Thus } (\mathcal{S}_j)_F \approx (\mathcal{S}_j)_P + \delta \mathcal{S}_j.$$

Instead of using  $(\mathcal{S}_j)_P$ , the prestep value of  $\mathcal{S}_j$ , in the difference equation, we want to use  $(\mathcal{S}_j)_F$ , the poststep value. We should, in principle, use the above equation to calculate  $\delta \mathcal{S}_j$ . This, however, involves the poststep concentrations of the other species as well as  $j$ , and would make the difference equations unpleasantly complex.

What we do then is neglect all of the terms but the first, and write,

$$(\mathcal{S}_j)_F \approx (\mathcal{S}_j)_P + \frac{\partial \mathcal{S}_j}{\partial \varphi_j} \delta \varphi_j.$$

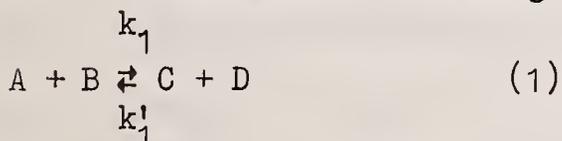
Spalding has found this approximation to be satisfactory. In actuality, we need only a rough estimate, because when the steady-state condition is reached  $\mathcal{S}_j$  ceases to change with time. In fact, it would probably be possible to solve the difference equations using only the prestep value for  $\mathcal{S}_j$ , but then much smaller time steps would be needed.

This treatment is unnecessary for the other source terms because they change little with time.

We shall first show how the chemical source term is calculated. A typical two reaction mechanism will be used to illustrate the method. Next, the source term for the energy flux will be calculated, and finally, the source terms arising from the diffusion of average molecular weight.

a) Calculation of the chemical source term

To show how the species source term is calculated, consider a reaction mechanism consisting of two reactions



Reaction (1) is a second order reaction in both the forward and reverse directions, while reaction (2) is third order in the forward direction and second order in the reverse. These represent the two types of reactions which we shall consider in the flame chemistry.

Consider the rate at which species A is formed. This is

$$\dot{[A]} = -k_1[A][B] + k'_1[C][D] - 2k_2[A]^2[M] + 2k'_2[E][M]. \quad (A43)$$

The bracketed quantities signify concentrations in moles/m<sup>3</sup>. We wish to express  $\mathcal{S}(A)$ , the source term for species A, in terms of the concentration variables  $\varphi$  (moles/kg). From the table on page A15, we see that

$$\mathcal{S}_A = R_A / (\rho M_A) \quad \text{moles kg}^{-1} \text{s}^{-1}.$$

(Remember that  $R_A$  is the mass production rate of A in kg m<sup>-3</sup> s<sup>-1</sup>.)

From a dimensional analysis, we have

$$\mathcal{S}_A = [\dot{A}]/\rho, \quad \text{and} \quad [A] = \rho\varphi_A. \quad (\text{A44})$$

Thus we need only divide A43 by  $\rho$  and substitute for each of the concentrations their equivalent values in terms of  $\varphi$ ; A43 becomes,

$$\mathcal{S}_A = [\dot{A}]/\rho = -k_1\rho\varphi_A\varphi_B + k_1'\rho\varphi_C\varphi_B - 2k_2\rho^2\varphi_A^2\varphi_M + 2k_2'\rho\varphi_E\varphi_M \quad (\text{A45})$$

Let us now calculate  $S(\text{species}) = \eta \int_{-}^{+} \mathcal{S} d\omega$ . To do this we will assume that  $\mathcal{S}$  is constant throughout the control volume. Thus we can write,

$$S(\text{species}) = \eta \mathcal{S}(\omega_+ - \omega_-) = \eta \mathcal{S}_{\frac{1}{2}}(\omega_{++} - \omega_{--}) = \eta \mathcal{S}_{\frac{1}{2}} \Omega \quad (\text{A46})$$

Note that we could have assumed that  $\mathcal{S}$  was linear between the grid points as we did for the concentration variables  $\varphi$  (see Fig. A2). This would have given (see A30),

$$S(\text{species}) = \eta \frac{1}{8} (3\Omega \mathcal{S} + \Omega_+ \mathcal{S}_{++} + \Omega_- \mathcal{S}_{--}) \quad (\text{A47})$$

Thus A47 uses the values of  $\mathcal{S}$  at three grid points to approximate the integral  $\int \mathcal{S} d\omega$ , whereas A46 uses only the value at one point. We first work out the implications of the one point approximation in terms of our two reaction model A42. This is the approximation used in the computer program. Afterward, we shall show how the three point approximation could be incorporated into the calculation.

1) One point approximation to the chemical source term integral.

As discussed earlier in this section, we want to express the chemical source term as the sum of two terms,

$$S(\text{species}) = S_P + S_F\varphi = \frac{1}{2}\eta \mathcal{S} \Omega. \quad (\text{A48})$$

here  $\varphi$  denotes the poststep value of the concentration of the particular species, e.g., species A in A45; and we use A46 to approximate  $S(\text{species})$ . Let us expand  $\mathcal{S}$  in a Taylor series in  $\varphi$  about the point  $\varphi_P$ , the pre-step value.

$$\begin{aligned} \mathcal{S}(\varphi) &\approx \mathcal{S}(\varphi_P) + \left(\frac{d\mathcal{S}}{d\varphi}\right)_P(\varphi - \varphi_P) = \left[\mathcal{S}(\varphi_P) - \left(\frac{d\mathcal{S}}{d\varphi}\right)_P\varphi_P\right] + \left(\frac{d\mathcal{S}}{d\varphi}\right)_P\varphi \\ &= \sigma_P + \sigma_F\varphi, \end{aligned} \quad (\text{A49})$$

where

$$\sigma_P = \mathcal{A}(\varphi_P) - \left(\frac{d\mathcal{A}}{d\varphi}\right)_P \varphi_P \quad \text{and} \quad \sigma_F = \left(\frac{d\mathcal{A}}{d\varphi}\right)_P. \quad (\text{A50})$$

All these terms with the P subscript have prestep values.

As an example, let us calculate  $\sigma_P$  and  $\sigma_F$  for the two reaction model A42. Taking the derivative of A45 with respect to  $\varphi_A$  we get,

$$\sigma_F(A) = [-k_1 \rho \varphi_B - 4k_2 \rho^2 \varphi_M \varphi_A]_P, \quad (\text{A51})$$

and

$$\sigma_P(A) = [k_1' \rho \varphi_C \varphi_D + 2k_2 \rho^2 \varphi_A^2 \varphi_M + 2k_2' \rho \varphi_E \varphi_M]_P. \quad (\text{A52})$$

The subscript P means that all of the quantities have prestep values.

For this case, we thus have,

$$S(A) = \frac{1}{2} \eta \Omega \sigma_P(A) + \frac{1}{2} \eta \Omega \sigma_F(A) (\varphi_A)_F = S_P(A) + S_F(A) (\varphi_A)_F, \quad (\text{A53})$$

where  $(\varphi_A)_F$  is the poststep value of  $\varphi_A$ .

2) Three point approximation to the chemical source term integral.

If we were to approximate  $S(\text{species})$  by A47, the finite difference coefficients of A40 for the species equations become,

$$A' \equiv 2T_+ - L_+ - \frac{1}{4}(P + G)\Omega_+ + \frac{1}{4}\eta\sigma_{F++}\Omega_+$$

$$B' \equiv 2T_- + L_- - \frac{1}{4}(P + G)\Omega_- + \frac{1}{4}\eta\sigma_{F--}\Omega_-$$

$$C' \equiv \frac{1}{4}P(3\varphi_P \Omega + \varphi_{P++}\Omega_+ + \varphi_{P--}\Omega_-) \\ + 2S_m(\text{species}) + \frac{3}{4}\eta\sigma_P \Omega + \frac{1}{4}\eta\sigma_{P++}\Omega_+ + \frac{1}{4}\eta\sigma_{P--}\Omega_-$$

$$D \equiv 2(T_+ + T_-) + L_+ - L_- + \frac{3}{4}(P + G)\Omega - \frac{3}{4}\eta\sigma_F \Omega. \quad (\text{A54})$$

These equations are not significantly more complicated than A40, and should be more accurate. They have not, however, been incorporated into the present computer program.

b) Calculation of the energy source term

The energy source term is easily calculated. From A35 and the table on page A15, we have

$$\begin{aligned}
 S &= \eta \int_{-}^{+} \mathcal{S} d\omega = \frac{1}{\eta} \int_{-}^{+} \frac{\partial}{\partial \omega} \left[ \sum_j h_j^* \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \frac{\partial \varphi_j}{\partial \omega} \right] d\omega \\
 &= \frac{1}{\eta} \left[ \sum_j h_j^* \left( \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \frac{\partial \varphi_j}{\partial \omega} \right) \right]_{+} - \frac{1}{\eta} \left[ \sum_j h_j^* \left( \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \frac{\partial \varphi_j}{\partial \omega} \right) \right]_{-} \quad (A55)
 \end{aligned}$$

As usual the  $\pm$  subscripts refer to values at the control volume boundaries. For the first term we have

$$\left[ h_j^* \left( \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \right) \right]_{+} = \frac{1}{2} \left[ h_j^* \left( \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \right) \right]_{++} + \frac{1}{2} \left[ h_j^* \left( \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \right) \right]_{-}$$

and for the derivative

$$\left( \frac{\partial \varphi_j}{\partial \omega} \right)_{+} = \frac{\varphi_{j++} - \varphi_j}{\omega_{++} - \omega}$$

The - boundary is treated the same way.

c) Calculation of source terms arising from diffusion of molecular weight

From A35 and the table on page A15, we have for the source terms arising from the diffusion of the average molecular weight

$$S_m = \eta \int_{-}^{+} \mathcal{S}_m d\omega = \frac{1}{\eta} \left[ \sum_j h_j^* \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \omega} \right]_{+} - \frac{1}{\eta} \left[ \sum_j h_j^* \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \omega} \right]_{-} \quad (A56)$$

for the energy equation, and

$$S_m = \frac{1}{\eta} \left[ \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \omega} \right]_{+} - \frac{1}{\eta} \left[ \Delta_j \rho^2 \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial \omega} \right]_{-} \quad (A57)$$

for the species equation.

It is worth noting at this point that we have been somewhat careless about how we calculate quantities at the control volume boundaries. Throughout Appendix H and I, we have frequently been confronted with expressions involving products of various quantities

which are to be evaluated at the control volume boundaries. For example, let  $Q_{\pm} = (q_1 q_2 q_3 \dots q_n)_{\pm}$  where the  $q_k$  are various quantities like  $\rho$ ,  $h_j^*$ ,  $\Delta_j$ ,  $\lambda$ , etc. All of these quantities can be expressed in terms of the values of  $\varphi_j$  and  $h$  at the grid points; indeed, this is how they are evaluated at the end of every time step. This, of course, yields their values at the grid points. The values at the control volume boundaries are always taken to be the average of the values at the two grid points on each side of the boundary in question. Thus, for the + boundary we take

$$q_{k+} = \frac{1}{2}(q_{k++} + q_k)$$

To calculate  $Q_+$  we often take products of the  $q_{k+}$ .

$$Q_+ = q_{1+} q_{2+} q_{3+} \dots q_{n+}$$

Actually we should evaluate  $Q_+$  from the expression

$$Q_+ = \frac{1}{2} [(q_1 q_2 q_3 \dots q_n)_{++} + (q_1 q_2 q_3 \dots q_n)]$$

If the  $q_k$  do not change much from grid point to grid point, the two ways of calculating  $Q_+$  would give about the same answer.

Let us consider a case where  $Q$  is a product of two terms and compare the two methods of evaluating  $Q_+$ .

Suppose that

$$q_{1++} = q_1 + \Delta q_1 \quad \text{and} \quad q_{2++} = q_2 + \Delta q_2$$

The correct way to calculate  $Q_+$  gives

$$Q_+ = \frac{1}{2}(q_1 q_2)_{++} + \frac{1}{2}(q_1 q_2) = q_1 q_2 + \frac{1}{2}(q_1 \Delta q_2 + q_2 \Delta q_1) + \frac{1}{2} \Delta q_1 \Delta q_2$$

The approximate method gives

$$Q_+(\text{appr}) = \frac{1}{2}(q_{1++} + q_1) \frac{1}{2}(q_{2++} + q_2) = q_1 q_2 + \frac{1}{2}(q_1 \Delta q_2 + q_2 \Delta q_1) + \frac{1}{4} \Delta q_1 \Delta q_2$$

$$\text{Thus } Q_+(\text{appr}) = Q_+ - \frac{1}{4} \Delta q_1 \Delta q_2$$

For  $\Delta q_k$  values encountered in the present calculations, the difference between the two methods of calculating products at the boundaries is negligible.

## APPENDIX J

## CALCULATION OF THE ENTRAINMENT RATES

The so-called entrainment rates are the mass flow rates across the hot and cold boundaries. In the steady-state flame, these will naturally be equal to each other. During the calculation, however, they are automatically adjusted to keep the grid centered on the flame front. As discussed in Appendix F, the mass flow rate across the hot boundary is

$$a = -d\psi_H/dt \equiv \dot{m}_H$$

Across the cold boundary it is

$$a + b = -d\psi_C/dt \equiv \dot{m}_C$$

Consider a position somewhere near the hot boundary; let  $\varphi_{HH}$  be the value of a concentration of one of the species at this point. Let  $\varphi_{CC}$  be its value at a point near the cold boundary. The entrainment rates are calculated from the following formulas

$$\dot{m}_C = \dot{m}_m \left[ \frac{\varphi_{CC} - \varphi_C}{\varphi_H - \varphi_C} \times 100 \right]^v \quad (A58)$$

$$\dot{m}_H = \dot{m}_m \left[ 2 - \left\{ \frac{\varphi_H - \varphi_{HH}}{\varphi_H - \varphi_C} \times 100 \right\}^v \right] \quad (A59)$$

$$\dot{m}_m \equiv - \left| \eta \int_0^1 \frac{R_j}{\rho M_j} d\omega \right| / |\varphi_H - \varphi_C| \quad (A60)$$

where  $\varphi_C$  and  $\varphi_H$  are the values of  $\varphi$  at the cold and hot boundaries.

Note that  $\varphi_C$ ,  $\varphi_{CC}$ ,  $\varphi_H$ , and  $\varphi_{HH}$  have an implied subscript  $j$  which denotes the species used to calculate the entrainment rates. In principle, any species could be used because the integral in  $\dot{m}_m$  will be the same for all the species. However, it is best to choose one of the major species for this purpose since the numerical evaluation of this integral will not be as accurate if the species is a trace one.

To see how A58 and A59 automatically control the entrainment rates,

and thus the grid width, consider the situation in which the concentrations  $\varphi_{CC}$  and  $\varphi_{HH}$  differed from  $\varphi_C$  and  $\varphi_H$  by 1% of the total difference in  $\varphi$  across the grid. Then  $(\varphi_{CC} - \varphi_C)/(\varphi_H - \varphi_C)$  and  $(\varphi_H - \varphi_{HH})/(\varphi_H - \varphi_C)$  would equal 0.01 and from A58 and A59 we have  $\dot{m}_C = \dot{m}_H = \dot{m}_m$ . This would be the situation in the steady-state. Suppose this condition arose by accident at some early time in the calculation. Suppose also that as the calculation proceeded  $(\varphi_{CC} - \varphi_C)/(\varphi_H - \varphi_C)$  increased. Then  $\dot{m}_C$  would become somewhat greater than  $\dot{m}_m$  and cold gas would be pulled into the flame front at an increased rate. If  $\dot{m}_H$  had not changed, this would mean that the grid would start increasing in width. Increasing the grid width would tend to lower  $\varphi_{CC}$  for future times thus bringing  $\dot{m}_C$  back to  $\dot{m}_m$  thereby slowing down the rate of increase of grid size. Through the use of A58 and A59, the grid is automatically adjusted after every time step in such a way as to make  $(\varphi_{CC} - \varphi_C)/(\varphi_H - \varphi_C)$  and  $(\varphi_H - \varphi_{HH})/(\varphi_H - \varphi_C)$  tend to 0.01. This condition will finally be reached in the steady-state when all the  $\varphi$  cease to change with time.

The parameter  $\nu$  in A58 and A59 is used to control the effect that departures in the 1% condition have on  $\dot{m}_C$  and  $\dot{m}_H$ . For a stable calculation  $\nu$  must be of the order of 0.1. Note that  $\nu$  is purely a numerical device. We desire only to have some relation between  $\dot{m}_{C/H}$  and the  $\varphi$  values such that  $\dot{m}_{C/H} \rightarrow \dot{m}_m$  when the  $\varphi$  reach certain specified values. Other relationships which served a similar purpose could no doubt have been devised to perform the functions of A58 and A59.

It remains to show how the integral  $\int (R_j/\rho M_j) dw$  is calculated. Recalling that  $R_j/\rho M_j = \mathcal{S}_j$  (see page A21), we used the three point approximation A47 to give

$$\int_0^1 \mathcal{S}_j dw = \frac{1}{8}(3\alpha \mathcal{S}_j + \alpha_+ \mathcal{S}_{j++} + \alpha_- \mathcal{S}_{j--}). \quad (\text{A61})$$

The  $\mathcal{S}_j$  values used in A61 are calculated from the values of  $\varphi$  at the end of each time step.

## APPENDIX K

## BOUNDARY CONDITIONS

Consider the finite-difference forms of the flame equations in final form. This is Eq. A43 derived in Appendix H.

$$\Phi_i = A_i^* \Phi_{i+1} + B_i^* \quad (A43)$$

The subscripts refer to the grid points. There is one of these equations for each species and one for the enthalpy. To solve these equations we must assign a value to  $\Phi_{N+2}$ , the grid point on the cold side of the flame. Furthermore, to calculate  $B_3^* \equiv B_3 \Phi_2 + C_3$  we need to specify a value for  $\Phi_2$ . This section is about how  $\Phi_2$  and  $\Phi_{N+2}$  are to be chosen. The values will depend on the boundary conditions. We consider two types of flames; a) the freely propagating flame, and b) a flame stabilized on a burner.

a) Boundary conditions for a freely propagating flame

On the cold side of the species equations we set  $\varphi_{N+2}$  equal to their values in the cold incoming gas. On the hot side, we assume that the concentration gradient vanishes; thus we set  $\varphi_2 = \varphi_3$ . For the enthalpy, we calculate  $h_{N+2}$  from the temperature and composition of the cold gas mixture. Since the freely propagating flame is considered to be adiabatic, the value on the hot side is the same as that on the cold side. Thus we set  $h_2 = h_{N+2}$ .

b) Boundary conditions for a flame stabilized on a burner

The situation here is considerably more complicated than that occurring in the freely propagating flame. Consider the cold side of the grid. Locate the grid point N+2 at the surface of the burner. Figure A3 shows this portion of the grid along with the burner and the last control volume. The burner is considered to be made of some porous material which allows free passage of stable species, but not radicals. Cold reactant gases at a temperature  $T_\infty$  enter the porous

material at a rate  $\dot{m}_\infty$  ( $\text{kg}/\text{m}^2\text{s}$ ). The composition and enthalpy of the mixture at this point is  $\varphi_{j\infty}$  and  $h_\infty$ . Since the porous plug is absorbing heat from the flame, there will be a temperature gradient within it. The temperature of the gas will thus increase to some extent as it passes through the plug. It is also conceivable that the concentrations

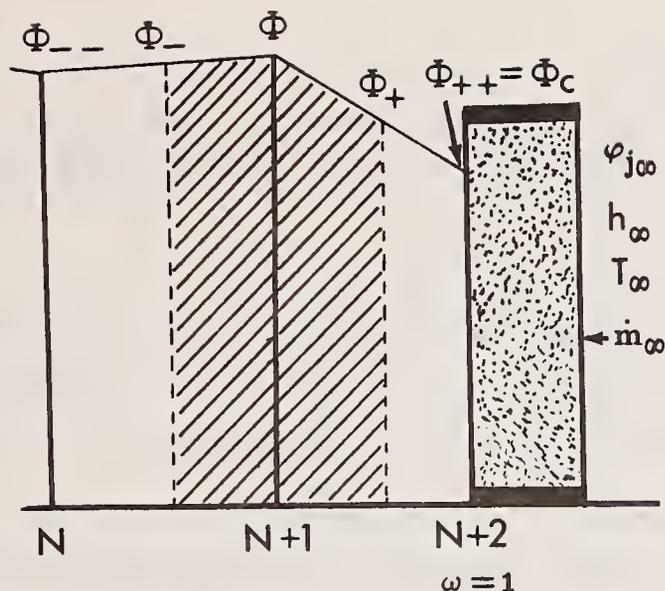


Figure A3.

$\varphi_j$  on the warm side of the plug may be somewhat different from  $\varphi_{j\infty}$  because of diffusion into and from the flame zone. Thus on the warm side of the plug which is located at the last grid point the gas properties have the values  $\varphi_{jC}$ ,  $h_C$ , and  $T_C$ . These values are not known beforehand, but must be determined by solving the flame equations.

Consider first the boundary conditions for the species equations. Suppose  $Y_{jC}$  is the mass fraction of species  $j$  at the flame side of the plug. The mass flux of  $j$  at this point will be  $(v + V_j)_C(\rho Y_j)_C$ . This must be equal to the rate at which  $j$  emerges from the plug. This will be equal to the rate at which  $j$  enters the cold side of the plug, i.e.,  $(\rho v Y_j)_\infty$ . As an additional source or sink for  $j$  we might also consider radicals diffusing out of the flame zone and combining on the burner surface. This effect is likely to be of importance, however, only for H atoms because of their large diffusion coefficient. To include this process in the model we define a surface destruction or production rate for the species  $j$ . Call this  $\Theta_j$ . If  $\Theta_j$  is in units of moles/ $\text{m}^2\text{s}$ , then  $M_j \Theta_j$  will be the mass rate in  $\text{kg}/\text{m}^2\text{s}$ . For the particular case of H atoms recombining to give  $\text{H}_2$ , we can let  $\Theta_H = -B_H (C_H)_C$ , where  $(C_H)_C$  is the concentration of H at the burner surface, and  $B_H$  is a rate constant for surface recombination. We will let negative values of  $\Theta$

correspond to destruction of the species. For the  $H_2$  formed as a result of this, we would have  $\Theta_{H_2} = \frac{1}{2}B_H(C_H)_C$ . In general then, there will be an additional source of  $j$  equal to  $-M_j\Theta_j$ , and so we must have

$$(v + V_j)_C(\rho Y_j)_C = (\rho v Y_j)_\infty - M_j\Theta_j \quad (A62)$$

Since  $v$  is negative and denotes a flux from right to left along the  $y$  axis,  $-M_j\Theta_j$  for  $\Theta_j > 0$  also gives a negative flux which corresponds to production of  $j$ .

Since  $\rho v = \dot{m}_\infty$  throughout the flame (note that  $\dot{m}_\infty$  will be specified as part of the experimental conditions), A62 becomes

$$\dot{m}_\infty Y_{jC} + (\rho V_j Y_j)_C = \dot{m}_\infty Y_{j\infty} - M_j\Theta_j \quad (A63)$$

Inserting the expression for  $V_j$  (Eq. A5) into A63, we get

$$\left( \Delta_j \rho \frac{Y_j}{x_j} \frac{\partial x_j}{\partial y} \right)_C = \dot{m}_\infty (Y_{jC} - Y_{j\infty}) + M_j\Theta_j$$

Using the identity shown on page A9 and dividing the resulting equation by  $M_j$  gives

$$\left( \Delta_j \rho \frac{\partial \varphi_j}{\partial y} + \Delta_j \rho \frac{\varphi_j}{\langle M \rangle} \frac{\partial \langle M \rangle}{\partial y} \right)_C = \dot{m}_\infty (\varphi_{jC} - \varphi_{j\infty}) + M_j\Theta_j$$

If the gradient of  $\langle M \rangle$  is neglected, this becomes

$$\left( \Delta_j \rho \frac{\partial \varphi_j}{\partial y} \right)_C = \dot{m}_\infty (\varphi_{jC} - \varphi_{j\infty}) + M_j\Theta_j$$

Transforming to the  $\omega, t$  system with the relation  $\partial/\partial y = (\rho/\eta)\partial/\partial \omega$  gives

$$\frac{1}{\eta} \left( \Delta_j \rho \frac{\partial \varphi_j}{\partial \omega} \right)_C = \dot{m}_\infty (\varphi_{jC} - \varphi_{j\infty}) + M_j\Theta_j \quad (A64)$$

Since the  $\varphi$  profile is assumed to be linear between grid points (see Fig. A3), the concentration gradient at point C, i.e.,  $i=N+2$ , will be the same as that at the mid-point between  $i=N+1$  and  $N+2$ . Thus

$$\left( \frac{\partial \varphi_j}{\partial \omega} \right)_C = \left( \frac{\partial \varphi_j}{\partial \omega} \right)_+ = \frac{\varphi_{j++} - \varphi_j}{\omega_{++} - \omega} = \frac{\varphi_{jN+2} - \varphi_{jN+1}}{\omega_{N+2} - \omega_{N+1}} \quad (A65)$$

Inserting this into A64 and solving for  $\varphi_{jN+2} = \varphi_{jC}$ , we get

$$\varphi_{jN+2} = \left[ \frac{\Delta_j \rho^2}{\eta(\omega_{N+2} - \omega_{N+1})} \right] \varphi_{jN+1} - \dot{m}_\infty \varphi_{j\infty} + M_j \Theta_j$$

$$\div \left[ \frac{\Delta_j \rho^2}{\eta(\omega_{N+2} - \omega_{N+1})} - \dot{m}_\infty \right] \quad (\text{A66})$$

At the hot boundary we assume  $\partial\varphi/\partial\omega = 0$ , and thus we set  $\varphi_2 = \varphi_3$  as in the freely propagating flame.

Next we consider the boundary conditions on the energy equation. The net flux of energy at the burner surface will be the sum of that caused by the mass flow of the reactant mixture, the flow caused by diffusion of species to and from the surface, and the flow arising from a temperature gradient in the gas at the surface. Let  $(F_h)_C$  be this net energy flux at the burner.

$$(F_h)_C = -\left(\lambda \frac{\partial T}{\partial y}\right)_C + \sum_j \left[ (v + V_j) \rho Y_j h_j \right]_C$$

$$= -\left(\lambda \frac{\partial T}{\partial y}\right)_C + \dot{m}_\infty h_C + \sum_j (V_j \rho Y_j h_j)_C \quad (\text{A67})$$

Introducing the expression for  $V_j$  and the identity on page A9, neglecting the gradient of  $\langle M \rangle$ , converting to  $\varphi_j$  concentration units, and going to the  $\omega, t$  system, we get

$$(F_h)_C = -\frac{1}{\eta} \left( \frac{\lambda \rho}{C_p} \frac{\partial h}{\partial \omega} \right)_C + \dot{m}_\infty h_C - \frac{1}{\eta} \sum_j \left[ \left\{ \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \right\} h_j^* \frac{\partial \varphi_j}{\partial \omega} \right]_C \quad (\text{A68})$$

Since this is the net flux of energy into the cold side of the flame front, it must be equal to the net flux out of the flame on the hot side. Because we assume that  $\partial T/\partial y$  and  $\partial \varphi_j/\partial y = 0$  there, this flux out is  $\dot{m}_\infty h_H$  where  $h_H$  is the enthalpy of the hot gas mixture. Assuming a linear variation of  $h$  between grid points gives

$$\left( \frac{\partial h}{\partial \omega} \right)_C = \left( \frac{\partial h}{\partial \omega} \right)_+ = \frac{h_{N+2} - h_{N+1}}{\omega_{N+2} - \omega_{N+1}} = \frac{h_C - h_{N+1}}{\omega_C - \omega_{N+1}}$$

Inserting this into A68, and setting  $(F_h)_C = \dot{m}_\infty h_H$ , gives

$$\dot{m}_\infty h_H = - \frac{1}{\eta} \left\{ \frac{\lambda \rho}{C_p} \right\}_C \left\{ \frac{h_C - h_{N+1}}{\omega_C - \omega_{N+1}} \right\} + \dot{m}_\infty h_C - \frac{1}{\eta} \sum_j \left[ \left\{ \Delta_j \rho^2 - \frac{\lambda \rho}{C_p} \right\} h_j^* \frac{\partial \varphi_j}{\partial \omega} \right]_C \quad (\text{A69})$$

This equation can be solved to give  $h_{N+2}$  ( $=h_C$ ) in terms of  $h_{N+1}$  and  $h_H$ , and other known quantities. The enthalpy at the hot side can be calculated from the measured final flame temperature and its composition. From  $h_C$  and the calculated composition  $(\varphi_j)_C$  at the burner surface, we can also evaluate  $T_C$ .

## APPENDIX L.

TRANSFORMATION FROM THE  $\omega, t$  COORDINATE SYSTEM BACK TO THE  $y, t$  SYSTEM

The steady-state solution of the species and energy equations A22 and A23 in the  $\omega, t$  coordinate system yields values of  $\varphi_j$  and  $h$  as functions of  $\omega$ , where  $0 \leq \omega \leq 1$ . If we want to compare these profiles with experimental ones we must express  $\varphi_j$  and  $h$  in terms of  $y$ , the laboratory spatial variable.

The variable  $\varphi$  is a function of  $\psi$  and  $t$ , and  $\psi$  is a function of  $y$  and  $t$ ; thus,

$$\omega = \omega(\psi, t) = \omega(\psi(y, t), t)$$

A small change in  $\psi$  can be expressed as

$$d\psi = \frac{\partial \psi}{\partial y} dy + \frac{\partial \psi}{\partial t} dt = \rho dy - \rho v dt$$

Here we have used A18, the expressions defining  $\psi$ . A small change in  $\omega$  can likewise be written

$$\begin{aligned} d\omega &= \frac{\partial \omega}{\partial \psi} d\psi + \frac{\partial \omega}{\partial t} dt = \frac{\partial \omega}{\partial \psi} (\rho dy - \rho v dt) + \frac{\partial \omega}{\partial t} dt \\ &= \frac{\partial \omega}{\partial \psi} \rho dy + \left( -\rho v \frac{\partial \omega}{\partial \psi} + \frac{\partial \omega}{\partial t} \right) dt \end{aligned}$$

Using the facts that  $\frac{\partial \omega}{\partial \psi} = \frac{1}{\eta}$  and  $\frac{\partial \omega}{\partial t} = \frac{a + b\omega}{\eta}$ , we get

$$d\omega = \frac{\rho}{\eta} dy + \left( -\frac{\rho v}{\eta} + \frac{a + b\omega}{\eta} \right) dt$$

In the steady state  $b \rightarrow 0$  and  $a \rightarrow \dot{m}_C = \rho v$ . Therefore  $d\omega \rightarrow \frac{\rho}{\eta} dy$ .

Solving this for  $dy$  and integrating from  $\omega = 0$  (the hot side) to some arbitrary  $\omega$  value, we get

$$\int_{y_H}^y dy = \eta \int_0^\omega \frac{d\omega}{\rho} = y - y_H$$

In the present calculation, we arbitrarily set  $y_H = 0$ . Thus we can calculate the value of  $y$  which corresponds to a particular value of  $\omega$  by evaluating the integral

$$y = \eta \int_0^\omega \frac{d\omega}{\rho}$$

This integral is approximated in the computer program by the summation

$$y_j = 2\eta \sum_{i=2}^j \frac{(\omega_{i+1} - \omega_i)}{\rho_{i+1} - \rho_i}, \quad (\text{A70})$$

where the subscripts refer to the grid points.

## APPENDIX M.

## CALCULATION OF THERMAL AND TRANSPORT PROPERTIES FOR PURE SPECIES

An expression for the effective diffusion coefficient  $\Delta_j$  of a particular species in the flame has been derived in Appendix C (eq.A9). This gives  $\Delta_j$  in terms of the concentrations of the various species and the binary diffusion coefficients for all possible pairs of species. In practice, the summation in the denominator of A9 need not be evaluated over all the species since  $x_i$ , the mole fraction, is large only for the major species. Thus, we must calculate only the binary diffusion coefficients between major species, and between major and minor species. Those coefficients involving pairs of minor species need not be evaluated. In Appendix D we showed how the thermal conductivity of the flame was calculated. The expression used contains the thermal conductivities of the pure species, and also their heat capacities. The heat capacities of the pure species are also needed to calculate  $C_p$ , the average heat capacity of the flame. This quantity occurs in the energy equation. For the energy equation, we also require values for the enthalpies of the pure species. Thus, we must evaluate the following properties involving the pure species (or pairs of species in the diffusion case):

- a. Binary diffusion coefficients  $D_{ij}$ .
- b. Thermal conductivities  $\lambda_j$ .
- c. Heat capacities  $C_{pj}$ .
- d. Enthalpies  $h_j$  (or  $h_j^* = h_j M_j$ ).

a) Binary diffusion coefficients

For the diffusion coefficients, we used Lennard-Jones potentials. This potential function has two parameters  $\sigma_j$  and  $\epsilon_j/k$ .  $\sigma_j$  gives the distance in Å where the potential goes from repulsive to attractive, and  $\epsilon_j/k$  in °K gives the depth of the potential well.  $\sigma_j$  and  $\epsilon_j$  refer to the interaction between two molecules of the same species. For the diffusion coefficients, we require analogous parameters for the interaction between molecules of different species. If  $\sigma_i$ ,  $\sigma_j$  and  $\epsilon_i$ ,  $\epsilon_j$  are the parameters for the species  $i$  and  $j$ , we take as the

parameters for the interaction between  $i$  and  $j$  to be

$$\sigma_{ij} = \frac{1}{2}(\sigma_i + \sigma_j) \quad \text{and} \quad \epsilon_{ij} = \sqrt{\epsilon_i \epsilon_j}.$$

The formula for the binary diffusion coefficient is

$$D_{ij} = \frac{1.86 \times 10^{-7} [(M_i + M_j)/M_i M_j]^{\frac{1}{2}} T^{1.5}}{P \sigma_{ij}^2 \Omega_{ij}(1,1)^*} \quad (\text{A71})$$

where  $P$  is the pressure in atmospheres,  $M_i$  and  $M_j$  are the molecular weights, and  $\Omega_{ij}(1,1)^*$  is a function of a reduced temperature  $T^* = Tk/\epsilon_{ij}$ . To a good approximation this function is proportional to  $T^a$  where  $a$  takes on two constant values, one in the range  $T \geq 3\epsilon_{ij}/k$  and another for  $T < 3\epsilon_{ij}/k$ . We have calculated  $D_{ij}$  from either of the following formulas depending on whether the temperature was greater or less than  $3\epsilon_{ij}/k$ .

$$D_{ij} = \frac{1.66 \times 10^{-7} [(M_i + M_j)/M_i M_j]^{\frac{1}{2}} T^{1.67}}{P \sigma_{ij}^2 (\epsilon_{ij}/k)^{0.17}} \quad T \geq 3\epsilon_{ij}/k \quad (\text{A72})$$

$$D_{ij} = \frac{1.26 \times 10^{-7} [(M_i + M_j)/M_i M_j]^{\frac{1}{2}} T^{1.94}}{P \sigma_{ij}^2 (\epsilon_{ij}/k)^{0.44}} \quad T < 3\epsilon_{ij}/k \quad (\text{A73})$$

These formulas will give  $D_{ij}$  in units of  $\text{m}^2/\text{s}$ .

#### b) Thermal conductivities

Lennard-Jones potentials can also be used to calculate the thermal conductivities of the pure species. The formula for the thermal conductivity is similar to that for the diffusion coefficients. In a like manner we have found that it is possible to express it approximately in terms of a constant power of  $T$ , where the appropriate value of the exponent depends on whether the temperature is above or below  $3\epsilon_j/k$ .

The formulas obtained were,

$$\lambda_j = \frac{1.604 \times 10^{-4} M_j^{-\frac{1}{2}} T^{0.67}}{\sigma_j^2 (\epsilon_j/k)^{0.17}} \quad T \geq 3\epsilon_j/k \quad (\text{A74})$$

$$\lambda_j = \frac{1.231 \cdot 10^{-4} M_j^{-\frac{1}{2}} T^{0.94}}{\sigma_j^2 (\epsilon_j/k)^{0.44}} \quad T < 3\epsilon_j/k \quad (A75)$$

These two formulas give the thermal conductivities in units of cal/cm-deg-s. To be usable in the program they would have to be converted to J/m-deg-s units.

In the present version of the program, experimental thermal conductivity data were used for the major species in the  $H_2-O_2$  flame. The data were fit by a least squares calculation to a power series in  $T$  and the least squares coefficients were used in the program to calculate the  $\lambda_j$  values.

#### c) Heat Capacities

Heat capacity data from the JANAF Tables were fit by least squares calculations to a power series in  $T$ . Terms up to the second power were used. Thus, heat capacities were calculated from the formula

$$C_{pj} = d_j + e_j T + f_j T^2 \quad (A76)$$

where  $d_j$ ,  $e_j$ , and  $f_j$  have values such that the  $C_{pj}$  are in units of J/kg-deg.

#### d) Enthalpies

The enthalpy of a species in J/kg is given by the formula

$$h_j = h_j^R + \int_{T_R}^T C_{pj} dT' \quad (A77)$$

where  $h_j^R$  is the enthalpy at the reference temperature  $T_R$ .

Substituting A76 into A77, we get

$$h_j = (h_j^R - d_j T_R - \frac{1}{2} e_j T_R^2 - \frac{1}{3} f_j T_R^3) + d_j T + \frac{1}{2} e_j T^2 + \frac{1}{3} f_j T^3 \quad (A78a)$$

$$= h_j^O + d_j T + \frac{1}{2} e_j T^2 + \frac{1}{3} f_j T^3 \quad (A78b)$$

This formula was used in the program to calculate  $h_j$  values. Values of  $h_j^O$  were calculated from  $h_j^R$  values taken from the JANAF Tables, the  $T_R$  values, and the heat capacity coefficients  $d_j$ ,  $e_j$ , and  $f_j$ .

APPENDIX N.  
INITIAL PROFILES

The mixture of cold gases through which the flame will propagate is naturally thermodynamically unstable. Nevertheless, in most systems of practical interest the rate of approach to equilibrium at low temperatures is negligibly slow. This is because the free radical reactions which destroy the reactants normally have large activation energies. There are two ways to speed up these reactions. One can raise the temperature of the system or introduce a sufficiently high concentration of the necessary radicals. For a flame steadily propagating into a cold unstable gas mixture, both of these processes occur. The cold gases receive heat by thermal conduction and radicals by molecular diffusion from the flame region. To solve the time dependent flame equations, one must begin with the system in such a state that the radical reactions are proceeding at an appreciable rate. Furthermore, it is desirable to start the calculation with concentration and energy profiles which are similar to those expected in the final steady state to minimize the number of steps required for the integration.

These goals were satisfied by starting the calculations with S-shaped profiles for temperature and for the concentration of the major species. These profiles were obtained by using the following function of  $\omega$ , the spatial variable in the Spalding coordinate system:

$$P_d(\omega) = 10\omega^3 - 15\omega^4 + 6\omega^5 \quad (A79)$$

This has the boundary values,  $P_d(0) = 0$ , and  $P_d(1) = 1$ . Since  $\omega = 0$  corresponds to the hot side of the flame, A79 represents a decay profile. Formation profiles were obtained from the function

$$P_f(\omega) = 10(1-\omega)^3 - 15(1-\omega)^4 + 6(1-\omega)^5 \quad (A80)$$

At the start of the calculation, we specify the mole fractions of a major species on the hot and cold sides of the flame. The former is calculated by assuming that the overall reaction has gone to completion. Also specified is whether the species decays or grows as we go from the cold to the hot side of the flame.

For example, suppose the mole fraction of species  $j$  is  $x_{jC}$  and  $x_{jH}$  for  $\omega = 1$  and  $0$ , and also suppose that  $x_{jC} > x_{jH}$ , i.e., we have a decay profile; then

$$x_j(\omega) = x_{jH} + (x_{jC} - x_{jH})(10\omega^3 - 15\omega^4 + 6\omega^5)$$

To calculate an initial temperature profile, we first calculate the enthalpy on the cold side from the cold side temperature  $T_C$  with the expression,

$$h_C = \sum_{j=j_R+1}^n M_j \varphi_{jC} (h_j^0 + d_j T_C + \frac{1}{2} e_j T_C^2 + \frac{1}{3} f_j T_C^3) \quad (A81)$$

$\varphi_{jC}$  is the concentration of  $j$  on the cold side in units of mole/kg. We then set the enthalpy on the hot side equal to that on the cold side. (Note that  $j_R$  is the index of the last radical species; i.e., we sum over the major species only.) The expression for the hot side enthalpy is

$$\begin{aligned} h_H = h_C &= \sum_{j=j_R+1}^n M_j \varphi_{jH} (h_j^0 + d_j T_H + \frac{1}{2} e_j T_H^2 + \frac{1}{3} f_j T_H^3) \\ &= A + B T_H + \frac{1}{2} C T_H^2 + \frac{1}{3} D T_H^3 \end{aligned} \quad (A82)$$

where

$$A = \sum_{j=j_R+1}^n M_j \varphi_{jH} h_j^0$$

$$B = \sum_{j=j_R+1}^n M_j \varphi_{jH} d_j$$

$$C = \sum_{j=j_R+1}^n M_j \varphi_{jH} e_j$$

$$D = \sum_{j=j_R+1}^n M_j \varphi_{jH} f_j$$

We want to calculate  $T_H$  in terms of  $h_H$ . This was done by finding the appropriate root of the equation

$$F(T_H) = A + B T_H + \frac{1}{2} C T_H^2 + \frac{1}{3} D T_H^3 - h_H = 0$$

A good first approximation to the desired root can be gotten by neglecting the non-linear portion of A82. This gives  $T_H^{(1)} = (h_H - A)/B$ .

The Newton-Raphson method was used to get a better approximation.

The second approximation is

$$\begin{aligned} T_H^{(2)} &= T_H^{(1)} - F(T_H^{(1)})/F'(T_H^{(1)}) \\ &= T_H^{(1)} + \left\{ h_H - (A + BT_H^{(1)} + \frac{1}{2}CT_H^{(1)2} + \frac{1}{3}DT_H^{(1)3}) \right\} / (B + CT_H^{(1)} + DT_H^{(1)2}) \end{aligned}$$

In practice,  $T_H^{(3)}$  was a sufficiently accurate value for  $T_H$ .

The Newton-Raphson method was also used in the main part of the program to calculate the temperature from the enthalpy value at a particular grid point.

An S-shaped starting temperature profile was calculated from the formula

$$T(\omega) = T_C + (T_H - T_C)(10(1-\omega)^3 - 15(1-\omega)^4 + 6(1-\omega)^5)$$

For the enthalpy, a constant starting profile  $h(\omega) = h_C$  was used. It should be noted that this profile is not consistent with the starting temperature profile. The enthalpy profile should be calculated from the temperature and concentration starting profiles. It would be worthwhile to examine what effect, if any, the starting enthalpy profile has on the course of the calculation.

The radical concentrations were set to zero on the cold boundary and remained so throughout the calculation, since the cold boundary concentrations of all species are never modified. Throughout the rest of the flame they initially were set to some constant value comparable to their average expected steady-state concentrations. For flames like  $O_3$  decomposition and  $H_2 - Br_2$  the initial radical concentrations can be set to zero throughout the flame zone. This is possible because the radical generating reactions like  $Br_2 + M \rightleftharpoons 2Br + M$  proceed rapidly on the hot side of the flame. For the  $H_2 - O_2$  flame however, such reactions are too slow for cool flames to initially generate enough radicals. The flame dies out before the radical concentrations become high enough to sustain the flame. Thus, one must begin the calculation with non-zero radical concentrations.

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