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TESTING AND RATING CONVECTORS

Commercial Standard CS140-47

(Issued March 3, 1947) Effective Date for New Production from March 1, 1948



A RECORDED VOLUNTARY STANDARD OF THE TRADE

UNITED STATES DEPARTMENT OF COMMERCE W. AVERELL HARRIMAN, Secretary

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COMMERCIAL STANDARDS

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Organized in 1927, the Division of Trade Standards has assisted many industries in the development of Commercial Standards for a wide variety of commodities. A list of previously established Commercial Standards appears herein.

COMMERCIAL STANDARD FOR TESTING AND RATING CONVECTORS

On March 15, 1945, at the instance of the Convector Manufacturers Association and the Institute of Boiler and Radiator Manufacturers, a proposed commercial standard for testing and rating convectors was circulated to representative user organizations, testing laboratories, Government agencies, distributors and manufacturers for comment. Following adjustment in the light of comment, a recommended commercial standard was circulated on July 19, 1946, to the entire trade for written acceptance.

The trade has since approved the standard as shown herein for issuance by the U. S. Department of Commerce.

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COMMERCIAL STANDARD CS140-47

for

TESTING AND RATING CONVECTORS

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1. GENERAL

PURPOSE

1. The purpose of this commercial standard is to establish standard methods of test for the output of convectors, that is, the condensation capacity of steam convectors and the water heat capacity of hot-water convectors, and methods for determining, designating, and guaranteeing convector ratings.

SCOPE

2. This standard covers definitions, requirements, and methods of testing and rating cast-iron and nonferrous steam and hot-water convectors. It also covers a uniform method of manifesting compliance with the standard and a means for checking convector ratings for approval.

DEFINITIONS

3. Convector.—The term "convector," as used in this standard, shall apply to any type of steam- or water-heated room heater which operates with gravity recirculated room air, which is surrounded on all sides by an enclosure having an air-inlet opening at its bottom end and an air-outlet opening at its top end, and which is installed inside the heated room or recessed in the wall of the heated room.

4. Stack height.—"Stack height," as used in this standard, is the vertical distance measured from the lowest point of the main body of the convector to (a) the upper edge of the free openings of a front outlet grille or opening, or (b) the underside of a horizontal top outlet grille, or (c) the center of an inclined outlet grille.

5. Inside enclosure depth.—"Inside enclosure depth," as used in this standard, is the minimum free distance from the inside of the enclosure front to the inside of the enclosure back, taken at the level of the convector.

6. Inside enclosure length.—"Inside enclosure length," as used in this standard, is the minimum free inside length of the enclosure, taken at the level of the convector. Where baffle plates are used at the end, the inside enclosure length is the minimum distance between the inner edges of such baffle plates.

7. Over-all enclosure height.—"Over-all enclosure height," as used in this standard, is the height from the bottom of the enclosure to its highest point.

8. Opening heights or widths.—"Opening heights or widths" (including inlet and outlet openings or grilles), as used in this standard, is the distance from the lower or front edge of the free opening to the upper or rear edge of the free opening or grille.

9. *Free area.*—"Free area," as used in this standard, is the total minimum area of the opening, or the openings in the grille, through which air can pass.

10. *Capacity.*—"Capacity," as used in this standard, is the condensation capacity determined in accordance with the provisions prescribed in part 2, or the water heat capacity as prescribed in part 3.

11. Rating.

11a. Top outlet convectors.—The rating of a top outlet convector shall be the capacity as defined in paragraph 10.

11b. Front outlet convectors.—The rating of a front outlet convector shall be the capacity, as defined in paragraph 10, plus an addition for heating effect not to exceed the percentage shown in table 3.

11c. *Inclined outlet convectors.*—The rating of an inclined outlet convector shall be the capacity, as defined in paragraph 10, plus an addition for heating effect as prescribed in paragraph 34.

REQUIREMENTS

12. Rating tests.

12a. Tests for determining steam ratings shall conform to the requirements prescribed in part 2 of this standard.

12b. Tests for determining water ratings shall conform to the requirements prescribed in paragraphs 17, 17a, 17b, 17c, 17d, 17e and in part 3 of this standard.

13. Minimum test capacities.—Tests made for rating purposes shall be conducted with convectors of at least 25 square feet steam (240 Btu per square foot) or water (150 Btu per square foot) capacity, except that convectors of less than 25-square-feet capacity may be used for test if the over-all length of the convector exceeds 45 inches, or if the convector is designed to operate as a unit of one size only.

14. Types and depths.—Each convector type and depth shall be tested.

15. *Heights.*—Tests shall be conducted with at least two heights of both top and front outlet enclosures for each convector type and each depth. One of the tested enclosures shall be equal to or lower than the lowest height for that type and depth cataloged by the manufacturer. The maximum enclosure height to be tested shall be equal to or higher than the maximum for that type and depth cataloged by the manufacturer. Intermediate sizes may be tested.

16. Lengths.—One length only of each convector type and depth need be tested.

17. Hot-water tests.—Tests for determining water ratings shall conform to the requirements prescribed in part 3 of this standard.

17a. Water-heating ratings based on a water-temperature drop of 20°F through the convector and including a rating for 170°F average water temperature shall be published by the manufacturer for each hot-water convector. Tests for these ratings shall be made with a water-temperature drop through the convector of 20°F, ± 15 percent, and shall include at least one test made with an average water temperature of not more than 175°F nor less than 165°F.

17b. Hot-water tests shall be made at average temperatures within 10 percent of the minimum and maximum temperature listed by the manufacturer.

17c. Tests shall be made with intervals between average water temperature not exceeding 30°F.

17d. If additional rating data for different temperature drops are also published, the temperature drop shall be stated, and tests shall be made within the range and with the temperature intervals prescribed in this paragraph 17. During these tests the temperature drop shall be maintained within 15 percent of the published temperature drop.

17e. If the water convector is similar in design to the steam convector, water ratings may be expressed in Btu per square foot of catalog steam ratings. A water convector is considered similar to a steam convector if the tubes are similarly arranged and if its fins have substantially the same size, shape, thickness, and spacing, and if the total actual heating surface of the water convector does not vary more than 5 percent from the surface of the corresponding size steam convector. Where this similarity exists, only one size of the water convector need be tested for rating. This tested convector shall be of average depth and length, and shall be tested in an enclosure of approximately 26-inch over-all height and of the same style as used for the steam tests of the equivalent size of steam convector. These water tests shall be made for all water temperatures and temperature drops specified in this paragraph 17. (For method of rating untested hot-water convectors, see paragraph 35b of part 4.)

18. Test room or booth.—Testing of steam convectors and hot-water convectors shall be conducted in one of the following types and sizes of room.

18a. Warm-wall booth.

- (1) A warm-wall booth is a room with one side open, located in a larger room. The floor of the booth shall be at least 1 foot and not more than 4 feet above the larger room floor. The open side of the booth shall have a shield projecting down 1 foot from the ceiling. The air in the booth shall be free from draft, except that created by the convector under test in the course of its normal operation.
- (2) The test booth shall be protected from the influence of uncontrolled heat sources. The distance between any test booth wall and the wall of the surrounding room shall be not less than 2 feet and the ceiling of the test booth shall be not less than 1 foot from the ceiling of the larger room. The air temperature in this larger room shall be taken at

the mid-point of each of the three closed sides of the test booth at a level of 30 inches above the floor of the test booth at a distance of 12 inches from the test booth walls and shall show a variation not to exceed $\pm 3 \deg F$ during the course of a test. This temperature at no time during the test shall be less than 50°F.

- (3) The convector shall be installed in accordance with the manufacturer's instructions, in an enclosure as furnished by the manufacturer or built to the dimensions and of the material specified by the manufacturer. It shall be located at the approximate center of the wall opposite the open side of the booth, and the back of the enclosure shall be placed tightly against this wall.
- 18b. Cooled-wall room.
 - (1) A cooled-wall room is a room with all sides closed, in which the inner portions of walls, ceiling, or floor are cooled by air circulating over the outer surface of the inner wall for the purpose of removing heat and maintaining suitable air temperatures in the test room. The temperature of the inside surfaces of walls, floor, or ceiling at no point shall be lower than 55°F during the test.
 - (2) The convector shall be installed in accordance with the manufacturer's instructions in an enclosure furnished by the manufacturer or built to the dimensions and of the material specified by the manufacturer. It shall be located at the approximate center of a cooled wall. A $\frac{1}{2}$ -inch-thick insulation board of at least the size of the enclosure shall be placed tightly against the wall, and the back of the enclosure shall be placed tightly against this insulation.
 - (3) The convector shall be set on one piece of nonporous, nonmetallic material of not to exceed ¼-inch thickness, which is placed against the wall and projects at least 1 foot beyond the front and sides of the enclosure.
- 18c. Cold room.
 - (1) A cold room is a room with all sides closed, which has two or more walls exposed to an air space having a temperature less than 50° F but not less than -10° F. The walls, floor, and ceiling of this test room must be of customary good building construction. The walls, ceiling, and floor exposed to the cold air shall have a heat-transmission coefficient not to exceed 0.27 Btu per square foot per degree Fahrenheit difference per hour. The walls, floor, and ceiling of such a room not exposed to the cold air shall have a total heat exchange not to exceed 5 percent of the convector capacity. At least one of the exposed walls shall have a window of commercial construction and at least 10-square-feet area with the top of the window stool located approximately 30 inches above the floor. The total exposed window and door area shall not exceed 25 percent of the exposed area, including wall, window, and door.

(2) Convectors shall be installed in accordance with the manufacturer's instructions, in an enclosure furnished by the manufacturer or built to the dimensions and of the material specified by the manufacturer. The back of the enclosure shall be placed tightly against an exposed wall. Convectors with enclosure heights of 26 inches or less shall be located underneath the window center. Convectors with higher enclosures shall be located at an exposed wall, but not immediately in front of the window.

18d. Size and construction.—The test room or booth shall have a floor area of more than 100 square feet and less than 300 square feet, and no side wall shall be less than 9 feet long. The ceiling height shall be not less than 8 feet and not more than 10 feet. The floor shall be tight and constructed of commercial wood flooring. Nonmetallic walls and ceiling shall be used. The inside of the walls and ceiling shall be either plastered or painted with a flat oil paint.

2. METHODS OF TESTING STEAM CONVECTORS

19. Test conditions.

19a. *Relative humidity and air motion.*—It is inadvisable for relative humidity in the test room or booth at the time of test to exceed 75 percent. Air motion shall be limited to natural circulation.

19b. Steam supply.—Steam shall be supplied to the convector at a pressure corresponding to a saturated-steam temperature of not less than 214°F nor more than 217°F and shall have a superheat of not less than 2 deg F nor more than 5 deg F. The steam-supply temperature shall be measured by a thermometer accurate within $\frac{1}{2}$ deg F directly exposed to the steam and located within 12 inches of the convector. The steam pressure shall be measured by a liquid-filled manometer connected to the supply pipe. The supply piping shall be well insulated and of such size as to cause only a negligible pressure drop between manometer and convector. The piping inside the test room or booth shall be kept to a minimum. (Fig. 1 illustrates a suitable piping arrangement.)

19c. Condensate piping.—The condensate piping shall be well insulated and shall freely drain the condensate from the convector to a receptacle. Suitable seals shall be provided in this condensate pipe to prevent steam from issuing from the end of this piping. This pipe shall terminate outside the test room or booth and be provided with an air vent. Steam which might escape from this vent shall be conducted outside the test room or booth.

19d. Air temperature.—The convector shall be tested with an inlet air temperature of not less than 60°F nor more than 75°F. This temperature shall be measured at three or more points spaced not more than 8 inches apart throughout the length of the inlet-air opening midway between the top and bottom and 18 inches in front of the inlet. The outer thermometers shall be not more than 3 inches from the ends of the air inlet. The last 2 inches of the temperature sensitive end, if more than 0.03 inch in diameter, shall be shielded against radiation by bright-metal shields of such construction as not to interfere with the air flow. (See fig. 2 for suggested construction.) The thermometer ¹ used in the test shall be accurate within $\frac{1}{2}$ deg F. The sensitive end of the thermometer shall be 'The term "thermometer," as used in this standard, applies to any temperature-measuring device.

not more than $\frac{5}{16}$ inch in diameter. It is recommended that temperature readings be taken in the center of the room or booth at levels of 3 inches, 30 inches, and 60 inches above the floor and 3 inches below the ceiling.

19e. Air venting.—The convector shall be vented continuously during the test by suitable means, and the expelled air, gases, and steam shall be discharged outside the test room. The vent in the condensate line mentioned in paragraph 19c of this part 2 may be used for this purpose. It is recommended that the air-vent hole be not larger than 0.03 inch in diameter.

20. Test procedure.

20a. Start of test.—The test shall be started only after a state of equilibrium has been reached. Such a state of equilibrium may be considered as obtained if, for a period of at least 30 minutes, the inlet air temperature has not varied more than 1 deg F and the rate of condensation has not varied more than 3 percent.

20b. Duration of \bar{test} .—The test shall be conducted for not less than 1 hour.

20c. Condensate.—The total condensate shall be collected and weighed to 0.01-pound accuracy. At least two measurements of condensate should be made at half-hour intervals within the test time, and the condensation rate obtained on these measurements shall not vary more than 3 percent. Intermediate readings shall be taken every 15 minutes during the tests in a warm-wall booth.

20d. *Temperature readings.*—The inlet air and steam temperatures shall be read at least at the beginning and at the end of the test. Intermediate readings shall be taken every 15 minutes during the tests in a warm-wall booth. These readings shall not vary more than a total of 1 deg F. The average of the temperatures shall be used for calculation.

21. No load test.—The condensing capacity of the part of the supply and condensate piping which drains into the condensate collector shall be determined by a separate test, simulating test conditions, and the amount of condensation thus determined shall be deducted from the gross condensation of the convector.

22. Calculation of condensation capacity.—The condensation capacity of a convector under test conditions shall be determined by the formula:

$$H_{ts} = W_s \times h_{ta},$$

where

 H_{ts} = Condensation capacity for test conditions, Btu per hour

- $W_s =$ Net weight of condensate, expressed in pounds per hour, (gross weight of condensate minus deduction for supply and condensate piping)
- h_{fg} = Latent heat of evaporation of steam, corresponding to the saturated steam temperature in the convector during test (see table 1).

Temperature	Absolute steam pressure	Absolute steam pressure	Latent heat (h_{fg})
°F ^a 214 215 ^a 216	lb/sq~in 15.289 15.595 15.901	in. Hg 31.129 31.752 32.375 32.010	Btu/lb 969.0 968.4 967.8

TABLE 1.—Latent heat of	evaporation of steam
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"Abstracted, by permission, from "Thermodynamic Properties of Steam" by J. H. Keenan and F. G. Keyes, published by John Wiley & Sons, Inc.

23. Correction factor for standard conditions.—The correction factor for converting the capacity during test conditions to the capacity at standard conditions shall be determined by the formula:

$$C_{s} = \left[\frac{215 - 65}{t_{s} - t_{l}}\right]^{1.5} = \left[\frac{150}{t_{s} - t_{l}}\right]^{1.5},$$

where

 $C_s =$ Correction factor

 $t_s =$ Saturated-steam temperature during the test, °F

 t_l = Average inlet-air temperature during test, °F.

For convenience, these correction factors are given in table 2 for the range of temperature differences permitted by this standard.

$$C_{s} = \left[\frac{150}{t_{s} - t_{i}}\right]^{1.5}.$$
 Range $\begin{cases} t_{i}: 60.0^{\circ} \text{ to } 75.0^{\circ}\text{F}.\\ t_{s}: 214.0^{\circ} \text{ to } 217.0^{\circ}\text{F}. \end{cases}$

ts—ti	C_s	$t_s - t_l$	C_{s}
157.0	0.9339	147.5 .	1.0254
156.5	.9383	147.0	1.0307
156.0	.9428	146.5	1.0361
155.5	.9473	146.0	1.0414
155.0	.9518	145.5	1.0467
154.5	.9567	145.0	1.0522
154.0	.9612	144.5	1.0577
153.5	.9660	144.0	1.0632
153.0	.9708	143.5	1.0687
152.5	.9754	143.0	1.0744
480.0.			
152.0	.9802	142.5	1.0800
151.5	.9852	142.0	1.0856
151.0	.9901	141.5	1.0915
150.5	.9951	141.0	1.0973
150.0	1.0000	140.5	1.1031
140 5	1 0049	140.0	1 1001
140.0	1 0100	120.5	1 1159
149.0	1 0159	109.0	1 1910
148.0	1.0152	199.0	1.1210
148.0	1.0204		

TABLE 2.—Correction factors

24. Condensation capacity under standard conditions:—The condensation capacity under standard conditions (H_s) shall be determined as follows:

$$H_s = C_s \times H_{ts}.$$

25. Sources of error in steam convector testing.—The major sources of error are as follows:

- (a) Entrained water brought into the convector with the steam.
- (b) Improper measuring of condensate caused by heat loss of supply and condensate piping.
- (c) Loss of condensate during the process of collecting by spillage or evaporation.
- (d) Incomplete venting of the convector.
- (e) Excessive air currents inside test room or booth due to disturbances.
- (f) Wet or insufficient insulation on piping.

- (g) Incorrect calibration of thermometers and scales.
- (h) Starting test before equilibrium is obtained.
- (i) Inaccurate inlet air temperature readings due to improper shielding.

3. METHODS OF TESTING HOT-WATER CONVECTORS

26. Test conditions.

26a. *Relative humidity and air motion.*—It is inadvisable for relative humidity in the test room or booth at the time of test to exceed 75 percent. Air motion shall be limited to natural circulation.

26b. Air temperature.—The convector shall be tested with an inlet air temperature of not less than 60°F nor more than 75°F. This temperature shall be measured at three or more points spaced not more than 8 inches apart throughout the length of the inlet-air opening midway between the top and bottom and 18 inches in front of the inlet. The outer thermometers shall be not more than 3 inches from the ends of the air inlet. The last 2 inches of the temperature sensitive end, if more than 0.03 inch in diameter, shall be shielded against radiation by bright-metal shields of such construction as not to interfere with the air flow. (See fig. 2 for suggested construction.) The thermometer used in the test shall be accurate within $\frac{1}{2}$ deg F. The sensitive end of the thermometer shall be not more than $\frac{5}{16}$ inch in diameter. It is recommended that temperature readings be taken in the center of the room or booth at levels of 3 inches, 30 inches, and 60 inches above the floor and 3 inches below the ceiling.

26c. Water supply.

- (1) During the test the hot water shall be supplied to the convector at a rate not varying more than 2 percent, and the water temperature shall not vary more than 1 deg F.
- (2) The temperature of the water entering and leaving the convector shall be measured by thermometers in direct contact with the water. These thermometers shall be accurate to ¼ deg F and shall be located within 3 feet of the convector. The piping between the convector and these thermometers shall have an external surface not to exceed 3 percent of the convector heating surface and shall be insulated with equivalent to 1-inch hairfelt or better. (Fig. 3 illustrates a suitable method of water supply.)
- (3) It is very important that the water in the inlet and outlet supply piping be thoroughly mixed at the point of temperature measurement to avoid errors due to stratification.

26d. *Air venting.*—The convector shall be vented before starting the test by means of a manually operated air vent through the regular opening provided.

27. Test procedure.

27a. Start of test.—The test shall be started after a state of equilibrium has been reached. Such a state of equilibrium may be considered as obtained if the inlet air temperature has not varied more than 1 deg F and the water temperature not more than 1 deg F for a period of 30 minutes, and the rate of water flow has not varied more than 2 percent.

27b. Duration of test.-A test shall be conducted for not less than 10 minutes. At least two such tests shall be made consecutively. The water heat capacity determined by these tests corrected to standard conditions shall not vary more than 3 percent.

27c. Readinas.

- (1) The total water flow during the test periods shall be collected and weighed to an accuracy of at least $\frac{1}{2}$ percent.
- (2) The inlet air temperatures shall be read at least at the beginning and at the end of each test. Intermediate readings may be taken. The average of the air-temperature readings taken at the beginning of the tests shall not vary more than 1 deg F from the average temperature readings taken at the end of the tests. The average of all the air temperatures shall be used for calculation.
- (3) The water temperatures shall be read at the beginning and end of each test and during the test at intervals of not more than 2 minutes. These readings shall not vary more than 1 deg F. The average of these temperatures shall be used for calculation.

28. Calculation of water heat capacity.—The water heat capacity of the convector under test conditions shall be determined by the formula

$$H = W(\theta_1 - \theta_2),$$

where

H = Water heat capacity for test conditions. Btu per hour

W = Weight of water, pounds per hour

 θ_1 = Average entering water temperature, °F θ_2 = Average leaving water temperature, °F.

29. Correction for standard conditions.—Plot a performance curve on which the abscissa shows the difference between average water temperature and average inlet air temperature and the ordinate shows test heat capacities (H), Btu per hour. From the values of average water temperatures used in the published ratings, deduct 65° F and for these differences. as abscissa values on the chart, read the Btu per hour from the curve. The values so obtained are the water heat capacities for standard air conditions (65°F) and for the selected average water temperatures.

30. Sources of error in hot water convector testing.—The major sources of error are as follows:

- (a) Incomplete venting of the convector
- (b) Excessive air currents inside test room or booth due to disturbances
- (c) Wet or insufficient insulation on piping
- (d) Incorrect calibration of thermometers and scales
- (e) Starting test before equilibrium is obtained
- (f) Inaccurate water temperature readings due to stratification
- (g) Inaccurate inlet air temperature readings due to improper shielding.

31. Water friction head.—Tests to determine the friction head through the convector may be made with water of room temperature. The connecting pipes shall be the same size as the convector tapping. Tests shall be made for the conditions prescribed in item 14 on form C-7. Figure 4

shows suggested apparatus for making this test for hot-water convectors. Figure 5 shows the type of piezometer that may be used in connection with the test setup given in figure 4.



FIGURE 1.—Suggested connections and equipment for supplying steam to the convector and measuring the condensate.

To simplify illustration, this diagram shows all apparatus in a single plane. It is essential that steam separator and seal be located immediately outside of test booth with as short intervening piping as possible.







FIGURE 3.—Suggested method for testing heat emission of hot water conrectors.



FIGURE 4.—Suggested method for testing friction through hot water convector element.



FIGURE 5.—Detail of piezometer rings shown in figure 4.

4. METHODS OF RATING

32. Top outlet convectors.—The ratings of convectors with top outlets shall not exceed the capacity, as defined in paragraph 10, or determined as prescribed in paragraphs 35, 35a, and 35b.

33. Front outlet convectors.—The ratings of convectors with front outlets shall not exceed the capacity, as defined in paragraph 10, or determined as prescribed in paragraphs 35, 35a, and 35b, plus the percentages listed in table 3. The percentages for sizes not listed in table 3 shall be interpolated from the two nearest figures given in the table.

Over-all enclosure height	Add to capacity percentages not exceeding those listed below to arrive at rating
	Percent
38 or more	0
36	1
34	2
32	3
31	
30	5
90	e a
49	0
28	1
27	8
26	9
25	10
24	11
23	12
22	13
91	14
20 én 1- en	17

TABLE 3.—Maximum additions to capacity

34. *Inclined outlet convectors.*—The ratings of inclined outlet convectors shall not exceed the capacity as defined in paragraph 10 or determined as prescribed in paragraph 36, plus the percentages in table 3 multiplied by

Angle of outlet to horizontal.

For example, a 45° inclined outlet of an enclosure 25 inches high allows an addition to the capacity of

$$\frac{45}{90} \times 10 = 5$$
 percent.

35. Capacities of untested sizes.

- 35a. Steam convectors.
 - Capacities of convectors with untested enclosure heights shall be determined by lineal interpolation between tested heights. (See paragraph 15.)
 - (2) Capacities of convectors of untested lengths shall be determined by multiplying the capacity of the tested convector of the same type and depth by the ratio obtained by dividing the heating surface of the untested convector by the heating surface of the tested convector. The heating surface shall include all convector primary and secondary heating surface exposed to circulating air, including sur-

face of the headers. This extension of the test results is only permissible if for all lengths, the enclosure depth, the stack height, and opening heights and widths are the same; also, if the length of inlet and outlet grilles or openings and free area are changed in the abovementioned ratio. (See paragraph 38.)

35b. Hot-water convectors.—If the water convector is similar to the steam convector (as defined in paragraph 17e), the relation between capacities of tested and untested sizes of the water convector shall be identical to the relation between the capacities of the corresponding sizes of the steam convector.

36. Capacities of untested inclined outlet convectors.—The capacity of a convector with inclined outlet enclosure where the inclined outlet has an inclination of 60° or less from the horizontal shall be the same as an identical convector which has been tested with top outlet enclosure, the height of which is equal to the distance from the center of the inclined outlet to the bottom of the enclosure.

37. Reduction in ratings for variations in enclosure material.—The material of enclosure ends and back may be different from those used during the test. If a metal front panel was used during the test and is replaced by a nonmetallic panel, the rating shall be reduced 5 percent.

38. Reductions in ratings for dimensional variations.

38a. Where inside enclosure depth does not exceed depth of test enclosure by more than 2 percent, no reduction. For each additional 1 percent depth, deduct 2 percent.

38b. Where stack height is lower than stack height used during test by 3 percent or less, no reduction. For each additional 1 percent reduction in stack height, deduct $\frac{1}{2}$ percent.

38c. Where inside enclosure length does not exceed length of enclosure used during test by more than 2 percent, no reduction. For each additional 1 percent increase in length, deduct 1 percent.

38d. Where any opening height or width or the free area is less than that used during test by 10 percent or less, no reduction. For each additional 1 percent reduction in any opening height or width or the free area, deduct 1 percent.

5. LABELING AND APPROVALS

39. The procedure outlined below will become available to all manufacturers, whether members of the sponsor Associations or not, in connection with approval of convector ratings.

40. Compliance.—In order that the purchaser may be assured of obtaining convectors rated according to this standard, manufacturers may, individually or in concert with their trade association, declare that the ratings of convectors have been determined in conformity with this standard. Manufacturers whose convectors are rated in conformity with the provisions of this standard and whose ratings have been approved as prescribed in paragraphs 41a and 41b may include in their sales literature an 1 on invoices and contracts the following statement:

The ratings of these convectors have been determined in conformance with Commercial Standard CS140-47, as developed cooperatively by the trade and the National Bureau of Standards, U. S. Department of Commerce, and the said ratings have been approved by the Convector Rating Committee. 41. Approval of ratings.

41a. In order that convectors may be purchased with full confidence that the ratings used by the manufacturer have been determined strictly in accordance with Commercial Standard CS140-47, the manufacturers who are members of The Institute of Boiler and Radiator Manufacturers and of the Convector Manufacturers Association, respectively, have appointed two of three members of a Convector Rating Committee with authority to receive and analyze test data which the manufacturer has used in determining his convector ratings. The third member has been appointed by the Division of Trade Standards, National Bureau of Standards. The Division of Trade Standards, National Bureau of Standards, has appointed a secretary of the Convector Rating Committee, to whom manufacturers who desire approval of ratings will submit their test data and requested ratings. The secretary will be responsible for referring said data to the Convector Rating Committee and receiving from that committee their determinations as to whether the ratings requested by the manufacturer represent accurate results based on the provisions of this commercial standard.

41b. The use of the compliance statement covered in paragraph 40 may be construed as evidence that said committee has approved the ratings used by the manufacturer. Approval of the committee will be furnished to the manufacturer in writing, signed by the secretary of the committee.

42. Data required for approval.—Parts 1, 2, 3 and 4 set forth in detail the conditions to be observed in determining convector ratings. Tests may be conducted by the manufacturer or at any laboratory selected by him, but all the observations taken and the results obtained during the test period are to be recorded and submitted on forms prescribed below, which will be provided at cost by the secretary of the committee upon request.

42a. Test reports.

- (1) Forms C-6 and C-7 are to be used for reporting test results on steam and water convectors, respectively. The data on these forms are to be recorded from individual manufacturer's log sheets, and the signature of the manufacturer will be construed as a certification that the data reported on these forms accurately represent the results obtained on tests conducted according to the provisions and within the limitations of this standard.
- (2) On steam tests, Form C-6 shall be filed covering, on one sheet, the various tested enclosure heights for a given "Type and Nominal Depth." Each "Type and Nominal Depth" is to be reported on a separate sheet.
- (3) On water tests, Form C-7 shall be filed for the "Type and Nominal Depth" of a convector of the over-all enclosure height used in test. Separate sheets are to be filed covering each convector type and depth and for each style and enclosure tested. Graphs are to accompanyeach copy of Form C-7 that is submitted, in accordance with the provisions of paragraph 29.

42b. Dimensional data reports.—Forms C-3, C-4, and C-5 are to be submitted for each cataloged convector and enclosure. Separate sheets of these forms are to be submitted if the dimensions of steam and water convectors differ.

42c. Request for approval of ratings.

- (1) Forms C-1 and C-2 constitute official request from the manufacturer to the committee for approval of ratings that are to be included in trade literature, and shall be filed IN DUPLICATE.
- (2) One form C-1 only need be used to cover steam and hotwater ratings if the steam and water convectors are similar in all respects as defined in paragraph 17e. If steam and water convectors differ, separate copies of form C-1 must be filed covering steam and water ratings, respectively.
- (3) Form C-2 is to be used only where water heat ratings are expressed in trade literature in Btu per square foot of catalog steam rating and where steam and water convectors are similar in all respects as defined in paragraph 17e.

43. Use of approved ratings.—Receipt by the manufacturer of copies of forms C-1 and C-2 signed by the secretary of the committee will constitute official approval for the use of those ratings, and thereafter no ratings for those convectors in excess of the approved ratings shall be used by the manufacturer. Two copies of trade literature incorporating these ratings shall be filed by the manufacturer with the secretary of the committee as soon as available. The manufacturers' trade literature shall state the percentages which have been added to condensation capacity or water heat capacity in determining catalog ratings of front outlet and inclined outlet enclosures. This statement shall be printed adjacent to the catalog ratings, substantially as follows:

These ratings represent the condensation capacity (or water heat capacity) to which have been added the following percentages in accordance with the provisions of Commercial Standard CS140-47.

Secretary, Convector Rating Committee, Room 1001, 60 East Forty-Second Street, New York 17, N. Y. (Form C-1)

REQUEST FOR APPROVAL OF RATINGS (Square Feet) Note.—This form is to be used for both steam and hot water ratings and only one form need be used if steam and water convectors are similar in all respects as defined in paragraph 17e of the commercial stand-ard. If steam and water convectors differ, use separate sheets and cross out "steam" or "water" after the name of the convector. Water ratings shall be based on 170°F average water temperature with 20° drop through the convector. arough the convector. Convector Steam. Water.

forms....., dated.....

	RATINGS, SQUARE FEET													
	Type and nominal depth	Tested length inches	Location of outlet	Er	nclo	sure	hei	ghts l	istee	l in c	atal	og (i	nch	es)
Item 18, form C-6 or C-7 Catalog rating			Front Front											••
Item 18, form C-6 or C-7			Тор	•	•	•	•					•	•	•
Catalog rating Item 18. form C-6 or C-7			Front		-		_			_			—	
Catalog rating			Front							_				
Item 18, form C-6 or C-7 Catalog rating		·····	Тор Тор				•					•		
Item 18, form C-6 or C-7			Front	•		•								
Catalog rating			Front Top							-			-	
Catalog rating Approval of catalog ratings lis	ted above i	s requeste	Top d. The rat	ings	of	thes	e co	nvect	or t	ypes	for 1	engt	hs a	ind
undersigned manufacturer has surface is adequately heated v	s determine	ed that on the pressu	all lengths	f the	d d e he	epth	s of g m	thes	e cor n.	vect	ors	the h	neat	ing
Manufacturer		.By					-Tit	le						

Date	
The ratings	listed above are hereby approved.
Date	Secretary, Convector Bating Committee.

Secretary, Convector Rating Committee, Room 1001, 60 East Forty-Second Street, New York 17, N. Y. (Form C-
REQUEST FOR APPROVAL OF WATER RATINGS (Btu per Square Foot)
Note.—This form is to be used for hot water convectors where water heat ratings are expressed in trac literature in Btu per sq ft of cataloged steam ratings and where the steam and water convecto are similar in all respects as defined in part one, paragraph 17e of the commercial standard. Convector: NameType and nominal depth.
Tested in enclosure of inches over-all height with outl
3. Heating surface of steam convector of same type and length
6. Rating of this steam convector (item 18 on form C-6)item 3
7. Rating of steam convector of same length as water convector (item $6 \times \frac{1}{1 \text{ item } 5}$)sq ft
 Published average water temperature, °F (list all cataloged water temperatures)
 10. Water heat capacity, Btu/hr (read from graph (item 1) for each temperature shown in item 9) 11. Rating, Btu/hr (item 10+percentage covered in program 23 or 24)
12. Rating per square foot steam rating, (item 11 Btu/sq ft
Approval is requested of ratings shown in item 12 for the temperature shown in item 8. Our
ManufacturerByTitle
The ratings listed above are hereby approved.
Date
Secretary, Convector Rating Committee, Room 1001, 60 East Forty-Second Street, New York 17, N. Y. (Form C-
DESCRIPTION OFCONVECTOR AND ENCLOSURE Note.—If physical dimensions listed herein apply only to steam or water convector, state which
Construction of heating element State whether assembled or one piece
Material of tubes
Material of nns Details of convector Nors.—The following details of convector are to be filled in only if fins and tubes are assembled. Method of bonding
Type
No. of fins per inch
Describe inserts.
Remarks:
Enclosure Material of back, top and sides Material of front Remarks
Manufacturer
ByTitle

Secretary, Convector Rating Committee, Room 1001, 60 East Forty-Second Street, " New York 17, N. Y.



NOTE .--- If physical dimensions listed herein apply only to steam or water convector, state which

Convec-	Installa-			Shortes	t length	Tested	length	Longest length			
tor type	tion height, A	Depth, B	Height, C	Over-all length, L, min	Heating surface, sq ft	Over-all length, L	Heating surface, sq ft	Over-all length, L, max	Heating surface, sq ft		
				·							
Manufactu	irer		Date.	Ву			Title				

CONVECTOR DIMENSIONS

(Form C-4)

Secretary, Convector Rating Committee, Room 1001, 60 East Forty-Second Street, New York 17, N. Y.

(Form C-5)

CONVECTOR

Height,		Stack	height					
	Top	Front	Inclined	l outlet	Denth	Over-all	Free	
н	outlet, ST	outlet, SF	SI	Angle of inclination, α	Deptn, D	OL*	EL*	
·····								
	Height, H	Height, H	Height, Height, Height, Height, ST ST SF	Height, Height, H	Height, Heig	$\begin{array}{c c c c c c c c c c c c c c c c c c c $	$\begin{array}{c c c c c c c c c c c c c c c c c c c $	

ENCLOSURE DIMENSIONS-Continued

			Inlet grille		Outlet grille						
Convector type	Height, H	Height, BH	Length, BL*	Free area, %	Height, GH	Width, GW	Length, GL*	Free area, %			
							• -				
		,									
Manufacture	er		Date			Title					

Secretary, Convector Rating Committee, Room 1001, 60 East Forty-Second Street, New York 17, N. Y.

(Form C-6)

Norz.—This form is to be used for reporting tests on steam convectors only. Convector: NameType and nominal depthType of test room 1. Tested enclosure height, inches 2. Location of outlet 3. Tested convector length, inches 4. Number of tests made with same enclosure 5. Barometric pressure	
Convector: NameType and nominal depthType of test room	
1. Tested enclosure height, inches. 2. Location of outlet. 3. Tested convector length, inches. 4. Number of tests made with same enclosure. 5. Barometric pressure.	
Location of outlet Tested convector length, inches Number of tests made with same enclosure Barometric pressure	-
Tosted convector length, inches Tosted convector length, inches Sarometric pressure	
 Vested convector rengen, inclusive and enclosure	• -
5. Barometric pressure	
J. Darometric pressure	
6 Steem inlet temperature °F	
7. Steam processing (h abcaluta)	
9. Steam temperature commence align to proceed on ST	
o. Interim temperature corresponding to pressure, r	
3. The air temperature, F	
10. Steam temperature – Infet air temperature (Item 8 – Item 9)	
11. Duration of test, in-	-
12. Total weight of condensate during test, 10.	
13. Condensation rate, lb per hr (item 12/item 11)	
14. Latent heat of steam (Hfg), Btu/lb	
15. Condensation capacity at test, Btu/hr (item 14×item 13)	
16. Condensation capacity for standard conditions, Btu/hr	
(part two, paragraph 24 of the commercial standard)	
17. Condensation capacity for standard conditions, sq ft (item 16/240)	
18. Rating, sq ft (item 17+percentage covered in paragraph 33 or 34)	
NOTE	-
Manufacturer By Title	
Date	

Secretary, Convector Rating Committee, Room 1001, 60 East Forty-Second Street, New York 17, N. Y.

(Form C-7)

		4							
1.	Intended nominal temperature drop through convector, °F								
2.	Tested convector length, inches					1			
3.	Measured heating surface of this length, sq ft								
4.	Number of tests made with same water temperature								
5.	Average inlet water temperature, °F								
6.	Average outlet water temperature, °F								
7.	Average water temperature, °F $\left(\frac{\text{item } 5 + \text{item } 6}{2}\right)$								
8	Water temperature drop °F (item 5-item 6)	ł						1 1	
9.	Water flow, lb/hr								
10.	Water heat capacity at test condition, Btu/hr (item 8×item9)								
11.	Average inlet air temperature, °F								
12.	Average water-inlet air temperature, °F (item 7-item 11)								

13. Plot for each intended temperature drop items 10 and 12 on a curve in accordance with paragraph 29 of to the each intended temperature drop items to and its on a curve in accordance with paragraph 25 of commercial standard. If convector length used with high water temperatures differs from length used with low temperatures, plot each separately. Extend curve of short convector into untested temper-ature range by multiplying capacities read from curve of long convector by the ratio: - heating surface of short convector heating surface of long convector and plotting the values so obtained on graph of short convector.

Resistance in inches water of longest convector for flow rate required at a rating for 170°F average water temperature in 26 in. high enclosure with

For 10°F temperature drop....flow, lb/hr. For 20°F temperature drop....flow, lb/hr.

top outlet Fill in items below only where separate water rating tables are published independent of steam rating tables

perature drop (read from graph). Percentage added for heating effect for enclosure height and type shown above. (See paragraph __Btu/hr 16 33 or 34) 17. Rating, Btu/hr (item 15+addition for heating effect shown as a percentage in item 16)......Btu/hr

18. Rating, sq ft (item 17÷150)		 	 sqft
Manufacturer	Bv	 .Title	
I	Date		

EFFECTIVE DATE

44. The standard is effective for new production from March 1, 1948.

STANDING COMMITTEE

45. The following individuals comprise the membership of the standing committee, which is to review, prior to circulation for acceptance, revisions proposed to keep the standard abreast of progress. Comment concerning the standard and suggestions for revision may be addressed to any member of the committee or to the Division of Trade Standards, National Bureau of Standards, which acts as secretary for the standing committee.

R. N. TRANE. Chairman

Manufacturers:

E. L. BRODERICK, American Radiator & Standard Sanitary Corp., 675 Bronx River Road, Yonkers 4, N. Y.

R. G. DIXON, Modine Manufacturing Co., Racine, Wis.
R. E. FERRY, The Institute of Boiler & Radiator Manufacturers, 60 East 42d St. New York 17, N. Y.
L. N. HUNTER, The National Radiator Co., Johnstown, Pa. JOHN W. MCELGIN, J. J. Nesbitt Co., Holmesburg, Philadelphia, Pa. (Representing the Convector Manufacturers Association.)
M. W. MCRAE Comp. Co. 4100 South Vederic Association.)

M. W. McRAE, Crane Co., 4100 South Kedzie Avenue, Chicago 5, Ill. R. N. TRANE, The Trane Co., La Crosse, Wis.

Distributors and Installers:

C. S. RAMBO, The Central Supply Association, 228 North La Salle St., Chicago 1, Ill.
H. F. WELDIN, 3101 Market St., Wilmington, Del. (Representing National Association of Master Plumbers of the U. S., Inc.)
J. H. ZINK, JR., Heat & Power Corp., 424 East Saratoga St., Baltimore 2, Md. (Representing the Heating, Piping & Air Conditioning Contractors National

Association.)

PLUMBING & HEATING WHOLESALERS OF NEW ENGLAND, INC., 31 St. James Ave., Boston 16, Mass. (Invited to name a representative.)

Users:

RHEES BURKET, 1223 Connecticut Ave., N. W., Washington, D. C. (Representing The American Institute of Architects.)

S. J. FENTIMAN, Veterans Administration, Washington 25, D. C.

R. K. THULMAN, Federal Housing Administration, Washington 25, D. C.

Laboratories:

R. S. DILL, National Bureau of Standards, Washington 25, D. C.

W. S. HARRIS, University of Illinois, Urbana, Ill.

HISTORY OF PROJECT

46. In May 1941 a conference of convector manufacturers, representatives of Government agencies and other organizations was sponsored by the Subcommittee on Heating and Ventilating, Central Housing Committee, FHA, for the purpose of determining a generally acceptable basis for the solution of problems involved in the testing and rating of convectors.

47. Subsequently, the Joint Convector Code Committee, consisting of representatives of the Convector Manufacturers Association and the Institute of Boiler and Radiator Manufacturers, assisted by advisory members of the committee, representing the National Bureau of Standards and the University of Illinois, developed a code for testing and rating convectors.

48. Under date of December 9, 1944, the draft so developed was submitted by the Joint Convector Code Committee to the National Bureau of Standards with a request for cooperation in the establishment of a commercial standard.

49. After minor changes in the draft the Division of Trade Standards circulated, on March 15, 1945, copies of the proposed commercial standard to manufacturers and representative user, distributor, and testing organizations, and to government agencies for review and advance comment.

50. Following adjustment of the requirements to represent the composite views of all interested groups, the recommended standard was circulated on July 19, 1946 to the trade for written acceptance.

51. Upon receipt of official acceptances representing a satisfactory majority of the trade and in the absence of active, valid opposition from any quarter, announcement was issued on March 3, 1947, that the standard would become effective for new production from March 1, 1948.

52. In accordance with advices from the Joint Convector Code Committee, this standard represents only the first step in a general program. Convector ratings heretofore published by manufacturers, generally have consisted of a condensation or water heat capacity plus a heating effect valuation, intended to represent the relation between the useful output of a convector, in the comfort zone of a room, and the total input as measured by steam condensation or water heat capacity. Manufacturers. regarding heating effect as a justifiable part of convector rating, have not agreed on a method for evaluating its magnitude. The committee's ultimate task, therefore, is to formulate methods of test which will accurately measure the complete performance of convectors of any design in advance of installation. Development of this ultimate objective will require a longer period of investigation and research. In the interim, the committee has made available a method for rating convectors which takes into consideration the condensation or water heat capacity and an evaluation of heating effect, as shown in table 3.

53. The following named individuals constituted the Joint Convector Code Committee:

Representing the Convector Manufacturers Association:

R. N. TRANE (chairman), The Trane Co., La Crosse, Wis. A. G. DIXON, Modine Mfg. Co., Racine, Wis.

Representing the Institute of Boiler and Radiator Manufacturers:

L. N. HUNTER. The National Radiator Co., Johnstown, Pa.

M. W. MCRAE, Crane Co., Chicago, Ill.

Advisory Members:

R. S. DILL, National Bureau of Standards, Washington, D. C.

M. K. FAHNESTOCK, University of Illinois, Urbana, Ill.

Secretary:

R. E. FERRY, The Institute of Boiler and Radiator Manufacturers, 60 East Forty-Second Street, New York 17, N.Y.

ACCEPTANCE OF COMMERCIAL STANDARD

If acceptance has not previously been filed, this sheet properly filled in, signed, and returned will provide for the recording of your organization as an acceptor of this commercial standard.

Date

Division of Trade Standards. National Bureau of Standards. Washington 25, D. C.

Sirs

We believe that the Commercial Standard CS140-47 constitutes a useful standard of practice, and we individually plan to utilize it as far as practicable in the

production ¹ distribution ¹ purchase¹ testing¹

of convectors.

We reserve the right to depart from it as we deem advisable.

We understand, of course, that only those articles which actually comply with the standard in all respects can be identified or labeled as conforming thereto.

Signature of authorized officer______(In ink)

(Kindly typewrite or print the following lines)

Name and title of above officer

Organization____

(Fill in exactly as it should be listed)

Street address

City, Zone, and State

¹Underscore which one. Please see that separate acceptances are filed for all subsidiary companies and affiliates which should be listed separately as acceptors. In the case of related interests, trade associations, trade papers, etc., desiring to record their general support, the words "General Support" should be added after the signature.

TO THE ACCEPTOR

The following statements answer the usual questions arising in connection with the acceptance and its significance:

1. Enforcement.—Commercial standards are commodity specifications voluntarily established by mutual consent of those concerned. They present a common basis of understanding between the producer, distributor, and consumer and should not be confused with any plan of governmental regulation or control. The United States Department of Commerce has no regulatory power in the enforcement of their provisions, but since they represent the will of the interested groups as a whole, their provisions through usage soon become established as trade customs, and are made effective through incorporation into sales contracts by means of labels, invoices and the like.

2. The acceptor's responsibility.—The purpose of commercial standards is to establish for specific commodities, nationally recognized grades or consumer criteria and the benefits therefrom will be measurable in direct proportion to their general recognition and actual use. Instances will occur when it may be necessary to deviate from the standard and the signing of an acceptance does not preclude such departures; however, such signature indicates an intention to follow the commercial standard where practicable, in the production, distribution, or consumption of the article in question.

3. The Department's responsibility.—The major function performed by the Department of Commerce in the voluntary establishment of commercial standards on a Nation-wide basis is fourfold: first, to act as an unbiased coordinator to bring all interested parties together for the mutually satisfactory adjustment of trade standards; second, to supply such assistance and advice as past experience with similar programs may suggest; third, to canvass and record the extent of acceptance and adherence to the standard on the part of producers, distributors, and users; and fourth, after acceptance, to publish and promulgate the standard for the information and guidance of buyers and sellers of the commodity.

4. Announcement and promulgation.—When the standard has been endorsed by a satisfactory majority of production or consumption in the absence of active, valid opposition, the success of the project is announced. If, however, in the opinion of the standing committee or the Department of Commerce, the support of any standard is inadequate, the right is reserved to withhold promulgation and publication.

ACCEPTORS

54. The organizations listed below have individually accepted this Commercial Standard as their standard of practice in testing and rating convectors. Such endorsement does not signify that they may not find it necessary to deviate from the standard. the requirements of this standard. Therefore, specific evidence of conformity should be obtained where required.

ASSOCIATIONS (General Support)

- (General Support) American Association of Engineers, Chicago, Ill. American Specification Institute, Chicago, Ill. British Columbia Research Council, Vancouver, B. C., Canada. Convector Manufacturers Association, Chicago, Ill. Fuel Oil Distributers Association of New Jersey, Newark, N. J. Heating, Piping & Air Conditioning Contractors District of Columbia Association, Inc., Washing-ton D. C.
- District of Columbia Association, Inc., Washing-ton, D. C. Heating, Piping & Air Conditioning Contractors National Association, New York, N. Y. Institute of Boiler & Radiator Manufacturers, The, New York, N.Y. Oil-Heat Institute of America, Inc., New York,
- N. Y.
- Western Plumbing Officials Association, Los An-geles, Calif.

FIRMS

- FIRMS Acme Electric Heating Co., Boston, Mass. Adams, Franklin O., Tampa, Fla. Ahern Co., John F., Fond du Lae, Wis. Air Conditioning, Inc., Bethesda, Md. Akron Plumbing & Heating Co., Akron, Ohio. Alear Supply Co., Inc., Rochester, N. Y. Allen Co., A. G., Indianapolis, Ind. (General upport.) Armour Research Foundation, Chicago, III. Automatic Heat, Inc., Philadelphia, Pa. Automatic Heat & Air Conditioning Corp., Joliet, III. Ill.

- III. Baker, Smith & Co., Inc., New York, N. Y. Barber Co., Wm. C., Churchville, N. Y. Barber Co., Wm. C., Churchville, N. Y. Beggs Co., W. E., Scattle, Wash. Beiden Porter Co., Minneapolis, Minn. Better Living Co., Jackson, Miss. Beuttler, William, Sioux City, Iowa. Biddlel Urchasing Co., New York, N. Y. Bjorkman Bros. Co., Minneapolis, Minn. Bon Heating Co., Milwaukee, Wis. Boyd Engineering Co., Albuquerque, N. Mex., and El Paso, Tex. Braman, Dow & Co., Boston, Mass. Bridgeport Brass Co., Bridgeport, Conn. (General support.) support.)

- support.) Brooks-Borg, Des Moines, Iowa. Bucky, Fred W., Jr., Jacksonville, Fla. Buerky, Fred W., Jr., Jacksonville, Fla. Bull, Ralph N., Sparta, N. J. Burnham Boiler Corp., Irvington, N. Y. Campbell Heating Co., E. K., Kansas City, Mo. Cannon & Mullen, Salt Lake City, Utah. Carnegie Institute of Technology, Pittsburgh, Pa. (General support.) Chapin, Rollin C., Minneapolis, Minn. (General support.) Chapin, Rollin C., Minneapolis, Munn, (General support.) Cincinnati, City of, Cincinnati, Ohio. Clarke Bros., Indianapolis, Ind. Colbert Co., Inc., Maurice J., Washington, D. C. Conrad Bros., Chicago, Ill. Conrad & Cummings, Binghamton, N. Y. Consolidated Heating & Ventilating Co., Los Ange-los Colif

- les, Calif.

- Corriveaux, F.—Home & Industrial Service, Sche-neetady, N. Y. Cram & Ferguson, Boston, Mass. Crane Co., Chicago, Ill. Crosbie Co., The, Washington, D. C. Curtsinger, L. P., Eugene, Oreg. Dallman Supply Co., San Francisco, Calif. Daly & Sons, Inc., M. J., Waterbury, Conn. Danforth Co., John W., Buffalo, N. Y. Daniel-Morris Co., Inc., New York, N. Y. DeJarnette, Charles W., Des Moines, Iowa. Detroit, University of, Detroit, Mich. Dierks & Co., Inc., A., Brooklyn, N. Y. Distriet Engineering Co., Washington, D. C. Donovan, John J., Berkeley, Calif. Dudley, James G., New York, N. Y. Dunham Co., C. A., Chicago, Ill. Dunkirk Radiator Corp., Dunkirk, N. Y. Electrical Testing Laboratories, Inc., New York, N. Y. N.Y

- N. Y. Elliott-Lewis Co., Inc., Philadelphia, Pa. Emery Industries, Inc., Cincinnati, Ohio. Evans, Inc., John P., Washington, D. C. Fargo Foundry Co., Fargo, N. Dak. Fitzsimmons & McAllister, Inc., Albany, N. Y. Flannagan, Eric G., Henderson, N. C. Geisler & Co., F. E., Pittsburgh, Pa. General Electric Co., Scheneetady, N. Y. Glauber, Inc., Sam S., New York, N. Y. Graves Engineering Co., Nashville, Tenn. (General support.)

- Graves & Graves, Chicago, III. Graves & Graves, Chicago, III. Greene Co., C. W., Worcester, Mass. Gregory Heating & Plumbing Co., J. M., Jackson-ville, Fla

- ville, Fla. Guimarin & Co., W. B., Columbia, S. C. Hahn, Stanley W., Cleveland, Ohio. Haldeman, Inc., Harry F., Los Angeles, Calif. Hanley Co., S. V., Milwaukee, Wis. Harley, Ellington & Day, Inc., Detroit, Mich. Harris, Jay, New York, N. Y. Hasness, Carlisle D., Harrisburg, Pa. Helfensteller, Hirsch & Watson, St. Louis, Mo. Hodgdon, Charles, San Gabriel, Calif. Hone, Frank L., Jr. San Dieco, Calif.

- Heneinstener, Airsen & Watson, S. Louis, Mo. Hodgdon, Charles, San Gabriel, Calif. Hope, Frank L., Jr., San Diego, Calif. Hubbard, Lange & Heek, Inc., New York, N. Y. Hughes & Co., Spokane, Wash. Hughes Heating Co., Memphis, Tenn. Ideal Furnace Co., Detroit, Mich. International Heater Co., Utica, N. Y. Iron Fireman Sales Corp., Washington, D. C. Jacobs, Lionel L., Wayne, Pa. (General support.) Jaehnig & Peoples, Inc., Newark, N. J. Johnson, Larsen & Co., Detroit, Mich. Kahn, Albert. Associated Architects & Engineers. Inc., Detroit, Mich. Keine & O'Brien, Warren, Ohio. Kien veally Co., V. J., Boston, Mass. Kiefer Plumbing Co., J. E., Denver, Colo. Kilham, Hopkins & Greeley, Boston, Mass. Krauser-Boyd, Inc., North Tonawanda, N. Y. Kyel, Herbert S., Charleston, W. Va. (General support.)

Support.) Larriek, Thomas, Athens, Ohio. Latenser & Sons, John, Omaha, Nebr. Latenser & Sons, John, Omaha, Nebr. Laver, Earl W., Utica, N. Y. Law, Law, Potter & Nystrom, Madison, Wis. Leidy Electric Co., Inc., Phillipsburg, N. J.

25

- Lowiston Hardware & Plumbing Supply Co., Lewiston Hardware & Plumbing Supply Lewiston, Maine. Liniger Co., Inc., Marion, Ind. Little & Son, C. J., Youngstown, Ohio. Lochman Co., Edward W., Kansas City, Mo. Loch, Laurence M., White Plains, N. Y. Lohman Bros., Los Augeles, Calif. Lowry Electric Co., Inc., Williamsport, Pa. Warm & Co., Hutchingon Kans

- Lowry Electric Co., Inč., Williamsport, Pa. Manu & Co., Hutchinson, Kans. Martino Co., A. R., Waterbury, Conn. Massena & duPont, Wilmington, Del. Master Plumber & Heating Contractor, Brooklyn, N. Y. (General support.) Maupai Co., Inc., R. G., Jersey City, N. J. McCord Corp., Detroit, Mich. McQuay, Inc., Minneapolis, Minn. Mechanical Construction Corp., Hibbing, Minn. Mehring & Hanson Co., Washington, D. C. Michigan State College, E. Lansing, Mieh. Mincasolis, City of, Engineering Department.

- Minneapolis, City of, Engineering Department, Minneapolis, Minn.
- Minnesota, University of, Minneapolis, Minn.
- Modine Manufacturing Co., Racine, Wis. Monroe Air Conditioning Co., Inc., Rochester, N. Y Montgomery Ward, Chicago, Ill. Mooser, William, San Francisco, Calif.
- Moran Plumbing & Heating Service, Clifford, Highland Park. Ill.

- Mueller Furnace Co., L. J., Milwaukee, Wis. Mueller, Hair & Hetterich, Hamilton, Ohio. Mutual Heating & Plumbing Co., Buffalo, N. Y.
- National Radiator Co., The, Johnstown, Pa. Nesbitt, Inc., John J., Holmesburg, Philadelphia, Pa
- Neuhaus Heating & Ventilating Co., H. J., Chicago,
- III. New York Testing Labor tories, Inc., New York, N. Y.
- N. I. Noland Co., Inc., Newport News. Va North Dakota Agricultural College, Fargo, N. Dak. Northern Heating & Plumbing Co., Inc., Laconia,
- N. H. Northwestern Heating & Plumbing Co., Evanston,
- III.

- ^{111.} Officer, Gwynn, Lafayette, Calif. Ohio Pipe & Supply Co., Inc., Cleveland, Ohio. Oklahoma, University of, Norman, Okla. Orange Memorial Hospital, Orange, N. J. Oviatt Plumbing & Heating Dist. Co., Inc., Troy, N. Y.
- Pacific Coast Heating & Appliance Co., Portland, Oreg.
- Pehrson & Associates, G. A., Spokane, Wash. Pennsylvania Engineering Co., Philadelphia, Pa. Pennsylvania Hospital, Philadelphia, Pa.

- Perper, George W., Philadelphia, Pa. Perfect Air Conditioning Co., Washington, D. C. Prause, W. K., Charleston, S. C.
- Premier Engineering Co., Inc., Chicago, Ill. Rariden, Henry M., Bedford, Ind.
- CS No.
 - Item 0-40. Commercial standards and their value to business (third edition)
 - 1-42. Clinical thermometers (third edition)

 - 2-30. Mopsticks. 3-40. Stoddard solvent (third edition)
 - 4-29. Staple porcelain (all-clay) plumbing fixtures
 - tures.
 5-46. Pipe nipples; brass, copper, steel and wrought-iron (second edition).
 6-31. Wrought-iron pipe nipples (second edi-tion). Superseded by CS5-46.
 7-29 Standard weight malleable iron or steel screwed unions.
 8-41. Gage blanks (third edition).
 9-33. Builders' template hardware (second edi-tion).

 - tion).
 - 10-29. Brass pipe nipples. Superseded by CS5-46.
 - 11-41. Moisture regains of cotton yarns (second edition). 12-40. Fnel oils (fifth edition).

- Raub Supply Co., Lancaster, Pa. Rautman Plumbing & Heating, Seattle, Wash. Rearick Brothers Heating & Supply Co., Gary, Ind. Richmond Radiator Co., Uniontown, Pa. Ritchie & Associates, James H., Boston, Mass. Rochester Fire College, Rochester, N. Y. Rome Turney Radiator Co., Rome, N. Y. (General
- support.)
- Royal Steam Heater Co., Gardner, Mass.

- Support.) Royal Steam Heater Co., Gardner, Mass. Sawyer Heating Co., Detroit, Mich. Schulke, William H., Moline, Ill. Schuylkill Valley Oil Co., Pottstown, Pa. Sears, Roebuck & Co., Chicago, Ill. Sheet Metal Products Co., Inc., Shamokin, Pa. Slyder-Clough Heating Co., Inc., Washington, D. C. Smith Co., Inc., The H. B., Westfield, Mass. Smith, Hinchman & Grylls, Detroit, Mich. Specification Record, Chicago, Ill. Sprinchorn & Co., Jamestown, N. Y. Stoetzel, Ralph, Chicago, Ill. Swarthmore Heating Service, Swarthmore, Pa. Taylor, Ellery K., Haddonfield, N. J. Temple, Seth J. & Arthur, Davenport, Iowa. Tennessee, University of, Engineering Experiment Station, Knoxville, Tenn. support.)

- support.) Tharp & Son, Chas. E., Ft. Wayne, Ind. Thorne, Henry Calder, Ithaca, N. Y. Trance Co., The, La Crosse, Wis. Tuttle & Bailey, Inc., New Britain, Conn. U. S. Supply Co., Kansas City, Mo. United States Radiator Corp., Detroit, Mich. United States Testing Co., Inc., Hoboken, N. J. (General support)
- (General support.) Utica Radiator Corp., Utica, N. Y. Viking Manufacturing Corp., The, Cleveland, Ohio. Virginia Polytechnic Institute, Blacksburg, Va. (General support.)
- Wahpeton Plumbing & Heating Co., Wahpeton, N. Dak.

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- (second edition).
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 38-32. Hospital rubber sheeting.
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- 40-32. Surgeons' rubber gloves. 41-32. Surgeons' latex gloves. 42-43. Structural fiber insulating board (third edition). 43-32. Grading of sulphonated oils.

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- 54-35. Colors and finishes for cast stone.
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- 69-38. Pine oil disinfectant.
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- 72-38. Household insecticide (liquid spray type) 73-45. Old growth Douglas fir standard stock
- doors (third edition).
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- with CS78-40), 80-41. Electric direction signal systems other than semaphore type for commercial and other vehicles subject to special motor vehicle laws (after market). 81-41. Adverse-weather lamps for vehicles (after
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- 4-41. Calking lead

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- 112-43. Homogeneous fiber wallboard.
- 113-44. Oil-burning floor furnaces equipped with vaporizing pot-type burners.
- 114-43. Hospital sheeting for mattress protection.
- 115-44. Porcelain-enameled tanks for domestic use.
- 116-44. Bituminized-fibre drain and sewer pipe. 117-44. Mineral wool; blankets, blocks, insulating
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 - 128-45. Men's sport shirt sizes—woven fabrics (other than those marked with regular neckband sizes). 129-46. Materials for safety wearing apparel.

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	130-46.	Color	materials	for	art	education	in
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types—testing and reporting.

132-46. Hardware cloth. 133-46. Woven wire netting.

134-46. Cast aluminum cooking utensils (metal

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135-46. Men's shirt sizes (exclusive of work shirts).

Item 136-46. Blankets for hospitals (wool and wool and cotton).

137-46. Size n easurements for men's and boys' shorts (woven fabrics).

138-47. Insect wire screening.

139-47. Nork gloves.
140-47. Convectors: testing and rating.
141-47. Sine bars, blocks, plates and fixtures.

NOTICE.—Those interested in commercial standards with a view toward accepting them as a basis of everyday practice may secure copies of the above standards, while the supply lasts, by addressing the Division of Trade Standards, National Bureau of Standards, Washington 25, D. C.

¹Where "(E)" precedes the CS number, it indicates an emergency commercial standard, drafted under war conditions with a view toward early revision.

☆ U. S. GOVERNMENT PRINTING OFFICE: 1947-744578